Resonance dynamics in compressible cavity flows using time-resolved velocity and surface pressure fields

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The resonance modes in Mach 0.94 turbulent flow over a cavity having a lengthto-depth ratio of five were explored using time-resolved particle image velocimetry (TR-PIV) and time-resolved pressure sensitive paint (TR-PSP). Mode switching was quantified in the velocity field simultaneous with the pressure field. As the mode number increased from one through three, the resonance activity moved from a region downstream within the recirculation region to areas further upstream in the shear layer, an observation consistent with linear stability analysis. The second and third modes contained organized structures associated with shear layer vortices. Coherent structures occurring in the velocity field during modes two and three exhibited a clear modulation in size with streamwise distance. The streamwise periodicity was attributable to the interference of downstream-propagating vortical disturbances with upstream-travelling acoustic waves. The coherent structure oscillations were approximately 180° out of phase with the modal surface pressure fluctuations, analogous to a standing wave. Modal propagation (or phase) velocities, based on cross-correlations of bandpass-filtered velocity fields were found for each mode. The phase velocities also showed streamwise periodicity and were greatest at regions of maximum constructive interference where coherent structures were the largest. Overall, the phase velocities increased with modal frequency, which coincided with the modal activity residing at higher portions of the cavity where the local mean flow velocity was elevated. Together, the TR-PIV and TR-PSP provide unique details not only on the distribution of modal activity throughout the cavity, but also new understanding of the resonance mechanism as observed in the velocity field.

Key words: aeroacoustics, compressible flows, high-speed flow

1. Introduction

The flow over an open cavity generates large pressure fluctuations associated with broadband turbulence and cavity resonance (Rossiter 1964), which can result in high levels of structural vibrations (Wagner *et al.* 2015*b*, 2016; Barone & Arunajatesan 2016). Since cavity flow is analogous in many ways to an open aircraft bay in flight,

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the subject has received much attention over the last sixty years as detailed in the review articles of Rockwell & Naudascher (1978, 1979), Rowley & Williams (2006) and Cattefesta *et al.* (2008). In subsonic flows, when the cavity length-to-depth ratio L/D is less than approximately seven (Tracy & Plentovich 1997), an interaction of the cavity's acoustic field and free shear layer form a feedback loop that results in resonant oscillations. The resonant modes are primarily longitudinal and in simple rectangular geometries, the tonal frequencies are reasonably predicted by the semi-empirical relation provided in the seminal work of Rossiter (1964).

In comparison to modal frequencies, prediction of the modal amplitudes is much more difficult as the pressure magnitudes are sensitive functions of the flow conditions (boundary layer properties, Mach number, etc.) as well as the cavity geometry (L/D)and length-to-width ratio L/W, etc.). Changes to geometric parameters such as L/Walso lead to modifications of cavity turbulence levels (Beresh *et al.* 2015*b*,*c*) and the shear layer growth rate (Beresh, Wagner & Casper 2016), which in turn can alter the overall broadband fluid dynamic loading. Geometric complexities associated with real aircraft bays, such as inflow ramps and doors, can also alter and enhance resonance tones (Casper *et al.* 2015) resulting in three-dimensional flow features (Ukeiley *et al.* 2008).

Other challenges lie in understanding the temporal behaviour of resonating cavity flows. Cavity flow dynamics involves complex interactions of the mixing layer, recirculating flow, turbulence and acoustic waves. Although a given cavity flow can be associated with several tonal frequencies, the tones do not necessarily occur simultaneously. Rather the dominant cavity mode can vary with time - a process known as 'mode switching'. The idea for mode switching was first proposed in the earlier works of Krishnamurty (1955) and Sarohia (1977), later proven in experiments (Kegerise 1999; Kegerise, Spina & Cattefesta 2004; Murray & Ukeiley 2007; Gueniat, Pastur & Lusseyran 2014; de Vicente et al. 2014a,b) and observed in the simulations of Larchevêque et al. (2004). Fairly recently, direct numerical simulation (DNS) by Brès & Colonius (2008) suggested that spanwise instabilities can lead to low-frequency modulations of Rossiter modes. An additional unique view on cavity dynamics was given by Delprat (2006) who theorized that cavity modes are not unique; rather they are modulations of a fundamental loop frequency. Despite these examples of recent progress, the dynamics of cavity flows, particularly in the high-speed regime, is an evolving research area. A common, quantitative consensus on the physical mechanisms of cavity tone modulations, and the relationship between resonance pressure fluctuations and cavity flow field dynamics in general, has yet to be achieved.

Experimental data capturing cavity flow field dynamics are required to develop and validate new theoretical models. Flow visualization methods show that large-scale coherent structures propagate in the shear layer of resonating, compressible cavity flows (Rossiter 1964; Kegerise 1999; Garg & Cattafesta 2001; Kegerise *et al.* 2004; Murray & Ukeiley 2007). More quantitatively, in incompressible cavity flows, time-resolved particle image velocimetry (TR-PIV) and laser Doppler velocimetry (LDV) measurements have revealed periodic velocity variations associated with resonance (Hassan, Labraga & Keirsbulck 2007; Basley *et al.* 2011; Gueniat *et al.* 2014) and have been used to quantify low-frequency, large-scale flapping motions in the shear layer (Liu & Katz 2013).

In compressible flows, however, PIV measurements are typically made at repetition rates far too low to capture dynamics in a time-resolved fashion. To overcome this, researchers have pieced together spatial distributions using time-resolved point

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measurements of velocity and density (e.g. Kegerise 1999). Stochastic estimation methods that create low-dimensional representations of the velocity field informed by time-resolved surface pressure data from sensors have also proved useful. For instance, Murray & Ukeiley (2007) were able to successfully track the propagation of coherent flow structures over surface pressure sensors at transonic Mach numbers. In another example, Murray, Raspet & Ukeiley (2011) used stochastic methods to suggest that Rossiter pressure fluctuations were primarily influenced by mean shear velocity terms in Poisson's equation as opposed to nonlinear, turbulent velocity fluctuation terms.

Even with these methods, it is difficult to elucidate flow structure and dynamics in the compressible flow over a cavity with measurements having either low temporal (10 Hz PIV) or low spatial resolution (i.e. sensors, hot wires). Fortunately, recent advances in measurement technology have set the stage for high repetition rate measurements of cavity flow field dynamics using TR-PIV (Beresh *et al.* 2015*a*), as well as cavity surface pressures with time-resolved pressure sensitive paint (TR-PSP) (Flaherty *et al.* 2013; Casper *et al.* 2017). In the current work, both of these pacing measurement technologies, along with spectral analysis, are utilized to increase understanding of cavity resonance. Data are presented at a free-stream Mach number of 0.94 where the first three resonance modes are strong and have similar amplitude. Importantly, the presence of three robust resonance tones allows for detailed comparisons of resonant flow structure as a function of mode number to be made.

The temporal and spatial resolution of the TR-PIV allows for the dynamics to be captured over nearly the entire cavity flow field in an unparalleled fashion. Mode switching, as measured in the pressure field, is first investigated in the velocity field as a function of both mode number and spatial location. The focus of the paper then shifts to temporal and spatial modulation of coherent structure in the velocity field. Variations in resonance activity throughout the shear layer and the recirculation region as a function of mode number are assessed using fields based on power spectral density (PSD) and are explained in the context of linear stability analysis. The relationship of coherent structure size to the cavity acoustics, as determined using the TR-PSP, is then used to explain how velocity fluctuations vary over the cavity length. Finally, modal phase velocity fields are used to show how the modal propagation velocity varies throughout the cavity for the three modes of interest. Collectively, these results allow the behaviour of coherent structures as identified in the velocity field to be placed within the context of the resonant processes in the cavity.

2. Experimental program

2.1. Trisonic wind tunnel (TWT)

Experiments were conducted in the blowdown-to-atmosphere TWT. Trisonic refers to the fact that the TWT can operate at subsonic, transonic and supersonic Mach numbers. The facility uses air as the test gas and has a test section enclosed in a pressurized plenum. Data were obtained at a free-stream Mach number M_{∞} of 0.94 $(U_{\infty} = 311 \text{ m s}^{-1})$, which was determined using a free-stream velocity measurement at the cavity entrance location and the assumption of isentropic expansion. Forty-nine wind tunnel runs were made at this condition. Typical flow conditions for the experiments conducted in the 305 mm² × 305 mm² test section are given in table 1. The tunnel wall boundary layers developed naturally and were fully turbulent upon arrival at the test section. Previous PIV measurements have indicated that the 99 %

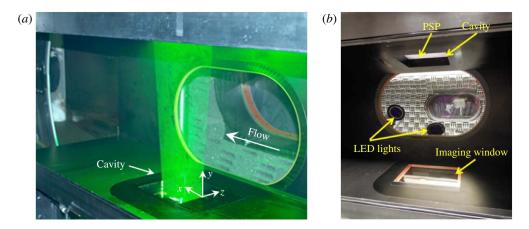


FIGURE 1. Cavity photos: (a) cavity in the lower wall of the test section and the PIV laser sheet and (b) cavity installed in the upper test section wall for PSP experiments.

 M_{∞} P_0 (kPa) T_0 (K) $Re \times 10^{-6}$ (m⁻¹) 0.94 60 321 13 TABLE 1. Typical TWT experimental conditions. T_0 , stagnation temperature; R_e , Reynolds number.

wall boundary layer thickness at the cavity entrance is approximately 13 mm or approximately half the cavity depth. The boundary layer displacement and momentum thicknesses were approximately 1.7 mm and 1.4 mm, respectively.

In subsonic cavity flows it is necessary to mitigate the undesirable acoustical interference associated with solid wind tunnel walls by placing an acoustic absorber in the wall opposite to the cavity (Song 2008; Murray, Sallstrom & Ukeiley 2009; Wagner *et al.* 2015*a*). The spanwise walls of subsonic test sections can also lead to acoustical contamination if untreated (Wagner *et al.* 2015*a*). To reduce these effects, the wall opposite the cavity and one spanwise wall were replaced with acoustic dampeners, which consisted of a porous wall, used typically for transonic testing, backed by acoustical absorption foam. The opposite spanwise wall was solid with a window providing a view for the PIV cameras. This acoustic dampener configuration was found to significantly reduce acoustical contamination in comparison to those using either a solid opposite or solid spanwise wall.

2.2. Cavity hardware

The cavity had a length L of 127 mm, width W of 127 mm and depth D of 25.4 mm and was installed in the lower wall of the test section in experiments using PIV (figure 1*a*). In separate experiments using TR-PSP, the cavity was instead installed in the ceiling of the wind tunnel and the cavity floor was viewed through windows in the lower walls of the test section and plenum (figure 1*b*). In both cases the leading edge of the cavity was 410 mm downstream of the test section entrance and the turbulent boundary layer had similar properties. With an L/D of 5, the cavity flow category is 'open' and expected to resonate (Tracy & Plentovich 1997). The streamwise (*x*), wall-normal (y), spanwise (z) coordinate system originates at the spanwise centre of the cavity leading edge (figure 1a). The wind tunnel floor is defined to be at y = 0, with positive y pointing away from cavity.

2.3. Unsteady pressures

Two dynamic pressure sensors (Kulite XCQ-062-30A) having a range of approximately 207 kPa and a flat frequency response up until approximately 50 kHz were placed in the spanwise centre of the cavity fore and the aft wall. The fore wall sensor was located at y = -0.5D and the aft wall probe was at a position of -0.63D. The sampling frequency f_s was 200 kHz and the data were lowpass filtered at 50 kHz. One million samples were obtained at each location. Previous experiments have confirmed the experiments to be repeatable having a precision uncertainty on the mean of approximately 0.5 % (Wagner *et al.* 2015*b*).

High-frequency PSP from Innovative Scientific Solutions, Inc. (ISSI) was sprayed onto the entire floor of the cavity (figure 1*b*). The paint used a platinum-tetra-fluorophenyl-porphyrin (PtTFPP) luminophore added on top of a porous ceramic binder. The porosity of the base layer added surface area to allow fast response of the paint. Specifically, a frequency response of approximately 10 kHz (Casper *et al.* 2017) was possible making it essentially time resolved for the resonance dynamics characterized herein. Moreover, the paint measurements have been shown to be in good agreement with pressure sensors (Casper *et al.* 2017).

Three ISSI LM2XX-DM-400 water-cooled light arrays excited the PSP over a range of approximately 400 nm \pm 30 nm. The lights were placed in the plenum surrounding the tunnel to locate them as close as possible to the cavity. The majority of emission from the paint was centred at a wavelength near 650 nm. A high-speed camera (Photron SA-Z) fitted with a 50 mm lens was used to acquire full-view images of the cavity at framing rates of 20 kHz. At 20 kHz, the camera had a resolution of 1024 pixel \times 1024 pixel with a quantum efficiency of 49% at 630 nm. A 590 nm longpass filter removed the 400 nm excitation light and 20000 images were acquired and used for spectral analysis.

Raw images were post-processed using an adaptive Wiener filter in the Mathworks software package Matlab. This acted as a lowpass filter to reduce additive noise in the images. Static calibrations were used to convert the intensity ratios to pressure ratios, yielding the unsteady pressure fluctuations within the cavity. Additional details on the calibration and TR-PSP set-up can be found in Casper *et al.* (2017).

2.4. Time-resolved particle image velocimetry

Seeding was provided by a smoke generator that delivered a large quantity of mineral oil particles to the tunnel stagnation chamber. Measurement of the particle response across a shock wave has shown the particle size to be approximately 0.8 μ m, which returns Stokes numbers of approximately 0.04 based on *a posteriori* measurements of typical cavity shear layer eddies. The particles, therefore, rapidly attain the local velocity even in the presence of velocity gradients in the shear layer (Samimy & Lele 1991). Visual inspection of the PIV images showed adequate seeding in all portions of the cavity, including the recirculation region.

A quasi-continuous burst-mode laser (QuasiModo-1000, Spectral Energies, LLC) with both diode and flashlamp Nd:YAG amplifiers was used to produce a high energy pulse train at a wavelength of 532 nm. The laser design is based on a master oscillator power amplifier architecture and is similar to previously reported pulse-burst

lasers (Slipchenko *et al.* 2012, 2014). As detailed in Beresh *et al.* (2015*a*), the laser is capable of producing doublets with the variable inter-pulse spacing time and at varying pulse frequencies (repetition rates). Here, the laser was operated at 37.5 kHz with doublets separated in time by 3 μ s. A 10.2 ms burst was generated every 8 s and the energy per pulse was approximately 27 mJ. Five bursts per wind tunnel run brought the total number of independent bursts (independent time series) to 245 and the total number of image pairs to approximately 94000. The beam was shaped into a laser sheet covering the streamwise length of the cavity having a spanwise thickness of 1.5 mm and was located at the spanwise centre of the cavity.

Images were acquired using two high-speed cameras (Photron SA-Zs). Each camera was operated at 75 kHz at a resolution of 640 pixel \times 360 pixel. The two pulses in a doublet were frame straddled around the inter-frame transfer time of the cameras to produce separate images for cross-correlation analysis. As a result, the PIV repetition rate was half that of the framing rate, or 37.5 kHz. In a manner similar to Beresh *et al.* (2015*a*), the two cameras were angled downward at 12° to peer into the cavity and placed side by side to extend the field of view in the streamwise direction. The views were joined using a linear interpolation method to give the vectors equidistant spacing across the entire field. The combined field of view was approximately 120 \times 30 mm. The cameras imaged approximately 50% of the cavity depth in contrast to the 100% depth covered in previous 10 Hz stereoscopic PIV experiments, which utilized a mirror in the test section plenum to increase optical access (Beresh *et al.* 2015*c*). Resonance dynamics as determined from the TR-PIV is compared to that given using this 10 Hz dataset.

The image pairs were processed using the LaVision PIV software package DaVis v8.2 to a final interrogation window size of 24 pixel \times 24 pixel in the same fashion as described in Beresh *et al.* (2015*a*).

3. Analysis methods

3.1. Power spectral densities

Power spectral densities of the pressure and velocity data were computed in Matlab using the Welch windowing algorithm (pwelch function). A window overlap of 50% was used. PSDs based on the pressure sensor data utilized a window length Δf of 10 Hz, whereas the window lengths in the TR-PSP and TR-PIV PSDs were 50 Hz and 100 Hz, respectively.

For the velocity data, the square root of the PSD is reported, which is normalized by the free-stream velocity U_{∞} to yield units of Hz^{-0.5}. In the case of the pressure data, frequency spectra are reported in decibels as sound pressure level (SPL):

$$SPL(f) = 10 \log_{10} \left[\frac{PSD_p(f) \Delta f_{ref}}{p_{ref}^2} \right], \qquad (3.1)$$

where PSD_p represents the power spectral density of pressure, p_{ref} is a reference pressure set to the standard 20 µPa and Δf_{ref} is a reference frequency set here to 1 Hz. Spatial distributions of the PSD amplitude at frequencies corresponding to cavity tones are shown herein to highlight the spatial variation associated with resonance in both the pressure (TR-PSP) and velocity (TR-PIV) fields.

3.2. Bandpass filtering of TR-PIV

The time-resolved velocity fields were bandpass filtered in the time domain about each of the first three cavity tone frequencies using a Butterworth filter based on the

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algorithm in Russell (2006). This algorithm uses a cascade of first-order filters and hence is precise for narrow-band applications. The filter results in very little signal attenuation at the desired frequency band and is designed to produce zero phase shift. A bandwidth of 100 Hz was chosen to narrowly focus on the resonance frequencies.

3.3. Conditioning of the TR-PIV data (active mode definition)

The bandpass-filtered velocities (v_{fm} , where *m* is integer mode number) are used to condition the TR-PIV data as follows. The standard deviation of the wall-normal filtered velocity was calculated for each of the 245 TR-PIV series at a location observed to contain prominent resonance peaks in the TR-PIV PSDs. As described in the results, in the case of modes two and three, a location of x = 3.6D and y = 0.1D was chosen to satisfy this criterion. In contrast, turbulent fluctuations in the vicinity of mode one overwhelmed any resonance peak at this location. Thus, for mode one a point above the cavity and more upstream (x = 2D and y = 0.6D) was chosen where mode one did exhibit a prominent PSD peak. An 'active' mode is defined to occur when at the conditioning location, the standard deviation of v_{fm} is in the top forty per cent in comparison to all 245 time sequences. This leaves about 100 cases of active modes for subsequent spectral and cross-correlation analysis. Unless otherwise specified only active cases were used in the analysis.

3.4. Continuous wavelet transform

Joint-time frequency analysis (JTFA) on the pressure sensor and time-resolved velocity data was conducted using a continuous wavelet transform (CWT). This technique shows the frequency content of the disturbances as a function of time. The transform provides good time resolution for identifying intermittent mode switching within the cavity. As in Kegerise *et al.* (2004), the transform utilized a Morlet mother wavelet. The mathematical formulation to provide the CWT coefficients is described in detail in Torrence & Compo (1998). Moreover, the CWT coefficients were computed using a Matlab script that utilized an algorithm based on the discrete Fourier transform. The script was also provided by Torrence & Compo (1998). Finally, results are presented as an intermittency function:

$$I_n(s) = \frac{|W_n(s)|^2}{\langle |W_n(s)|^2 \rangle_s},$$
(3.2)

where $W_n(s)$ is the CWT coefficient. This serves to normalize the transform with respect to the average spectrum over the wavelet scale (frequency range) of interest.

The wavelet transform was applied to both the pressure sensor data and the TR-PIV data with two primary purposes: (i) to investigate whether the mode switching observed in the pressure signals is also evident in simultaneous PIV fields and (ii) to conditionally average the previous 10 Hz dataset of Beresh *et al.* (2015*c*).

3.5. Determination of coherent flow structure using 10 Hz PIV

Coherent flow structures during resonance, based on the previous 10 Hz PIV data (Beresh *et al.* 2015*b*), were found by conditionally averaging the velocity fields. The fore wall pressure sensor was used in this process since turbulent fluctuations are lower in the upstream portion of the cavity. The wavelet transform was applied to the pressure sensor time traces. The traces were then effectively bandpass filtered by

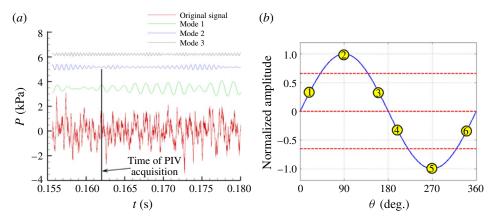


FIGURE 2. Bandpass filtering of pressure sensor data: (a) unfiltered and bandpass-filtered traces of the fore wall pressure sensor in the vicinity of PIV acquisition time and (b) illustration of the binning regions used to produce phase-averaged vector fields.

setting wavelet transform coefficients other than those at the cavity tone frequencies to zero and then computing an inverse wavelet transform. Importantly, this served to filter the pressure data without a time delay to maintain synchronization with the 10 Hz PIV data. An example of bandpass-filtered pressures using this methodology is shown in figure 2(a), where each trace is offset in pressure for clarity. The filtering is clearly able to reveal modulation of modal amplitude.

The bandpass-filtered pressures were used to conditionally average the simultaneously acquired 10 Hz PIV data as follows. For a given mode, the amplitude of the filtered trace at the PIV acquisition time was first determined. The local maximum amplitude in the 25 ms surrounding the PIV time was also found. If the amplitude at the PIV time was at least 50% of the local maximum then the mode was considered active and the PIV vector field was used for conditional averaging. As illustrated in figure 2(b), the waveforms were binned into six phases. In the current paper, only conditionally averaged fields in the vicinity of the peak (region 2 at 90°) and the trough (region 5 at 270°) are shown for brevity.

Approximately 10% (300) of the 10 Hz vector fields were used for each phase-averaged result. Finally, the mean wall-normal velocity field (V) was subtracted from the phase-averaged field ($V_{\theta m}$) to reveal the coherent flow structure (Hussain 1986) associated with resonance tones.

3.6. Convective and phase velocities based on cross-correlations

Cross-correlation coefficients between time sequences of wall-normal velocity v at points separated by fixed streamwise distance Δx and variable time delay Δt were calculated.

The time delay at which the correlation was at a maximum and the known streamwise separation were used to compute the streamwise convection velocity U_c between the points. The analysis was performed in a row-by-row fashion, at every point within the row, thereby revealing spatial distributions of streamwise convective velocity. This methodology captures U_c over the entire measurement domain, although the data are essentially smoothed by a moving-average, lowpass filter.

The cross-correlation analysis was also performed on the bandpass-filtered data corresponding to cavity modes one-three $(v_{f1}-v_{f3})$. In this case, the velocity returned

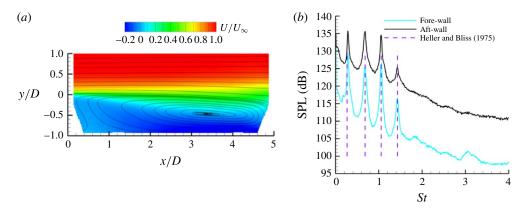


FIGURE 3. Basic cavity flow properties: (a) mean streamwise velocity U contours with superposed streamlines (Beresh *et al.* 2015*c*) and (b) SPL at the fore wall and aft wall of the cavity. Dashed lines represent frequencies predicted by the modified Rossiter relation of Heller & Bliss (1975).

is by definition the phase velocity U_{pm} , where U_p is phase velocity and *m* is integer mode number.

The streamwise separation Δx between measurements locations was varied between ten and twenty vector spacings (0.45D–0.90D). A spacing less than approximately ten vectors was generally too low for the temporal resolution of the measurements, whereas a spacing greater than approximately twenty vectors overly smoothed the convective velocity fields. Comparisons between calculations using separations within this range typically fell within the precision uncertainty of the measurements quantified in the results. Hence, the smaller spacing of 0.45D was utilized to maximize spatial resolution with one exception being the unfiltered case where a larger spacing of 0.68D was used to reduce a non-physical jump in U_c observed at the interface between the two side-by-side measurements.

4. Results and discussion

4.1. Basic cavity flow properties

Some basic cavity characteristics and definitions are first given here prior to discussing resonance dynamics. The mean flow field over nearly the entire height and length of the cavity, obtained from 10 Hz, stereoscopic PIV measurements (Beresh *et al.* 2015*c*), is shown in figure 3(a).

The flow in figure 3(a) demonstrates the expected behaviour of a mixing layer that grows with streamwise distance (Murray *et al.* 2009). A single recirculation region centred at approximately x=3.4D is also evident. Below the cavity lip line (y=0), the flow is turned downward in the mixing layer. Eventually the mixing layer impinges on the aft wall to produce the acoustical disturbances responsible for the resonance feedback loop.

Sound pressure level SPL spectra obtained at the fore wall and aft wall of the cavity are shown in figure 3(b). The spectra were calculated using the pressure transducers. At Mach 0.94, the first three Rossiter modes have similar amplitude. A fourth cavity tone is also observed, but discussion of this mode is omitted from the paper as its behaviour is consistent with modes two and three. The expected frequencies of the

т	f_m (kHz)	St_m	ω_m
1	0.69	0.28	0.07
2	1.67	0.68	0.16
3	2.58	1.04	0.25
4	3.50	1.43	0.34
TABLE 2. Cavity tone frequencies.			

cavity tones can be given with the Rossiter (1964) relation modified by Heller & Bliss (1975):

$$St_{m} = \frac{f_{m}L}{U_{\infty}} = \frac{m - \alpha}{\left[M_{\infty}\left(1 + \frac{\gamma - 1}{2}M_{\infty}\right)^{-1/2} + \frac{1}{\kappa}\right]},$$
(4.1)

where f is frequency, U_{∞} is free-stream velocity and γ is the ratio of specific heats. This semi-empirical equation is based on the assumption that vortical disturbances associated with resonance propagate downstream at a constant convection velocity of $\kappa \times U_{\infty}$. When a disturbance reaches the aft wall, there is a phase lag α (in fractions of a wavelength), which occurs before an upstream-propagating disturbance is formed. According to this view, the upstream-propagating disturbance then moves at the speed of sound based on the stagnation temperature a_0 . Throughout the literature, commonly used constants for phase lag and convective velocity are 0.25 and 0.57, respectively (Tracy & Plentovich 1997; Murray *et al.* 2009). Although the semi-empirical relation is a simplification of the cavity resonance physics (e.g. it is known that disturbances in the shear layer do not move at constant velocity (Larchevêque *et al.* 2004; Hassan *et al.* 2007)), it often provides good predictions of cavity tone frequencies. The current work is no exception. For instance, as shown in figure 3(b), using the common values for α and κ gives agreement between the measured and predicted frequencies to within 2%.

Finally, for linear stability considerations presented later, it is also useful to normalize the modal frequencies as circular shear layer forcing frequency:

$$\omega_m = \frac{2\pi f_m \delta_{\Omega,0}}{U_\infty},\tag{4.2}$$

where $\delta_{\Omega,0}$ is the initial vorticity thickness at the cavity entrance, which was measured to be 4.8 mm (Beresh *et al.* 2016). The cavity frequencies f_m along with their non-dimensional counterparts St_m and ω_m are summarized in table 2.

4.2. Modulations of cavity pressure and velocity

The dynamic events during cavity flows do not occur uniformly with time. Rather, modal amplitudes are known to vary temporally through mode switching. Examples of this tonal modulation are shown in figure 4, which gives the time-varying frequency content of pressure measured at the fore wall during two separate wind tunnel experiments. Contour levels are CWT coefficients normalized according to (3.2). The average used for normalization in each case is that over the time and frequency range corresponding to each panel. Dashed lines appear in the figures to mark the first three cavity mode frequencies $(f_1, f_2, \text{ and } f_3)$. Modes two and three exhibit

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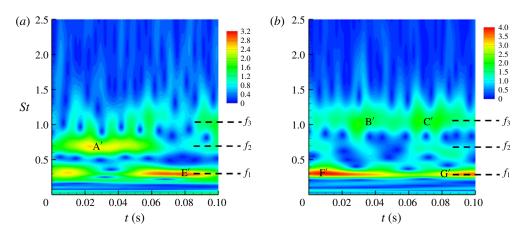


FIGURE 4. Time-frequency content of fore wall pressure from two separate Mach 0.94 time sequences.

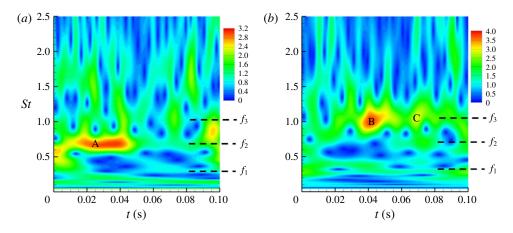


FIGURE 5. Time-frequency content of wall-normal velocity at x = 3.6D and y = 0.1D: (a) same time sequence as figure 4(a) and (b) same time sequence as figure 4(b).

substantial differences between the two cases. For example, mode two is more active in figure 4(a). Specifically, on average, the CWT amplitude at mode two is 1.9 times greater than that in figure 4(b). On the other hand, mode three is more active in figure 4(b) and has a CWT amplitude 1.8 times greater than that in figure 4(a).

The corresponding variation of frequency content in the time-resolved velocity data is shown in figure 5. The JTFA was performed on the wall-normal velocity component v at a location of x = 3.6D and y = 0.1D where modes two and three exhibited prominent peaks at in the TR-PIV PSDs (subsequently shown in § 4.7). Note that as discussed in § 3.3, this is the same location used to condition the TR-PIV data to define active modes. The left panel corresponds to the same time sequence of figure 4(a) and the right panel corresponds to that in figure 4(b). Similar to the pressure frequency content, the third cavity tone is observed to be much more active in the right panel than in the left (the amplitude at mode three is two times greater than that in figure 5a). Also, like the pressure data, mode two contains greater fluctuations in the left panel (the amplitude at mode two is 1.8 times greater than that

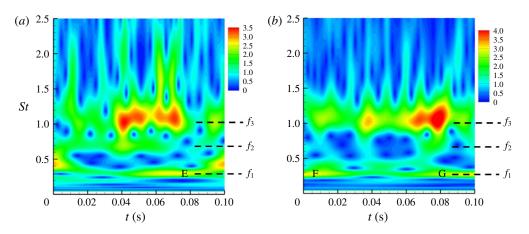


FIGURE 6. Time-frequency content of wall-normal velocity at x = 2.0D and y = 0.6D: (a) same time sequence as figure 4(a) and (b) same time sequence as figure 4(b).

in figure 5b). In both figures 4(a) and in 5(a), mode two exhibits a peak around a time of 0.003 s (labels A and A'). Though less pronounced, some correlation between figures 4(b) and 5(b) can also be seen for the third cavity mode (B-B' and C-C').

On the other hand, the correlation between the pressure and velocity figures at f_1 is more limited. At mode one, turbulent fluctuations overwhelm resonance at this particular velocity location as is detailed subsequently in § 4.7. A clearer correlation between the pressure and velocity fields can be observed if instead the location of x = 2.0D and y = 0.6D is chosen. At this location, mode one exhibits a prominent peak in the TR-PIV PSDs (shown later in § 4.7), which is the reason this location was also used to condition the TR-PIV data. Normalized CWT coefficients at x = 2.0D and y = 0.6D are shown in figure 6, where a stronger similarity to the mode one pressure fluctuations (label E in figure 6*a*) is also present in the pressure fluctuations (label E in figure 6*a*) is also present in the pressure fluctuations (label E in figure 4*a*). Similarly, the mode one peaks in figure 6*(b)* (labels F and G) occur at similar times to those in figure 4*(b)* (F' and G').

4.3. Coherent structures based on 10 Hz PIV

Coherent flow structures during resonance, based on the previous 10 Hz PIV data of Beresh *et al.* (2015*c*), are first briefly presented. The primary purpose of this is to ensure that coherent structures returned using a more conventional process (Hussain 1986) are similar to those given using the bandpass-filtered TR-PIV.

Coherent wall-normal velocity fields $(V_{\theta} - V)$ corresponding to resonance modes two and three are shown in figure 7. As expected, moving from the case when the modal pressure is at a peak $(\theta = 90^{\circ})$ to when it is at a trough $(\theta = 270^{\circ})$ results in a velocity sign inversion. The coherent velocities show a spatial modulation in structure size with streamwise distance. For instance, the mode two fields show a significant growth in coherent structures to occur in the vicinity of x = 3D, while a decrease in structure size occurs in the aft end of the cavity for x > 4D. Modulation is also seen for mode three, particularly at $\theta_3 = 270^{\circ}$, where structure size increases in the region 1D < x < 2D and in the region 3D < x < 4D. These observations are explained subsequently using the TR-PIV and TR-PSP data.

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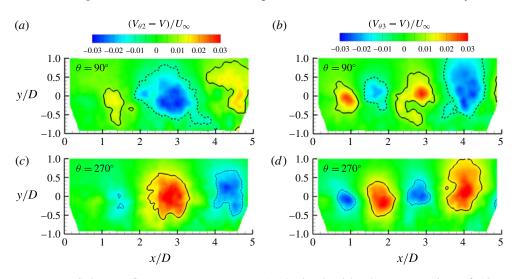


FIGURE 7. Coherent flow structures $(V_{\theta} - V)$ obtained with phase averaging of 10 Hz PIV data: mode two (a,c) and mode three (b,d). Solid and dashed lines correspond to $0.017U_{\infty}$ and $-0.017U_{\infty}$, respectively.

4.4. Resonant flow structure using TR-PIV

4.4.1. Bandpass-filtered sequences

To characterize resonance dynamics, the TR-PIV data were bandpass filtered about the modal frequencies. This allows for the contributions of each cavity mode to be separated and analysed. As discussed subsequently, modes two and three show behaviour more consistent with the classical view (Rossiter 1964) that resonance occurs in part through the downstream propagation of vortical disturbances. Thus, these modes are described prior to mode one.

An example of flow structure during resonance is shown in figure 8, which gives the bandpass-filtered, wall-normal velocity v_{f2} (*a,c,e,g,i,k*) and streamwise velocity u_{f2} (*b,d,f,h,j,l*) fields corresponding to a cycle of mode two (St = 0.68) lasting for a period T_2 . The figure was generated from a single experiment, namely the same as in figures 4(*a*) and 5(*a*) when mode two was active and is representative of other mode two cycles. The wall-normal velocities associated with mode two reveal a clear convection of organized structure as time increases. At t = 0, a region of organized positive vertical velocity appears near x/D = 1 (label A). With continuing time, this structure convects downstream reaching a position near x/D = 3.4 at $t = 0.87T_2$. Upon completion of a full cycle at T_2 (not shown) it is apparent that the upstream region of positive vertical velocity (A) moves downstream to become the larger region of positive vertical velocity (A'). Similar observations are seen in the regions of negative vertical velocities (B and B').

The streamwise modal velocities exhibit a more complex flow structure than the wall normal. A region of negative velocity tends to reside below a region of higher velocity and *vice versa*, an observation consistent with propagating vortical structures. Regardless, similar to the wall-normal component, an overall downstream convection of negative (C and C') and positive velocities (D and D') is also observed in the streamwise velocities.

Bandpass-filtered velocities corresponding to a mode three cycle (St = 1.04) of resonance having period T_3 are displayed in figure 9. The figure was generated from

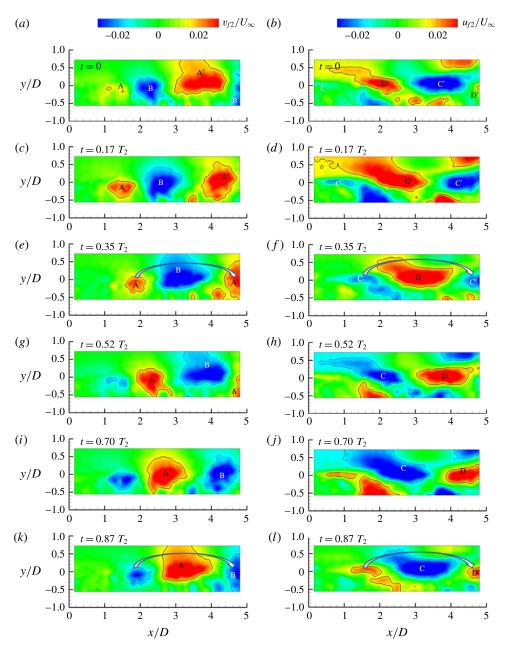
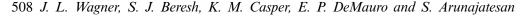


FIGURE 8. Mode two (St = 0.68) bandpass-filtered velocities: wall normal (a,c,e,g,i,k) and streamwise (b,d,f,h,j,l). Solid and dashed lines correspond to $0.017U_{\infty}$ and $-0.017U_{\infty}$, respectively.

the same time sequence as in figures 4(b) and 5(b) when mode three was active and is representative of other mode three cycles. Like mode two, an obvious convection of wall-normal flow structures is seen with increasing time, though as expected, the structures are smaller and more closely spaced, consistent with a lower wavelength. Once again, in the streamwise case, regions of opposite signed velocity appear under



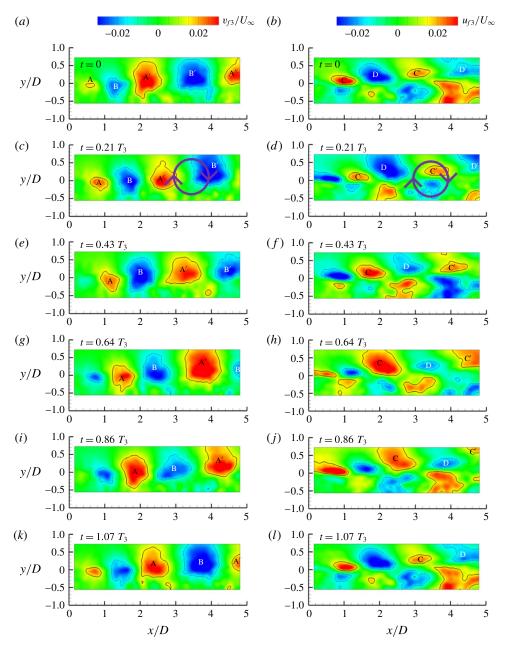


FIGURE 9. Mode three (St = 1.04) bandpass-filtered velocities: wall normal (a,c,e,g,i,k) and streamwise (b,d,f,h,j,l). Solid and dashed lines correspond to $0.017U_{\infty}$ and $-0.017U_{\infty}$, respectively.

the structures propagating in the shear layer. Again this observation is related to the fact that resonance occurs through the propagation of vortices as is emphasized with the annotations in the $t = 0.21T_3$ panels.

As the structures propagate downstream they reside at higher wall-normal locations within the cavity. A similar upward trajectory of resonant flow structures has been

observed in several incompressible (Lee, Seena & Sung 2010; Seena & Sung 2013) and compressible (Li, Nonomura & Fujii 2013) numerical studies. A possible explanation for this observation may be related to the fact that the thickness of turbulent mixing layer grows with downstream distance. If turbulent fluctuations serve to disrupt the coherent vortices associated with resonance, then the height at which vortices are less susceptible to disruption would continually increase with downstream distance as the turbulent mixing layer grows. Thus, a structure at a greater y/D may be statistically more likely to remain intact with downstream propagation.

Inspection of the mode two and mode three bandpass-filtered sequences reveals a modulation of structure size with streamwise distance. This is summarized subsequently with a more rigorous approach that calculates structure size as a function of streamwise distance based on phase-averaged fields of the TR-PIV data.

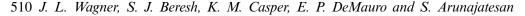
Bandpass-filtered velocity fields corresponding to cavity mode one (St = 0.28) are displayed in figure 10. The time sequence was obtained during the same experiment as in figure 4(*b*), when mode one was active. Like modes two and three, a convection of wall-normal flow structure is evident. As time progresses through the full cycle period T_1 , the regions of positive (A) and negative (B) wall-normal velocity move downstream. A major difference with respect to the two higher-frequency modes, however, is that this convection takes place at a lower wall-normal location. In fact, comparison to the mean flow field in figure 3 suggests that wall-normal flow structures are moving through a region in the shear layer more influenced by recirculation.

A convection of resonant flow structure is also seen in the streamwise filtered velocity u_{f1} . As time progresses, regions of negative (C) and positive (D) streamwise velocity propagate downstream once again lower in the cavity. In comparison to the wall-normal modal velocities, however, the streamwise velocities exhibit larger variations above y = 0. For instance, at $t = 0.20T_1$, a large region of positive velocity region can also be seen at $t = 0.60T_1$. These regions were observed to occur with downstream propagation beginning at the upstream limit of the measurement domain. The velocity fluctuations are perhaps related to a large-scale flapping of the shear layer that has been observed in several incompressible (Rockwell & Knisely 1978; Knisely & Rockwell 1982; Najm & Ghoneim 1991; Lin & Rockwell 2001; Liu & Katz 2013) and compressible (Murray *et al.* 2009) cavity flows. This flapping has been suggested to be associated with low-frequency dynamics in the recirculation region.

An additional look at mode one flow structure is given figure 11 where coherent streamwise velocity fields $(U_{\theta 1} - U)$ are shown. The fields shown are 180° out of phase corresponding to instances where mode one is a peak with respect to the upstream pressure sensor (90°) and when it is a trough (270°). Similar to the bandpass-filtered sequence in figure 10, mode one is seen to occur with coherent motion over a significant streamwise extent of the cavity length as well as in the recirculation region.

4.4.2. Phase-averaged bandpass-filtered TR-PIV fields

The bandpass-filtered TR-PIV data at modes two and three were phase averaged to resolve the resonant flow structure in a more statistical fashion. In this case, the averaging was achieved using the TR-PIV data alone without comparison to the pressure sensor data. The signal location for the phase averaging was the same as that used for conditioning, namely x = 3.6D, y = 0.1D. Phase-averaged, bandpass-filtered velocity fields corresponding to modes two and three are shown in figure 12. A single



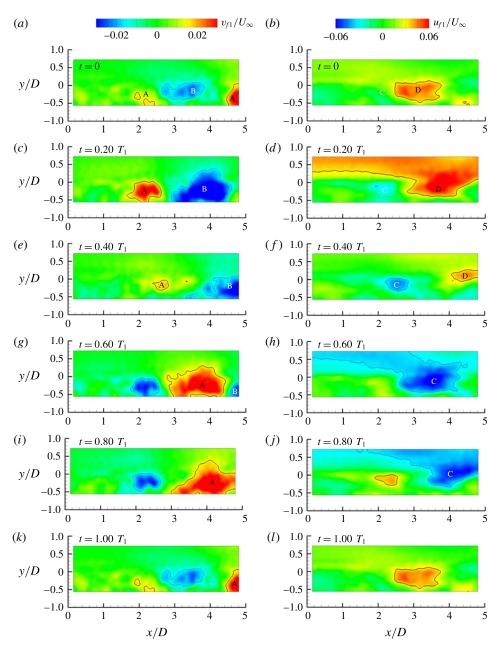


FIGURE 10. Mode one bandpass-filtered velocities: wall normal (a,c,e,g,i,k) and streamwise (b,d,f,h,j,l). Solid and dashed lines correspond to $0.017U_{\infty}$ and $-0.017U_{\infty}$ in the wall-normal panels and $0.034U_{\infty}$ and $-0.034U_{\infty}$ in the streamwise panels.

phase (or time) is shown for each mode. The phase was chosen for each mode such that the fields qualitatively matched those based on the 10 Hz PIV data shown in figure 7.

The fields yield similar results to the previous sequences displayed in figures 8 and 9 that were based on single experiments. In the streamwise velocities (figure 12a,b),

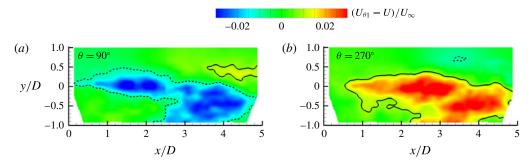


FIGURE 11. Mode one coherent flow structures $(U_{\theta 1} - U)$ obtained with phase averaging of 10 Hz PIV data. Solid and dashed lines correspond to $0.017U_{\infty}$ and $-0.017U_{\infty}$, respectively.

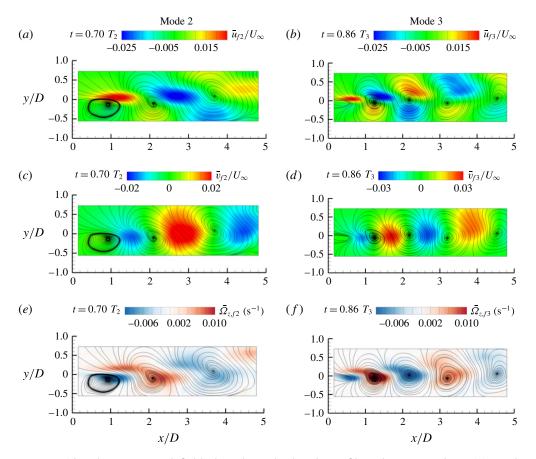


FIGURE 12. Phase-averaged fields based on the bandpass-filtered TR-PIV data: (a) mode two streamwise velocity, (b) mode three streamwise velocity, (c) mode two wall-normal velocity, (d) mode three wall-normal velocity, (e) mode two spanwise vorticity and (f) mode three spanwise vorticity. Streamlines are overlaid on each figure.

regions of oppositely signed velocity tend to reside under those higher in the shear layer, an observation which is clearer in the case of mode three (figure 12b). In the wall-normal velocity fields (figure 12c,d), large structures with alternating sign are

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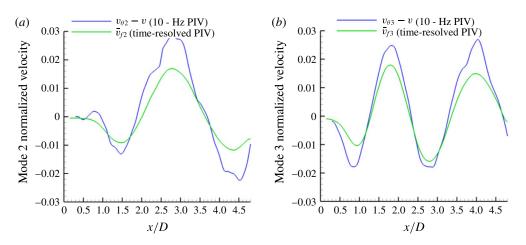


FIGURE 13. Modal wall-normal velocities, normalized by U_{∞} , obtained from phase averaging the bandpass-filtered TR-PIV data and those found by phase averaging previous 10 Hz PIV data: (a) mode two and (b) mode three.

prevalent throughout the shear layer. The streamlines overlaid on each panel show the expected result that resonance occurs with the propagation of vortical disturbances. Plots of phase-averaged spanwise vorticity (figure 12e, f), based on bandpass-filtered vorticity fields, further confirm this to be the case.

4.5. Comparison of flow structure using 10 Hz PIV and TR-PIV

The TR-PIV bandpass-filtered and the 10 Hz PIV phase-averaged fields look qualitatively similar. To facilitate quantitative comparisons, the wall-normal velocity fields in figure 12 (TR-PIV) and in figure 7 (10 Hz PIV) were averaged over the wall-normal coordinate as is shown in figure 13. In the case of both mode two (figure 13a) and mode three (figure 13b) the independent methods produce excellent agreement in terms of phase and wavelength indicating that either technique is well suited to produce coherent structures associated with resonance. The differences in amplitude likely stem from the different methodologies used to produce the 10 Hz PIV and TR-PIV phase averages.

An obvious advantage that the TR-PIV gives in comparison to the 10 Hz PIV is substantially improved temporal resolution. This allows for allows for PSDs to be calculated at each available vector location as well as calculations of convective velocity fields. Such insight would be impossible with the 10 Hz data. The TR-PIV is also able to provide much more detailed information on coherent structure modulation as opposed to the 10 Hz data since many more phases can be obtained.

4.6. Modulation of coherent structure size with streamwise distance

An in-depth look at coherent structure modulation provided by the TR-PIV data is shown in figures 14 and 15. In each case, the coherent structures over an entire resonance cycle are superposed. This effectively captures the envelope of resonant flow structure size. The figures were generated using the phase-averaged, bandpass-filtered velocities described in §4.4.2. The contours are lines of constant velocity with the thresholds chosen to most effectively maximize dynamic range in the figure (i.e. show the smallest and largest structures simultaneously).

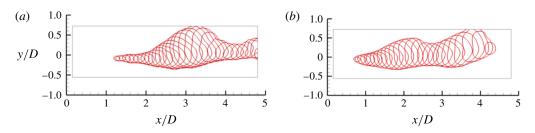


FIGURE 14. Modulation of resonant flow structure based on bandpass-filtered, phaseaveraged, wall-normal velocity fields: (a) mode two and (b) mode three. The contours in each figure are lines of constant $0.017U_{\infty}$ showing superposed structures for an entire resonance cycle.

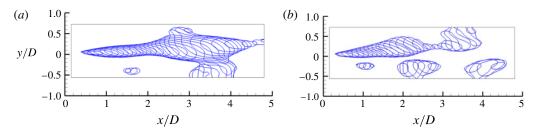


FIGURE 15. Modulation of resonant flow structure based on bandpass-filtered, phaseaveraged, streamwise velocity fields: (a) mode two and (b) mode three. The contours in each figure are lines of constant $0.015U_{\infty}$ showing superposed structures for an entire resonance cycle.

Coherent structures based on the wall-normal component of velocity are displayed in figure 14. During mode two (figure 14*a*), the coherent structure size increases to a maximum near x = 3D before reaching a local minimum near x = 4D. Mode three (figure 14*b*) also shows variation in structure size with streamwise distance, albeit over an apparently shorter wavelength. For both modes, this modulation occurs primarily in the shear layer.

Coherent structures returned using the streamwise velocity component are given in figure 15. Like the sequences of bandpass-filtered velocity, the distribution of streamwise coherent flow structure is more complex than the wall normal. Both mode two (figure 15*a*) and mode three (figure 15*b*) demonstrate significant variation in structure size within the mixing layer. Additionally, however, the structure size also varies in the region under the mixing layer. This is because the streamwise distributions are influenced by vortical structures where positive regions lie under negative regions and *vice versa*.

Structure size is quantified further in figure 16 where the height of the wall-normal velocity structures as a function of axial position is given. The size here is defined using the envelope in figure 14. The profiles returned using the streamwise velocity structures of figure 15 are very similar and are omitted here for brevity. Note that the modulation of structure size seen in figure 16 in the context of resonance dynamics is discussed subsequently along with the PSD profiles in the figure.

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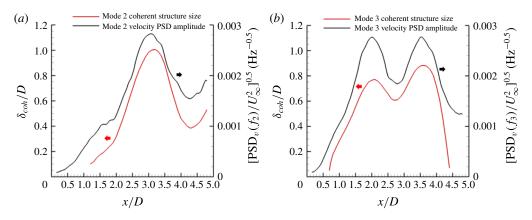


FIGURE 16. Resonant flow structure size using the wall-normal velocity fields (left ordinate) compared to the modal PSD amplitude averaged across the height of the measurement domain (right ordinate): (a) mode two and (b) mode three.

4.7. Spatial distributions of velocity PSD modal amplitude

Many of the preceding observations related to the bandpass-filtered velocity fields can be summarized concisely with spatial distributions of PSD amplitude at the given cavity mode frequencies. Along these lines, PSD amplitude distributions of wall-normal and streamwise velocity at frequencies corresponding to cavity modes one through three (i.e. $PSD_v(f_m)$ and $PSD_u(f_m)$) are shown in figure 17. The contours display the square root of PSD amplitude as to maintain units of normalized velocity.

In the case of mode one (figure 17a,b), the PSD peaks lie near the recirculation region (figure 3a). As the cavity mode number increases to two and three, the extent of resonance activity grows to reach higher portions within the cavity mixing layer. Such an observation is consistent with the bandpass-filtered results, which in general showed that resonant flow structures propagated at higher wall-normal locations as the mode number was increased.

As the mode number increases, the extent of modal activity not only moves upward within the cavity, but also upstream. This is especially evident in the wall-normal distributions. This observation can be explained using linear stability analysis. The overall gain describing the amplitude of an instability wave in an acoustically forced free shear layer is the product of a receptivity factor, describing the creation of the instability wave, and an exponential growth rate factor, describing subsequent growth of the instability wave. Both of these factors are frequency dependent; however, for subsonic and transonic Mach numbers, the optimal frequency is very close to the most unstable instability mode frequency (Barone 2003). For the present Mach number, the optimal non-dimensional circular forcing frequency ω (4.2) is approximately 0.4 (Barone 2003). Therefore, as displayed in table 2, when the mode frequency increases, ω_m approaches the optimal value. Recall that ω_m is based on the initial vorticity thickness within the cavity. This means that the shear layer is initially more receptive to higher-frequency instabilities, which explains why the modal activity moves upstream with mode number.

A spatial periodicity, more obviously present in the wall-normal PSD distributions, is evident. As the mode number increases, the spatial wavelength decreases accordingly. Kegerise (1999) also noted a spatial periodicity above the cavity using hot wire and optical deflectometry measurements. Moreover, Garg & Cattafesta (2001)

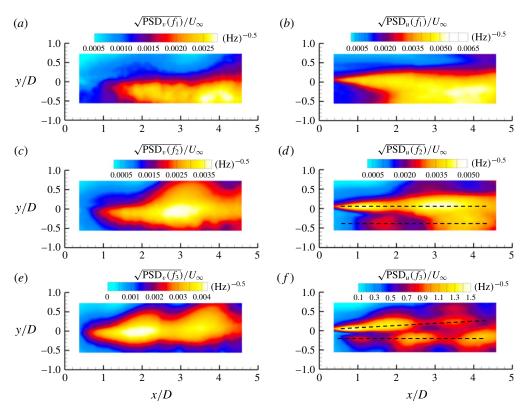


FIGURE 17. Spatial distributions showing the square root of velocity PSD amplitudes at the cavity tone frequencies normalized by U_{∞} : (a) mode one, wall normal, (b) mode one, streamwise, (c) mode two, wall normal, (d) mode two, streamwise, (e) mode three, wall normal and (f) mode three, streamwise.

observed coherent structures in the density field above a resonating cavity flow. The current work seeks to further build on these observations by explaining the mechanism for the periodicity with the TR-PIV measurements.

The streamwise PSD distributions exhibit periodicity, although in comparison to the wall-normal distributions, a much greater dependence on y is observed. In the case of both mode two (figure 17d) and mode three (figure 17f), a distinct waveform is observed near y = 0, while another is present within the cavity. Dashed lines appear in each panel to clarify the locations of the upper and lower waveforms. The mode two and mode three streamwise contour plots each indicate there to be a phase shift between the upper and lower waveforms. This is quantified further in figure 18, which shows line plots of streamwise PSD amplitude along the two different dashed lines annotated in figure 17(d). Near the aft end of the cavity there is a phase shift approximately 0.3 times the mode two wavelength. As annotated in figure, this phase shift is apparent when the peak-to-peak distance or the peak-to-trough distance is used. An intuitive, though qualitative, explanation for this observation lies in the phase delay built into the Rossiter model (4.1). The Rossiter model prescribes a phase delay between the time when a vortical structure impinges on the aft wall and when an acoustical disturbance propagates upstream, typically near a value corresponding to a phase lag of 0.25λ . The fact that a similar phase delay is observed here between the

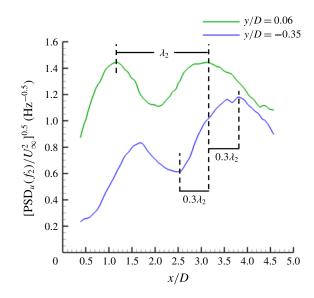


FIGURE 18. Mode two profiles of streamwise velocity PSD amplitude at two different y locations corresponding to the dashed lines in figure 17(d). As illustrated above, the mode two wavelength λ_2 is estimated using the shear layer waveform along y/D = 0.06.

upper shear layer wave and that lower within the cavity indicates that the streamwise PSD amplitude plots may simply be providing visualization of the phase lag.

The mode two and mode three PSD amplitude contours are visually similar to figures 14 and 15, which plot variation of coherent structure size. This suggests that the PSD amplitude distributions are influenced primarily by velocity fluctuations associated with the modulation of coherent flow structure. To estimate the integrated behaviour across cavity height, the wall-normal velocity PSD amplitude distributions of figure 17 were averaged across the entire wall-normal extent of the measurement region. In figure 16, line plots of the resulting averaged PSD amplitude are compared to the coherent structure size returned using the bandpass-filtered, phase-averaged TR-PIV fields. As is shown in this figure, the cavity modal PSD distributions closely follow the coherent structure size indicating they are dominated by resonant fluctuations. It should be noted, however, that turbulent contributions can play a role in the spatial distributions for weaker resonant tones (Beresh *et al.* 2017). Moreover, mode one PSD amplitude distributions are more influenced by broadband fluctuations.

To further investigate mode one acoustics, PSDs of wall-normal velocity at five points within the cavity are shown in figure 19. At locations of y = 0 or lower, peaks corresponding to mode one tend to have little prominence. In fact, in the recirculation region (x = 4D and y = -0.3D), broadband fluctuations have completely overwhelmed any evidence of a peak at f_1 . On the other hand, the points above y = 0 do show evidence of mode one in the spectra. The relevance of this point will become apparent when comparing to the mode one pressure fluctuations given by the TR-PSP.

4.8. Relationship between the modal pressure and velocity fields

Distributions of surface pressure fluctuations along the cavity floor given from the TR-PSP are presented in figure 20(a-c). The spatial distributions show the SPL value at each of the three modal frequencies (i.e. $SPL(f_m)$). Note that the lines at z/D = -1.8 and the semicircles near z/D = -2.5 are measurement artefacts where shading

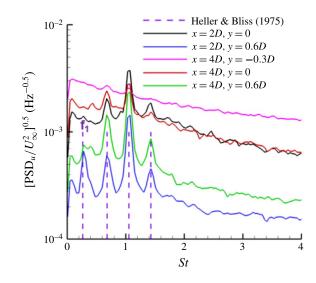


FIGURE 19. Mode one PSD spectra at five points within the cavity using all 245 time sequences.

by the wind tunnel and cavity geometry prevented adequate lighting of the cavity floor. Overall, the pressure distributions are fairly one-dimensional and show little variation in pressure fluctuations across the spanwise dimension of the cavity. Like the velocity PSD amplitude distributions, the spatial wavelengths of the SPL fields decrease with mode number. For instance, the mode one distribution exhibits a single minimum near x = 2D, whereas modes two and three have two and three local minima, respectively.

The primary purpose of the TR-PSP here is to use its excellent spatial and temporal resolution to map pressure variations as a function of streamwise distance. This allows for detailed comparisons to the TR-PIV to better understand resonance dynamics. Towards this end, the streamwise variation of PSD amplitude at the spanwise centre of the cavity is plotted for modes one (figure 20d), two (figure 20e) and three (figure 20f). Several previous cavity studies (Tracy & Plentovich 1997; Lee 2010; Flaherty *et al.* 2013; Casper *et al.* 2017) have observed similar spatial distributions, which look similar to patterns produced by standing waves.

Casper *et al.* (2017) recently showed that these spatial amplitude variations can be better explained using Rossiter's model of cavity resonance which consists of downstream-propagating shear layer disturbances and weaker upstream-propagating acoustic waves. Because the waves moving upstream and downstream have different speeds, the phase velocity is not zero and a true standing wave is not generated. However, an interference pattern is developed by the wave superposition. Importantly, since the interfering waves have the same frequency, the group velocity of the wave summation is zero. In other words, the minima and maxima do not move with time (Casper *et al.* 2017).

Of prime importance is how the velocity field is coupled to the pressure field. (Even though the velocity and pressure data were obtained on opposite walls of the wind tunnel, the incipient boundary layers and cavity acoustics are essentially identical at the two locations making for effective comparisons.) The streamwise velocity PSD amplitudes of figure 17, averaged across the wall-normal direction of the measurement, are also shown in figure 20. For modes two and three, the averaging was performed over the entire wall-normal extent of the measurement, whereas for mode one, the region above y=0 was used to avoid the influence of turbulent fluctuations (figure 19).

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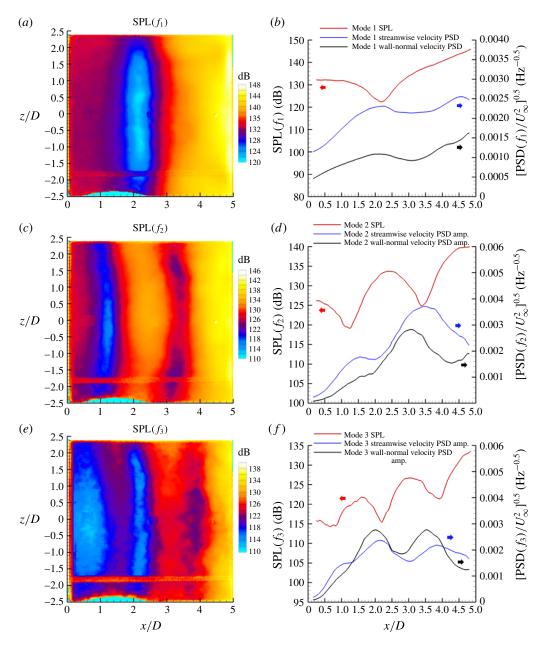


FIGURE 20. Spatial distributions of cavity floor SPL amplitudes corresponding to modes (a) one, (b) two and (c) three. Streamwise profiles of centreline SPL amplitude compared to streamwise and wall-normal velocity PSD amplitudes for modes the for modes (d) one, (e) two and (f) three.

In the case of modes two (figure 20e) and three (figure 20f), the locations of local minima in the pressure PSDs correspond to local maxima in the streamwise velocity PSDs. Similarly, local maxima in the pressure PSDs correspond to local minima in the streamwise modal velocities. Because the streamwise modal pressure distributions have fixed local minima and maxima (Casper *et al.* 2017), these observations can be

explained with analogy to a standing wave, where nodes in modal pressure correspond to antinodes in streamwise modal velocity and *vice versa*. In other words, the very same constructive and destructive interference responsible for the periodic pressure distributions is apparent in the velocity fields due to cavity acoustics.

Wall-normal modal velocity PSD amplitudes, averaged across the measurement height, are also plotted in figure 20. At modes two and three, the wall-normal distributions take on a wave-like form similar to the streamwise distributions. Since cavity resonance occurs with the propagation of vortical structures, a coupling between the streamwise and wall-normal velocity fluctuations through rotation is to be expected. Though in close proximity, local wall-normal velocity maxima no longer occur at the location of pressure minima. Rather, the wall-normal waveforms precede the streamwise waveforms in x. This is consistent with regions of upwash preceding streamwise velocity variations (figure 12).

Similar behaviour is observed at mode one where a local maximum is observed in the streamwise and wall-normal velocity amplitude profiles near the SPL minimum. The increasing velocity fluctuations at x > 3D are likely explained by increasing turbulence as the aft wall is approached (e.g. Zhuang *et al.* 2006). Likewise, a diminishing contribution of mode one resonance with increasing x was also observed in figure 19 in the spectra corresponding to y = 0.6D.

The preceding experimental observations suggest a 180-degree phase difference to exist between the modal pressure and velocity fluctuations. In conjunction with Rossiter-based models to predict pressure fluctuation distributions (Kegerise 1999; Casper *et al.* 2017), this can be used to predict velocity fluctuation distributions at resonance frequencies in the flow above the cavity.

4.9. Convective and modal propagation velocities

4.9.1. Convective velocities based on unfiltered velocities

The spatial distribution of convective velocity within the cavity using the unfiltered velocity data is displayed in figure 21(a). The convection velocity is based on correlations of wall-normal velocity since v is less influenced by upstream-propagating pressure waves than u and is thereby more likely to give a better measurement of U_c (Larchevêque *et al.* 2004). The non-physical jump seen near the centre of the figure is related to the fact that the two vector fields were combined to produce the single field.

The unfiltered convective velocity demonstrates some expected trends. At a constant streamwise station, U_c increases with y. Moreover, at constant y, the convective velocity tends to increase with increasing streamwise distance. These trends are consistent with the mean flow field (figure 3), which shows the shear layer dives downward into the cavity with increasing streamwise distance to increase the convective velocity at a given y location. The convective velocity profile at y = 0 is plotted in figure 21(b), where the precision uncertainty based on 95% confidence intervals is approximately $\pm 0.01U_{\infty}$. The convective velocity increases as x increases from 1D through 4D. Over this same region, the slope of the profile decreases until a plateau region is reached between approximately 3D and 3.8D. The convection velocity then begins to decrease for the rest of the measurement domain. As discussed in Larchevêque *et al.* (2004), the relatively constant acceleration observed in the upstream portion of the cavity corresponds to a region where the flow is predominantly two dimensional, whereas the sudden decrease in the downstream portion is associated with the aft wall.

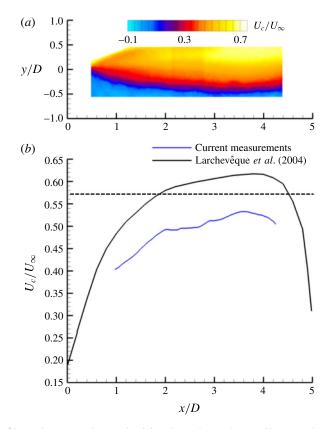


FIGURE 21. Unfiltered convective velocities based on the wall-normal component of raw velocity: (a) distribution within the cavity and (b) centreline profile compared to the simulations of Larchevêque *et al.* (2004). Dashed line corresponds to $U_c = 0.57U_{\infty}$.

These measurements are in line with previous cavity flow studies. Hassan *et al.* (2007), for example, measured similar convective velocity profiles in incompressible cavity flows, including a region of increasing slope, a plateau region, and a region of decreasing velocity following the plateau. The average convective velocities in Hassan *et al.* (2007) resided between approximately $0.40U_{\infty}$ and $0.50U_{\infty}$, similar to the values reported in the incompressible studies of Haigermoser *et al.* (2008) and Bian *et al.* (2011). In comparison, in the available measurement region, the average convective velocity predict the tone frequencies. It is possible that a new combination of constants using the lower measured U_c could instead be used, although such an investigation is beyond the scope of the current work. The simulated convective velocity profile of Larchevêque *et al.* (2004), computed at similar flow conditions ($M_{\infty} = 0.85$, L/D = 5) to those herein, is also shown in figure 21. The current measurements show similar trends, although the simulated profile has an overall higher average convective velocity of $0.52U_{\infty}$.

4.9.2. Modal propagation (phase) velocities

Modal propagation (phase) velocities (U_p) , based on the cross-correlation of bandpass-filtered, wall-normal velocity fields are shown in figure 22. The non-physical jump is no longer present, indicating the source of the discontinuity was in the

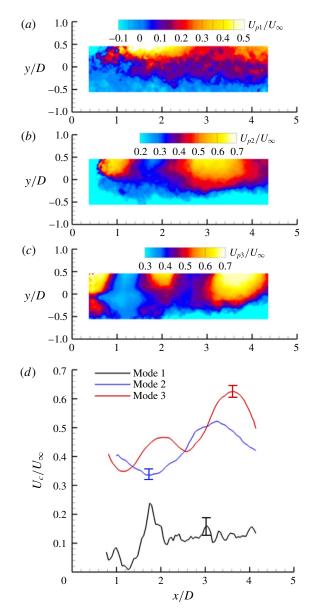


FIGURE 22. Modal phase velocities based on the wall-normal component of filtered velocity: (a) mode 1, (b) mode 2, (c) mode 3 and (d) streamwise profiles at y = 0. Error bars represent the precision uncertainty with 95% confidence intervals.

mean flow. The propagation velocity fields present a unique view of resonance dynamics in cavity flows. Unlike the measurements of unfiltered velocity data shown here in figure 21 and in other studies, a periodicity is observed when the bandpass-filtered velocities are used to compute the cross-correlations. There is a wavelength in the propagation velocity, which decreases with increasing mode number in a fashion analogous to the decreasing spatial wavelengths in the modal velocity and surface pressure fluctuations.

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Propagation velocity profiles obtained at y = 0 for all three cavity modes are plotted in figure 22(d). On average, the propagation velocity increases with cavity mode number. Specifically, the average U_p of modes one, two and three in the figure is $0.11U_{\infty}$, $0.42U_{\infty}$ and $0.47U_{\infty}$, respectively. This is likely related to the fact that as the cavity mode number increases, the coherent structures tend to reside at higher locations within the flow field reaching a higher portion of the shear layer where the mean local velocity is higher (figure 3a).

The propagation velocity distributions in figure 22(d) corresponding to modes two and three show a clear periodicity. Casper *et al.* (2017) also measured a periodic propagation velocity based on correlations of bandpass-filtered surface pressures using TR-PSP. Moreover, their wave interference model, based on Rossiter's interpretation of upstream-travelling acoustic disturbances superposed with downstream-travelling shear layer disturbances, predicted oscillations in propagation velocity. The modal propagation velocity exhibits periodicity because the superposed waves travel at different velocities. Since the propagation velocity represents motion of a resulting travelling wave at a distinct frequency, it is more appropriate to refer to the distributions in figure 22 as modal phase velocities. Here, the fact the phase velocity in the shear layer (y = 0) is positive for modes one through three indicates that the downstream-propagating disturbances overwhelm their upstream-propagating counterparts at this location.

5. Discussion of resonance mechanism elucidated by the TR-PIV

Since the seminal work of Rossiter (1964) it is has been well known that resonance in high-speed cavity flows occurs through the propagation of vortical structures within the mixing layer. Naturally, many additional studies throughout the years have used flow visualization methods to follow motion of these large-scale coherent structures in compressible cavity flows (Kegerise 1999; Garg & Cattafesta 2001; Kegerise *et al.* 2004; Murray & Ukeiley 2007). Quantitative studies, using low-speed PIV, have also been able to track vortical structures throughout the mixing layer by using either ensemble averaging or stochastic estimation methods (Murray & Ukeiley 2007; Murray *et al.* 2011). Although impressive insight has been gleaned from these studies, a lack of temporally resolved, quantitative flow field data have precluded critical observations regarding the behaviour of coherent structures as identified in the velocity field to be placed within the context of the resonant processes. More specifically, previous work has been unable to explain how coherent structures propagate and are affected by the resonance as a function of mode number and cavity position.

The current work adds to the compressible cavity flow knowledge base through the use of spatially and temporally resolved data. Many of the key findings of the paper are effectively captured and summarized in figure 23. The figure focuses on properties derived from the wall-normal component of the TR-PIV, though, as was discussed, a similar analysis using the streamwise velocity component reaches the same conclusions. The pressure profiles demonstrate local minima and maxima consistent with the Rossiter mechanism. In other words, the pressure variations are the result of downstream-propagating vortical disturbances superposed with upstream-travelling acoustical disturbances.

At both modal frequencies of figure 23, a clear modulation of coherent structure size with streamwise location is observed because of the cavity resonance. The very same modulation is also observed in the velocity PSD amplitude profiles, indicating

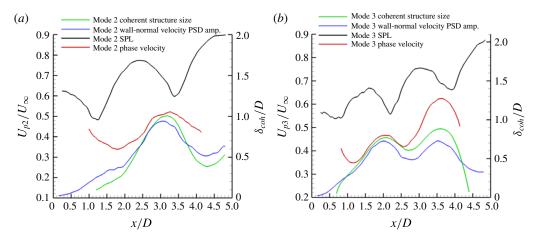


FIGURE 23. Streamwise distributions of coherent structure size (right ordinate), wallnormal velocity PSD amplitude, SPL and phase velocity (left ordinate) at (a) mode two and (b) mode three. To maintain clarity, the SPL and velocity PSD amplitude scales are not given here, although they are the same as in figure 20.

that the PSD profiles are most influenced by the resonance fluctuations within the cavity. Of course such an observation is dependent on the strength of a given resonance tone and on any conditioning algorithm used to select prominent resonance cycles. Both the velocity PSD amplitude traces and the coherent structure size profiles are approximately 180° out of phase with the pressure profiles. This phase shift is analogous to that in a standing wave resonator.

The phase velocity profiles shed further light on the resonance dynamics. Regions of maximum phase velocity correspond to regions of maximum constructive interference where the coherent structure size is the largest. Importantly, these measurements and observations can be used to estimate the velocity field fluctuations in the compressible flow over a cavity. Such an understanding is a step towards predicting the localized structural response within aircraft bay flows.

6. Conclusions

The unsteady resonance characteristics of Mach 0.94 flow over a cavity were explored with TR-PIV using a pulse-burst laser and surface pressure measurements using TR-PSP. These novel, time-resolved data allowed the cavity dynamics to be studied without phase or ensemble averaging. Observations of mode switching in the velocity field simultaneous with the pressure field were observed for the first time. The ability to detect mode switching, however, was found to be dependent on location within the cavity and mode number.

The resonance dynamics of the first three cavity modes was probed with bandpass filtering, which was able to reveal the coherent flow structures associated with each mode. The first Rossiter mode appeared to occur through large-scale motions in the shear layer as well as in recirculation region. The second and third modes contained coherent structures consistent with convecting vortical disturbances in the shear layer.

The TR-PSP data revealed an expected streamwise periodicity in the modal pressure fluctuations attributable to the downstream-propagating vortical disturbances and upstream-travelling acoustic waves known to occur with the Rossiter mechanism. Importantly, a strong, periodic variation of coherent structure size was observed to occur with streamwise distance. The coherent structure modulation occurred 180° out of phase with respect to the surface pressure fluctuations, similar to a standing wave. This result, along with Rossiter-based models (e.g. Kegerise 1999; Casper *et al.* 2017) can be used to predict modal velocity fluctuations in the compressible flow above a cavity.

The TR-PIV velocity fields were able to provide new details regarding the spatial distribution of resonance activity as a function of mode number. As the mode number increased from one through three, the resonance activity moved from a region downstream coinciding with the recirculation region to areas further upstream in the shear layer near the leading edge of the cavity. This observation can be explained using linear stability analysis, which suggests that the optimal, non-dimensional circular forcing frequency makes the initial portions of the shear layer more susceptible to instabilities at higher modal frequencies. The resonance activity also moved upward, away from the cavity floor, with increasing mode number.

Modal phase velocity fields were found with cross-correlations of the bandpassfiltered velocities. The phase velocities provided additional details on cavity resonance dynamics. The phase velocities exhibited peaks in regions where coherent structures were the largest corresponding to locations where constructive interference of downstream- and upstream-propagating waves was at a maximum. Furthermore, the phase velocity was found to decrease with decreasing mode number. This was caused by the fact the modal activity resided at locations lower in the cavity as the mode frequency was decreased. In other words, at lower locations, the local mean flow velocity is diminished explaining the decreased phase velocities.

Together the TR-PIV and TR-PSP measurements enabled the exploration of the resonant nature of compressible cavity flows. These diagnostics provided unique details not only on the distribution of modal activity throughout the cavity, but also new understanding of the resonance mechanism as observed in the velocity field. Such information is critical to determine structural response in the high-speed flow over an aircraft bay.

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REFERENCES

- BASLEY, J., PASTUR, L. R., LUSSEYRAN, F., FAURE, T. M. & DELPRAT, N. 2011 Experimental investigation of global structures in an incompressible cavity flow using time-resolved PIV. *Exp. Fluids* 50 (4), 905–918.
- BARONE, M. 2003 Receptivity of compressible mixing layers. PhD thesis, Department of Aeronautics and Astronautics, Stanford University, Stanford, USA.
- BARONE, M. & ARUNAJATESAN, S. 2016 Pressure loadings in rectangular cavity with and without a captive store. J. Aircraft 53 (4), 982–991.
- BERESH, S. J., KEARNEY, S. P., WAGNER, J. L., GUILDENBECHER, D. G., HENFLING, J. F., SPILLERS, R. W., PRUETT, B. O. M., JIANG, N., SLIPCHENKO, M., MANCE, J. et al. 2015a Pulse-burst PIV in a high-speed wind tunnel. *Meas. Sci. Technol.* 26 (9), 095305.

- BERESH, S. J., WAGNER, J. L., PRUETT, B. O. M., HENFLING, J. F. & SPILLERS, R. W. 2015b Supersonic flow over a finite-width rectangular cavity. *AIAA J.* **53** (2), 296–310.
- BERESH, S. J., WAGNER, J. L., PRUETT, B. O. M., HENFLING, J. F. & SPILLERS, R. W. 2015c Width effects in transonic flow over a rectangular cavity. *AIAA J.* 53 (12), 3831–3834.
- BERESH, S. J., WAGNER, J. L. & CASPER, K. M. 2016 Compressibility effects in the shear layer over a rectangular cavity. J. Fluid Mech. 808 (12), 116–152.
- BERESH, S. J., WAGNER, J. L., CASPER, K. M., DEMAURO, E. P., HENFLING, J. F. & SPILLERS, R. W. 2017 Spatial distribution of resonance in the velocity field for transonic flow over a rectangular cavity. AIAA J. https://arc.aiaa.org/doi/10.2514/1.J056106.
- BIAN, S., DRISCOLL, J. F., ELBING, B. R. & CECCIO, S. L. 2011 Time resolved flow-field measurements of a turbulent mixing layer over a rectangular cavity. *Exp. Fluids* 51 (1), 51–63.
- BRÈS, G. A. & COLONIUS, T. 2008 Three-dimensional instabilities in compressible flow over open cavities. J. Fluid Mech. 599, 309–339.
- CASPER, K. M., WAGNER, J. L., BERESH, S. J., HENFLING, J. F., SPILLERS, R. W. & PRUETT, B. O. M. 2015 Complex geometry effects on cavity resonance. AIAA J. 54 (1), 320–330.
- CASPER, K. M., WAGNER, J. L., BERESH, S. J., SPILLERS, R. W. & HENFLING, J. F. 2017 Spatial distribution of pressure resonance in compressible cavity flow. J. Fluid Mech.; AIAA Paper 2017-1476.
- CATTEFESTA, L. N., SONG, Q., WILLIAMS, D. R., ROWLEY, C. W. & ALVI, F. S. 2008 Active control of flow-induced cavity oscillations. *Prog. Aerosp. Sci.* 44 (7), 479–502.
- DELPRAT, N. 2006 A simple spectral model for a complex amplitude modulation process? *Phys. Fluids* **18** (7), 309–339.
- FLAHERTY, W., REEDY, M. T., ELLIOT, G. S., AUSTIN, J. M., SCHMIT, R. F. & CRAFTON, J. 2013 Investigation of cavity flow using fast-response pressure sensitive paint. AIAA Paper 2013-0678.
- GARG, S. & CATTAFESTA, L. N. 2001 Quantitative Schlieren measurements of coherent structures in a cavity shear layer. *Exp. Fluids* **30** (2), 123–134.
- GUENIAT, F., PASTUR, L. & LUSSEYRAN, F. R. 2014 Investigating mode competition and threedimensional features from two-dimensional velocity fields in an open cavity flow by modal decompositions. *Phys. Fluids* **20** (10), 105101.
- HAIGERMOSER, C., VESELY, L., NOVARA, M. & ONORATO, M. 2008 A time-resolved particle image velocimetry investigation of a cavity flow with a thick incoming turbulent boundary layer. *Phys. Fluids* **20** (10), 105101.
- HASSAN, M., LABRAGA, L. & KEIRSBULCK, L. 2007 Aero-acoustic oscillations inside large deep cavities. In *16th Australian Fluid Mechanics Conference (AFMC)*, pp. 421–428. School of Engineering, The University of Queensland.
- HELLER, H. H. & BLISS, D. B. 1975 The physical mechanism for flow-induced pressure fluctuations in cavities and concepts for their suppression. *AIAA Paper* 75-491.
- HUSSAIN, A. K. M. 1986 Coherent structures and turbulence. J. Fluid Mech. 173, 303-356.
- KEGERISE, M. A. 1999 An experimental investigation of flow induced cavity oscillations. PhD thesis, Department of Mechanical Engineering, Syracuse University, New York, USA.
- KEGERISE, M. A., SPINA, G. S. & CATTEFESTA, L. N. 2004 Mode-switching and nonlinear effects in compressible flow over a cavity. *Phys. Fluids* 16 (3), 678–687.
- KNISELY, C. & ROCKWELL, D. 1982 Self-sustained low-frequency components in an impinging shear layer. J. Fluid Mech. 116, 413–432.
- KRISHNAMURTY, K. 1955 Acoustic radiation from two-dimensional rectangular cut-outs in aerodynamic surfaces. *NACA TN* 3487.
- LARCHEVÊQUE, L., SAGAUT, P., LE, T. H. & COMPTE, P. 2004 Large-eddy simulation of compressible flow in a three-dimensional open cavity at high Reynolds number. J. Fluid Mech. 516, 265–301.
- LEE, B. H. K. 2010 Effect of captive stores on internal weapons bay floor pressure distributions. J. Aircraft 47 (2), 732–735.
- LEE, S. B., SEENA, A. & SUNG, H. J. 2010 Self-sustained oscillations of turbulent flow in an open cavity. J. Aircraft 47 (3), 820–834.

- LI, W., NONOMURA, T. & FUJII, K. 2013 On the feedback mechanism in supersonic cavity flows. *Phys. Fluids* **25** (5), 056101.
- LIN, J. C. & ROCKWELL, D. 2001 Organized oscillations in initially turbulent flow past a cavity. AIAA J. 39 (6), 1139–1151.
- LIU, X. & KATZ, J. 2013 Vortex-corner interactions in a cavity shear layer elucidated by time-resolved measurements of the pressure field. *J. Fluid Mech.* **728**, 417–457.
- MURRAY, N. & UKEILEY, L. 2007 Modified quadratic stochastic estimation of resonating subsonic cavity flow. J. Turbul. 8 (53), 1–22.
- MURRAY, N., RASPET, R. & UKEILEY, L. 2011 Contributions of turbulence to subsonic cavity flow wall pressures. *Phys. Fluids* 23 (1), 015104.
- MURRAY, N., SALLSTROM, E. & UKEILEY, L. 2009 Properties of subsonic open cavity flow fields. *Phys. Fluids* **21** (9), 095103.
- NAJM, H. N. & GHONEIM, A. F. 1991 Numerical simulation of the convective instability in a dump combustor. AIAA J. 29 (6), 911–919.
- ROCKWELL, D. & KNISELY, C. 1978 The organized nature of flow impingement upon a corner. J. Fluid Mech. 93 (3), 413–432.
- ROCKWELL, D. & NAUDASCHER, E. 1978 Review-self-sustaining oscillations of flow past cavities. Trans. ASME J. Fluids Engng 100 (2), 152–165.
- ROCKWELL, D. & NAUDASCHER, E. 1979 Self-sustaining oscillations of impinging free shear layers. Annu. Rev. Fluid Mech. 11 (1), 67–94.
- ROWLEY, C. W. & WILLIAMS, D. R. 2006 Dynamics and control of high-Reynolds-number flow over open cavities. *Annu. Rev. Fluid Mech.* **38** (1), 251–276.
- ROSSITER, J. E. 1964 Wind-tunnel experiments on the flow over rectangular cavities at subsonic and transonic speeds. *Aeronautical Research Council Report* 3438.
- RUSSELL, D. R. 2006 Development of a time-domain, variable-period surface-wave magnitude measurement procedure for application at regional and teleseismic distances, part I: theory. *Bull. Seismol. Soc. Am.* 96 (2), 665–677.
- SAROHIA, V. 1977 Experimental investigation of oscillations in flows over shallow cavities. *AIAA J.* **15** (7), 984–991.
- SAMIMY, M. & LELE, S. K. 1991 Motion of particles with inertia in a compressible free shear layer. *Phys. Fluids* **3** (8), 1915–1923.
- SEENA, A. & SUNG, H. J. 2013 Spatiotemporal representation of the dynamic modes in turbulent cavity flows. *Intl J. Heat Fluid Flow* **4**, 1–13.
- SLIPCHENKO, M. N., MILLER, J. D., ROY, S., GORD, J. R., DANCZYK, S. A. & MEYER, T. R. 2012 Quasi-continuous burst-mode laser for high-speed planar imaging. Opt. Lett. 37 (8), 1346–1348.
- SLIPCHENKO, M. N., MILLER, J. D., ROY, S., GORD, J. R. & MEYER, T. R. 2014 All-diode-pumped quasi-continuous burst-mode laser for extended high-speed planar imaging. *Opt. Express* 21 (1), 681–689.
- SONG, Q. 2008 Closed-loop control of flow-induced cavity oscillations. PhD thesis, Department of Mechanical and Aerospace Engineering, University of Florida, Gainesville, USA.
- TORRENCE, C. & COMPO, G. P. 1998 A practical guide to wavelet analysis. Bull. Am. Meteorol. Soc. 79 (1), 61-78.
- TRACY, M. B. & PLENTOVICH, E. B. 1997 Cavity unsteady-pressure measurements at subsonic and transonic speeds. *NASA Tech. Paper* 3669.
- UKEILEY, L., SHEEHAN, M., COIFFET, F., ALVI, F., ARUNAJATESAN, S. & JANSEN, B. 2008 Control of pressure loads in geometrically complex cavities. J. Aircraft **45** (3), 1014–1024.
- DE VICENTE, J., BASLEY, J., MESEGUER-GARRIDO, F., SORIA, J. & THEOFILIS, V. 2014*a* Three-dimensional instabilities over a rectangular open cavity: from linear stability analysis to experimentation. *J. Fluid Mech.* **748**, 189–220.
- DE VICENTE, J., BASLEY, J., MESEGUER-GARRIDO, F., SORIA, J. & THEOFILIS, V. 2014b Three-dimensional instabilities over a rectangular open cavity: from linear stability analysis to experimentation. J. Fluid Mech. 748, 189–220.

- WAGNER, J. L., CASPER, K. M., BERESH, S. J., HENFLING, J. F., SPILLERS, R. W. & PRUETT, B. O. M. 2015a Mitigation of wind tunnel wall interactions in subsonic cavity flows. *Exp. Fluids* 56 (3), 59.
- WAGNER, J. L., CASPER, K. M., BERESH, S. J., HUNTER, P. S., HENFLING, J. F., SPILLERS, R. W. & PRUETT, B. O. M. 2015b Fluid-structure interactions in compressible cavity flows. *Phys. Fluids* 27 (6), 066102.
- WAGNER, J. L., CASPER, K. M., BERESH, S. J., HUNTER, P. S., SPILLERS, R. W. & HENFLING, J. F. 2016 Response of a store with tunable natural frequencies in compressible cavity flow. AIAA J. 54 (8), 2351–2360.
- ZHUANG, N., ALVI, F. S., ALKISLAR, M. B. & SHIH, C. 2006 Supersonic cavity flows and their control. AIAA J. 44 (9), 2118–2128.