



Generalised solutions to the Benjamin problem

Vladimir V. Ostapenko^{1,2,†}

¹Lavrentyev Institute of Hydrodynamics of the Siberian Branch of the Russian Academy of Sciences, Lavrentyev Avenue 15, Novosibirsk, 630090, Russia

²Novosibirsk State University, Pirogova street 2, Novosibirsk, 630090, Russia

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We generalise the classical Benjamin solution (Benjamin, *J. Fluid Mech.*, vol. 31, 1968, pp. 209–248) modelling the flow in a horizontal duct of finite depth in situations where the flow contains a region spanning the depth of the duct, and a region in which the surface detaches from the ceiling of the duct as a free surface. It is shown that the Benjamin solution belongs to a one-parameter family of similar solutions, which are divided into two types: solutions that describe potential flows where the free surface of the fluid is deflected from the duct ceiling at a zero angle; and solutions that admit the formation of a vortex flow region in the vicinity of the point of fluid separation from the duct ceiling. It is shown that this one-parameter family of solutions is the limit of a two-parameter family of solutions in which part of the uniform flow energy is converted into energy of the small-scale fluid motion. Based on the local hydrostatic approximation, the applicability of the constructed solutions is discussed.

Key words: channel flow, vortex dynamics

1. Introduction

This paper presents a theoretical study of steady gravity currents in a horizontal rectangular duct of finite depth in situations where the flow contains a region spanning the depth of the duct, and a region in which the surface detaches from the ceiling of the duct as a free surface. Such currents represent the limiting case of more complex two-layer gravity currents when the density of the lighter fluid tends to zero. This kind of fluid flow is seen to occur in power plant tailrace tunnels, storm-water and sewage systems, qanat systems (gently sloping underground aqueducts), irrigation systems, and pipelines with air vents or undersized surge tanks (Sundquist & Papadakis 1983).

In his classical paper, Benjamin (1968) investigated theoretically a steady gravity current of an incompressible fluid inside a horizontally aligned rectangular duct and

[†] Email address for correspondence: ostigil@mail.ru

circular tube. As a result of these studies, approximate solutions were constructed that simulate wave flows arising after the instantaneous removal of the barrier bounding a semi-infinite duct or tube filled with liquid. Wallis, Crowley & Hagi (1977) investigated experimentally the water outflow from a horizontal pipe using a basic hydraulic approach. They showed that both viscosity and surface-tension effects are negligible provided the pipe diameter is larger than 100 mm. Comparable results followed also from Wilkinson (1982), who carried out an experimental study of the motion of an air cavity into a long horizontal duct with a rectangular cross-section. Baines & Wilkinson (1986) investigated the air propagation into an inclined rectangular duct and analytically predicted the cavity shape of their experiments. Hager (1999) studied cavity outflow from a nearly horizontal pipe, based on detailed experimentation and a hydraulic approach. Atrabi *et al.* (2015) proposed and applied for numerical simulation two one-dimensional mathematical models of transient flows with the propagation of an interface in a water-filled duct.

A generalisation of the problem of fluid outflow arising after the instantaneous removal of the barrier bounding a semi-infinite rectangular duct filled with liquid is the problem of the two-layer gravity currents produced by lock exchange, i.e. currents that occur after the instantaneous removal of the barrier separating two resting liquids of different densities filling a rectangular duct. A rather large number of papers have been devoted to theoretical, numerical and experimental study of the second problem; see, for example, Shin, Dalziel & Linden (2004), Ungarish (2010), Borden & Meiburg (2013), Baines (2016) and Konopliv *et al.* (2016). Whereas a characteristic feature of the solutions to the first problem (simulating the outflow of a single-layer fluid) is the existence of a stagnation point A at which the free surface of the liquid is deflected from the duct ceiling at a finite angle, the flow in the vicinity of this point is potential and, as a consequence, the fluid velocity at point A coincides with the movement velocity of this point. The stagnation-point condition was effectively used by Korobkin (2013) for modelling wave flows induced by lifting of a flat body from the free boundary of an infinitely deep liquid. Wave flows induced by lifting of a rectangular beam partly immersed in shallow water were considered by Ostapenko & Kovyrkina (2017) in the first approximation of shallow-water theory (Friedrichs 1948; Stocker 1957) without using the stagnation-point condition.

The Benjamin solution (1968) given in § 2 was obtained under conditions that uniquely determine the parameters of uniform flows at the inlet and outlet of a rectangular duct. These conditions suggest that the fluid is ideal, its flow is potential and liquid separation from the duct ceiling occurs at a stagnation point A. In this paper, we generalise the Benjamin solution, successively weakening these conditions. In § 3, we cancel the stagnation-point condition, which allows us to obtain a one-parameter family of solutions describing the potential flows where the free surface of the fluid is deflected from the duct ceiling at a zero angle. In § 4, we reject the potential-flow condition, which makes it possible to construct a one-parameter family of solutions that admit the formation of a vortex-flow region in the vicinity of the point A of fluid separation from the duct ceiling. In § 5, we take into account the effect of viscous friction in the vicinity of the separation point A, which allows us to obtain a two-parameter family of stable solutions in which part of the uniform flow energy is converted into energy of the small-scale fluid motion. In § 6, the applicability of the constructed solutions is given using the local hydrostatic approximation proposed by Ostapenko (2018).

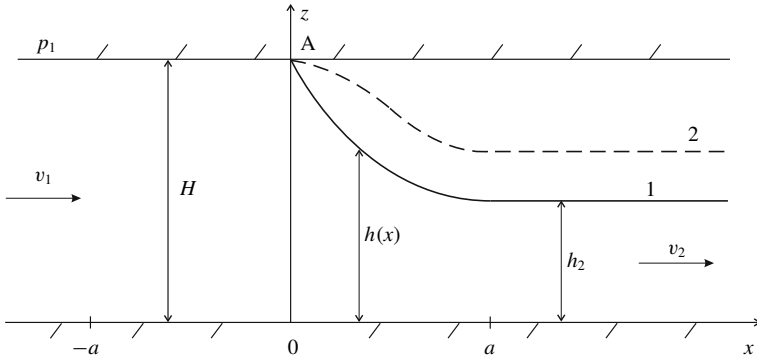


FIGURE 1. Potential flow of an ideal incompressible fluid flowing from a horizontally aligned rectangular duct. The free fluid surface is the Benjamin solution (solid line 1) and the other solution is where the free surface of the fluid is deflected from the duct ceiling at a zero angle (dotted line 2).

2. Benjamin solution

Consider a plane-parallel steady gravity current of an incompressible fluid inside a horizontally aligned rectangular duct. In the coordinate system shown in figure 1, the fluid depth $h(x)$ of this current satisfies the conditions

$$h(x) = H, \quad x \leq 0, \quad \text{and} \quad h(x) < H, \quad x > 0, \quad (2.1a,b)$$

where H is the duct height. Let us assume that the flow becomes uniform at a certain distance a from the vertical section $x = 0$ where the upper boundary of the fluid is deflected from the duct ceiling. The flow uniformity in these areas means that

$$h(x) = h_2, \quad x \geq a, \quad \text{and} \quad \tilde{p}(x) = p(x, h(x)) = p_1, \quad x \leq -a, \quad (2.2a,b)$$

$$u(x, z) = \begin{cases} v_1, & x \leq -a, \\ v_2, & x \geq a, \end{cases} \quad \text{and} \quad w(x, z) = 0, \quad |x| \geq a. \quad (2.3)$$

Here $p(x, z)$ and $\tilde{p}(x)$ are the specific pressures in the fluid and on its surface; $u(x, z)$ is the horizontal velocity of the fluid directed along the x axis; $w(x, z)$ is the vertical velocity of the fluid; and h_2, p_1, v_1 and v_2 are some given values.

We assume that the atmospheric pressure on the free surface of the fluid is equal to zero, i.e. $\tilde{p}(x) = 0$ at $x \geq 0$, and that the fluid surface $z = h(x)$ is a streamline with the horizontal velocity of the fluid at the stagnation point $A = (0, H)$ being

$$u_0 = u(0, H) = 0. \quad (2.4)$$

Benjamin (1968) showed that in this case, from the Bernoulli equation along the surface streamline (with allowance for the mass and momentum conservation laws on the segment $|x| \leq a$), it follows that the parameters of a piecewise-constant solution (2.2) and (2.3) are determined uniquely and are calculated by the following:

$$h_2 = H/2, \quad v_1 = c_1/2, \quad v_2 = \sqrt{2}c_2, \quad p_1 = -c_1^2/8, \quad (2.5a-d)$$

where $c_1 = \sqrt{gH}$ and $c_2 = \sqrt{gh_2}$ are the velocities of propagation of small perturbations within the framework of the shallow-water theory and g is the acceleration due to gravity. It follows from (2.5) that the flow is subcritical ($v_1 < c_1$) in the domain $x \leq -a$ and supercritical ($v_2 > c_2$) in the domain $x \geq a$.

3. One-parameter family of approximate solutions obtained within the framework of the model of an ideal incompressible fluid

Within the framework of the ideal incompressible fluid model, the plane-parallel steady flow is described by the continuity equation

$$u_x + w_z = 0 \tag{3.1}$$

and by the steady Euler equations

$$uu_x + wu_z + p_x = 0, \quad uw_x + ww_z + p_z + g = 0 \tag{3.2a,b}$$

for the horizontal and vertical components of the fluid velocity. A consequence of equations (3.2) is the Bernoulli equation

$$B = \frac{u^2 + w^2}{2} + gz + p = \text{const.} \tag{3.3}$$

along each fluid streamline. For the potential flow $u_z - w_x = 0$, the constant on the right-hand side of (3.3) is identical for all streamlines. From (3.1) and (3.2) follow the equations

$$(u^2 + p)_x + (uw)_z = 0, \quad (uw)_x + (w^2 + p + gz)_z = 0, \tag{3.4a,b}$$

which are conservation laws for the horizontal and vertical momenta of the fluid written in differential form.

First we will simulate the flow under consideration by the solution

$$u(x, z), \quad w(x, z), \quad p(x, z), \quad 0 \leq z \leq h(x), \tag{3.5a-d}$$

of equations (3.1) and (3.2), satisfying the conditions (2.1) and the following boundary conditions at infinity:

$$\lim_{x \rightarrow +\infty} h(x) = h_2, \quad \lim_{x \rightarrow -\infty} p(x, H) = p_1, \tag{3.6a,b}$$

$$\lim_{x \rightarrow -\infty} u(x, z) = v_1, \quad \lim_{x \rightarrow +\infty} u(x, z) = v_2, \quad \lim_{x \rightarrow \pm\infty} w(x, z) = 0, \tag{3.7a-c}$$

which are consistent with the conditions (2.2) and (2.3). Assuming that the fluid surface $z = h(x)$ is the streamline in solution (3.5), we obtain the kinematic condition on this surface as

$$w(x, h) = u(x, h)h_x. \tag{3.8}$$

Let us integrate the differential equations (3.1) and (3.4a) with respect to z from 0 to h and with respect to x from x_1 to x_2 . Taking into account the kinematic condition (3.8) and the no-normal-flow condition on the duct bottom,

$$w(x, 0) = 0, \tag{3.9}$$

we obtain on the spatial interval $[x_1, x_2]$ the integral conservation laws for the mass and horizontal momentum of the fluid:

$$\left(\int_0^h u \, dz \right) \Big|_{x_1}^{x_2} = 0, \quad \left(\int_0^h (u^2 + p) \, dz \right) \Big|_{x_1}^{x_2} = \int_{x_1}^{x_2} \tilde{p} h_x \, dx. \tag{3.10a,b}$$

Generalised solutions to the Benjamin problem

Based on conditions (2.1), we have

$$h_x = 0, \quad x < 0, \quad \text{and} \quad \tilde{p} = 0, \quad x \geq 0; \quad (3.11a,b)$$

correspondingly, for our problem we obtain

$$\int_{x_1}^{x_2} \tilde{p} h_x \, dx = 0. \quad (3.12)$$

Let us assume that the solution (3.5) approximately satisfies conditions (2.2) and (2.3). In this case, the Bernoulli equation (3.3) written with respect to the surface streamline $z = h(x)$ yields the following approximate relation:

$$v_1^2/2 + gH + p_1 = v_2^2/2 + gh_2. \quad (3.13)$$

As the pressure of the fluid in a uniform flow obeys the hydrostatic law,

$$p = \tilde{p} + g(h - z) \quad \Rightarrow \quad \int_0^h p \, dz = \tilde{p}h + gh^2/2,$$

then the approximate expressions for solution (3.5), which follow from the conservation laws (3.10) at $x_1 < -a$ and $x_2 > a$, are

$$q = H v_1 = h_2 v_2, \quad H v_1^2 + gH^2/2 + H p_1 = h_2 v_2^2 + gh_2^2/2, \quad (3.14a,b)$$

where q is the flow rate of the fluid.

If pressure $p_1 < 0$, the system of equations (3.13) and (3.14) admit the one-parameter family of solutions

$$v_1 = \frac{c_1 h_2}{H} = h_2 \sqrt{\frac{g}{H}}, \quad v_2 = c_1 = \sqrt{gH}, \quad h_2 = H - \sqrt{\frac{2H|p_1|}{g}}, \quad (3.15a-c)$$

in which the parameter is p_1 . The depth positive condition $h_2 > 0$ leads to the following restriction on this parameter:

$$-c_1^2/2 < p_1 < 0. \quad (3.16)$$

It follows from (3.15) that the flow is subcritical in the domain $x \leq -a$ and supercritical in the domain $x \geq a$ for all values of the pressure p_1 satisfying inequalities (3.16). In this case, the velocity v_2 does not depend on the value p_1 , but the velocity v_1 , depth h_2 and flow rate q decrease monotonically to zero with p decreasing to $-c_1^2/2$.

For the correctness of solution (3.15), the Bernoulli equation (3.3) has to be satisfied along the entire surface streamline $z = h(x)$, in particular, at the point $A = (0, H)$, where the surface pressure is $\tilde{p}(0) = 0$. From here, taking into account equations (3.13) and (3.14), we obtain

$$\frac{u_0^2}{2} = \frac{v_1^2}{2} + p_1 = \frac{gh_2^2}{2H} + p_1 = \frac{gH}{2} - \sqrt{2gH|p_1|} \geq 0 \quad \Rightarrow \quad |p_1| \leq \frac{gH}{8}. \quad (3.17a,b)$$

As a result, at $u_0 = 0$ we obtain the Benjamin solution (2.5), and at $u_0 > 0$ we obtain a family of solutions whose parameters satisfy the inequalities

$$-c_1^2/8 < p_1 < 0, \quad H/2 < h_2 < H, \quad c_1/2 < v_1 < c_1. \quad (3.18a,b)$$

Thus, the Benjamin solution is the limiting solution for the one-parameter family of solutions (3.15) and (3.18) that describe potential flows where the free surface of the fluid is deflected from the duct ceiling at a zero angle (dotted curve 2 in figure 1). Using the method proposed by Plotnikov & Toland (2004), it can be shown that, within the framework of potential flows of an ideal fluid, the free surface of the liquid in the Benjamin solution (solid curve 1 in figure 1) is deflected from the duct ceiling at an angle of 60° .

4. Solutions that allow the formation of a vortex-flow region in the vicinity of the liquid separation point from the duct ceiling

We describe the flows that model the solutions (3.15) satisfying the condition

$$-c_1^2/2 < p_1 < -c_1^2/8, \quad (4.1)$$

which leads to a violation of the inequality (3.17a). For such solutions, the fluid surface $z = h(x)$ is not a single streamline on the interval $[-a, a]$. In this case the Bernoulli equation (3.3) used to derive (3.13) is not satisfied on the fluid surface $z = h(x)$ over the entire interval $[-a, a]$. Moreover, as the kinematic condition (3.8) is not satisfied on the entire interval $[-a, a]$, the integral conservation laws (3.10) for $x_1 < -a$ and $x_2 > a$ cannot be derived from the differential continuity and Euler equations (3.1) and (3.2). Thus, to describe such flows, the integral mass and momentum conservation laws (3.10) should be taken as the basic relations, which do not follow from the differential equations of the ideal incompressible fluid in the general case.

Under condition (4.1) a vortex-flow region is formed in the neighbourhood of the point $A = (0, H)$, where the fluid separates from the duct ceiling (figure 2). Let us assume that this region has the form

$$W_1 = \{(x, z) : b_1 < x < b_2, \mu(x) < z < h(x)\},$$

where $-a < b_1 < 0, a > b_2 > 0$ and the function $z = \mu(x)$ satisfies the conditions

$$\mu(b_1) = h(b_1) = H, \quad \mu(b_2) = h(b_2) < H.$$

We will also assume that the function $z = \mu(x)$ is part of the streamline

$$z = \eta(x) = \begin{cases} H, & x \leq b_1, \\ \mu(x), & b_1 \leq x \leq b_2, \\ h(x), & x \geq b_2, \end{cases} \quad (4.2)$$

separating the vortex-flow region W_1 from the potential-flow region $W_2 = W \setminus W_1$, where

$$W = \{(x, z) : 0 \leq z \leq h(x, t)\}$$

is the area of existence of the solution (3.5)–(3.7). This assumption allows to use the formula (3.13) obtained from Bernoulli equation (3.3) and formulae (3.14) obtained

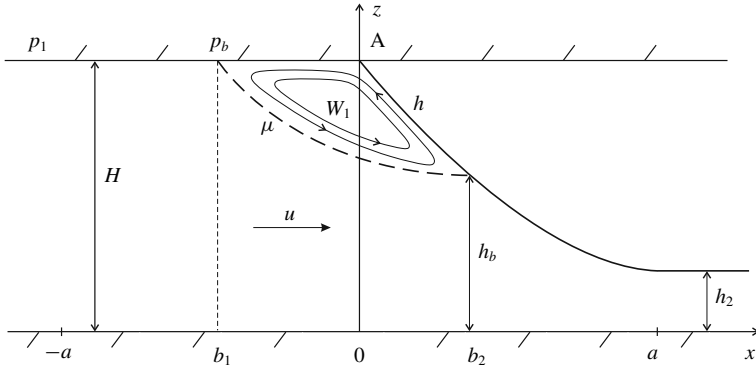


FIGURE 2. The flows in which the vortex-flow region W_1 is formed in the vicinity of the point A.

from the integral conservation laws (3.10) to construct solutions (3.15) in which the pressure p_1 satisfies the inequalities (4.1). The parameters of such solutions satisfy the conditions

$$-c_1^2/2 < p_1 < -c_1^2/8, \quad 0 < h_2 < H/2, \quad 0 < v_1 < c_1/2.$$

If streamline (4.2) has discontinuous derivatives at the points $x = b_i$, i.e. satisfies the conditions

$$\lim_{x \rightarrow b_1+0} \mu'(x) < 0, \quad \lim_{x \rightarrow b_2-0} \mu'(x) > h'(b_2),$$

then it follows from the continuity of the fluid velocity at these points that

$$u(b_1, H) = w(b_1, H) = 0, \quad u(b_2, h_b) = w(b_2, h_b) = 0, \quad h_b = h(b_2). \quad (4.3a,b)$$

Using the Bernoulli equation (3.3) written for streamline (4.2) at points with coordinates $(-\infty, H)$, (b_1, H) , (b_2, h_b) and $(+\infty, h_2)$ taking into account conditions (3.6), (3.7) and (4.3) we have

$$v_1^2/2 + gH + p_1 = gH + p_b = gh_b = v_2^2/2 + gh_2. \quad (4.4)$$

From here, with allowance for formulae (3.15), we obtain the surface pressure

$$p_b = \tilde{p}(b_1) = gH/2 - \sqrt{2gH|p_1|} < 0$$

on the left boundary of the vortex-flow region W_1 and the depth

$$h_b = h(b_2) = 3H/2 - \sqrt{2H|p_1|/g} \quad (4.5)$$

on the right boundary of this region. It follows from (3.15) and (4.5) that $h_b - h_2 = H/2$ for any value of the pressure p_1 satisfying condition (4.1). In this case, the characteristic vertical size

$$H - h_b = \sqrt{2H|p_1|/g} - H/2$$

of the domain W_1 increases from 0 to $H/2$ as p_1 decreases from $-c_1^2/8$ to $-c_1^2/2$.

Note that the conditions (4.3) leading to equalities (4.4) provide the smallest possible size of the vortex-flow region W_1 .

5. Solutions in which part of the uniform flow energy is converted into energy of the small-scale fluid motion

In the general case, the solutions (2.1)–(2.3) of the considered problem depend on two parameters, namely, pressure p_1 and velocity v_1 , which determine the depth h_2 and velocity v_2 of the fluid flowing from the duct. In the one-parameter family of solutions (3.15) obtained under condition (3.16), the pressure p_1 and velocity v_1 are related by the following equality:

$$v_1 + \sqrt{2|p_1|} = c_1. \tag{5.1}$$

Assume that equality (5.1) is not satisfied. Then, the system of equations (3.13) and (3.14) becomes incompatible. A similar situation arises for the shocks that model hydraulic bores in shallow-water theory (Stocker 1957). On such shocks, the system of mass and total momentum conservation laws is incompatible with the energy conservation law, which at standing shocks is equivalent to the local momentum conservation law; its analogue in our problem is equation (3.13). Moreover, the criterion for the shock stability (Friedrichs & Lax 1971; Lax 1972) is energy inequality, due to which the energy decreases (is lost) when the fluid flows through the shock. For hydraulic bores, this means that part of the free-stream energy behind the bore front is transformed to the energy of small-scale fluid motion, which is not taken into account in the shallow-water theory.

We will apply this approach to construct energy-stable solutions (2.1)–(2.3) and (3.14), for which equality (5.1) does not hold. These solutions simulate flows in which the small-scale fluid motion occurs on the free surface of the liquid near the point of its separation from the duct ceiling. Figure 3 shows a case of surface wave breakdown in some interval (a_2, a_3) that leads to the formation of surface turbulent-vortex flow in the same interval (a_1, a_4) . Such small-scale fluid perturbations drift mostly downstream, gradually damping (due to the action of viscous friction) on approaching the horizontal boundaries of this region. Herein, we will neglect friction at the bottom and ceiling of the duct. With this in mind, we assume that for $x < a_1$ and $x > a_4$ the flow becomes potential, and also that for $x < -a < a_1$ and for $x > a > a_4$ it becomes uniform, i.e. satisfy the conditions (2.2) and (2.3). Similar assumptions are typically made in the modelling of hydraulic bores in the framework of shallow-water theory (Stocker 1957).

Given these assumptions, the parameters of the considered flows in the interval $[x_1, x_2]$, where $x_1 < -a$ and $x_2 > a$, satisfy the integral mass and total momentum conservation laws (3.10) from which, taking into account (2.1)–(2.3) and (3.12), we obtain equations (3.14). Excluding from these equations fluid velocities v_1 and v_2 , we have

$$\frac{q^2}{Hh_2}(H - h_2) = \frac{g}{2}(H^2 - h_2^2) + Hp_1. \tag{5.2}$$

Since for $h_2 < H$ the left side of this equation is positive, the right side must also be positive. From this we obtain the following restriction on the pressure:

$$p_1 > -\frac{g}{2H}(H^2 - h_2^2) = -\frac{c_1^2}{2} \left(1 - \frac{h_2^2}{H^2} \right). \tag{5.3}$$

A consequence of the system (3.1) and (3.2) is

$$(uB)_x + (wB)_y = 0, \tag{5.4}$$

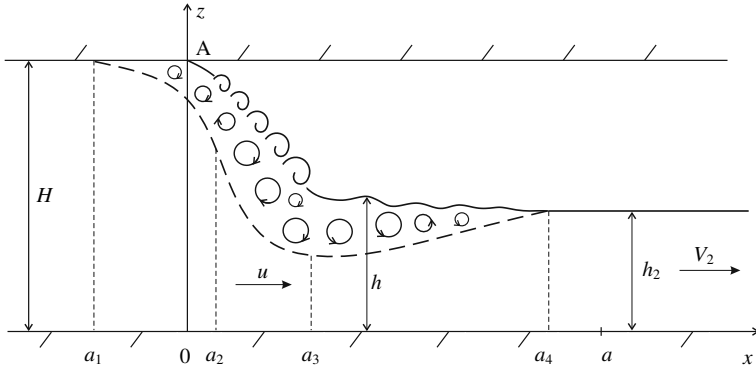


FIGURE 3. Flows in which waves break down on the free surface of the liquid near the point A, which leads to the formation of a surface region of the turbulent-vortex flow.

where B is the Bernoulli function in (3.3). This equation is a differential form of the energy conservation law of a liquid. Integrating equation (5.4) taking into account the conditions (3.8), (3.9) and (3.11), we obtain the integral energy conservation law

$$\left(\int_0^h u \left(\frac{u^2 + w^2}{2} + gz + p \right) dz \right) \Big|_{x_1}^{x_2} = 0. \quad (5.5)$$

When modelling the flows described in the previous section, this conservation law should be taken as the basic relation, which does not follow from the differential equations (3.1) and (3.2) in the general case. From the conservation law (5.5) at $x_1 < -a$ and $x_2 > a$, taking into account (2.2) and (2.3), we obtain

$$(q(v^2/2 + gh + \tilde{p})) \Big|_{x_1}^{x_2} = 0.$$

For the stability of the solutions (2.1)–(2.3), (3.14) and (5.3), considered in this section, it is necessary to fulfill the energy inequality

$$(q(v^2/2 + gh + \tilde{p})) \Big|_{x_1}^{x_2} < 0,$$

which (given the fact that $q = \text{const.} > 0$) can be rewritten in the form

$$v_1^2/2 + gH + p_1 > v_2^2/2 + gh_2. \quad (5.6)$$

Using formulae (3.14) and (5.2), we exclude from the inequality (5.6) the fluid velocities v_1 and v_2 . As a result, we obtain the following restriction on the pressure:

$$p_1 < -\frac{g}{2H}(H - h_2)^2 = -\frac{c_1^2}{2} \left(1 - \frac{h_2}{H} \right)^2.$$

Thus, under the condition

$$-\frac{c_1^2}{2} \left(1 - \frac{h_2}{H} \right)^2 < p_1 < -\frac{c_1^2}{2} \left(1 - \frac{h_2}{H} \right)^2, \quad (5.7)$$

equations (3.14) define a two-parameter family of energy-stable solutions (2.1)–(2.3). Note that the method for constructing these solutions is similar to the method using the Kutta condition in aerodynamics (Clancy 1975).

The determination of the parameters of a particular solution (3.14) and (5.7) is conveniently carried out as follows. First set the depth $h_2 \in (0, H)$, after which we select the pressure p_1 satisfying the inequalities (5.7). Next, from equation (5.2) we find the flow rate q , which we use to calculate the velocities $v_1 = q/H$ and $v_2 = q/h_2$. For example, if the depth $h_2 = H/2$, as in the Benjamin solution (2.5), then the inequality (5.7) takes the form

$$-3c_1^2/8 < p_1 < -c_1^2/4. \tag{5.8}$$

Choosing p_1 in the middle of the interval (5.8) and using equation (5.2) we obtain

$$p_1 = -\frac{5c_1^2}{16}, \quad q = \frac{Hc_1}{4}, \quad v_1 = \frac{c_1}{4}, \quad v_2 = \frac{c_1}{2} = \frac{c_2}{\sqrt{2}}. \tag{5.9a-d}$$

It is interesting to note that, in contrast to the one-parameter family of solutions (3.15) and (3.16), to which the Benjamin solution (2.5) belongs, in the solution (5.9) the flow is subcritical, for both $x < -a$ and $x > a$.

6. Justification of the applicability of the constructed solutions on the basis of the local hydrostatic approximation

The classical method of justifying the applicability of vertically averaged solutions of equations of the ideal incompressible fluid is based on the long-wave approximation (Friedrichs 1948), which implies that the fluid flow is potential and the characteristic depth of the flow H_0 is much smaller than the characteristic length of the surface waves L_0 , i.e. $\varepsilon = H_0^2/L_0^2 \ll 1$. For this reason (Stocker 1957), the spatial derivative of the depth h of the plane-parallel flow satisfies the condition $|h_x| \leq O(\sqrt{\varepsilon})$. However, in our case, this method is not applicable, since the flows studied in §§ 4 and 5 are not potential, and the potential flows considered in § 3 satisfy the condition $|h_x| \leq 1$ for all $x > 0$ only if $(H - h_2)/H \leq 1$. Therefore, to justify the applicability of the approximate solutions (2.1)–(2.3) constructed in this paper, we apply the local hydrostatic approximation proposed by Ostapenko (2018).

We will say that when the fluid flows out from a rectangular duct (figure 1) its parameters satisfy the local hydrostatic approximation at the point (x, z) , if the inequality $w^2(x, z)/c_1^2 < \delta \ll 1$ is satisfied at this point, where δ is a given small number. Let us divide the domain V of existence of the two-dimensional flow under consideration into two subdomains:

$$V_1(\delta) = \{(x, z) \in V : w^2(x, z)/c_1^2 < \delta\}, \quad V_2(\delta) = V \setminus V_1(\delta).$$

In the domain V_1 , we introduce dimensionless variables

$$x_* = \frac{\sqrt{\delta}}{H}x, \quad z_* = \frac{z}{H}, \quad h_* = \frac{h}{H}, \quad u_* = \frac{u}{c_1}, \quad w_* = \frac{w}{\sqrt{\delta}c_1}, \quad p_* = \frac{p}{c_1^2}, \tag{6.1a-f}$$

which, after the substitution $\delta = H_0^2/L_0^2$, $H_0 = H$, transform into the dimensionless variables of the long-wave approximation, where the variable L_0 can be considered as the characteristic wavelength along the streamlines passing inside the domain $W_1(\delta)$.

We will assume that the domain $V_2(\delta)$ in which the local hydrostatic approximation is not satisfied is bounded. With this in mind, the number a included in (2.2) and (2.3) is chosen so that for $|x| > a$ the flow is potential and condition $|x| < a$ is satisfied for all points $(x, z) \in V_2(\delta)$. In this case, the flows considered in §§ 3 and 4 are described by the integral mass, momentum and energy conservation laws (3.10) and (5.5), in which $x_1 < -a$ and $x_2 > a$. Writing down these conservation laws in dimensionless variables (6.1), by analogy with Ostapenko (2018), we obtain the relations

$$q|_{x_1}^{x_2} = 0, \quad (qv + h^2/2 + h\tilde{p})|_{x_1}^{x_2} = 0, \quad (v^2/2 + h + \tilde{p})|_{x_1}^{x_2} = 0, \quad (6.2a-c)$$

with accuracy no less than $O(\delta)$. In equations (6.2) the asterisk is omitted for brevity. For the flows studied in § 5, equation (6.2c) should be replaced by the energy inequality

$$(v^2/2 + h + \tilde{p})|_{x_1}^{x_2} < 0. \quad (6.3)$$

Given that acceleration $g = 1$ in dimensionless variables (6.1), the relations (6.2) imply formulae (3.13) and (3.14) and relations (6.2a,b) along with inequality (6.3) imply formula (5.2) and inequality (5.7). It follows that the approximate piecewise-constant solutions (2.1)–(2.3) that we have constructed in §§ 3–5 transmit the parameters of the considered flows with accuracy no less than $O(\delta)$.

7. Conclusion

In this paper, we construct three families of solutions (2.1)–(2.3) that generalise the classical Benjamin (1968) solution. The first family of solutions (3.15) and (3.18) describes the potential flows for which the stagnation-point condition is not satisfied (dotted line 2 in figure 1); the second family of solutions (3.15) and (4.1) admit the formation of a vortex-flow region in the vicinity of the separation point A (figure 2); and the third family of solutions (3.14) and (5.7) describes the flow in which part of the uniform flow energy is converted into energy of the small-scale fluid motion (figure 3). Since the third family of solutions is two-parameter, the whole set of constructed solutions depends on two input parameters; whereas the first two families of solutions, together with the Benjamin solution, form a single one-parameter family of solutions (3.15) and (3.16) satisfying the energy-conserving condition (5.1).

A characteristic feature of a steady flow described by solution (3.15) and (3.16) is that the uniform flow at $x > a$ is supercritical. It follows from the shallow-water theory (Stocker 1957) that the parameters of this uniform flow at the duct outlet are completely determined by the flow inside the duct, and therefore, for the correct modelling of such steady flow by the pseudo-unsteady method, it is necessary to set two boundary conditions (pressure p_1 and velocity v_1) for the uniform flow at the duct inlet and one should not set any boundary conditions for the uniform flow at the duct outlet. For the correct modelling of energy-losing steady flows (3.14) and (5.7), in which the uniform flow at $x > a$ is subcritical, the same as in the example (5.9), it is necessary to specify one boundary condition (for example, velocity v_1) for the uniform flow at the duct inlet and one boundary condition (for example, depth h_2) for the uniform flow at the duct outlet.

Let us pass to the coordinate system in which separation point A moves with velocity $-v_1$ and the velocity of the uniform flow in the left part of the duct becomes equal to zero. This is consistent with standard laboratory experiments (Wilkinson 1982) in which the fluid outflow is regulated only by the liquid depth h_2 at the duct outlet. Since the characteristic feature of such experiments is the formation of the stagnation point A, it is possible to obtain in these experiments only a single energy-conserving steady flow corresponding to the Benjamin solution (2.5).

Declaration of interests

The author report no conflict of interest.

References

- ATRABI, H. B., ASCE, S. M., HOSODA, T. & TADA, A. 2015 Simulation of air cavity advancing into a straight duct. *J. Hydraul. Engng* **141**, 04014068.
- BAINES, P. G. 2016 Internal hydraulic jumps in two-layer systems. *J. Fluid Mech.* **787**, 1–15.
- BAINES, W. D. & WILKINSON, D. L. 1986 The motion of large air bubbles in ducts of moderate slope. *J. Hydraul. Res.* **25**, 157–170.
- BENJAMIN, T. B. 1968 Gravity currents and related phenomena. *J. Fluid Mech.* **31**, 209–248.
- BORDEN, Z. & MEIBURG, E. 2013 Circulation-based models for Boussinesq internal bores. *J. Fluid Mech.* **726**, R1.
- CLANCY, L. J. 1975 *Aerodynamics*. Pitman.
- FRIEDRICH, K. O. 1948 On the derivation of shallow water theory. *J. Fluid Mech.* **1**, 109–134.
- FRIEDRICH, K. O. & LAX, P. D. 1971 Systems of conservation equation with convex extension. *Proc. Natl Acad. Sci. USA* **68**, 1686–1688.
- HAGER, W. H. 1999 Cavity outflow from a nearly horizontal pipe. *Intl J. Multiphase Flow* **25**, 349–364.
- KONOPLIV, N. A., SMITH, S. G., MCELWAIN, J. N. & MEIBURG, E. 2016 Modelling gravity currents without an energy closure. *J. Fluid Mech.* **789**, 806–829.
- KOROBKIN, A. A. 2013 A linearized model of water exit. *J. Fluid Mech.* **737**, 368–386.
- LAX, P. D. 1972 *Hyperbolic Systems of Conservation Laws and the Mathematical Theory of Shock Waves*. Society for Industrial and Applied Mathematics.
- OSTAPENKO, V. V. 2018 Justification of shallow-water theory. *Dokl. Phys.* **63**, 33–37.
- OSTAPENKO, V. V. & KOVYRKINA, O. A. 2017 Wave flows induced by lifting of a rectangular beam partly immersed in shallow water. *J. Fluid Mech.* **816**, 442–467.
- PLOTNIKOV, P. I. & TOLAND, J. F. 2004 Convexity of Stokes waves of extreme form. *Arch. Rat. Mech. Anal.* **171**, 349–416.
- SHIN, J. O., DALZIEL, S. B. & LINDEN, P. F. 2004 Gravity currents produced by lock exchange. *J. Fluid Mech.* **521**, 1–34.
- STOCKER, J. J. 1957 *Water Waves: the Mathematical Theory with Applications*. Wiley.
- SUNDQUIST, M. J. & PAPADAKIS, C. N. 1983 Surging in combined free surface-pressurized systems. *J. Transp. Eng.* **109**, 232–245.
- UNGARISH, M. 2010 Energy balances for gravity currents with a jump at the interface produced by lock release. *Acta Mechanica* **211**, 1–21.
- WALLIS, G. B., CROWLEY, C. J. & HAGI, Y. 1977 Conditions for a pipe to run full when discharging liquid into a space filled with gas. *Trans. ASME J. Fluids Engng* **99**, 405–413.
- WILKINSON, D. L. 1982 Motion of air cavities in long horizontal ducts. *J. Fluid Mech.* **118**, 109–122.