

Nonlinear exact coherent structures in pipe flow and their instabilities

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In this paper, we present computational results of some two-fold azimuthally symmetric travelling waves and their stability. Calculations over a range of Reynolds numbers (Re) reveal connections between a class of solutions computed by Wedin & Kerswell (*J. Fluid Mech.*, vol. 508, 2004, pp. 333–371) (henceforth called the WK solution) and the $Re \rightarrow \infty$ vortex–wave interaction theory of Hall & Smith (*J. Fluid Mech.*, vol. 227, 1991, pp. 641–666) and Hall & Sherwin (*J. Fluid Mech.*, vol. 661, 2010, pp. 178–205). In particular, the continuation of the WK solutions to larger values of Re shows that the WK solution bifurcates from a shift-and-rotate symmetric solution, which we call the WK2 state. The WK2 solution computed for $Re \leq 1.19 \times 10^6$ shows excellent agreement with the theoretical $Re^{-5/6}$, Re^{-1} and $O(1)$ scalings of the waves, rolls and streaks respectively. Furthermore, these states are found to have only two unstable modes in the large Re regime, with growth rates estimated to be $O(Re^{-0.42})$ and $O(Re^{-0.92})$, close to the theoretical $O(Re^{-1/2})$ and $O(Re^{-1})$ asymptotic results for edge and sinuous instability modes of vortex–wave interaction states (Deguchi & Hall, *J. Fluid Mech.*, vol. 802, 2016, pp. 634–666) in plane Couette flow. For the nonlinear viscous core states (Ozcakir *et al.*, *J. Fluid Mech.*, vol. 791, 2016, pp. 284–328), characterized by spatial a shrinking of the wave and roll structure towards the pipe centre with increasing Re , we continued the solution to $Re \leq 8 \times 10^6$ and we find only one unstable mode in the large Reynolds number regime, with growth rate scaling as $Re^{-0.46}$ within the class of symmetry-preserving disturbances.

Key words: transition to turbulence

1. Introduction

Transition to turbulence in pipes, channels and boundary-layer flows has attracted much recent interest from a dynamical system perspective. In particular, the investigation of invariant solutions has proved useful in understanding the transition mechanism. Travelling wave (TW) solutions are steady solutions of the Navier–Stokes equations in a reference frame moving at a constant speed. Such solutions have

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been calculated numerically by Nagata (1990), Waleffe (2001), Waleffe (2003), Wang, Gibson & Waleffe (2007), Gibson, Halcrow & Cvitanovic (2009) and Blackburn, Hall & Sherwin (2013) in plane Couette flow. In pipe geometry, Faisst & Eckhardt (2003), Wedin & Kerswell (2004), Pringle & Kerswell (2007), Viswanath (2007) and Willis *et al.* (2017) have reported similar states with different degrees of rotational symmetry.

The TW states do not arise from any finite Re bifurcation of the linearly stable laminar plane Couette or Hagen–Poiseuille flow, though they come increasingly close to the laminar state as $Re \rightarrow \infty$. Instead, these solutions, which are disconnected from plane Couette or Hagen–Poiseuille flow, appear as a saddle-node bifurcation with increasing Reynolds number. (When additional effects are included, such as suction/injection along the pipe walls, then these solutions can be connected in the parameter space.) The two branches emanating at the saddle-node bifurcation point are usually called the ‘upper’ and ‘lower’ branch depending on whether they have higher or lower drag, respectively. Note however that in the wave speed versus Re curve the upper and lower branches correspond to low and high drag respectively. The existence and stability of these states, and their connection in phase space, play an important role in understanding both transition to turbulence and the large Re behaviour of plane Couette and pipe flows. For example, there is numerical evidence from Duguet, Willis & Kerswell (2008*b*) and Budanur & Hof (2018) to suggest that the ‘lower-branch’ TW states can be part of the ‘edge’ state influencing the boundary of the basin of attraction of linearly stable pipe Poiseuille flow. On the other hand, the upper-branch solutions are more relevant to fully turbulent flow. Furthermore, when the unstable manifolds of some of these states are low-dimensional and slow in the sense of decreasing growth rates with increasing Re , they can correspond to coherent flows that are experimentally observable (see for example Hof *et al.* (2004)) in intermediate Reynolds number flows that migrate from one TW state to another, suggesting a connection in the phase space between the unstable manifold of one to the stable manifold of another. Such slow low-dimensional unstable manifolds, different from the ones reported here, have also been computed for a TW state in pipes by for example Viswanath & Cvitanovic (2009*b*). This raises the hope that one might use an appropriate control mechanism to stabilize a TW state with low drag.

Another class of invariant solutions referred to as relative periodic orbits are periodic solutions in a moving frame with constant phase shifts in two spatial directions. These may arise from Hopf bifurcations of TWs; see for example Viswanath (2007), Duguet, Pringle & Kerswell (2008*a*) and Chantry, Willis & Kerswell (2014). In a recent study, a collection of these invariant solutions in pipe flow were identified by Budanur *et al.* (2017) using visualizations in a symmetry-reduced state space by the method of slices.

For pipe flow, which is the concern of the present paper, we use cylindrical coordinates (r, θ, z) with cylinder axis aligned along the z -axis. We use the pipe radius and centreline velocity of the corresponding base flow to non-dimensionalize all quantities of interest and define Reynolds number in the usual way. The non-dimensional velocity in the Navier–Stokes TW solutions has the following form:

$$\mathbf{u} = \mathbf{v}_B(r) + \mathbf{U}(r, \theta) + \mathbf{v}_w(r, \theta, z - ct), \quad (1.1)$$

where $\mathbf{v}_B(r) = (1 - r^2)\hat{\mathbf{z}}$ is Hagen–Poiseuille flow and $\mathbf{v}_w(r, \theta, z - ct)$ is the wave component travelling at a phase speed c with 2π azimuthal and $2\pi/\alpha$ axial periodicity.

In the TW calculations reported here, the average axial pressure gradient is the same, $-(4/Re)\hat{z}$, as for the base flow \mathbf{v}_B . Furthermore, the wave component has zero average $\langle \mathbf{v}_w \rangle = 0$ over an axial period. If we write $\mathbf{U}(r, \theta) = (U(r, \theta), V(r, \theta), W(r, \theta))$ in cylindrical coordinates, $(U(r, \theta), V(r, \theta), 0)$ is referred to as the roll part of the flow and represents streamwise vortices, while $(0, 0, W(r, \theta))$ is termed the streak. A similar decomposition is possible for any predominantly unidirectional shear flow. The physical mechanism to sustain vortex–wave interaction (VWI) states for large Reynolds number is now well understood and can be traced back to the initial formulation of Hall & Smith (1991). While investigating boundary layers for large Re , Hall & Smith (1991) realized that in any shear flow, small $O(1/Re)$ roll components have an $O(1)$ effect on the $O(1)$ streaks, which can support neutral inviscid modes in the form of TWs. The Reynolds stress-type term arising from streamwise average of quadratic terms involving waves in turn drives the rolls. Subsequently Hall & Sherwin (2010) used the framework given by Hall & Smith (1991) in plane Couette flow to determine solutions of the parameter-free equations to confirm that the VWI asymptotic state is the high Reynolds number description of what had become known in the computational community as a self-sustaining process; see for example Waleffe (1995), Wedin & Kerswell (2004) and Wang *et al.* (2007).

The calculations of Hall & Sherwin (2010) and Deguchi & Hall (2014a) show remarkable agreement between the finite Reynolds number computations of the lower-branch modes and the asymptotic VWI approach even at Reynolds numbers as low as 500. This means that the asymptotic result that the dominant interaction involves only one streamwise wave mode turns out to be valid even at relatively small Re . However upper-branch modes calculated at large but finite Re approach the asymptotic state much more slowly.

Ozçakir *et al.* (2016) found that their computed solutions were in fact VWI states reflecting the scalings of Hall & Sherwin (2010) existing for pipe flows. That scaling was shown to be roughly in line with the numerical calculations of the TW states by Viswanath (2009a) in the large Re range. However, quantitative agreement with, for example, the Wedin & Kerswell (2004) (WK) TW solution was not possible since numerical calculations for WK solutions were not available for $Re > 1.1 \times 10^4$ due to computational limitations. Besides the VWI states based on the Hall & Sherwin (2010) asymptotics, Ozçakir *et al.* (2016) also identified two more possibilities. One was the possibility of a shrinking VWI state, where the vortex–wave collapses at a rate $Re^{-1/6}$ but the rolls, streaks and waves scale differently from the usual VWI state, though their three-way interaction is qualitatively similar. However such states have yet to be found in numerical calculations. A second possibility was that of a nonlinear viscous core (NVC) state shrinking towards the pipe centre at a rate $Re^{-1/4}$. That state might also be referred to as a centre mode and has properties qualitatively similar to the free-stream coherent structures identified by Deguchi & Hall (2014b) in boundary-layer flows. In this asymptotic case, there is no separation in scales between the rolls and transverse components of the waves, or between the streaks and longitudinal components of the waves. Finite Re numerical calculations by Ozçakir *et al.* (2016) produced two branches of TW solutions denoted by C1 and C2, which have a shrinking structure towards the centre of the pipe at large Re . Based on the then available numerical data for $Re \leq 1.3 \times 10^5$, Ozçakir *et al.* (2016) tentatively suggested that the C1 and C2 branches were finite Re realizations of the asymptotic NVC states; however, the agreement with the $Re \rightarrow \infty$ asymptotic theory was not conclusive. Subsequent calculations of the C1 branch up to $Re \leq 3.3 \times 10^6$

and the C2 branch up to $Re \leq 8 \times 10^6$ confirmed that this is indeed the case. The stability of these NVC states is a major part of the present paper.

Some results concerning the stability of the lower-branch TW states in Couette flow were given by Wang *et al.* (2007). It was found that a single unstable eigenmode exists and they estimated that with increasing Re the growth rate decayed like $Re^{-0.48}$. On the other hand the lower-branch calculations of Viswanath (2009a) in pipe flow resulted in two real unstable eigenmodes that scale as $Re^{-0.41}$ and $Re^{-0.87}$.

Deguchi & Hall (2016) investigated the stability of VWI states in Couette flow and found that they were susceptible to fast inviscid instabilities of the streak part of the flow, gentle meanderings of the VWI state and a mode described as an edge state mode having some properties of both the other modes. For Couette flow the modes have growth rates of size Re^0 , Re^{-1} and $Re^{-1/2}$ respectively and it was found that most of the lower-branch modes have a single unstable mode which was referred to as the edge mode which has growth rate scaling like $Re^{-1/2}$ agreeing well with the estimate $Re^{-0.48}$ given by Wang *et al.* (2007). However there is no reason why the slowest growing mode with wavenumber like Re^{-1} could not be unstable in other configurations. Therefore it seems likely that the estimates of Viswanath (2009a) correspond to an edge mode and a slow mode. The fact that states exist with just one unstable mode with small growth rate suggests that they may be observed in turbulent flows at relatively low Reynolds numbers; see for example Hof *et al.* (2004). On the other hand, the stability of large Reynolds number NVC states has not yet been investigated.

There are two distinct though related aims of the present paper. The first one is to extend our previous TW computations through greater efficiency to much higher Reynolds numbers, at least up to $Re = 1.19 \times 10^6$, than previously reported in Ozcakir *et al.* (2016). This is crucial in settling which states connect with the different possible high Reynolds number solutions. One major goal here is to show that the WK states can be continued to a higher Re where they connect to a shift-and-rotate symmetric state, which we refer to as the WK2 state. The WK branch bifurcates from a more symmetric WK2 at approximately $Re = 76438$. For sufficiently large Re up to $Re = 1.19 \times 10^6$, numerical evidence suggests WK2 is a VWI state with peak roll, wave and streak amplitudes scaling as Re^{-1} , $Re^{-5/6}$ and $O(1)$ respectively. We are unaware of any other computations of pipe flows at such large Re in the literature. The accuracy of our TW computations is also crucial in being able to compute reliable linear stability modes.

The other aim concerns the linear stability of the TWs. We limit our investigation to the modes with the same two-fold azimuthal symmetry and the same axial wavelength as the base state. These restrictions help us extend calculations to large enough values of Re so that comparison can be made with asymptotic predictions. These scalings are in agreement with the $Re^{-1/2}$ and Re^{-1} asymptotics for the edge and meandering modes predicted by Deguchi & Hall (2016). We do not, however, find any Re^0 unstable eigenmode within the class of pressure-gradient-preserving two-fold azimuthally symmetric disturbances.

Throughout the paper, the TW solutions we seek satisfy the dimensionless Navier–Stokes equations

$$\mathbf{u}_t + \mathbf{u} \cdot \nabla \mathbf{u} = -\nabla p + \frac{1}{Re} \Delta \mathbf{u}, \quad \nabla \cdot \mathbf{u} = 0, \quad (1.2)$$

in the form

$$\mathbf{u} = \mathbf{v}_B(r) + \mathbf{v}(r, \theta, z - ct), \tag{1.3}$$

$$p = p_B(r) + q(r, \theta, z - ct), \tag{1.4}$$

where $(\mathbf{v}_B, p_B) = ((1 - r^2)\hat{z}, -(4/Re)z)$. We assume that the perturbation velocity \mathbf{v} and the pressure q are 2π periodic in θ and $\tilde{z} := \alpha(z - ct)$ and satisfy the no-slip boundary condition on the wall $r = 1$.

In §2, we briefly describe the numerical formulation. In §3, we present results for both VWI and NVC states and describe their stability properties. For the most part, the stability investigation is limited to eigenmodes which result in no change to the mean pressure gradient relative to the base (Hagen–Poiseuille) flow. Nonetheless, in a limited range of Re , in order to make comparisons of growth rates with some previously reported calculations in the literature, we have also calculated eigenmodes that preserve mean mass flux instead of pressure gradient (Pringle, Duguet & Kerswell 2009). In §4 we discuss the stability results. Finally, we conclude with §5.

2. Numerical formulation

Our computational method is based on a Galerkin truncation of the Fourier modes in θ and $\tilde{z} = \alpha(z - ct)$ and a Chebyshev representation in r with appropriate radial basis functions Φ_j, Ψ_j for the axial and non-axial velocity components as given in Wedin & Kerswell (2004). This basis automatically satisfies the boundary conditions.

As mentioned in the introduction, TW states can be decomposed in the form $\mathbf{u} = \mathbf{v}_B(r) + \mathbf{v}(r, \theta, z - ct)$, where $\mathbf{v}_B(r) = (1 - r^2)\hat{z}$ is the base flow and $\mathbf{v}(r, \theta, z - ct)$ is the perturbation velocity with real phase speed c . It is clear from (1.2) that $\mathbf{v}(r, \theta, z - ct)$ satisfies

$$-c \frac{\partial \mathbf{v}}{\partial z} + (\mathbf{v} \cdot \nabla) \mathbf{v} = -\nabla q + \frac{1}{Re} \Delta \mathbf{v} - \mathbf{v}_B \cdot \nabla \mathbf{v} - \mathbf{v} \cdot \nabla \mathbf{v}_B, \quad \nabla \cdot \mathbf{v} = 0, \tag{2.1}$$

where the perturbed pressure q is determined by solving $\Delta q = \mathcal{N}[\mathbf{v}]$ with a consistent Neumann boundary condition $\partial q / \partial n = \mathcal{N}_b[\mathbf{v}]$ at the pipe wall $r = 1$. (Note that there is a typographical error in (2.10) and (2.11) of OzcaKir *et al.* (2016) and the term $-\mathbf{v}_B \cdot \nabla \mathbf{v}_B$ should not be present.) Here the operators are defined as

$$\mathcal{N}[\mathbf{v}] := \nabla \cdot [-\mathbf{v}_B \cdot \nabla \mathbf{v} - \mathbf{v} \cdot \nabla \mathbf{v}_B - \mathbf{v} \cdot \nabla \mathbf{v}], \tag{2.2a}$$

$$\mathcal{N}_b[\mathbf{v}] := \hat{\mathbf{r}} \cdot \left[\frac{1}{Re} \Delta \mathbf{v} \right]. \tag{2.2b}$$

In a cylindrical domain, the Navier–Stokes equations possesses a number of symmetries. If (u, v, w, p) is a solution of the Navier–Stokes equations then it can be shown that, if the transformations $\mathbb{R}_{k_0}, \mathbb{S}$ and \mathbb{Q}_{k_0} are applied to (u, v, w, p) , it produces another solution of the Navier–Stokes equations:

$$\left. \begin{aligned} \mathbb{R}_{k_0} : (u, v, w, p)(r, \theta, z) &\rightarrow (u, v, w, p)(r, \theta + 2\pi/k_0, z), \\ \mathbb{S} : (u, v, w, p)(r, \theta, z) &\rightarrow (u, -v, w, p)(r, -\theta, z + \pi/\alpha), \\ \mathbb{Q}_{k_0} : (u, v, w, p)(r, \theta, z) &\rightarrow (u, v, w, p)(r, \theta + \pi/k_0, z + \pi/\alpha). \end{aligned} \right\} \tag{2.3}$$

When applications of these transformations leave the flow invariant, then we say that the flow has the corresponding symmetry. We use the following truncated basis

representation for the velocity $\mathbf{v} = (u, v, w)^T$ suitable for k_0 -fold azimuthally symmetric (\mathbb{R}_{k_0}) TW states with shift-and-reflect (S) symmetry:

$$\begin{pmatrix} u \\ v \\ w \end{pmatrix} = \sum_{\substack{0 \leq j \leq N \\ 0 \leq k \leq M \\ 0 \leq l \text{ even} \leq P}} \begin{pmatrix} (u_{jkl}^{(1)} \cos \tilde{l}z + u_{jkl}^{(2)} \sin \tilde{l}z) \Phi_j(r; kk_0) \cos kk_0\theta \\ (v_{jkl}^{(1)} \cos \tilde{l}z + v_{jkl}^{(2)} \sin \tilde{l}z) \Phi_j(r; kk_0) \sin kk_0\theta \\ (w_{jkl}^{(1)} \sin \tilde{l}z + w_{jkl}^{(2)} \cos \tilde{l}z) \Psi_j(r; kk_0) \cos kk_0\theta \end{pmatrix} + \sum_{\substack{0 \leq j \leq N \\ 0 \leq k \leq M \\ 1 \leq l \text{ odd} \leq P}} \begin{pmatrix} (u_{jkl}^{(1)} \cos \tilde{l}z + u_{jkl}^{(2)} \sin \tilde{l}z) \Phi_j(r; kk_0) \sin kk_0\theta \\ (v_{jkl}^{(1)} \cos \tilde{l}z + v_{jkl}^{(2)} \sin \tilde{l}z) \Phi_j(r; kk_0) \cos kk_0\theta \\ (w_{jkl}^{(1)} \sin \tilde{l}z + w_{jkl}^{(2)} \cos \tilde{l}z) \Psi_j(r; kk_0) \sin kk_0\theta \end{pmatrix}. \tag{2.4}$$

Here N, M and P represent the truncation levels of the radial, azimuthal and axial basis elements respectively. Using these truncated velocities (2.4) in (2.1) at a given set of collocation points in r results in a nonlinear algebraic system for (\mathbf{X}, c) , where $\mathbf{X} = \{u_{jkl}^{(i)}, v_{jkl}^{(i)}, w_{jkl}^{(i)}\}_{j,k,l,i}$. Newton’s method is used to solve the resulting nonlinear set of equations using an efficient GMRES solver. The details of our numerics can be found in OzcaKir *et al.* (2016).

For linear stability calculations, we add an infinitesimal perturbation $\epsilon e^{\lambda t} \boldsymbol{\eta}(r, \theta, z - ct)$ to the already computed TW state $\mathbf{v}_B + \mathbf{v}$ and use the decomposition

$$\mathbf{u} = \mathbf{v}_B + \mathbf{v}(r, \theta, z - ct) + \epsilon e^{\lambda t} \boldsymbol{\eta}(r, \theta, z - ct). \tag{2.5}$$

Through linearization of (1.2) for small ϵ , we obtain the following equation for the eigenvalue λ and eigenfunction $\boldsymbol{\eta}(r, \theta, z - ct)$:

$$\lambda \boldsymbol{\eta} - c \boldsymbol{\eta}_z + (\mathbf{v} + \mathbf{v}_B) \cdot \nabla \boldsymbol{\eta} + \boldsymbol{\eta} \cdot \nabla (\mathbf{v} + \mathbf{v}_B) = -\nabla \hat{p} + \frac{1}{Re} \Delta \boldsymbol{\eta}, \quad \nabla \cdot \boldsymbol{\eta} = \mathbf{0}, \tag{2.6}$$

where $\hat{p}(r, \theta, z - ct)$ is the corresponding perturbation pressure.

The basis representation for $\boldsymbol{\eta}$ is the same as for \mathbf{v} in (2.4) and we employ a Galerkin truncation in the Fourier modes in θ and $\tilde{z} = \alpha(z - ct)$, and satisfy the resulting equations at a suitable set of collocation points in r . For the study of constant mean pressure gradient linear stability modes, we assume \hat{p} to be strictly periodic in \tilde{z} . On the other hand, for constant flux linear stability modes, we explicitly add $-q_0 \hat{z}$ on the right-hand side of (2.6), while still requiring \hat{p} to be periodic. The constant q_0 , in that case, can be expressed in terms of $\boldsymbol{\eta}$ by invoking the constant flux condition

$$\int_0^1 \langle \boldsymbol{\eta} \cdot \hat{\mathbf{z}} \rangle r \, dr = 0, \tag{2.7}$$

where $\langle \cdot \rangle$ denotes the average over θ and \tilde{z} over a period. This integration can be done explicitly and results in

$$q_0 = \frac{2}{Re} \left. \frac{\partial w_{0,0}}{\partial r} \right|_{r=1}, \tag{2.8}$$

where $w_{0,0}$ is the axial component of $\boldsymbol{\eta}$ averaged over θ and z . We did not impose the divergence condition on $\boldsymbol{\eta}$ directly in (2.6). Instead, in either case, we satisfy the Poisson equations for \hat{p} and the momentum equation in the same manner as discussed for TWs in (2.2a) and (2.2b). This leads to a Helmholtz equation for $\nabla \cdot \boldsymbol{\eta}$.

We checked that $\boldsymbol{\eta}$ is not a spurious eigenvector arising from λ being an eigenvalue of the Helmholtz equation by verifying that $\nabla \cdot \boldsymbol{\eta} = 0$, to within the expected numerical precision.

This procedure transforms (2.6) into an eigenvalue problem of the form $A\mathbf{y} = \lambda\mathbf{y}$ for the eigenvalue λ and eigenvector \mathbf{y} which is solved numerically using an eigenvalue solver followed by a Newton iteration process as described in appendix C.

3. Numerical results for TWs

The calculations presented here are limited to two-fold azimuthally symmetric TW states, i.e. $k_0 = 2$, with wavelength $\alpha = 1.55$. Even though our code allows us to impose any k_0 -fold symmetry for given α , we did not attempt to calculate other k_0 -fold symmetric TWs. This is firstly because our focus is to extend our previous numerical results and verify asymptotic scalings for those solutions calculated in Ozcakir *et al.* (2016) at the same values of k_0 and α . Also, numerical calculations suggest that k_0 -fold symmetric TWs with $k_0 = 1, 2$ exist at low Reynolds numbers through saddle-node bifurcations while higher k_0 -fold symmetric solutions appear at larger Re which makes the $k_0 = 1, 2$ states the most relevant in understanding transition. We performed calculations for various values of α in a limited range of Re around the saddle-node points of TWs which resulted in similar results. However the effect of varying α has not been investigated over a large range of values of Re . In the figures, different resolutions were used ranging between $(N, M, P) = (45, 8, 5)$ when $Re < 5000$ and $(N, M, P) = (220, 20, 12)$ when $Re \gg 10^5$. Significantly increasing each of N, M and P for particular Reynolds number in this range and comparing with baseline calculations showed that the calculation for c was accurate to four significant digits, whereas the velocity was accurate to at least three digits. A discussion of the resolution checks made is given in appendix B. We present numerical results for four distinct branches of TW solutions in the range $1556 < Re < 5 \times 10^5$ for the axial wavenumber $\alpha = 1.55$. We call these branches (i) a vortex-wave interaction state WK, (ii) WK with additional Ω_2 (shift-and-rotate) symmetry (WK2), (iii) a nonlinear viscous core state C1 and (iv) C1 with Ω_2 symmetry (C2). The phase speed c is shown in figure 1 as a function of Re for these branches for $\alpha = 1.55$, with the dotted blue, solid blue, dotted red and solid red curves denoting the WK, WK2, C1 and C2 branches of the solution respectively.

The WK solutions were calculated originally at relatively modest Reynolds numbers by Wedin & Kerswell (2004) and extended in Ozcakir *et al.* (2016). However, convergence in our prior calculations was limited to $Re \leq 11\,000$ since the available computational storage was limited at that time and the Newton solver run on a single processor was not as efficient as the current version. In the present paper, we report on a successful continuation to much larger Re , past a region of rapid variation, denoted by a hump in the dotted blue curve in figure 1. This solution branch when extended to larger Re is shown to bifurcate from a new solution branch which we call WK2. The bifurcation point corresponds to where the dotted blue and blue curves meet in figure 1 at $Re \approx 76\,438$.

Later we will describe in detail the large Re behaviour of WK2 that confirms that these are indeed finite Re realizations of the VWI asymptotic state. These states were calculated earlier in plane Couette flow by Hall & Sherwin (2010) whilst Ozcakir *et al.* (2016) showed that it was also applicable for pipe flows. However, to the best of our knowledge, this is the first time that calculations are presented for a large enough range of values of Re to conclusively confirm that possibility.

The NVC solutions C1 and C2 were reported in Ozcakir *et al.* (2016) for $Re \leq 1.3 \times 10^5$. Those solutions are marked by a shrinking structure of the waves and rolls towards the centre of the pipe as Re becomes larger. It is to be noted that, unlike C1, C2 has an additional shift-and-rotate symmetry. In the above paper, we gave an asymptotic description of the NVC for $Re \rightarrow \infty$ and presented numerical evidence to suggest that C1 and C2 were finite Re realizations of the NVC states, though the Reynolds number was not large enough to result in a convincing agreement between the asymptotics and numerics for all the physical quantities of interest. We have now extended our calculations to $Re = 3.3 \times 10^6$ for C1 and $Re = 8 \times 10^6$ for C2.

Also, the previous study of C1 and C2 focused primarily on a larger range of Re calculations of the lower-branch solutions in Ozcakir *et al.* (2016). The present calculations at moderate values of Re reveal that C1 originates through the bifurcation from C2 just above the turning point at $Re = 2298$ as shown in figure 1 and it remains very close to C2 when Re is large; see the upper inset in figure 1.

An analysis of the solution branches for various values of α near the bifurcation points suggests that WK2 and C2 correspond to the TWs labelled as ‘M2’ (A3) and ‘N2’ (C3) respectively in Pringle *et al.* (2009) where two classes, M-class and N-class, made up by a family of mirror-symmetric TWs with k_0 -rotational symmetry were discussed. The Mk_0 -class has a distinctive double-layered structure of fast and slow streaks across the pipe radius while the Nk_0 -class has a separation of fast streaks near the wall and slow streaks towards the pipe centre. This resembles some of the non-mirror-symmetric waves already known; see Faisst & Eckhardt (2003) and Wedin & Kerswell (2004). Figure 2 displays the normalized phase speed C versus the mean Reynolds number R_m for C2 at $\alpha = 1.2$ and WK2 at $\alpha = 2$ together with the solutions shown in Pringle *et al.* (2009) at the same values of α . It is clear that the C versus R_m curves are in close agreement.

3.1. Vortex–wave interaction states for $Re > 10\,000$

For the newly computed WK2 solution, the streamwise-averaged flow fields at $Re = 10\,000$, $\alpha = 1.55$, are shown in figure 3 with $(N, M, P) = (85, 12, 5)$. The roll, streak and wave components are displayed in a plane perpendicular to the pipe axis. The roll field $(U(r, \theta), V(r, \theta))$ and radial and azimuthal components $(u_w(r, \theta, \tilde{z}_0), v_w(r, \theta, \tilde{z}_0))$ of the wave velocity \mathbf{v}_w are depicted using arrows with the larger arrows corresponding to larger speeds; whilst intensities of the streak $W(r, \theta)$ and the axial component $w_w(r, \theta, \tilde{z}_0)$ of the wave velocity \mathbf{v}_w are represented by coloured contours at the cross-section $\tilde{z}_0 = 2\pi/\alpha$. In all solutions presented here the origin is fixed by imposing the same axial phase condition. The lighter colour corresponds to positive values of W in figure 3(a) and $w(r, \theta, \tilde{z})$ in figure 3(b), while darker colours correspond to negative values. Similar plots are shown for WK at $Re = 10\,000$ with the same resolution as in figure 4 for comparison. The streak velocity is plotted in the interval $[-0.459, 0.287]$ in figures 3(a) and 4(a) while the axial wave velocity is shown in the interval $[-0.036, 0.029]$ in figures 3(b) and 4(b). Note that the shift-and-rotate symmetry of WK2 results in streaks and rolls having four-fold azimuthal symmetry as seen in figure 3.

Computations of the lower-branch WK2 solutions at high values of Re allow for a more precise comparison between the numerical results and the $Re \rightarrow \infty$ asymptotic structure described in Ozcakir *et al.* (2016). In order to quantify the scaling features of WK2 for large Re , it is convenient to define the l th axial amplitude functions for rolls, streaks and waves as follows:

$$A_l^\pm(r, \theta) = \sqrt{u_l^{(1)}(r, \theta)^2 + v_l^{(1)}(r, \theta)^2 + u_l^{(2)}(r, \theta)^2 + v_l^{(2)}(r, \theta)^2}, \quad (3.1)$$

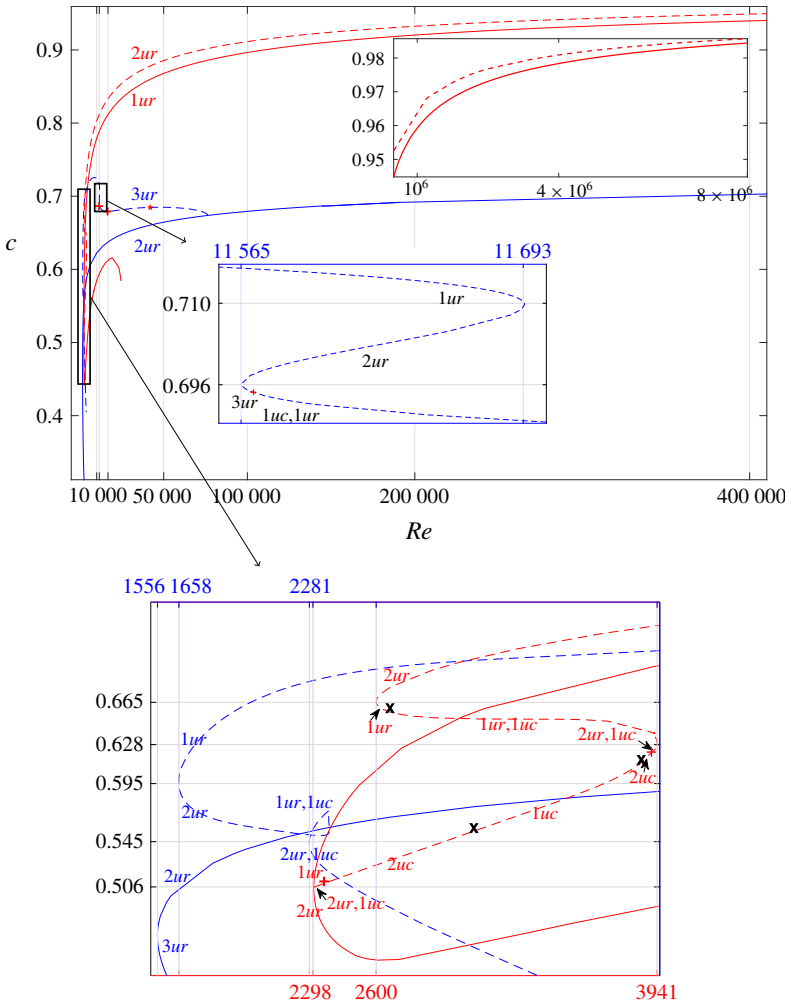


FIGURE 1. (Colour online) The wave speed c versus Re at $\alpha = 1.55$ for WK (blue, dashed), WK2 (blue, solid), C1 (red, dashed) and C2 (red, solid). Regions of large variation are shown in close-ups. The number of real (r) or complex (c) unstable (u) eigenmodes are displayed alongside each curve. \times symbols indicate locations of Hopf bifurcation points. On the C1 curve, $+$ symbols denote points where two unstable real eigenvalues join to form a complex eigenvalue pair. A red star on the WK curve indicates the local maximum value of c at $(Re, c) = (46\,300, 0.68494)$.

$$A_l^w(r, \theta) = \sqrt{w_l^{(1)}(r, \theta)^2 + w_l^{(2)}(r, \theta)^2}, \tag{3.2}$$

where

$$u_l^{(i)}(r, \theta) = \sum_{\substack{0 \leq j \leq N \\ 0 \leq k \leq M}} u_{jkl}^{(i)} \Phi_j(r; kk_0) \cos kk_0 \theta, \tag{3.3}$$

for $i = 1, 2$ (see (2.4)). The functions $v_l^{(i)}, w_l^{(i)}$ are defined in a similar manner.

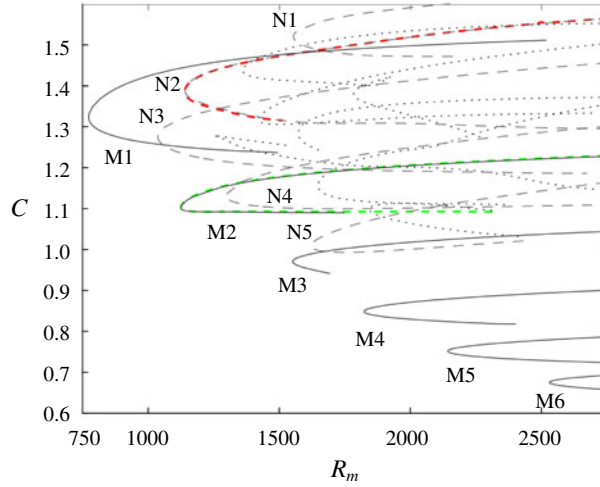


FIGURE 2. (Colour online) The wave speed C ($:= 2c_m$) versus R_m for the C2 branch (red) at $\alpha = 1.2$ and the WK2 branch (green) at $\alpha = 2$ compared to the N2 and M2 branches shown in figure 7 of Pringle *et al.* (2009), where (c_m, R_m) relates to (c, Re) through (A 8).

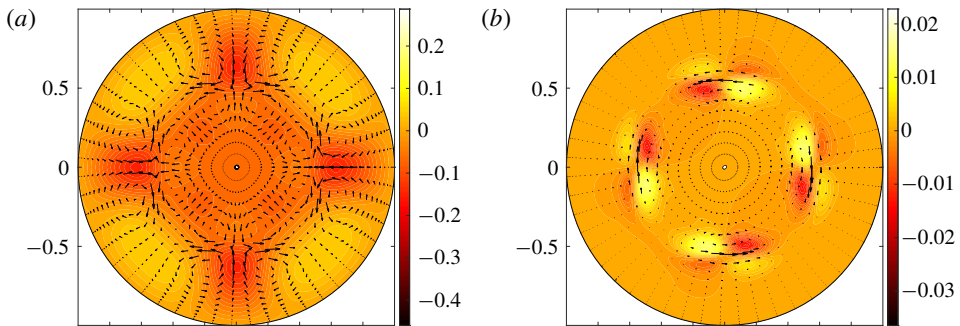


FIGURE 3. (Colour online) (a) Roll (U, V) and streak W profiles with $\max_{r,\theta} |(U, V)| = 0.0014196$ and $\max_{r,\theta} |W| = 0.14785$ and (b) wave $v_w(r, \theta, \tilde{z}_0)$ profile at $\tilde{z}_0 = 2\pi/\alpha$ with $\max_{r,\theta} |(u_w, v_w)| = 0.0055373$ and $\max_{r,\theta} |w_w| = 0.014165$ at $Re = 10\,000$ for WK2 at $\alpha = 1.55$ with resolution $(N, M, P) = (85, 12, 5)$.

Figure 5(a) shows the roll component sup-norms $\|U\|_\infty$ and $\|V\|_\infty$ and the streak sup-norms $\|W\|_\infty$, while figures 5(b) and 5(c) show the maximal perpendicular and streamwise wave amplitudes $\|A_l^w\|_\infty$, $\|A_l^\perp\|_\infty$ for the $l = 1, 2$ axial modes as a function of Re on a log–log scale respectively, where supremums are taken over r and θ in the computational domain. The linear fits on a log–log scale are based on a best-fit estimate of the computed data in the regime $1.5 \times 10^4 < Re < 1.19 \times 10^6$. The resulting numerical scales are displayed in table 1. The scalings of rolls are $Re^{-1.03}$ and $Re^{-0.96}$; whilst the streak scaling is $Re^{-0.04}$. These values agree remarkably well with VWI theory of Hall & Sherwin (2010) developed in the context of plane Couette flow with expected $O(Re^{-1})$ scale for the rolls and $O(1)$ for the streaks. The same scaling argument is shown to be valid for pipe flows in Ozcakir *et al.* (2016). The maximal perpendicular and axial wave amplitudes for the $l = 1$ mode are found numerically to scale as $Re^{-0.865}$ and $Re^{-0.81}$ respectively, which are consistent with

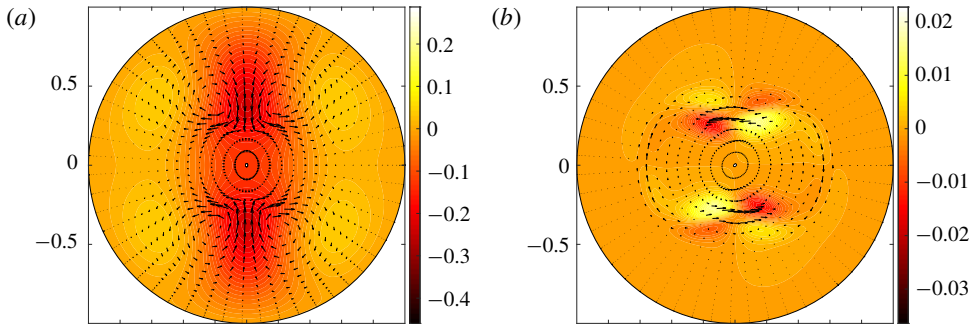


FIGURE 4. (Colour online) (a) Roll (U, V) and streak W profiles with $\max_{r,\theta} |(U, V)| = 0.00165$ and $\max_{r,\theta} |W| = 0.22976$ and (b) wave $\mathbf{v}_w(r, \theta, \tilde{z}_0)$ profile at $\tilde{z}_0 = 2\pi/\alpha$ with $\max_{r,\theta} |(u_w, v_w)| = 0.0052569$ and $\max_{r,\theta} |w_w| = 0.015851$ at $Re = 10\,000$ for WK at $\alpha = 1.55$ with $(N, M, P) = (85, 12, 5)$.

	$\ U\ _\infty$	$\ V\ _\infty$	$\ W\ _\infty$	$\ A_1^\perp\ _\infty$	$\ A_1^w\ _\infty$	$\ A_2^\perp\ _\infty$	$\ A_2^w\ _\infty$
	figure 5(a)			figure 5(b,c)		figure 5(b,c)	
Numerical scaling	1.03	0.96	0.04	0.865	0.81	1.48	1.25
VWI theory	1	1	0	5/6	5/6	5/3	5/3

TABLE 1. Comparison of the numerically inferred inverse Reynolds number scalings of flow quantities with the VWI predictions for WK2. The linear fits on a log–log scale are based on a best-fit estimate of the computed data in the regime $1.5 \times 10^4 < Re < 1.19 \times 10^6$. The corresponding expected asymptotic scales of each velocity component are shown in bold.

the VWI asymptotic predictions of the maximal wave amplitude of $O(Re^{-5/6})$ which is dominated by the $l = 1$ contribution occurring within a critical layer of width $O(Re^{-1/3})$ which decreases to $O(Re^{-7/6})$ outside the layer. However, the observed decay rates for the $l = 2$ mode were $Re^{-1.48}$ and $Re^{-1.25}$ respectively, which are not in such close agreement with the $Re^{-5/3}$ asymptotic prediction. This is likely because the effective Reynolds number gets smaller for the larger axial modes which means the higher Re calculations are needed to produce the asymptotic trends. Furthermore, the relative error of numerical calculations is larger for the higher axial modes since they are already small in magnitude.

It is also worth mentioning that, even though the WK state does not exist at large Re , the velocity scalings roughly agree with VWI theory in the regime $1.5 \times 10^4 < Re < 7 \times 10^5$ with $Re^{-1.005}$ and $Re^{-0.95}$ scaling of rolls and $Re^{-0.86}$ scaling of perpendicular wave amplitude. On the other hand, the streak scale is found to be $Re^{-0.21}$ while the streamwise wave amplitude scales as $Re^{-0.77}$, as against Re^0 and $Re^{-5/6}$ in the VWI theory. Thus, even though the WK state only appears to exist up to a certain Re beyond which it merges with the WK2 state, it still exhibits some but not all characteristics of the asymptotic VWI state. Thus, for instance, the wave remains dominated by a single axial mode over a wide range of Re .

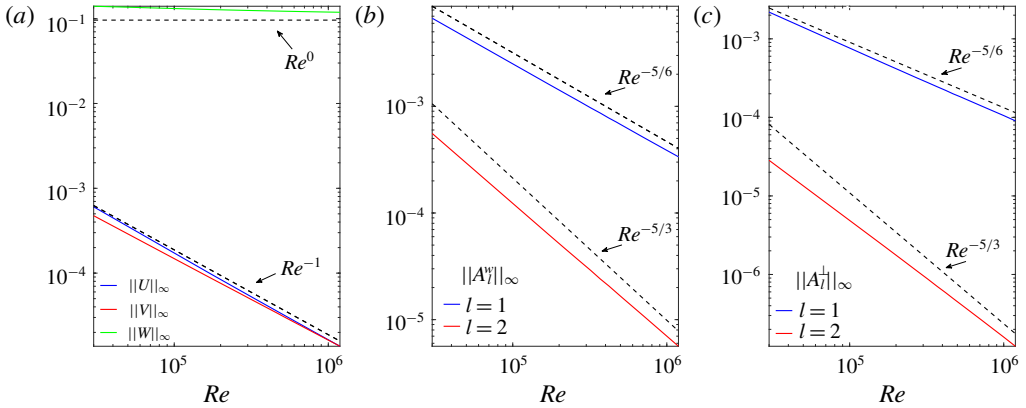


FIGURE 5. (Colour online) The WK2 scalings of (a) the roll components U (blue) and V (red) and streak W (green) and (b,c) the perpendicular and axial wave amplitudes at $l=1$ (blue), $l=2$ (red) where the supremum is taken over (r, θ) . Black dashed lines indicate the expected asymptotic scalings (Re^{-1} , Re^0 , $Re^{-5/6}$, $Re^{-5/3}$) for comparison.

4. Stability properties of the equilibrium states

In this section, we examine instabilities of the computed TW solution branches WK, WK2, C1 and C2. Since one of our primary concerns is to explore the large Re regime in order to compare and contrast with the asymptotic results for plane Couette flows, we limit ourselves to disturbances with two-fold azimuthal and shift-and-reflect symmetry of the TW states. This obviates the limitations in computer storage in a regime where the computation is most expensive. We limit ourselves mostly to studying disturbances that do not affect the mean pressure gradient, though, in some limited range of Re , we also consider constant-flux disturbances in order to check against earlier stability results. Instead of using a standard eigenvalue solver to determine all eigenvalues for the entire range of Re , which is very expensive computationally, for the most part we used a standard Newton solver (see appendix C) for equation (2.6) together with a normalization constraint to determine specific eigenvalues. An eigenvalue solver was only used at a particular set of parameter values below and above the bifurcation points to provide a rough initial guess for our Newton iteration scheme. Since our focus is on either the least stable or most unstable eigenvalues, we can then follow these through continuation techniques. Based on computational checks, we believe that the results presented here are accurate to at least two significant digits.

In figure 1 the notation 1ur corresponds to one real unstable and 1uc to one pair of unstable complex conjugate eigenvalues. Similarly 2ur indicates two real unstable eigenvalues; 1ur, 1uc indicates one real unstable and a pair of complex conjugate unstable eigenvalues, etc. By using these labels on each of the branches in figure 1, we are able to concisely summarize the stability features of all the solutions. It is to be noted that, as expected from earlier stability results in channels (Toh & Itano 2003; Wang *et al.* 2007) and pipes (Faisst & Eckhardt 2003; Kerswell & Tutty 2007; Viswanath & Cvitanovic 2009b), the unstable manifold is low-dimensional on the lower branch. Furthermore, we find a decrease in the size of the eigenvalues with Re , similar to that observed by Viswanath (2007) for a different ‘lower-branch’ TW solution. This low-dimensional ‘slow’ unstable manifold appears to be a generic

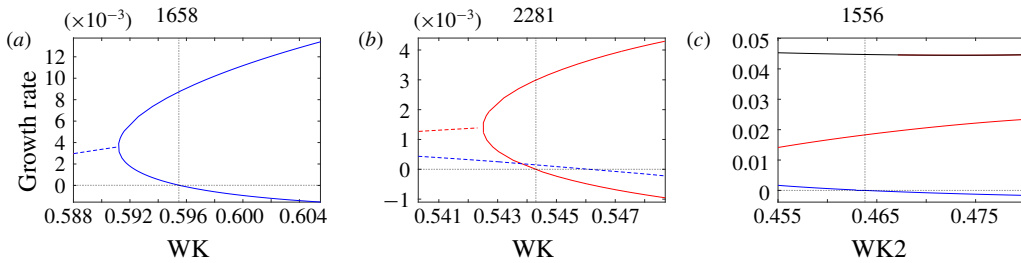


FIGURE 6. (Colour online) The growth rate ($Re\lambda$) versus wave speed c near the saddle-node points for the WK branch (*a,b*), at $Re = 1658$ and $Re = 2281$, and the WK2 branch (*c*) at $Re = 1556$. The curves indicate either real eigenvalues (solid curves) or a complex conjugate pair (dashed curves) as c changes. The intersection of the horizontal and vertical dotted lines corresponds to the position of the saddle-node points as shown in figure 1.

feature of ‘lower-branch’ TW solutions, whilst ‘upper-branch’ solutions typically have many unstable eigenmodes. We will later discuss the large Re asymptotics of the ‘lower branch’ in §4.2. On the other hand, the ‘upper branch’ is characterized by a more complex structure near the wall and requires higher resolution in both the axial and azimuthal directions which is very computationally demanding.

4.1. Instabilities at low/intermediate Re

Notice in figure 1 that in the smaller Re regime, the WK and WK2 solution branches have three saddle-node bifurcation points at $(Re, c) = (1658, 0.595)$, $(2281, 0.545)$ for WK and $(Re, c) = (1556, 0.464)$ for the WK2 solutions. Corresponding to these bifurcation points, it is clear from figure 6 that one real eigenvalue crosses zero exactly at the bifurcation point as expected.

On the other hand, when Re becomes large enough on the lower branch, WK goes through two more saddle-node bifurcations and makes an inverse S shape (see the middle inset in figure 1) where two more unstable real eigenvalues emerge resulting in a total of three unstable real eigenvalues. This happens in a narrow region around $Re = 11\,500$ where c is between 0.69 and 0.71 as shown in figure 7. If continued to higher values of Re , the lower branch of WK meets the lower WK2 branch at $Re \simeq 76\,438$ (see figure 1). A stability analysis of WK2 close to this point suggests that the shift-and-rotate symmetry-breaking neutral mode corresponds to the WK solution bifurcating from WK2. For larger Re , only WK2 survives and asymptotically approaches a VWI state.

In figures 8 and 9 the growth rates of the unstable eigenmodes of C1 and C2 are shown in a neighbourhood of the saddle-node points $(Re, c) = (2600, 0.665)$, $(3941, 0.6332)$ for C1 and $(Re, c) = (2298, 0.5047)$ for C2 (see the lower inset in figure 1). At the point where C1 appears as a bifurcation from C2, i.e. the point denoted by a star in figure 8, it has two unstable real and an unstable complex pair of eigenvalues. As c increases, C1 experiences two consecutive saddle-node bifurcations (see the insets in figure 8). These bifurcations appear as S-shapes in the c - Re curves in figure 1. Further, for the C1 solution in the interval $0.52 < c < 0.68$, we note that at three distinct locations marked by crosses in figure 8, the growth rate of a complex conjugate pair of eigenvalues changes sign; this suggests there are Hopf bifurcations to time-periodic states.

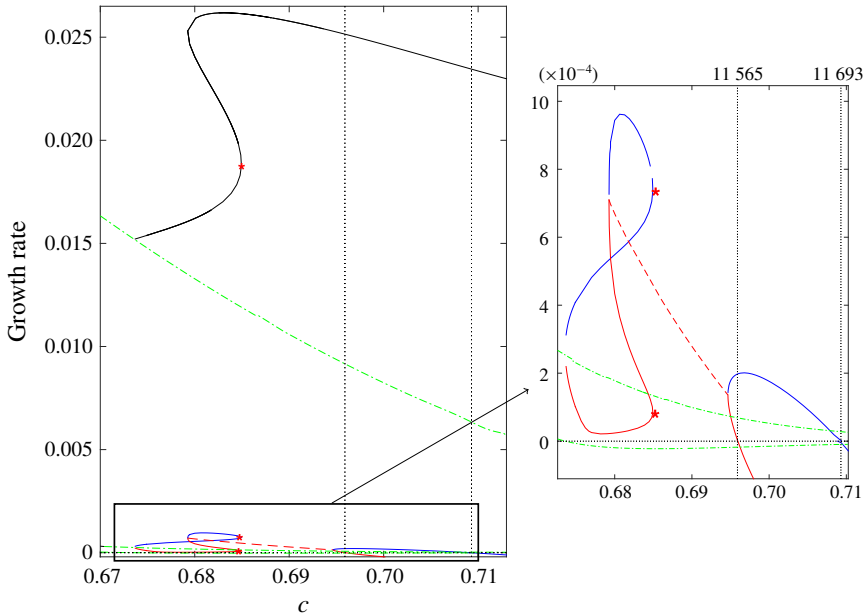


FIGURE 7. (Colour online) The growth rate versus wave speed c for the three most unstable eigenmodes for the lower-branch WK (solid red, black, blue) and WK2 (green dashed-dotted) curves for large Re in S-symmetric subspace. The curves indicate either real eigenvalues (solid) or a complex conjugate pair (red dashed) as Re changes. A red star indicates the local maximum value of c at $(Re, c) = (46\,300, 0.68494)$ in figure 1 which is marked with the same symbol. The inset shows a close-up around the bifurcations. The intersection of the horizontal and vertical dotted lines corresponds to the position of the saddle node.

4.2. Instabilities of lower-branch solutions for large Re

Here, we discuss the large Re calculations with a view to comparing them with the large Re -asymptotic stability theory. Though this theory was developed in the context of the plane Couette flows by Deguchi & Hall (2016), the scaling arguments are similar for the lower-branch states in pipe flow. The theory identified three types of unstable modes. The first and in some sense the most dangerous is a Rayleigh mode with different streamwise wavenumber and a growth rate $O(Re^0)$. The two other types of unstable modes are what the latter authors referred to as edge and slow modes with growth rates of size $O(Re^{-1/2})$ and $O(Re^{-1})$ respectively. The slow modes are rather gentle instabilities associated with a meandering of the roll–streak–wave flow. On the other hand, the ‘edge’ mode corresponds to an instability localized in a diffusion layer straddling the critical layer. Viswanath (2009a) investigated numerically at finite Re the stability properties of a solution branch different from the one reported here for the range up to $Re = 1.5 \times 10^4$. Based on a very rough fit with numerical data, he estimated $Re^{-0.41}$ and $Re^{-0.87}$ scalings for the unstable eigenmodes; these are not too different from the theoretical Deguchi–Hall scalings for the edge and slow modes respectively. Here, our calculations, which extend to a much larger Re range, show much closer agreement with the asymptotic stability theory and a linear fitting from the theory is almost indistinguishable from the numerical data on a log–log scale as shown in figure 11. This suggests that the Deguchi–Hall instability mechanisms hold

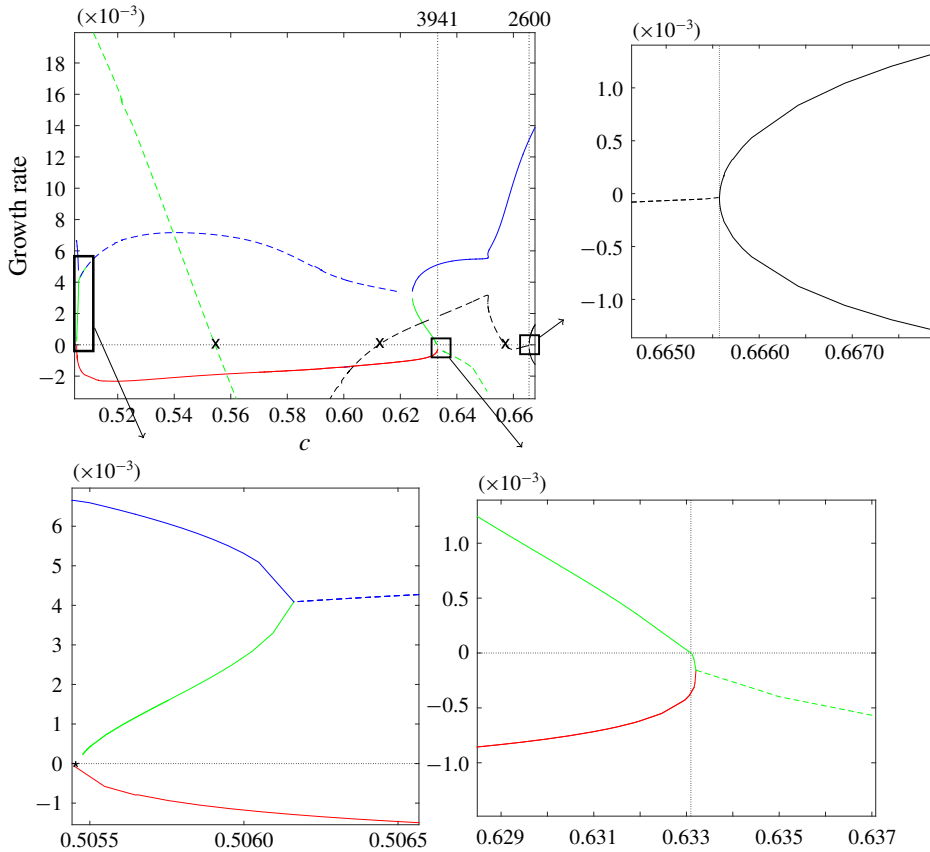


FIGURE 8. (Colour online) The growth rate versus wave speed c for the most unstable eigenmodes (red, black, green, blue) for C1 near the saddle-node bifurcation points at $Re = 2600, 3941$. The curves indicate either real eigenvalue (solid) or a complex conjugate pair (dashed) as c changes. Intersection of the horizontal and vertical dotted lines corresponds to the position of the saddle node. A black star indicates the location of symmetry-breaking bifurcation of C2 into C1 at about $c = 0.506$ in figure 1.

for all the lower-branch modes in pipes and channels. That is a direct consequence of the lower-branch states being described at large Reynolds numbers by the generic VWI theory of Hall & Smith (1991).

In § 3, we presented numerical evidence showing that the lower-branch WK solution exists up to $Re \approx 76\,438$ where it bifurcates from WK2. Therefore, it is appropriate to compare the $Re \rightarrow \infty$ asymptotic theory of Deguchi and Hall only with stability results for WK2, since WK does not exist beyond $Re \approx 76\,438$.

As discussed in § 3.1, WK2 has an asymptotic structure that consists of $O(1)$ streaks, $O(Re^{-1})$ streamwise rolls and wave components that peak to $O(Re^{-5/6})$ in a thin critical layer. Therefore, higher harmonics in the wave, which are at most $O(Re^{-5/3})$, play no role in the leading-order asymptotic Deguchi–Hall stability theory. We checked that this applies to WK2, which we identified as a finite Re realization of a VWI state, by comparing stability results for large Re corresponding to a single-mode axial approximation with a fully resolved TW solution. Figure 10 shows the linear stability results for a single axial mode WK2 solution approximation for

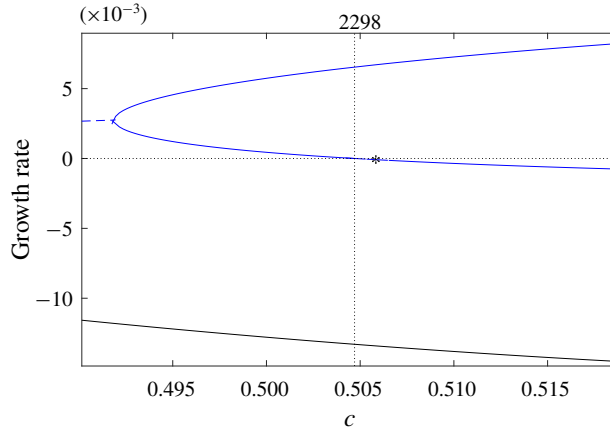


FIGURE 9. (Colour online) The growth rate versus wave speed c near the saddle-node bifurcation points at $Re = 2298$ for the three most unstable eigenmodes (blue, black) for the C2 solutions with \mathbb{S} and Ω_2 symmetry. A black star indicates the location of the symmetry-breaking bifurcation which results in the C1 branch.

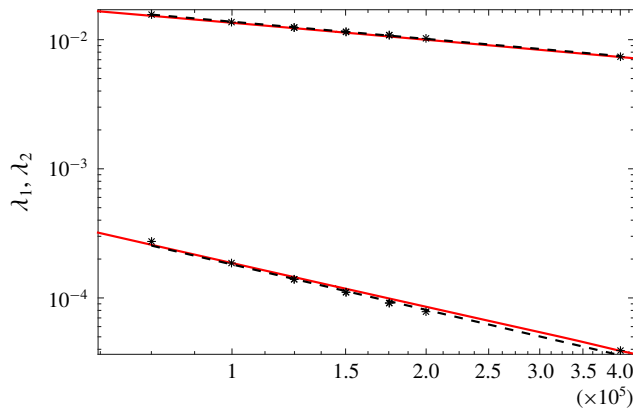


FIGURE 10. (Colour online) The growth rates λ_1, λ_2 for the edge and slow modes against Re on a log–log scale for the WK2 branches at $\alpha = 1.55$. The red line indicates stability of a fully resolved TW, while a black star indicates linear stability for single axial mode approximation to WK2. The black dashed line shows the linear fitting of growth rates for a single-mode approximation. Note that λ_1, λ_2 are real.

$6 \times 10^4 < Re < 4 \times 10^5$. Two unstable modes with real growth rates have been found. The linear fits of growth rates (black dotted line) corresponding to each unstable eigenmode, shown on a log–log scale, are based on a best-fit estimate of the computed data (black dots) which is close to the growth rates corresponding to a fully resolved WK2 solution in the range of Re shown as red solid lines.

Growth rates from a linear stability analysis of the fully resolved solution are shown in figure 11. The growth rates of the two unstable modes of WK2 decay like $Re^{-0.43}$ and $Re^{-1.13}$. This is in good agreement ($Re^{-0.42}$ and $Re^{-1.12}$) with the single axial mode approximation of a TW solution (see figure 10), and in rough agreement with the asymptotic $Re^{-1/2}$ and Re^{-1} scalings of Deguchi & Hall (2016) for the edge

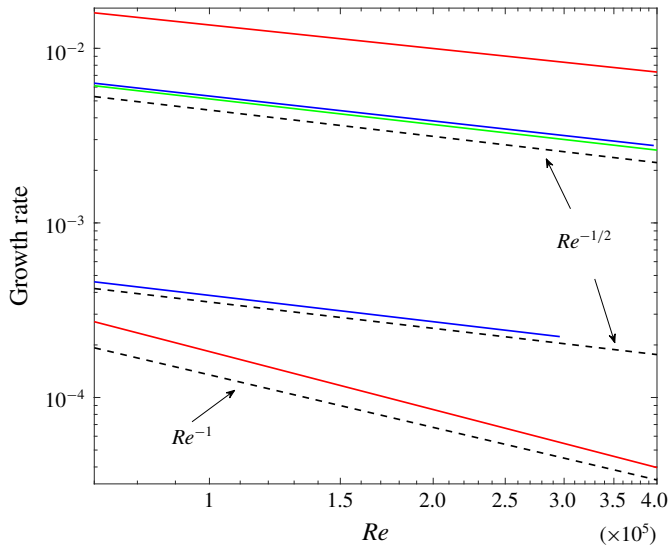


FIGURE 11. (Colour online) The growth rates of the unstable modes against Re on a log–log scale for WK2 (red), C1 (blue) and C2 (green) at $\alpha = 1.55$. The black dashed lines indicate the asymptotic scalings $Re^{-1/2}$ and Re^{-1} for comparison. The growth rates for the C1 branch are only shown for $Re < 3 \times 10^5$ due to computational storage limitations.

and slow instability modes. In spite of the seemingly large values of Re used in our numerical calculations, the agreement with the leading-order asymptotic theory is not close. The poor agreement is probably due to higher order corrections associated with small fractional powers of $1/Re$ in the asymptotic theory. In this context, it may be noted that even for the TW itself, agreement with the asymptotic theory was only conclusive after calculations in the range $1.5 \times 10^4 < Re < 1.19 \times 10^6$ reported in the first part of this paper, whereas previous calculations of WK for the range $5 \times 10^3 < Re < 1.1 \times 10^4$ in Ozcakir *et al.* (2016) were rather inconclusive.

A further check on the edge mode is available. A key property of the Deguchi–Hall edge instability mode is that the wave pressure eigenfunction is, to leading order, proportional to the equilibrium wave pressure of the VWI state. Figure 12 shows an excellent agreement between the pressure eigenfunction corresponding to the linearized stability problem and the pressure profile of the equilibrium WK2 state at $Re = 125\,000$.

Further, as shown in figure 13(a), the streak component of the eigenfunction associated with the edge states shows that the streak action is concentrated around the critical layer, which is similar to the viscous sublayer structure of the edge states found in Deguchi & Hall (2016).

As opposed to the VWI states, all the axial wave harmonics are of the same order in Re throughout a shrinking core for the NVC states. Computationally that means more computational space is required for the linear stability analysis at large Re values. Using available resources, we limited our attention to the calculation of C2 stability for $Re \leq 5 \times 10^6$ to disturbances that have the additional shift-and-rotate symmetry as the base state. The stability calculations for C1 are restricted to $Re \leq 3 \times 10^5$ due to computational limitations; these states have no shift-and-rotate symmetry and therefore eigenfunctions of the stability operator do not have this symmetry either. The stability features of C1 are shown in figure 11.

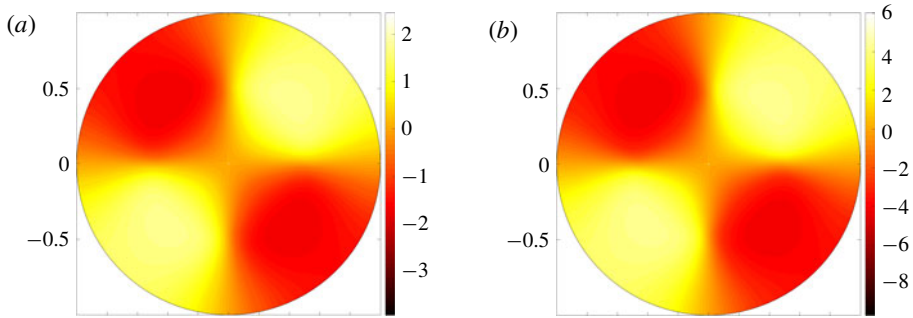


FIGURE 12. (Colour online) The pressure component of (a) equilibrium solution at $\tilde{z}_0 = 2\pi/\alpha$ at $Re = 125\,000$, with corresponding (b) pressure eigenfunction associated with edge mode of WK2.

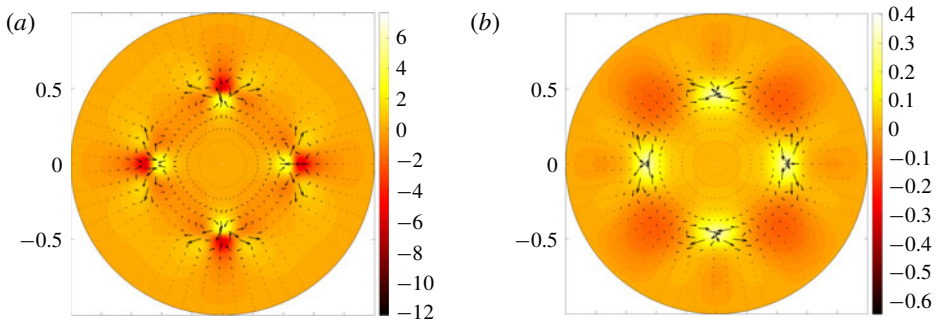


FIGURE 13. (Colour online) Roll and streak components of unstable eigenmodes (a) η_1 associated with edge and (b) η_2 slow modes of WK2 at $Re = 125\,000$.

For C1, the stability calculations suggest the existence of two real unstable modes with growth rates scaling like $Re^{-0.49}$ and $Re^{-0.46}$. For C2, there is only one real unstable mode with growth rate scaling like $Re^{-0.46}$ over the range $7 \times 10^4 \leq Re \leq 5 \times 10^6$. The streak components of the eigenfunctions associated with the unstable eigenmodes of C1 and C2 are shown in figure 14. It is clear from figure 14(a,c) that for C2 the most unstable eigenmode is concentrated near the core, while the most unstable eigenmode for C1 is spread throughout the pipe. Also, it is worth noting that the most unstable C mode in figure 14(a) has almost a four-fold azimuthally symmetric streak structure quite similar to that of figure 14(c). However the second eigenfunction has an apparent two-fold azimuthal symmetry. Based on these observations, we suspect that at some large Re value, the second most unstable C1 eigenvalue which has an eigenfunction spread across the pipe goes through a zero and becomes stable; and that this zero crossing corresponds to an azimuthal-symmetry-breaking bifurcation of C1 from C2 at some large value of Re . A confirmation of this scenario requires computations beyond our present resources. If this conjecture holds, only C2 survives as $Re \rightarrow \infty$. Moreover it has only one unstable mode within the class of symmetry-preserving disturbances; this implies that C2 is an edge state in this state space. Furthermore the growth rate for this unstable mode scales like $Re^{-1/2}$ which is in surprisingly good agreement with the Deguchi–Hall edge mode predictions since the NVC states have a very different

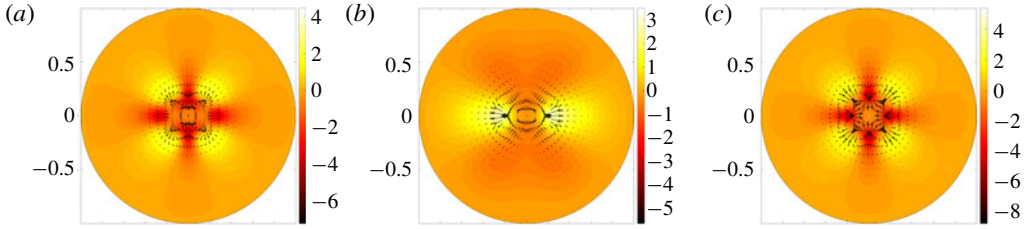


FIGURE 14. (Colour online) The roll and streak components of the unstable eigenmodes (a) η_1 associated with the edge and (b) η_2 slow modes of C1 at $Re = 102\,820$. (c) The roll and streak components of the unstable eigenmode η of C2 at $Re = 102\,820$.

asymptotic structure from the VWI states. However for more general perturbations it is not clear if it would remain an edge state.

4.3. Linear stability of C2 for constant mean mass flux

All calculations reported thus far correspond to the mean pressure gradient being held fixed. However, through a transformation of the variables and parameters (see (A 8)), a TW solution with the same mean pressure gradient as the base flow \mathbf{v}_B can be mapped onto a TW solution for the same mass flux as \mathbf{v}_B . Details of the mapping are given in appendix A. However, no such transformation exists for the stability modes. Imposition of a constant mean pressure gradient does not preserve mean mass flux and *vice versa*. Thus, these disturbance modes are fundamentally different. This has been recognized earlier in the literature in a broader context; see for example Soibelman & Meiron (1991). Our numerical calculations also confirm that constant pressure gradient modes do not satisfy the mass-flux condition (2.8) and *vice versa*.

We were motivated to conduct a limited linear stability study for constant mass flux in order to compare with the calculations reported in Pringle *et al.* (2009). As mentioned earlier in § 3, close agreement of the C versus Re curves shown in figure 2 suggests that the C2 solution curve that we first reported in Ozcair *et al.* (2016) is a large Re continuation of their ‘N2’ solution branch. However, when we attempted to reproduce their constant mass-flux stability results in the vicinity of saddle-node bifurcation point $R_m \approx 1441$ ($Re \approx 2040$) when $\alpha = 1.2$ in the S -symmetric space as shown in § 4.3, we noticed some discrepancies. Figure 15 shows stability calculations of N2 performed by Pringle *et al.* (2009) while figure 16 shows our stability results in the same regime. We find one real and two complex conjugate pairs of unstable eigenvalues on the upper branch that become two real and two complex conjugate pairs of unstable eigenvalues on the lower branch. However Pringle *et al.* (2009) report three real and a complex conjugate pair of unstable eigenvalues on the upper branch and two real and a complex conjugate pair of unstable eigenvalues on the lower branch. We checked our calculations by significantly increasing the number of modes in each of the radial, azimuthal and axial directions with no change in conclusion. The details of resolution checks can be found in appendix C. At this point, we are unsure about the cause of this discrepancy. One explanation is that, though the C versus R_m curves in figure 2 match closely, N2 is actually different from C2. Another possibility suggested by a referee is potential resolution issues in the Pringle calculations due to computational limitations at the time.

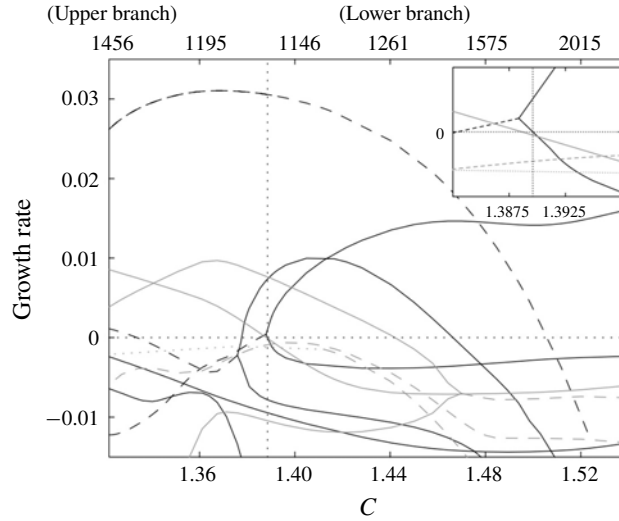


FIGURE 15. The growth rate calculations for C3 of Pringle *et al.* (2009) (which is referred to as C2 in this paper) branches at $\alpha = 1.2$ under the constant mass-flux constraint. The stability analysis was performed in the R_2 symmetric subspace in the vicinity of the saddle-node bifurcation at $R_m = 1141$ at $\alpha = 1.2$. Each curve indicates either real eigenvalue (solid curves) or a complex conjugate pair (dashed curves) as C changes. The black curves correspond to those that are symmetric under S , while the grey curves are antisymmetric under S . Intersection of the horizontal and vertical dotted lines corresponds to the position of the saddle node, of which the inset is a close-up. Saddle-node bifurcations are observed at $C = 1.33, 1.38, 1.385, 1.388$ and 1.389 and Hopf bifurcations are observed at $1.44, 1.47$ and 1.50 .

5. Conclusion

In this paper, we have reported on new numerical computations of TW solutions with shift-and-rotate symmetry. We presented some features of these solutions, including scaling of the lower-branch WK2 with Reynolds number in the range $1.5 \times 10^4 < Re < 1.19 \times 10^6$. Quantitative evidence suggests rather conclusively that the calculated lower-branch WK2 solution is indeed a finite Re realization of a VWI state with the expected Re^{-1} scale for rolls, $O(1)$ scale for streaks and a maximal wave amplitude of $O(Re^{-5/6})$.

From the agreement of c versus Re relation over a certain range of Re for particular axial wavenumber, it appears that the WK2 and C2 solutions are continuations of the so-called ‘N’- and ‘M’-class solutions of Pringle *et al.* (2009) with two-fold azimuthal symmetry. Their classification is based on fast/slow streak behaviour on a pipe cross-section with the M-class having a double-layer structure of fast and slow streaks across the pipe radius while the N-class have a separation of the fast streaks near the wall and towards the pipe centre. On the other hand, the evidence in this paper and in Ozcakir *et al.* (2016) suggests that the WK2 and C2 solutions are finite Re representations of VWI and NVC states respectively. It is very likely that other N- and M-class solutions of Pringle *et al.* (2009) could also be classified as VWI and NVC states in the asymptotic limit respectively.

Further, we determined computationally the linear stability properties of the TW solutions C1, C2, WK and WK2. We found that at the saddle-node bifurcation points

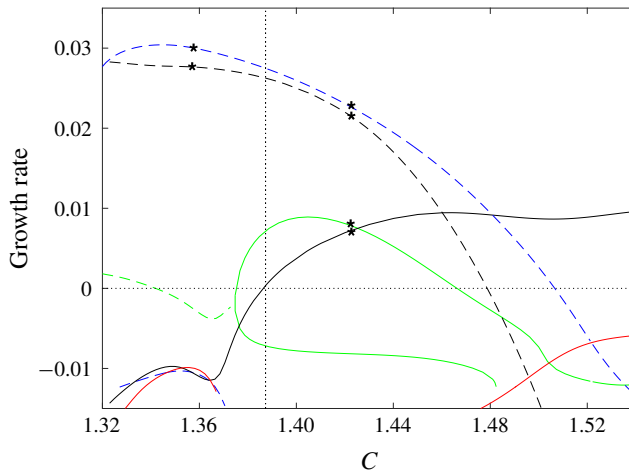


FIGURE 16. (Colour online) The growth rates of the constant mass-flux unstable modes of C2 in the vicinity of the saddle-node bifurcation at $R_m \approx 1141$ at $\alpha = 1.2$. The stability analysis was performed in the two-fold rotational (\mathbb{R}_2) subspace with \mathbb{S} -symmetry. Curves indicate either real eigenvalue (solid curves) or a complex conjugate pair (dashed curves) as C changes. Intersection of the horizontal and vertical dotted lines corresponds to the position of the saddle node shown in figure 2. Resolution checks of the unstable growth rates marked by stars at $R_m = 1190$ are given in appendix C.

in the c - Re plane, there is a zero crossing of one real eigenvalue. We confirmed that WK arises as a symmetry-breaking bifurcation from WK2. Our calculations show that C1 continues at least up to $Re = 2 \times 10^8$ and gets closer to C2 as Re gets larger, and observations on the unstable eigenmodes of C1 and C2 suggest a similar bifurcation scenario. Unfortunately our calculations did not have enough resolution to confirm whether these two branches join at very high values of Re .

We also identified Hopf bifurcation points for C1; these will give rise to relative periodic orbits similar to those discussed in Viswanath (2007), Duguet *et al.* (2008a) and Deguchi & Hall (2016), but we have not pursued them in this paper. Also, it is interesting to note that an initial edge state solution in the form of a WK lower branch has only one real unstable eigenvalue. When it is continued to larger $Re > 11\,000$, it gains additional unstable eigenvalues before it joins WK2. We confirmed that these lower-branch TWs have slow and low-dimensional unstable manifolds. Thus, WK2 has two unstable eigenvalues asymptotically scaling as $Re^{-0.43}$ and $Re^{-1.13}$, whereas the eigenvalue of C2 scales as $Re^{-0.46}$ suggesting their relevance to transition in turbulence. Thus, lower-branch C2 solutions are edge states that form part of the boundary separating laminar and turbulent flows.

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Appendix A. The relation between constant-pressure and mass-flux base flows

Calculations in the literature for the most part use constant flux, i.e. the periodically averaged component of the flux is the same as the Hagen–Poiseuille flow, rather than

maintaining the same mean pressure gradient. For TWs, there is a mapping of the Reynolds number and phase speed that maps the constant-pressure TW solution to equivalent constant mass-flux TW. This is well known, and is demonstrated below for completeness. Suppose, for constant flux, the TW solution $\bar{\mathbf{u}}$ is decomposed as $\bar{\mathbf{u}} = \tilde{\mathbf{v}}(r, \theta, z - c_m t) + w_0(r)\hat{\mathbf{z}}$, where

$$w_0(r) = -\frac{Q_0 R_m}{4}(1 - r^2) \tag{A 1}$$

is a pressure eigenfunction Poisseuille correction, and therefore there is no averaged flux contribution from $\tilde{\mathbf{v}}$. The constancy of the mass flux determines Q_0 in terms of $\tilde{\mathbf{v}}$. Substituting the expression for $\bar{\mathbf{u}}$ into the Navier–Stokes equations gives

$$-c_m \tilde{\mathbf{v}}_z + \tilde{\mathbf{v}} \cdot \nabla \tilde{\mathbf{v}} + (w_0 \hat{\mathbf{z}} + \mathbf{v}_B) \cdot \nabla \tilde{\mathbf{v}} + \tilde{\mathbf{v}} \cdot \nabla (w_0 \hat{\mathbf{z}} + \mathbf{v}_B) = -\nabla \bar{p} + \frac{1}{R_m} \Delta \tilde{\mathbf{v}}, \tag{A 2}$$

where c_m and R_m denote the phase speed and the mean Reynolds number, respectively, for the constant mean mass-flux calculations. We find

$$w_0 \hat{\mathbf{z}} + \mathbf{v}_B = \left(1 - \frac{Q_0 R_m}{4}\right) (1 - r^2) \hat{\mathbf{z}} = \mu \mathbf{v}_B, \quad \text{where } \mu = \left(1 - \frac{Q_0 R_m}{4}\right). \tag{A 3}$$

This implies that

$$-c_m \tilde{\mathbf{v}}_z + \tilde{\mathbf{v}} \cdot \nabla \tilde{\mathbf{v}} + \mu \mathbf{v}_B \cdot \nabla \tilde{\mathbf{v}} + \tilde{\mathbf{v}} \cdot \nabla \mu \mathbf{v}_B = -\nabla \bar{p} + \frac{1}{R_m} \Delta \tilde{\mathbf{v}}. \tag{A 4}$$

Next we define new variables $\tilde{\mathbf{v}} = \mu \mathbf{v}$, $\bar{p} = \mu^2 q$ and divide (A 4) by μ^2 , so that

$$-\frac{c_m}{\mu} \mathbf{v}_z + \mathbf{v} \cdot \nabla \mathbf{v} + \mathbf{v}_B \cdot \nabla \mathbf{v} + \mathbf{v} \cdot \nabla \mathbf{v}_B = -\nabla q + \frac{1}{R_m \mu} \Delta \mathbf{v}; \quad \nabla \cdot \mathbf{v} = 0, \tag{A 5}$$

which is the same as equation (2.6) for TWs with constant mean pressure gradient provided we identify

$$\frac{c_m}{\mu} = c, \quad \mu R_m = Re. \tag{A 5a,b}$$

In order to determine Q_0 and hence μ (noting the relation (A 3)), we invoke the mass-flux constraint

$$\mu \int_0^1 (\mathbf{v} \cdot \hat{\mathbf{z}})_{00} r \, dr = - \int_0^1 w_0(r) r \, dr = \frac{Q_0 R_m}{4} \int_0^1 (1 - r^2) r \, dr = \frac{Q_0 R_m}{16}, \tag{A 6}$$

which together with (A 3) implies that

$$Q_0 = \frac{16}{R_m} \frac{\mathcal{I}}{1 + 4\mathcal{I}} \quad \text{and} \quad \mu = \frac{1}{1 + 4\mathcal{I}}, \quad \text{where } \mathcal{I} = \int_0^1 \overline{(\mathbf{v} \cdot \hat{\mathbf{z}})} r \, dr, \tag{A 7a,b}$$

where $\overline{(\quad)}$ refers to azimuthal and axial period averaging.

Since the numerical evidence suggests that the Jacobian associated with (2.6) is non-singular except at bifurcation points, we therefore have a locally unique solution ensuring that the solution \mathbf{v} of (A 5) is the same constant-pressure solution of (2.6), provided we relate (c_m, R_m) to (c, Re) through (A 5) and (A 7). Thus the numerical calculation reported here for constant mean pressure gradient can be related to constant mass-flux calculations that appear elsewhere in the literature, see for example

(N_2, M_2, P_2)	(220, 20, 10)	(250, 20, 5)	(240, 26, 5)	(220, 20, 3)
c	2.9618×10^{-12}	9.3175×10^{-9}	4.4765×10^{-6}	1.3546×10^{-8}
$\ u\ $	6.4829×10^{-9}	6.9768×10^{-4}	4.4765×10^{-6}	6.9757×10^{-4}
$\ v\ $	4.1408×10^{-9}	0.0012	0.0051	0.0012
$\ w\ $	4.3832×10^{-11}	2.5490×10^{-6}	1.0352×10^{-4}	2.5658×10^{-6}
$\ p\ $	1.9397×10^{-8}	7.8178×10^{-6}	1.6466×10^{-4}	5.2176×10^{-6}

TABLE 2. The relative errors found by comparing the numerical solutions at various levels of truncation (N_2, M_2, P_2) with the baseline calculations with $(N_1, M_1, P_1) = (220, 20, 5)$ for the WK2 curve for $(Re, \alpha) = (710\,000, 1.55)$.

(N_2, M_2, P_2)	(145, 16, 8)	(165, 24, 10)	(225, 18, 10)
c	4.8971×10^{-7}	4.8976×10^{-7}	4.8975×10^{-7}
$\ u\ $	0.00031833	5.3869×10^{-5}	0.00028628
$\ v\ $	0.00046481	4.5058×10^{-5}	0.00040857
$\ w\ $	2.4079×10^{-5}	9.7147×10^{-6}	2.5107×10^{-5}
$\ p\ $	0.00015968	0.00010804	0.00014642

TABLE 3. The relative errors found by comparing the numerical solutions at various levels of truncation (N_2, M_2, P_2) with the baseline calculations with $(N_1, M_1, P_1) = (165, 18, 5)$ for the C2 curve for $(Re, \alpha) = (191\,020, 1.55)$.

Pringle *et al.* (2009), and the full transformation corresponds to

$$(\bar{u}, \bar{p}, c_m; R_m) = \left(\mu u - 4\mu \mathcal{I}(1 - r^2)\hat{z}, \mu^2 p, \mu c, \frac{Re}{\mu} \right) \quad \text{with } \mu, \mathcal{I} \text{ in (A 7)}. \quad (\text{A 8})$$

It is to be noted that μ in (A 7) is related to the mean flow velocity W_m appearing through the relation $\mu = 1/2W_m$. Therefore (A 8) is equivalent to $R_m = 2W_m Re$ and $c_m = c/2W_m$.

Appendix B. Resolution checks for TWs

In this appendix, we confirm the accuracy of the WK2, C1 and C2 TW calculations by comparing solutions at different truncations (N, M, P) . We performed detailed resolution checks at a few Re values when $k_0 = 2, \alpha = 1.55$. Results shown in the tables 2–7 display relative errors between the two compared solutions for phase speed, velocity components and pressure. In each table, we compare a TW solution calculated at truncation levels (N_1, M_1, P_1) and (N_2, M_2, P_2) .

Let X_1 be the solution at truncation level (N_1, M_1, P_1) and X_2 the solution at truncation level (N_2, M_2, P_2) , where $N_1 \leq N_2, M_1 \leq M_2$ and $P_1 \leq P_2$. The comparison is made by taking the difference between these two solutions by injecting the lower resolution solution X_1 into the higher dimension by adding zeros in the Fourier θ and z domain. Then each velocity component and pressure are computed in the real radial domain at the same radial collocation points using interpolation in the radial domain while keeping the solution in the Fourier θ and z space. The error comparison is done by computing relative errors using the infinity norm for each component in Fourier space separately. We denote $u = \{u_{jkl} | 0 \leq j \leq N, 0 \leq k \leq M, 0 \leq l \leq P\}$ and define v and w in the same manner.

(N_2, M_2, P_2)	(240, 20, 8)	(200, 20, 16)	(200, 26, 8)	(165, 18, 5)
c	6.2196×10^{-11}	2.3975×10^{-10}	3.7410×10^{-13}	8.7421×10^{-7}
$\ u\ $	5.4743×10^{-4}	3.9257×10^{-7}	2.7549×10^{-7}	0.0014
$\ v\ $	8.6642×10^{-4}	1.2161×10^{-7}	6.1228×10^{-8}	0.0023
$\ w\ $	2.5524×10^{-5}	1.3718×10^{-8}	1.1954×10^{-8}	9.7393×10^{-5}
$\ p\ $	5.0887×10^{-4}	2.9872×10^{-7}	6.5601×10^{-9}	6.8367×10^{-4}

TABLE 4. The relative errors found by comparing the numerical solutions at various levels of truncation (N_2, M_2, P_2) with the baseline calculations with $(N_1, M_1, P_1) = (220, 20, 8)$ for the C2 curve for $(Re, \alpha) = (1\ 026\ 020, 1.55)$.

(N_2, M_2, P_2)	(220, 16, 8)	(220, 28, 12)	(220, 20, 18)	(280, 20, 12)
c	3.098×10^{-6}	7.8267×10^{-12}	3.0994×10^{-6}	3.0997×10^{-6}
$\ u\ $	0.01002	3.3581×10^{-7}	0.010267	0.034912
$\ v\ $	0.013876	8.6652×10^{-8}	0.013514	0.052489
$\ w\ $	0.00083999	1.1745×10^{-8}	0.00087691	0.0015064
$\ p\ $	0.0078918	3.6936×10^{-8}	0.0078461	0.0083281

TABLE 5. The relative errors found by comparing the numerical solutions at various levels of truncation (N_2, M_2, P_2) with the baseline calculations with $(N_1, M_1, P_1) = (220, 20, 12)$ for the C2 curve for $(Re, \alpha) = (8\ 241\ 020, 1.55)$.

(N_2, M_2, P_2)	(200, 20, 24)	(250, 20, 16)	(170, 26, 16)	(200, 16, 12)
c	2.0407×10^{-7}	1.9212×10^{-7}	3.4636×10^{-7}	9.18388-10
$\ u\ $	2.6788×10^{-5}	0.0020165	0.003617	1.65228×10^{-6}
$\ v\ $	2.5178×10^{-5}	0.0033254	0.0059893	2.9987×10^{-7}
$\ w\ $	1.0714×10^{-5}	7.9784×10^{-5}	0.00015994	1.279578×10^{-7}
$\ p\ $	0.00090359	0.001207	0.027478	6.93519×10^{-7}

TABLE 6. The relative errors found by comparing the numerical solutions at various levels of truncation (N_2, M_2, P_2) with the baseline calculations with $(N_1, M_1, P_1) = (220, 20, 12)$ for the C1 curve for $(Re, \alpha) = (1\ 261\ 020, 1.55)$.

(N_2, M_2, P_2)	(280, 20, 12)	(220, 28, 12)	(220, 20, 20)	(220, 16, 12)
c	4.4335×10^{-7}	9.1582×10^{-10}	9.5719×10^{-10}	1.5764×10^{-9}
$\ u\ $	0.0041	1.9255×10^{-7}	2.0646×10^{-7}	1.6817×10^{-6}
$\ v\ $	0.0067	1.7464×10^{-7}	1.8106×10^{-7}	3.2513×10^{-7}
$\ w\ $	1.3777×10^{-4}	4.5631×10^{-8}	4.7229×10^{-8}	1.1944×10^{-7}
$\ p\ $	0.0022	3.5247×10^{-7}	3.6970×10^{-7}	8.0244×10^{-7}

TABLE 7. The relative errors found by comparing the numerical solutions at various levels of truncation (N_2, M_2, P_2) with the baseline calculations with $(N_1, M_1, P_1) = (220, 20, 12)$ for the C1 curve for $(Re, \alpha) = (3\ 263\ 020, 1.55)$.

It is clear from the findings that when Re gets larger, more radial N and azimuthal M modes are required to get good accuracy for WK2, C1 and C2, while a low value of $P=5$ is enough to achieve remarkable accuracy due to the single dominating axial mode property of VWI states.

(N_2, M_2, P_2)	(225, 16, 5)	(125, 28, 5)	(125, 16, 10)
λ_1	2.8514×10^{-7}	6.8565×10^{-8}	2.862×10^{-10}
$\ \tilde{u}_1\ $	0.00060697	0.0010804	1.5679×10^{-8}
$\ \tilde{v}_1\ $	0.00073511	0.00036073	1.0696×10^{-8}
$\ \tilde{w}_1\ $	0.00032027	0.00046045	3.2993×10^{-9}
λ_2	1.6915×10^{-5}	9.0423×10^{-6}	1.8717×10^{-7}
$\ \tilde{u}_2\ $	0.020097	0.0017363	1.7293×10^{-8}
$\ \tilde{v}_2\ $	0.029492	0.00057972	1.1222×10^{-8}
$\ \tilde{w}_2\ $	0.021792	0.001649	9.4181×10^{-8}

TABLE 8. The relative errors found by comparing the numerical solutions at various levels of truncation (N_2, M_2, P_2) with the baseline calculations with $(N_1, M_1, P_1) = (125, 16, 5)$ for the unstable eigenvalues $\lambda_1 = 0.01227, \lambda_2 = 0.00014474$ for the WK2 curve for $(Re, \alpha) = (125\,000, 1.55)$. Note that λ_1 corresponds to WK2 edge modes; the cross-sectional velocity and pressure profiles of the corresponding eigenfunction are given in figures 12 and 13.

(N_2, M_2, P_2)	(245, 16, 8)	(165, 28, 8)	(165, 16, 12)
λ_1	3.3746×10^{-11}	6.8365×10^{-9}	1.9062×10^{-9}
$\ \tilde{u}_1\ $	0.00015444	5.2622×10^{-7}	2.0642×10^{-7}
$\ \tilde{v}_1\ $	0.00020234	1.204×10^{-7}	7.1383×10^{-8}
$\ \tilde{w}_1\ $	0.00011014	1.8628×10^{-7}	7.3407×10^{-8}
λ_2	9.2387×10^{-10}	1.3284×10^{-7}	1.0805×10^{-8}
$\ \tilde{u}_2\ $	0.00021073	5.7046×10^{-7}	2.146×10^{-7}
$\ \tilde{v}_2\ $	0.00024949	6.4594×10^{-7}	7.0718×10^{-8}
$\ \tilde{w}_2\ $	0.00026735	9.604×10^{-7}	1.7931×10^{-7}

TABLE 9. The relative errors found by comparing the numerical solutions at various levels of truncation (N_2, M_2, P_2) with the baseline calculations with $(N_1, M_1, P_1) = (165, 16, 8)$ for the unstable eigenvalues $\lambda_1 = 0.0039247, \lambda_2 = 0.00027827$ for the C1 curve for $(Re, \alpha) = (191\,020, 1.55)$.

(N_2, M_2, P_2)	(225, 16, 8)	(145, 28, 8)	(145, 16, 12)
λ_1	1.311×10^{-10}	4.0829×10^{-9}	3.8367×10^{-10}
$\ \tilde{u}_1\ $	0.00022143	1.517×10^{-6}	2.795×10^{-7}
$\ \tilde{v}_1\ $	0.00029623	5.2331×10^{-7}	9.31×10^{-8}
$\ \tilde{w}_1\ $	0.00010706	6.6183×10^{-7}	4.9492×10^{-8}

TABLE 10. The relative errors found by comparing the numerical solutions at various levels of truncation (N_2, M_2, P_2) with the baseline calculations with $(N_1, M_1, P_1) = (145, 16, 8)$ for the only unstable eigenvalue ($\lambda_1 = 0.0037345$) for the C2 curve for $(Re, \alpha) = (191\,020, 1.55)$.

Appendix C. Details of the stability problem and resolution checks for the growth rates

For small Re values a standard eigenvalue solver is used to compute eigenvalues with good numerical accuracy. However, when Re is large, this becomes a numerical challenge since a calculation of the TWs for high values of Re is computationally expensive due to the size of the Jacobian needed to resolve the solution. For the linear

(N_2, M_2, P_2)	(95, 16, 12)	(65, 24, 12)	(65, 16, 24)
λ_1	3.3275×10^{-12}	4.2304×10^{-7}	9.108×10^{-6}
$\ \tilde{u}_1\ $	9.1065×10^{-5}	0.00016624	0.0028156
$\ \tilde{v}_1\ $	3.7249×10^{-5}	1.6415×10^{-5}	0.0015999
$\ \tilde{w}_1\ $	5.4659×10^{-5}	7.4142×10^{-5}	0.00030233
λ_2	1.9315×10^{-12}	4.8291×10^{-7}	0.00021976
$\ \tilde{u}_2\ $	6.7283×10^{-5}	0.00010195	0.0090789
$\ \tilde{v}_2\ $	6.5355×10^{-5}	2.6398×10^{-5}	0.011366
$\ \tilde{w}_2\ $	8.2762×10^{-5}	0.00013701	0.0071189
λ_3	8.6174×10^{-11}	1.4469×10^{-6}	5.0084×10^{-5}
$\ \tilde{u}_3\ $	0.0001305	0.00010523	0.0045826
$\ \tilde{v}_3\ $	0.00012388	2.8511×10^{-5}	0.0021821
$\ \tilde{w}_3\ $	0.00013892	9.3591×10^{-5}	0.00030091
λ_4	2.0889×10^{-10}	2.6119×10^{-6}	2.1153×10^{-5}
$\ \tilde{u}_4\ $	0.00022181	5.1387×10^{-5}	0.026387
$\ \tilde{v}_4\ $	9.3608×10^{-5}	5.1387×10^{-5}	0.016377
$\ \tilde{w}_4\ $	0.00034078	3.7822×10^{-5}	0.0047094

TABLE 11. The relative errors found by comparing the numerical solutions at various levels of truncation (N_2, M_2, P_2) with the baseline calculations with $(N_1, M_1, P_1) = (65, 16, 12)$ for the unstable eigenvalues $\lambda_1 = 0.045528 + 0.15917i$, $\lambda_2 = 0.043037 + 0.42324i$, $\lambda_3 = 0.015814$, $\lambda_4 = 0.014294$, for the mass-flux-preserving lower C2 branch for $\alpha = 1.2$ given in figure 2 for $(R_m, C) = (1190, 1.4218)$. These unstable eigenvalues are marked with stars in figure 16.

(N_2, M_2, P_2)	(95, 16, 12)	(65, 24, 12)	(65, 16, 24)
λ_1	6.6669×10^{-10}	4.3001×10^{-6}	0.00063003
$\ \tilde{u}_1\ $	0.00031151	0.0033991	0.016814
$\ \tilde{v}_1\ $	0.00018061	0.00023451	0.015286
$\ \tilde{w}_1\ $	0.0001777	0.0018094	0.0041389
λ_2	1.1437×10^{-9}	3.2987×10^{-6}	0.00049332
$\ \tilde{u}_2\ $	0.0004615	0.0011232	0.050851
$\ \tilde{v}_2\ $	0.00033636	0.00013109	0.020596
$\ \tilde{w}_2\ $	0.00030505	0.0012058	0.088373

TABLE 12. The relative errors found by comparing the numerical solutions at various levels of truncation (N_2, M_2, P_2) with the baseline calculations with $(N_1, M_1, P_1) = (65, 16, 12)$ for the unstable eigenvalues $\lambda_1 = 0.060732 + 0.15325i$, $\lambda_2 = 0.055847 + 0.42618i$ for the mass-flux-preserving upper C2 branch for $\alpha = 1.2$ given in figure 2 for $(R_m, C) = (1190, 1.3568)$. These unstable eigenvalues are marked with stars in figure 16.

stability of these converged TW solutions at high Re , a higher resolution has to be chosen; therefore the matrix created from the linearized equation (2.6) will have to be at least as big as the Jacobian for the converged TW. For example, when $Re = 125\,000$, TWs are calculated with $(N, M, P) = (125, 16, 5)$. Using the linearized equation (2.6) for the stability analysis about these TWs at the same truncation level will create a matrix of size $68\,419 \times 68\,419$ for shift-and-reflect symmetric states WK, C1 and $34\,399 \times 34\,399$ for WK2 and C2 which have additional shift-and-rotate symmetry.

For this reason, we solve the eigenvalue problem on a coarse grid only at a few small Re values around the bifurcation points in figure 1. The resulting eigenvalues are useful in determining the generic features of the system, i.e. number of unstable real and complex eigenvalues. Then, the first few most unstable eigenvalues and the corresponding eigenfunctions are used as rough initial guesses $(\lambda_0, \mathbf{y}_0)$ for the stability problem at larger Re in a Newton iteration scheme to solve for a specific eigenvalue–eigenvector pair (λ, \mathbf{y}) satisfying $\mathbf{A}\mathbf{y} - \lambda\mathbf{y} = 0$. Through a continuation procedure in Re , this method allows us to compute eigenvalues and eigenfunctions very accurately using GMRES at each Newton iteration without constructing a huge matrix \mathbf{A} . It should be noted that in the parameter space (Re or c) when a complex conjugate pair of eigenvalues in the form $\lambda = \text{Re}\lambda \pm i\text{Im}\lambda$, with corresponding eigenvectors $\mathbf{y} = \text{Re}\mathbf{y} \pm i\text{Im}\mathbf{y}$, coalesce and become real, the nonlinear Newton solver was modified by treating $\text{Re}\lambda$, $\text{Im}\lambda$, $\text{Re}\mathbf{y}$ and $\text{Im}\mathbf{y}$ as distinct real unknowns. This resulted in a Jacobian matrix $\hat{\mathbf{A}}$ that is twice the size of matrix \mathbf{A} , but allows for the continuation of an eigenvalue–eigenvector pair past the transition point from real to complex, or *vice versa*.

Results shown in the tables display the relative errors at a few Re values between the two compared solutions for the i th most unstable eigenvalue (growth rate) λ_i , velocity components $(\tilde{u}_i, \tilde{v}_i, \tilde{w}_i)$ of the corresponding eigenfunction y_i . In each table, we compare linear stability calculations of a given TW solution calculated at truncation levels (N_1, M_1, P_1) and (N_2, M_2, P_2) using the same approach used in appendix B.

In tables 8–10 we confirm the accuracy of the growth rate calculations for WK2, C1 and C2 TWs shown in figure 11 by comparing solutions at different truncations (N, M, P) , where we imposed constant pressure gradient condition. The growth rate results presented are accurate to at least seven significant digits based on these computational verifications, while the corresponding eigenfunctions are only accurate to at least two digits.

Furthermore, in tables 11 and 12, we present numerical resolution checks for linear stability results under constant mass-flux condition for the unstable growth rates marked with stars in figure 16 at $Re = 1190$. The results provide numerical evidence for the accuracy of calculations presented in this paper.

REFERENCES

- BLACKBURN, H. M., HALL, P. & SHERWIN, S. J. 2013 Lower branch equilibria in Couette flow: the emergence of canonical states for arbitrary shear flows. *J. Fluid Mech.* **726**, R2.
- BUDANUR, N. B. & HOF, B. 2018 Complexity of the laminar-turbulent boundary in pipe flow. *Phys. Fluids* **3**, 054401.
- BUDANUR, N. B., SHORT, K. Y., FARAZMAND, M., WILLIS, A. P. & CVITANOVIC, P. 2017 Relative periodic orbits form the backbone of turbulent pipe flow. *J. Fluid Mech.* **833**, 274–301.
- CHANTRY, M., WILLIS, A. P. & KERSWELL, R. R. 2014 Genesis of streamwise-localized solutions from globally periodic traveling waves in pipe flow. *Phys. Rev. Lett.* **112**, 164501.
- DEGUCHI, K. & HALL, P. 2014a The high-Reynolds-number asymptotic development of nonlinear equilibrium states in plane Couette flow. *J. Fluid Mech.* **750**, 99–112.
- DEGUCHI, K. & HALL, P. 2014b Free-stream coherent structures in parallel boundary-layer flows. *J. Fluid Mech.* **752**, 602–625.
- DEGUCHI, K. & HALL, P. 2016 On the instabilities of vortex–wave interaction states. *J. Fluid Mech.* **802**, 634–666.
- DUGUET, Y., PRINGLE, C. C. T. & KERSWELL, R. R. 2008a Relative periodic orbits in transitional pipe flow. *Phys. Fluids* **20**, 1141102.

- DUGUET, Y., WILLIS, A. P. & KERSWELL, R. R. 2008*b* Transition in pipe flow: the saddle structure on the boundary of turbulence. *J. Fluid Mech.* **613**, 255–274.
- FAISST, H. & ECKHARDT, B. 2003 Traveling waves in pipe flow. *Phys. Rev. Lett.* **91**, 224502.
- GIBSON, J. F., HALCROW, J. & CVITANOVIC, P. 2009 Equilibrium and travelling-wave solutions of plane Couette flow. *J. Fluid Mech.* **638**, 243–266.
- HALL, P. & SHERWIN, S. 2010 Streamwise vortices in shear flows: harbingers of transition and the skeleton of coherent structures. *J. Fluid Mech.* **661**, 178–205.
- HALL, P. & SMITH, F. 1991 On strongly nonlinear vortex/wave interactions in boundary layer transition. *J. Fluid Mech.* **227**, 641–666.
- HOF, B., VAN DOORNE, C., WESTERWEEL, J., NIEUWSTADT, F., FAISST, H., ECKHARDT, B., WEDIN, H., KERSWELL, R. & WALEFFE, F. 2004 Experimental observation of nonlinear traveling waves in the turbulent pipe flow. *Science* **305** (5690), 1594–1598.
- KERSWELL, R. & TUTTY, O. 2007 Recurrence of traveling waves in transitional pipe flow. *J. Fluid Mech.* **584**, 69–102.
- NAGATA, M. 1990 Three dimensional finite-amplitude solutions in plane Couette flow: bifurcation from infinity. *J. Fluid Mech.* **217**, 519–527.
- OZCAKIR, O., TANVEER, S., HALL, P. & OVERMAN, E. A. 2016 Travelling waves in pipe flow. *J. Fluid Mech.* **791**, 284–328.
- PRINGLE, C. C. T., DUGUET, Y. & KERSWELL, R. R. 2009 Highly symmetric travelling waves in pipe flow. *Phil. Trans. R. Soc. Lond. A* **367** (1888), 457–472.
- PRINGLE, C. C. T. & KERSWELL, R. R. 2007 Asymmetric, helical and mirror-symmetric travelling waves in pipe flow. *Phys. Rev. Lett.* **99**, 074502.
- SOIBELMAN, I. & MEIRON, D. 1991 Finite-amplitude bifurcations in plane Poiseuille flow: two-dimensional Hopf bifurcation. *J. Fluid Mech.* **229**, 389–416.
- TOH, S. & ITANO, T. 2003 A periodic-like solution in channel flow. *J. Fluid Mech.* **481**, 67–76.
- VISWANATH, D. 2007 Recurrent motions within plane Couette turbulence. *J. Fluid Mech.* **580**, 339–358.
- VISWANATH, D. 2009 Critical layer in pipe flow at high Reynolds number. *Phil. Trans. R. Soc. Lond. A* **580**, 561–576.
- VISWANATH, D. & CVITANOVIC, P. 2009 Stable manifolds and the transition to turbulence in pipe flow. *J. Fluid Mech.* **627**, 215–233.
- WALEFFE, F. 1995 Hydrodynamic stability and turbulence: beyond transients to a self-sustaining process. *Stud. Appl. Maths* **95** (3), 319–343.
- WALEFFE, F. 2001 Exact coherent structures in channel flow. *J. Fluid Mech.* **435**, 93–102.
- WALEFFE, F. 2003 Homotopy of exact coherent structures in plane shear flows. *Phys. Fluids* **15** (6), 1517–1534.
- WANG, J., GIBSON, J. & WALEFFE, F. 2007 Lower branch coherent states in shear flows: transition and control. *Phys. Rev. Lett.* **98** (20), 204501.
- WEDIN, H. & KERSWELL, R. 2004 Exact coherent structures in pipe flow: travelling wave solutions. *J. Fluid Mech.* **508**, 333–371.
- WILLIS, A. P., DUGUET, Y., OMEL'CHENKO, O. & WOLFRUM, M. 2017 Surfing the edge: using feedback control to find nonlinear solutions. *J. Fluid Mech.* **831**, 579–591.