

Transfer functions for flow predictions in wall-bounded turbulence

Kenzo Sasaki^{1,2,3,†}, Ricardo Vinuesa^{2,3}, André V. G. Cavalieri¹, Philipp Schlatter^{2,3} and Dan S. Henningson^{1,2,3}

¹Instituto Tecnológico de Aeronáutica, Aerodynamics Department, São José dos Campos, 12228900, Brazil

²Linné FLOW Centre, KTH Mechanics, SE-100 44 Stockholm, Sweden

³Swedish e-Science Research Centre (SeRC), Stockholm, Sweden

(Received 24 July 2018; revised 1 November 2018; accepted 23 December 2018; first published online 11 February 2019)

Three methods are evaluated to estimate the streamwise velocity fluctuations of a zero-pressure-gradient turbulent boundary layer of momentum-thickness-based Reynolds number up to $Re_\theta \simeq 8200$, using as input velocity fluctuations at different wall-normal positions. A system identification approach is considered where large-eddy simulation data are used to build single and multiple-input linear and nonlinear transfer functions. Such transfer functions are then treated as convolution kernels and may be used as models for the prediction of the fluctuations. Good agreement between predicted and reference data is observed when the streamwise velocity in the near-wall region is estimated from fluctuations in the outer region. Both the unsteady behaviour of the fluctuations and the spectral content of the data are properly predicted. It is shown that approximately 45% of the energy in the near-wall peak is linearly correlated with the outer-layer structures, for the reference case $Re_\theta = 4430$. These identified transfer functions allow insight into the causality between the different wall-normal locations in a turbulent boundary layer along with an estimation of the tilting angle of the large-scale structures. Differences in accuracy of the methods (single- and multiple-input linear and nonlinear) are assessed by evaluating the coherence of the structures between wall-normally separated positions. It is shown that the large-scale fluctuations are coherent between the outer and inner layers, by means of an interactions which strengthens with increasing Reynolds number, whereas the finer-scale fluctuations are only coherent within the near-wall region. This enables the possibility of considering the wall-shear stress as an input measurement, which would more easily allow the implementation of these methods in experimental applications. A parametric study was also performed by evaluating the effect of the Reynolds number, wall-normal positions and input quantities considered in the model. Since the methods vary in terms of their complexity for implementation, computational expense and accuracy, the technique of choice will depend on the application under consideration. We also assessed the possibility of designing and testing the models at different Reynolds numbers, where it is shown that the prediction of the near-wall peak from wall-shear-stress measurements is practically unaffected even for a one order of magnitude change in the corresponding Reynolds number of the design and test, indicating that the interaction between the near-wall peak fluctuations and the wall is approximately Reynolds-number independent. Furthermore, given the

† Email address for correspondence: kenzo@ita.br

performance of such methods in the prediction of flow features in turbulent boundary layers, they have a good potential for implementation in experiments and realistic flow control applications, where the prediction of the near-wall peak led to correlations above 0.80 when wall-shear stress was used in a multiple-input or nonlinear scheme. Errors of the order of 20 % were also observed in the determination of the near-wall spectral peak, depending on the employed method.

Key words: turbulence modelling, turbulent boundary layers

1. Introduction

Turbulent boundary layers account for up to 50 % of the drag and therefore fuel consumption of modern aircraft (Schrauf 2005). Turbulent flows also lead to enhanced exchange processes, such as mixing and heat transfer, present, for instance, in large-scale meteorological phenomena. Such characteristics have consequently led to efforts concerning a better understanding of the dynamics of these flows, both in terms of predictive models and their use to control the fluctuations on the near-wall region, where the highest values of shear and turbulent production are present (Marusic, Mathis & Hutchins 2010).

Given their practical importance, several studies have aimed over the last decades at achieving a deeper understanding of the turbulent structures in the near-wall region of wall-bounded flows. The current understanding of the flow behaviour in this region involves a self-sustained turbulence regeneration cycle (Waleffe 1995, 1997; Jiménez & Pinelli 1999; Panton 2001; Moehlis, Faisst & Eckhardt 2004). Coherent turbulent structures comprise a hierarchical ordering along the wall-normal direction (Flores & Jiménez 2010; Jiménez 2012, 2013) and exhibit long life times in the streamwise direction, as can be recognized from works of Favre, Gaviglio & Dumas (1967), Blackwelder & Kovasznay (1972), Wark & Nagib (1991), Kim & Adrian (1999) and Hutchins & Marusic (2007). Moreover, large-scale turbulent structures make a significant contribution to the turbulent kinetic energy and Reynolds stresses in the outer layer (Komminaho, Lundbladh & Johansson 1996; Guala, Hommema & Adrian 2006) and have been recently shown to imprint their presence in the near-wall region (Hoyas & Jiménez 2006), an effect modelled via an amplitude modulation (Mathis, Hutchins & Marusic 2009; Marusic *et al.* 2010; Schlatter & Örlü 2010; Bernardini & Pirozzoli 2011; Vinuesa *et al.* 2015; Dogan *et al.* 2019).

Approaches based on the linearization of the Navier–Stokes operator have been recently shown to enable the extraction of some of the features of these large-scale structures; in such methods the equations are linearized around the mean turbulent flow (as opposed to the linearization around a laminar base flow, as is the case for transitional problems). This could be considered analogously to the laminar base flow (Crighton & Gaster 1976) of the most usual analysis of transitional cases. Of particular importance was the realization that even stable flows may exhibit strong transient amplifications, which are related to the non-normality of the Navier–Stokes operator (Trefethen *et al.* 1993; Farrell & Ioannou 1996; Schmid & Henningson 2012). The transient growth analysis has also been used to study turbulent flows, as in the studies by del Álamo & Jiménez (2006) and Pujals *et al.* (2009) in channels and Cossu, Pujals & Depardon (2009) in boundary layers, all of which deal with a linearization of the equations around the turbulent mean profile.

In related works, but using a different strategy, these large-scale structures have been studied by means of a resolvent-based method (Hwang & Cossu 2010; McKeon & Sharma 2010; McKeon, Sharma & Jacobi 2013; McKeon 2017). The basic idea consists of performing a Reynolds decomposition and replacing the nonlinear fluctuation terms of the Navier–Stokes equations by a forcing to the linear operator. Doing so, the nonlinear forcing is treated as an unknown, and the most amplified flow responses to such forcings are sought. The resolvent operator is then used to extract information from the flow and it has been shown to reproduce relevant features, such as the spectral proper orthogonal decomposition (SPOD) modes of the streamwise velocity fluctuations of a turbulent boundary layer over an airfoil and in a turbulent free jet (Abreu, Cavalieri & Wolf 2017; Towne, Schmidt & Colonius 2018).

The works cited above focus on frequency-domain characteristics of the flow. The model of Marusic and co-workers (Mathis *et al.* 2009; Marusic *et al.* 2010; Mathis, Hutchins & Marusic 2011) treats the interaction among different wall-normal directions as an amplitude modulation, and is able to predict with compelling agreement the statistics of the fluctuations in the near-wall region from measurements in the logarithmic layer. This model is, however, unable to accurately capture the instantaneous flow field, one of the necessary ingredients for flow control applications. Examples of models which are able to do such time-domain, instantaneous predictions may be found in the recent works of Baars, Hutchins & Marusic (2016) and Illingworth, Monty & Marusic (2018), which make use of spectral stochastic estimation and linear estimation via a state observer, respectively, to obtain time-domain predictions for the large-scale structures in a turbulent boundary layer.

In this work we propose to follow a somewhat different strategy based on the cross-spectral density between various wall-normal separated positions. This approach allows us to obtain linear and nonlinear frequency response functions which, when inversed Fourier transformed, lead to convolution kernels; these will be referred to as transfer functions (TFs). This method has already been used by this group to model a high-Reynolds-number turbulent jet (Sasaki *et al.* 2017) and as a reduced-order model for control of transitional flows (Sasaki *et al.* 2018*a,b*). All these previous studies have led to very satisfactory results.

Since the proposed method, when applied to a boundary layer, relies uniquely on the temporal and spanwise data, it allows a considerable flexibility in terms of the definition of inputs and outputs, both in terms of wall-normal positions and quantities to be estimated. One interesting feature that is explored in this study is the use of measurements at the wall to estimate the flow at higher wall-normal locations, which could have direct implications for experimental implementations. ‘Off-design’ evaluations, where the models are constructed and tested at different Reynolds numbers, are also explored here. The possibility to apply the methods at much higher Reynolds numbers, reducing the number of measurements to be conducted, is expected to have a significant appeal to experimental and practical applications.

Finally, the formalism written in terms of transfer functions permits to extract the relationship that exists between the different inputs and outputs of the system, allowing to obtain the footprint of the large-scale outer-region wall structures on the near-wall region.

The database under study is the highly resolved and well-validated large-eddy simulation (LES) zero-pressure-gradient (ZPG) turbulent boundary layer (TBL) by Eitel-Amor, Örlü & Schlatter (2014), where the highest available Reynolds number based on the momentum thickness is $Re_\theta = 8300$.

This article is organized as follows: in § 2 the LES database is presented, § 3 presents the calculated coherence for different wall-normal positions and justifies the application of linear methods to this flow. The linear and nonlinear transfer functions under consideration are presented in § 4 whereas the results and analysis are discussed in § 5, along with a comparison of the different methods. The off-design and robustness analysis are performed in § 6 and a comparison with other methods available in the literature is given in § 7. The conclusions of the study are presented in § 8. Finally, the appendices present details of the frequency-domain tools used in this work, namely the ensemble averaging for the calculation of spectra and the spectral conditioning for the decorrelation of inputs.

2. Description of the numerical database

The set-up of the LES by Eitel-Amor *et al.* (2014) corresponds to a flat plate where a statistically two-dimensional ZPG TBL, with density ρ , viscosity ν and free-stream velocity U_∞ develops along the streamwise direction. The dimensions of the domain are $L_x \times L_y \times L_z = 13\,500 \times 400 \times 540$ in the streamwise, wall-normal and spanwise directions, where non-dimensionalization with the displacement thickness at the inlet δ_0^* is considered. Here we employ a Reynolds decomposition of the velocity field into a spanwise and temporal average $U(x, y)$ and the fluctuating velocities, $u(x, y, z, t)$, $v(x, y, z, t)$ and $w(x, y, z, t)$. The outer velocity scale is U_∞ , and the outer length scale is δ_{99} , i.e. the wall-normal position where the mean velocity $U(x, y)$ reaches 99% of U_∞ . The Reynolds number $Re_\theta = U_\infty \theta / \nu$ is then defined in terms of the momentum thickness θ . The inner scale is based on the friction velocity $u_\tau = \sqrt{\tau_w / \rho}$, which is obtained from the mean shear stress at the wall, τ_w and the inner length scale is the viscous length $l_* = \nu / u_\tau$. Inner scaling will be denoted by the superscript $+$. The friction Reynolds number is then defined as $Re_\tau = u_\tau \delta_{99} / \nu$.

Following a procedure similar to that in wind-tunnel experiments, laminar flow enters the domain and is then forced to transition via a tripping (see Schlatter & Örlü (2012) for a complete description of the method). The flow is tripped close to the leading edge, at $Re_\theta = 180$, where the tripping is more efficient. Transition to turbulence is considered to be complete at $Re_\theta \geq 600$, as shown in Eitel-Amor *et al.* (2014). The highest Reynolds number to be considered in this work is of $Re_\theta = 8200$, located upstream of the end of the computational domain in order to avoid outflow effects.

This study will focus on the data at the streamwise position corresponding to $Re_\theta = 4430$ ($Re_\tau = 1324$), however the cases corresponding to $Re_\theta = 2240$ ($Re_\tau = 704$) and $Re_\theta = 8200$ ($Re_\tau = 2370$) will also be analysed in order to evaluate the effect of the Reynolds number in the predictions and to perform the off-design studies.

The number of collocation points employed in the simulation is $13\,824 \times 513 \times 1152$ in the streamwise, wall-normal and spanwise directions. As shown by Eitel-Amor *et al.* (2014) this resolution corresponds to a very finely resolved LES, in excellent agreement with DNS studies of ZPG TBL. The spectral code SIMSON (Chevalier, Lundbladh & Henningson 2007) is used for this simulation, and was previously used in a number of studies by this group (Schlatter *et al.* 2009; Schlatter & Örlü 2010).

A total of 19410 time samples is used, with a spacing in time of $\Delta t^+ = 0.5$, which is appropriate for convergence of the statistics and is also adequate to correctly resolve the near-wall (which corresponds to a period $\lambda_t^+ \approx 100$) and outer ($\lambda_t \approx 10\delta_{99}$) peaks. The first half of the time samples is used to design the modelling methodologies and the other half to test them. The spanwise discretization results in 768 points separated

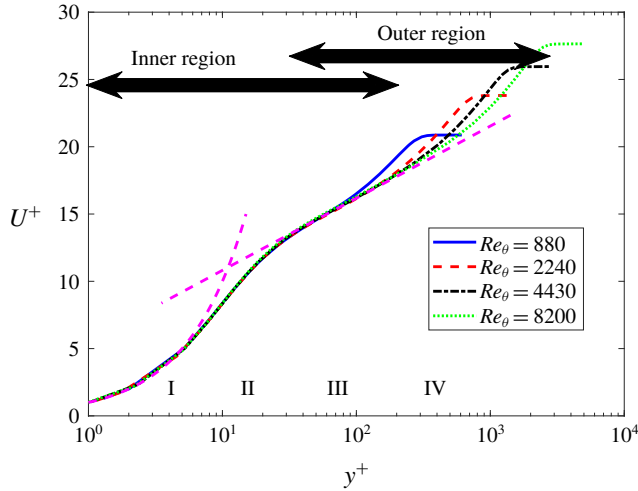


FIGURE 1. (Colour online) Inner-scaled mean velocity profiles for the four Reynolds numbers evaluated in the current study. Regions I, II, III and IV represent the viscous sublayer ($y^+ \lesssim 7$), buffer ($7 \lesssim y^+ \lesssim 30$) and logarithmic layers ($30 \lesssim y^+ \lesssim 220$) and wake region ($220 \lesssim y^+$ until the edge of the boundary layer, for the $Re_\theta = 8200$ case). The pink dashed lines correspond to the linear ($U^+ = y^+$) and logarithmic profiles ($U^+ = 1/\kappa \log(y^+) + B$, where $\kappa = 0.41$ and $B = 5.1$), respectively. The upper limit of the outer region corresponds to the $Re_\theta = 8200$ case, extending until the edge of the boundary layer.

by $\Delta z^+ = 12$, which is appropriate to capture the near-wall (with a typical spanwise wavelength $\lambda_z^+ \approx 100$) and outer ($\lambda_z \approx \delta_{99}$) structures. For further details concerning the convergence of statistics of this simulation the reader is referred to Eitel-Amor *et al.* (2014).

3. Pre-multiplied spectra and coherence

Prior to the construction of the transfer functions, an evaluation of the behaviour of the two-dimensional spectra and the coherence between sets of positions separated along the wall-normal direction was performed. The objective of such assessment was to evaluate the suitability of models for the prediction.

The focus will be on three locations in the wall-normal direction, $y^+ = 15, 50$ and 100 , which correspond to positions in the near-wall region, just outside the buffer layer and representative of the logarithmic region, respectively. This may be observed in figure 1 which shows the mean flow profile, scaled in inner units, for the four Reynolds numbers evaluated in this work. Regions I, II, III and IV represent the viscous sublayer, buffer and logarithmic layers and wake region, respectively. The linear and logarithmic profiles are shown to better highlight the behaviour of the mean velocity profile; the inner and outer regions (the latter shown for $Re_\theta = 8200$) are also shown on the upper part of the plot.

The inner-scaled, two-dimensional, pre-multiplied spectrum is defined as $E_{uu}\omega\beta/u_\tau^2$, where $E_{uu} = \langle \hat{u}\hat{u}^* \rangle$, with the double hat representing the double Fourier transform of u in the spanwise direction and time, the brackets $\langle \rangle$ denote an ensemble averaging as defined in appendix A, $*$ defines the complex conjugate, ω and β are the frequency and spanwise wavenumbers and $\lambda_t^+ = 2\pi/\omega$, $\lambda_z^+ = 2\pi/\beta$, given in inner units. Spectra

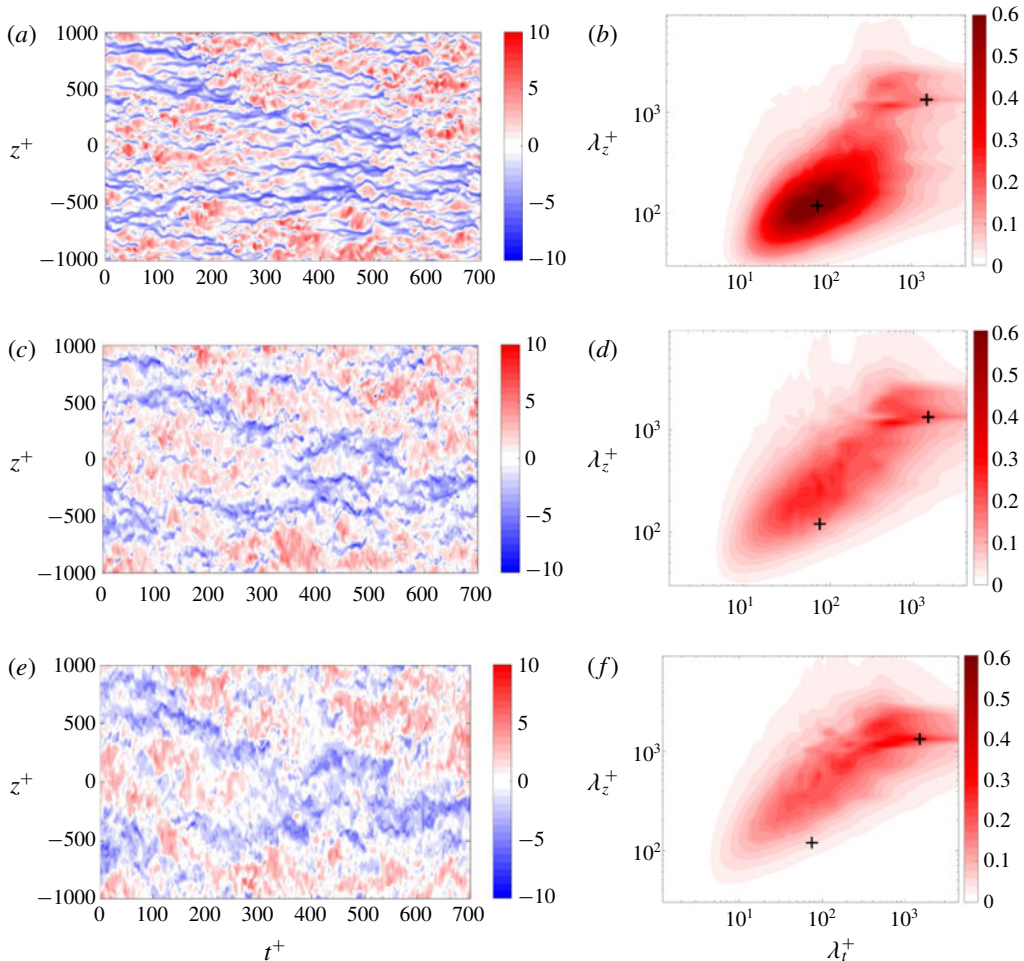


FIGURE 2. (Colour online) Behaviour of the streamwise velocity fluctuations and inner-scaled pre-multiplied two-dimensional spectra at different wall-normal positions for $Re_\theta = 4430$. The crosses correspond to reference locations for the inner $((\lambda_t^+, \lambda_z^+) \approx (100, 100))$ and outer peaks $((\lambda_t^+, \lambda_z^+) \approx (10\delta_{99}/U_\infty, \delta_{99}), (\lambda_t^+, \lambda_z^+) \approx (1400, 1300))$. The limits of the colour bar were kept the same between different plots to highlight the different amplitudes. (a) Instantaneous streamwise velocity fluctuations at $y^+ = 15$. (b) Pre-multiplied spectra at $y^+ = 15$. (c) Instantaneous streamwise velocity fluctuations at $y^+ = 50$. (d) Pre-multiplied spectra at $y^+ = 50$. (e) Instantaneous streamwise velocity fluctuations at $y^+ = 100$. (f) Pre-multiplied spectra at $y^+ = 100$.

have been averaged using Welch's method, using segments with 256 snapshots with 75% overlap and a time discretization $\Delta t^+ = 0.5$, with a triangular window leading to a frequency resolution of $\Delta\omega^+ = 0.05$. Figure 2 shows the time signal and pre-multiplied spectra of the streamwise velocity fluctuation at three locations, $y^+ = 15, 50$ and 100, for the $Re_\theta = 4430$, (corresponding to $Re_\tau = 1324$), case. Velocity fluctuations are normalized using the friction velocity u_τ in the results shown in figure 2, and also for all time series presented later in the paper.

The dominant spanwise wavelength of the fluctuations grows as one moves further in the wall-normal direction, which leads to a wider scale separation between the large-scale streaks in the outer region and the near-wall streaks, a feature which may be observed in the two-dimensional pre-multiplied spectra of figure 2. For $y^+ = 15$, representative of the buffer layer, the classical near-wall streaks with spacing $\lambda_z^+ \approx 100$ are observed, with a characteristic time $\lambda_t^+ \approx 100$. In the outer region a secondary peak, which scales in outer units, is observed with characteristic scales $\lambda_z \approx \delta_{99}$ and $\lambda_t \approx 10\delta_{99}/U_\infty$. In order to extract the relation between near-wall and outer positions, the coherence between two sets of synchronized signals separated in the wall-normal direction was computed using time series of the streamwise velocity fluctuation. For a statistically two-dimensional ZPG TBL, which is homogeneous in the spanwise direction, we define:

$$\gamma^2(y_{in}, y_{out}, \omega, \beta) = \frac{|\langle \hat{u}(y_{in}, \omega, \beta) \hat{u}^*(y_{out}, \omega, \beta) \rangle|^2}{\langle |\hat{u}(y_{in}, \omega, \beta)|^2 \rangle \langle |\hat{u}(y_{out}, \omega, \beta)|^2 \rangle}, \quad (3.1)$$

where y_{in}, y_{out} correspond to the wall-normal position of the input/output pair of time series that are considered in the analysis. The coherence is a quantity analogous to the correlation, but defined in the frequency domain (here also in wavenumber domain, taking advantage of homogeneity of the spanwise coordinate) and it represents a metric of linear behaviour between two signals, for a given (ω, β) pair. Its value is normalized between zero and one, where the former indicates a complete random behaviour and the latter an exact linear relation between the two signals (namely an amplitude change and a phase shift). Linear, time and span invariant models with single input and single output lead to unit coherence between input and output (Bendat & Piersol 2011); hence, coherence values close to one suggest the possibility of modelling the observed behaviour with a linear model.

The objective that we will pursue in the next sections is to predict the behaviour of the fluctuations in the near-wall region, such that the output position will be fixed at $y^+ = 15$, where a clear peak is observed in the pre-multiplied spectra shown in figure 2. Sample inputs are chosen at $y^+ = 50$ and 100, where the larger scales become prominent and in a third case using the wall-shear stress. The objective of the latter is to capture the finer-scale motions close to the wall.

Figure 3 shows the coherence for three pairs of positions. It is noticeable that for the higher wall-normal positions the long spanwise wavelengths are coherent. This observation is related to the wall footprint of large-scale motions that are attached to the wall, as in Marusic *et al.* (2010). We see that larger-scale structures at outer positions have a significant footprint at the buffer layer, with coherences ranging between 0.6 and 0.9 for wavelengths and periods corresponding to the outer peak. However, the near-wall peak ($\lambda_z^+, \lambda_t^+ = 100, 100$) is seen to be incoherent with outer-layer disturbances. In turn, the coherence using wall-shear stress as input exhibits values close to unity at the lower wavelengths, corresponding to the near-wall peak in the pre-multiplied spectra of figure 2; this indicates that shear-stress fluctuations may be an interesting candidate as an input measurement for prediction of buffer-layer fluctuations (Örlü & Schlatter 2011).

In order to better assess the behaviour of the coherence and its relation to the peak in the two-dimensional spectra, figure 4 presents the one-dimensional pre-multiplied temporal spectrum ($E_{uu}^{1D} \omega / u_\tau^2$) as a function of the wall-normal direction overlaid on the one-dimensional coherence,

$$\gamma_{1D}^2(y_{in}, y_{out}, \omega) = \frac{|\langle \hat{u}(y_{in}, \omega) \hat{u}^*(y_{out}, \omega) \rangle|^2}{\langle |\hat{u}(y_{in}, \omega)|^2 \rangle \langle |\hat{u}(y_{out}, \omega)|^2 \rangle}, \quad (3.2)$$

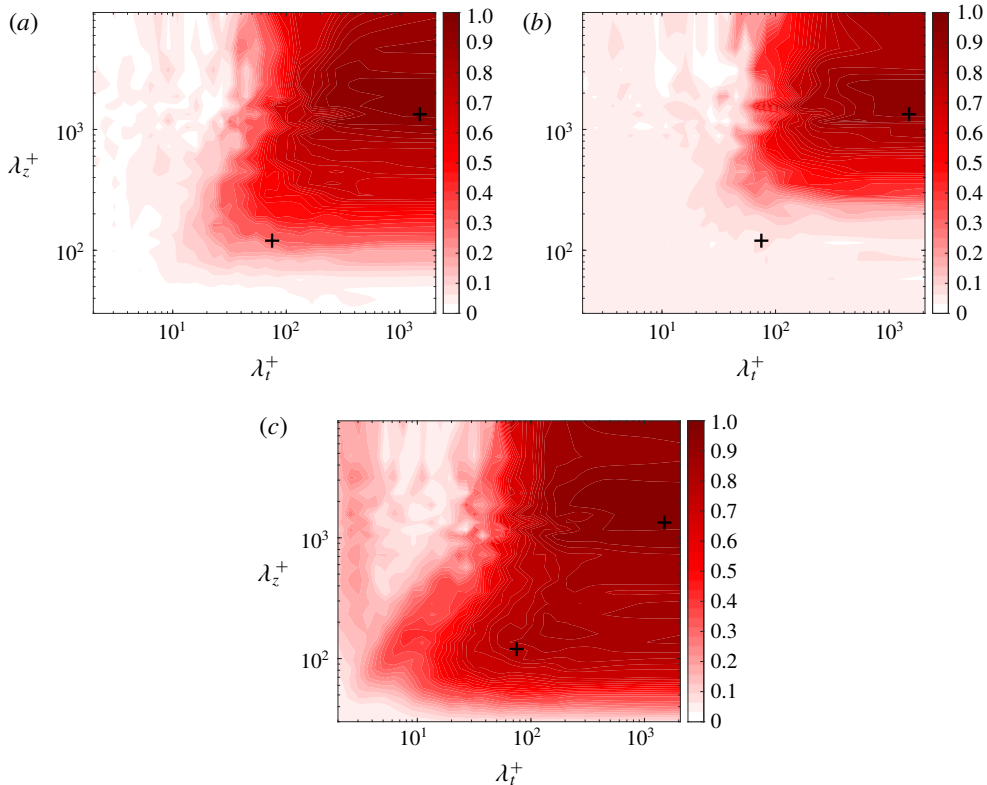


FIGURE 3. (Colour online) Coherence between two positions separated in the wall-normal direction, for $Re_\theta = 4430$. The crosses correspond to reference locations for the inner $((\lambda_t^+, \lambda_z^+) \approx (100, 100))$ and outer peaks $((\lambda_t, \lambda_z) \approx (10\delta_{99}/U_\infty, \delta_{99}), (\lambda_t^+, \lambda_z^+) \approx (1400, 1300))$. (a) Coherence between streamwise velocity fluctuations at $y^+ = 50$ and 15. (b) Coherence between streamwise velocity fluctuations $y^+ = 100$ and 15. (c) Coherence between wall-shear stress and streamwise velocity fluctuations at $y^+ = 15$.

which is calculated between a pair of positions at the same spanwise location, one is at a fixed wall-normal location and the other one is moved. The reference input position is in the viscous sublayer, i.e. at $y_{in}^+ = 5$ and two Reynolds numbers were considered, $Re_\theta = 4430$ ($Re_\tau = 1324$) and 8200 ($Re_\tau = 2370$). For both cases it is noticeable that the larger scales exhibit significant values of coherence with the near-wall region, illustrating the fact that the largest scales are indeed attached to the wall. It is also observed that, as the Reynolds number is increased, the larger scales remain coherent with the near-wall region until much further away from the wall, including the outer spectral peak. Wall-normal positions up to $y^+ \approx 500$ continue to exhibit coherence levels of 0.4 with the near wall indicating that the footprint of such large scales is clearly present for the $Re_\theta = 8200$ case. This enables the use of data in the outer region to predict specific flow features close to the wall. As for the finer scales, they present high levels of coherence only close to the wall, which permits the use of local wall-shear-stress data or even universal signals to predict the inner-peak behaviour.

The high values of coherence reported here motivate the use of predictive techniques for application in the turbulent zero-pressure-gradient boundary layer. Through this

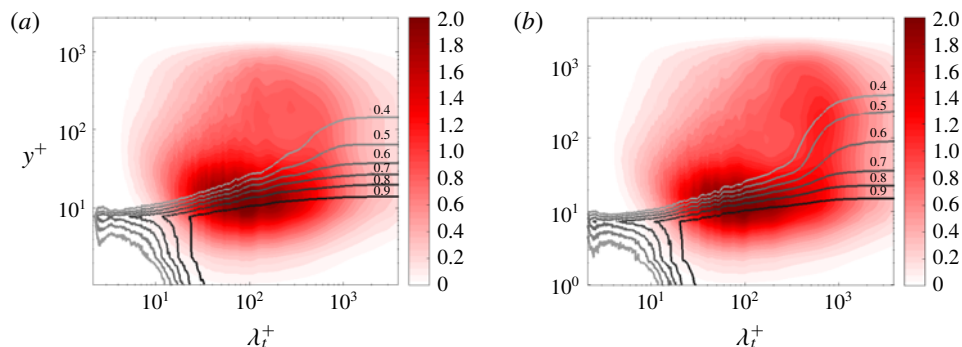


FIGURE 4. (Colour online) One-dimensional coherence (solid lines) overlaid on the one-dimensional pre-multiplied temporal spectrum as a function of the wall-normal direction for $Re_\theta = 4430$ and 8200 . Coherence levels of 0.4 until 0.9 are highlighted.

paper, such models will be defined in terms of linear and nonlinear transfer functions, which will be introduced in the next section.

Other studies using different methodologies, and based on experimental databases (Marusic *et al.* 2010; Baars *et al.* 2016), perform analyses up to $Re_\tau = 19\,000$ and observe that the footprint of the larger scales on the near-wall region remains. Particularly, the work of Baars *et al.* (2016) shows quantitative results based on the one-dimensional (1-D) coherence spectrum up to $Re_\tau = 13\,300$. These works supply evidence that the assumptions of the modelling methodologies herein developed would remain valid at considerably higher Reynolds numbers. Nevertheless, the footprint of the larger scales over the near-wall region and corresponding high coherence values are necessary ingredients for the use of the methods outlined in this work.

4. Linear and nonlinear transfer functions

4.1. Single-input linear transfer function

The basic supporting idea for the different identification methods that will be considered in this study is that there exists a function $f(I(z, t))$ that maps the relation between two wall-normal positions in the TBL, namely $I(z, t)$ and $O(z, t)$. These two sets of data, taken along the spanwise direction and time, will be referred to as the input and output of the system, respectively. A qualitative description of such relation may be given in terms of the block diagram shown in figure 5. It should be noted that no restrictions have been made regarding the variables to be considered or the linearity of the function f .

The methods therefore assume the availability of spanwise- and time-resolved measurements, which are available in a numerical simulation but could be more difficult to obtain experimentally. Such turbulence measurements above the wall would require the use of time-resolved particle image velocimetry (PIV), such as in the work of Cuvier *et al.* (2017). An alternative is to employ wall-shear stress sensors, as performed in closed-loop applications, such as in Lundell (2007).

The assumption in this section will be that there is a linear relationship between the two sets of measurements, such that the output may be predicted by means of a double convolution (along time and the spanwise direction) between the input measurement

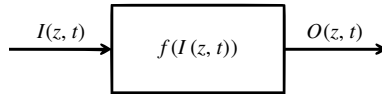


FIGURE 5. Block diagram describing the relationship between the input and output signals within the TBL.

and a kernel,

$$O(z, t) = \int_{-z_{max}/2}^{z_{max}/2} \int_{-\infty}^{\infty} g_{IO}(\zeta, \tau) I(z - \zeta, t - \tau) d\zeta d\tau, \tag{4.1}$$

which can be written in short as,

$$O(z, t) = g_{IO}(z, t) \otimes I(z, t), \tag{4.2}$$

where \otimes represents the double convolution. Here $g_{IO}(\zeta, \tau)$ is the convolution kernel, which, in accordance with the common nomenclature of the field, will be referred to as a transfer function in the remaining of this paper. The dummy variables (ζ, τ) , which are analogous to (z, t) , were introduced for the calculation of the convolution. In order to obtain $g_{IO}(\zeta, \tau)$, the problem is formulated in the frequency/spanwise wavenumber space, where the optimal frequency response, in the least squares sense, may be defined from the auto- and cross-spectra of the input and output signals (Bendat & Piersol 2011; Sasaki *et al.* 2017, 2018a):

$$G_{IO}(\beta, \omega) = \frac{S_{IO}(\beta, \omega)}{S_{II}(\beta, \omega)}, \tag{4.3}$$

where S_{II} and S_{IO} are the auto- and cross-spectra of the input and output, respectively. These quantities are calculated from the expected values of $\hat{I}(\beta, \omega)\hat{I}^*(\beta, \omega)$ and $\hat{I}(\beta, \omega)\hat{O}^*(\beta, \omega)$, obtained from an ensemble averaging. The details of the method are outlined in appendix A. It should be noted that since the spanwise direction is periodic in the simulation, the ensemble averaging along this direction corresponds directly to the Fourier transform from z to β . Equation (4.3) is referred to as the H_1 estimator of the system (Rocklin, Crowley & Vold 1985), and it minimizes the error due to measurement noise in the output. Other formulations, such as the H_2 or H_v estimators, exhibit different performances in terms of sensor noise minimization. They are, however, expected to perform equally well for this type of estimation, which does not consider the presence of measurement uncertainties.

One interesting property of the H_1 estimator is that it leads to a prediction error which is linearly uncorrelated with the input (Rocklin *et al.* 1985; Bendat 1993). Consider the input/output relationship, given in the frequency–wavenumber domain,

$$\hat{O}(\beta, \omega) = \hat{G}(\beta, \omega)\hat{I}(\beta, \omega) + \epsilon(\beta, \omega), \tag{4.4}$$

where ϵ will represent the error of the linear estimator. Post-multiplying equation (4.4) by the conjugate transpose of the input and taking the expected values, we obtain

$$S_{IO}(\beta, \omega) = \hat{G}(\beta, \omega)S_{II}(\beta, \omega) + S_{\epsilon,I}(\beta, \omega). \tag{4.5}$$

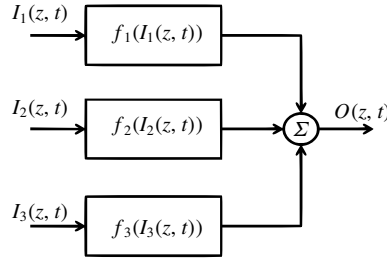


FIGURE 6. Block diagram describing the relationship between the multiple inputs and output of a system.

Inserting the H_1 estimator of (4.3) leads to $S_{\epsilon,x}(\beta, \omega) = 0$. Remaining correlated errors are related either to measurement noise, which can be considered negligible in this problem, or spectral leakage (Schoukens, Rolain & Pintelon 1997), which is minimized by use of long time series.

Performing an inverse Fourier transform of G_{IO} as:

$$g_{IO}(z, t) = \int_{-\infty}^{+\infty} \int_{-\beta_n}^{+\beta_n} G_{IO}(\beta, \omega) e^{i\beta\zeta} e^{-i\omega\tau} d\beta d\omega, \tag{4.6}$$

where $\pm\beta_n$ represent the spanwise Nyquist wavenumbers calculated based on the spanwise discretization, leads to $g_{IO}(\zeta, \tau)$. This represents an empirical, linear, time-domain transfer function, which can be used to estimate the evolution of the fields separated in the wall-normal direction.

4.2. Multiple-input linear transfer function

In this section the construction of a model with multiple inputs will be introduced. This method applies to problems where multiple time series are available to be used in the estimation of the output. A qualitative schematic of the estimation is shown in the block diagram of figure 6. In this figure a system with three inputs is considered, however the extension to any number of inputs is straightforward. The linear hypothesis is inherent to this formulation since the output is assumed to be determined directly from the superposition of the contributions from each input, $I_i(z, t)$, via the function $f_i(I_i(z, t))$, which is estimated in terms of a convolution kernel, g_{IO} . The estimation of the output is therefore obtained as follows:

$$O(z, t) = \sum_{i=1}^n I_i(z, t) \otimes g_{IO}(z, t). \tag{4.7}$$

As in the case of the single-input problem, the transfer functions are found in the (ω, β) space and then inverse Fourier transformed to the physical domain. For a system with n inputs, the transfer functions are obtained from the solution of the linear system:

$$\begin{pmatrix} S_{I_1O}(\beta, \omega) \\ S_{I_2O}(\beta, \omega) \\ \vdots \\ S_{I_nO}(\beta, \omega) \end{pmatrix} = \begin{pmatrix} S_{I_1I_1}(\beta, \omega) & S_{I_1I_2}(\beta, \omega) & \dots & S_{I_1I_n}(\beta, \omega) \\ S_{I_2I_1}(\beta, \omega) & S_{I_2I_2}(\beta, \omega) & \dots & S_{I_2I_n}(\beta, \omega) \\ \vdots & \vdots & \ddots & \vdots \\ S_{I_nI_1}(\beta, \omega) & S_{I_nI_2}(\beta, \omega) & \dots & S_{I_nI_n}(\beta, \omega) \end{pmatrix} \begin{pmatrix} G_{I_1O}(\beta, \omega) \\ G_{I_2O}(\beta, \omega) \\ \vdots \\ G_{I_nO}(\beta, \omega) \end{pmatrix}, \tag{4.8}$$

which is performed for each frequency independently. The terms $S_{i,O}$ define the cross-spectral density between the inputs and output, whereas $S_{i,i}$ are the auto-spectral density of the inputs, when $i=j$, or the cross-spectral density between different inputs, when $i \neq j$. The auto- and cross-spectral densities are also calculated by means of ensemble averaging, as described in appendix A.

Note that it could also be advantageous to deal with incoherent inputs, such that $S_{i,i} = 0$ if $i \neq j$, and only the diagonal terms in (4.8) remain. If the inputs are not correlated, each multiple-input–single-output (MISO) transfer function can be calculated directly as per (4.3), $G_{i,O} = S_{i,O}/S_{i,i}$, without the need to solve the matrix problem in (4.8). In order to decorrelate the inputs, the spectral conditioning technique (Bendat & Piersol 2011) may be used. This has been used in all multiple-input cases studied here and the technique to do so is outlined in appendix B.

Each transfer function has to be inverse Fourier transformed, as per (4.6), in order to perform the estimation via the sum of convolutions in (4.7).

4.3. Nonlinear transfer function

Now we introduce the concept of nonlinear spectral analysis, via the use of nonlinear transfer functions, to the prediction of the interaction between the input and output signals. One of the few uses of nonlinear estimation methods in fluid mechanics may be found in the work of Naguib, Wark & Juckenhöfel (2001), who made use of stochastic estimation using nonlinear flow sources. On the other hand, nonlinear spectral analysis has been applied in previous works by means of a Volterra series to the derivation of nonlinear output frequency responses, with the objective of deriving properties of nonlinear systems (Guillaume, Pintelon & Schoukens 1992; Lang & Billings 1996; Peng, Lang & Billings 2007). This technique assumes the expansion of the data into a power series, using an N -integral kernel, leading to higher-order spectra. It presents drawbacks related both to the prohibiting computational demand, which limits the size of the system, and the large errors which can only be avoided with very large amounts of data.

Here we will follow a somewhat different strategy, by considering *a priori* the existence of a nonlinear model between the defined input and output of the system and working on determining such model. The exact shape of the nonlinearities is unknown and they are sought with the objective of reproducing the observed behaviour. This method is thoroughly discussed in the works of Rice & Fitzpatrick (1988, 1991) and Bendat (1993), for time-dependent problems; in this study we extend the same ideas to application in a boundary layer with periodicity in the spanwise direction.

The nonlinear transfer-function technique considered here consists in replacing a single-input nonlinear system by a multiple-input linear system, where the linear identification techniques of the previous section may be applied. Consider a finite-memory, single-input–single-output nonlinear function $g(\cdot)$, which acts on $I(z, t)$, mapping it into the output $O(z, t)$:

$$O(z, t) = g(I(z, t)), \quad (4.9)$$

as shown schematically in the block diagram of figure 7. Such nonlinear system may be represented by a superposition of linear and nonlinear operations performed on the input, as given by:

$$O(z, t) = \sum_{i=1}^{N_{TF}} g_i(I(z, t)), \quad (4.10)$$

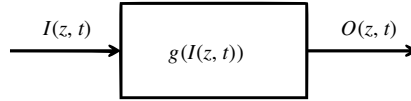


FIGURE 7. Block diagram corresponding to the nonlinear operation applied to $I(z, t)$.

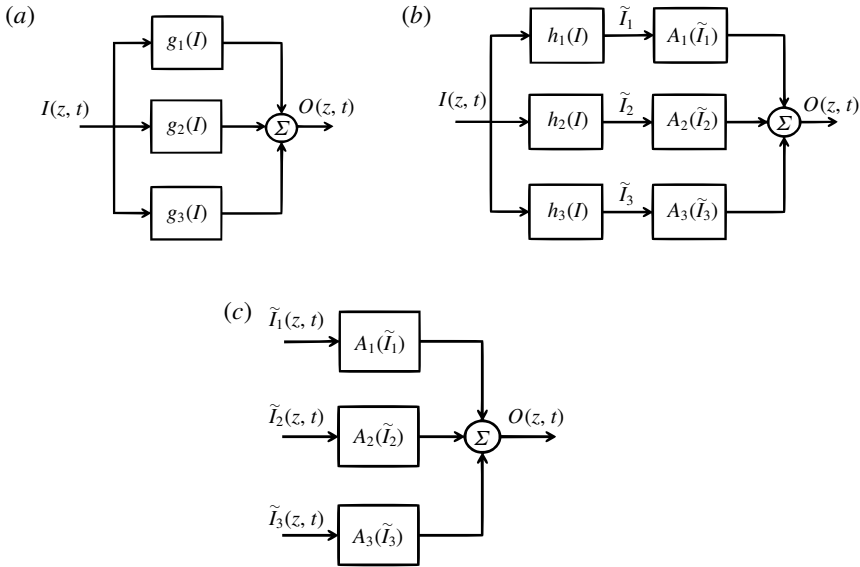


FIGURE 8. Summary of steps to transform a single-input nonlinear problem into a multiple-input linear system. (a) Nonlinear function is replaced by a sum of operations. (b) Each nonlinear function is broken into linear and nonlinear contributions. (c) The nonlinear operations are treated as known and the system is converted into the more usual MISO problem.

where g_i represents an operation performed on the signal $I(z, t)$ and N_{TF} is the number of transfer functions considered. Each one of the nonlinear operators g_i is now replaced by a nonlinear operator h_i and a linear operator A_i , such that:

$$g_i(I(z, t)) = A_i(h_i(I(z, t))). \tag{4.11}$$

We now define the signal $\tilde{I}_i(O, t)$, as the result of the nonlinear operation h_i applied to $I(O, t)$. If all nonlinearities are encapsulated in the operators h_i , the resulting system may now be treated as having inputs $\tilde{I}_i(O, t)$ and output $O(z, t)$, such that linear frequency-domain techniques are applied to the determination of the convolution kernels A_i , in the same manner as derived in the previous section, for a multiple-input–single-output (MISO) problem. Figure 8 illustrates the steps in obtaining this new system on a sample case where three functions were considered in the determination of the output. Note that the extension to any number of input functions is straightforward.

It should also be noted that when the exact shape of the nonlinearities is unknown, as will be the case here, the usual strategy is to deal with polynomial-like functions, with a degree sought such that a given performance is achieved; hence the method becomes more data driven than the previous approaches.

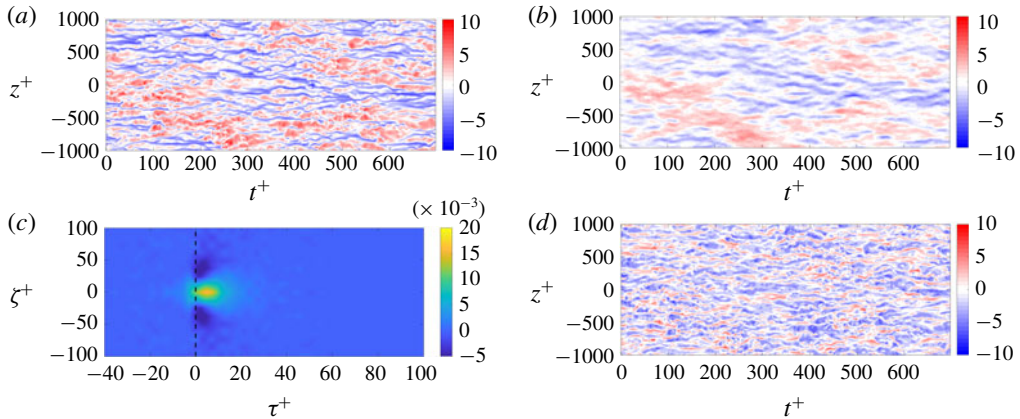


FIGURE 9. (Colour online) Comparison between LES data and estimated field, (a,b), for $Re_\theta = 4430$. The transfer function is shown in (c) and the error in the prediction ($O_{LES} - O_{est}$) in (d). In this case, $y_{in}^+ = 50$ and $y_{out}^+ = 15$ and the prediction was made with the linear transfer function. (a) Instantaneous streamwise velocity fluctuation u^+ from the original LES dataset. (b) Instantaneous streamwise velocity fluctuation estimated via the linear transfer function approach from a measurement at $y_{in}^+ = 50$. (c) Transfer function between input and output positions written in the spanwise and time coordinates. Dashed line highlights the zero time delay. (d) Residual from the single-input–single-output (SISO) linear prediction, $u_{LES}^+ - u_{est}^+$.

5. Results

Through this section the three methods for identification of the time/spanwise behaviour of the velocity fluctuations will be applied to the turbulent boundary layer of § 2. We will start with the simpler linear TF, then consider the case where more inputs are used to perform the estimation and, finally, consider the nonlinear transfer function. The performance of the three methods will be compared by using the pre-multiplied two-dimensional spectra.

The estimations will consider the streamwise velocity fluctuations as the output to be predicted and either streamwise velocity fluctuations or wall-shear stress as the input quantity. Nevertheless, the application to any other quantity is straightforward and demonstrated similar results as long as the same physical quantity was used as input and output (pressure, spanwise or wall-normal velocity fluctuations both as measured input and predicted quantity).

5.1. Linear transfer function

5.1.1. Sample predictions

In the results that follow, a sample prediction is considered, with the output (at $y^+ = 15$), and an input at the lower edge of the log layer, at $y^+ = 50$, for $Re_\theta = 4430$ ($Re_\tau = 1324$). For this case, the resulting transfer function, comparison of the time series and residual error are shown in figure 9.

The resulting transfer function is mostly causal, i.e. its peak and most of its non-zero content is observed for $\tau \geq 0$. The transfer function can then be truncated for $\tau < 0$ without degrading the performance of the model such that the output in this case is obtained only with past measurements of the input, as the integral in (4.1) can be restricted to positive time delays τ .

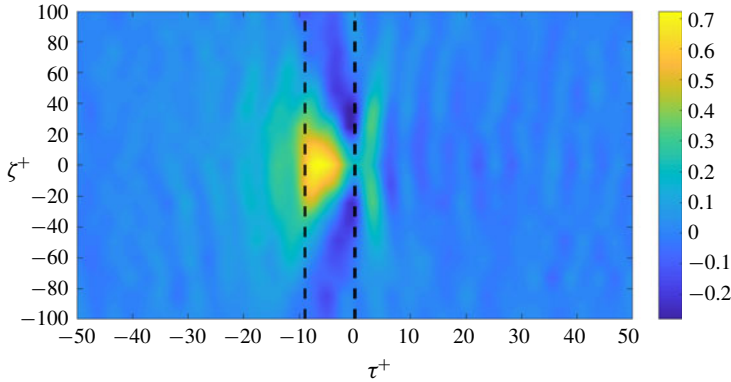


FIGURE 10. (Colour online) Resulting TF when the input is closer to the wall than the output, $y_{in}^+ = 15$, $y_{out}^+ = 50$, for $Re_\theta = 4430$. Zero time delay and value of Δt^+ considered in the estimation of the inclination angle of the structures are indicated by the dashed lines.

The time for the peak is related to the delay between the structures in the log layer and their footprint in the near-wall region. The physical interpretation of this is connected to the fact that the large-scale motions are not vertical, but inclined at an angle (Brown & Thomas 1977; Marusic & Heuer 2007). This results in the observed delay between the measurement of the streamwise velocity at a higher wall-normal position and its effect near the wall, for the same cross-section in the streamwise direction, a feature previously observed in other works (Marusic *et al.* 2010).

It is also noticeable that the transfer function is capable of predicting the behaviour of the large-scale fluctuations of typical wavelength $\lambda_z^+ \approx 1000$ in this case, which are coherent along the wall-normal direction. The resulting pre-multiplied spectra for LES and estimated field, together with the prediction error, will be shown in figure 19 for the different methods evaluated in this work.

Via the use of the linear TF, the estimated field lacks the small-scale, higher-frequency fluctuations, which behave incoherently with measurements in the log layer. This is observed in the residual, as in figure 9, in the time domain. It may be observed that this error basically contains the fine-scale fluctuations, which would be present in the absence of an interaction with the larger scales, present in the outer layer. Although not shown, the resulting error is incoherent with the input signal at $y_{in}^+ = 50$; thus such error could be related to the universal near-wall signal (Marusic *et al.* 2010).

For illustration, the input and output positions are inverted, and the resulting transfer function is shown in figure 10. The behaviour becomes non-causal, as most of the non-zero content of the TF is for negative values. This is related to the aforementioned tilting of the structures in the wall-normal direction. Hence, the resulting model can only be used for data-reconstruction purposes, and real-time (on-line) predictions, and therefore closed-loop control, cannot be performed. This fact could of course be overcome by moving the input measurement upstream. Figures 9(c) and 10 also allow for an estimation of the tilting angle of the structures. Considering a vertical separation between input and measurement of $\Delta y^+ = 35$, a delay of $\Delta t^+ = 9.4$ which is highlighted in figure 10 and a convective speed for the structures of $U_c^+ = 14$, which is between the mean velocities at $y^+ = 15$ and $y^+ = 50$ (see figure 1), the resulting inclination angle is of 14.9° . This estimated value is in very good agreement with previous observations (Brown & Thomas 1977; Marusic & Heuer 2007).

The results of this section are in accordance with the observations of Mathis *et al.* (2009, 2011) and Marusic *et al.* (2010), where it is shown that the fluctuations present at higher wall-normal positions imprint their signature near the wall. The model proposed in these works is capable of instantaneous predictions of the large-scale fluctuations, by modelling the interactions with near-wall structures through a large-scale footprint, corresponding to a linear superposition, and amplitude modulation of near-wall streaks. The behaviour of the finer-scale fluctuations (error signal in figure 9) can then only be predicted statistically, by means of a proposed universal signal, which would be present in the absence of such interactions between the different scales in a turbulent boundary layer. The linear transfer function proposed in this work is capturing such superposition effect of the large-scale structures on the near-wall region.

In the following section, the performance of the method for other wall-normal positions and Reynolds numbers will be evaluated. The construction of transfer functions using input/output data at different streamwise positions, although feasible, will not be pursued here, given the available streamwise fields in the database, which present a high separation between each other. Nevertheless, for short distances, the streamwise separation between positions is expected to mostly alter the time delay for the peak of the transfer function.

5.1.2. Performance for different wall-normal positions and Reynolds numbers

In order to evaluate the dependence of the linear predictions on the distance from the wall, four inputs were considered, $y_{in}^+ = 50, 75, 100$ and 200 , for different values of Re_θ . Two metrics were used to compare the LES and the various predicted fields in a consistent manner, namely the variance of the velocity fluctuations and the normalized correlation between prediction and nonlinear simulation which, for this case, is defined as follows,

$$Corr = \frac{\int_{-\pi}^{\pi} \int_{-\infty}^{\infty} O_{LES}(t, z) O_{est}(t, z) dt dz}{\sqrt{\int_{-\pi}^{\pi} \int_{-\infty}^{\infty} O_{LES}^2(t, z) dt dz} \sqrt{\int_{-\pi}^{\pi} \int_{-\infty}^{\infty} O_{est}^2(t, z) dt dz}}, \quad (5.1)$$

where $O_{LES}(t, z)$ and $O_{est}(t, z)$ are the LES and estimated fields, respectively. Note that this parameter varies between zero and unity, representing complete orthogonality and a perfect match between the two sets, respectively.

Figures 11 and 12 show the behaviour of the variance of the streamwise velocity and the correlations, respectively. Streamwise velocity was considered for both input and output. The predicted fields underestimate the value of the variances, which is in accordance with the fact that only the large structures, which exhibit a low frequency and wavenumber, are well captured by this method; the H_1 estimator leads to an output underpredicted by γ , the coherence function defined in (3.1). One could also interpret the results of figure 11 as the amount of energy in the near-wall region which is due to the wall-connected eddies in the outer positions, given by their linear footprint into the smaller scale structures, which is significant, accounting for up to 45% of the peak variance at $y^+ = 15$, for the highest Reynolds number and $y_{in}^+ = 50$, at the lower edge of the log layer.

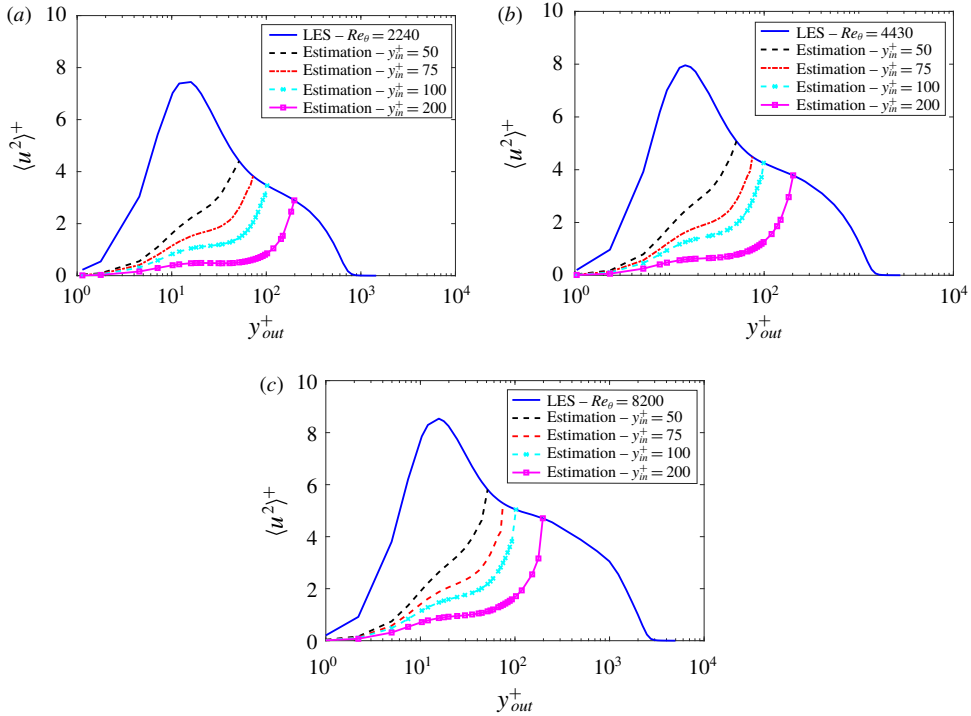


FIGURE 11. (Colour online) Comparison of the streamwise velocity variance of the estimated and LES fields for different input positions along the wall-normal direction at three Reynolds numbers. (a) $Re_\theta = 2240$. (b) $Re_\theta = 4430$. (c) $Re_\theta = 8200$.

Furthermore, the correlations exhibit relatively high values as long as input and output positions are close to each other. It is also noticeable that the performance of the model is fairly independent of the Reynolds number, at least for the evaluated cases, with the highest- Re case slightly outperforming the other two. This may be due to the wider scale separation which is present in this case, and also to the more significant levels at outer positions for increasing Reynolds number, related to the outer peak seen in the spectra (Smits, McKeon & Marusic 2011; Eitel-Amor *et al.* 2014). Finally, it is also noteworthy that even for the highest wall-normal positions under consideration the footprint of the fluctuations at these positions is still felt in the near-wall region, as shown in the correlation plots.

Figure 13 shows a sample of the linear transfer functions, for the three evaluated Reynolds numbers, calculated between positions $y_{in}^+ = 50$ and $y_{out}^+ = 15$. Note that the differences are mainly due to the amplitude and time delay to the maximum value, and are related to the fact that the interaction between wall-normal separated positions is slightly dependent on the Reynolds number of the flow. The information regarding the time delay for the peak comes directly from the inverse Fourier transform and $t^+ = tu_\tau/l^*$, with $l^* = \nu/u_\tau$.

5.1.3. An input on the wall

Results of the previous section allow us to extract the existing interaction between wall-normal separated positions. However, as highlighted in figure 3, the coherence between positions in the log layer and in the near-wall region lacks the fine scales,

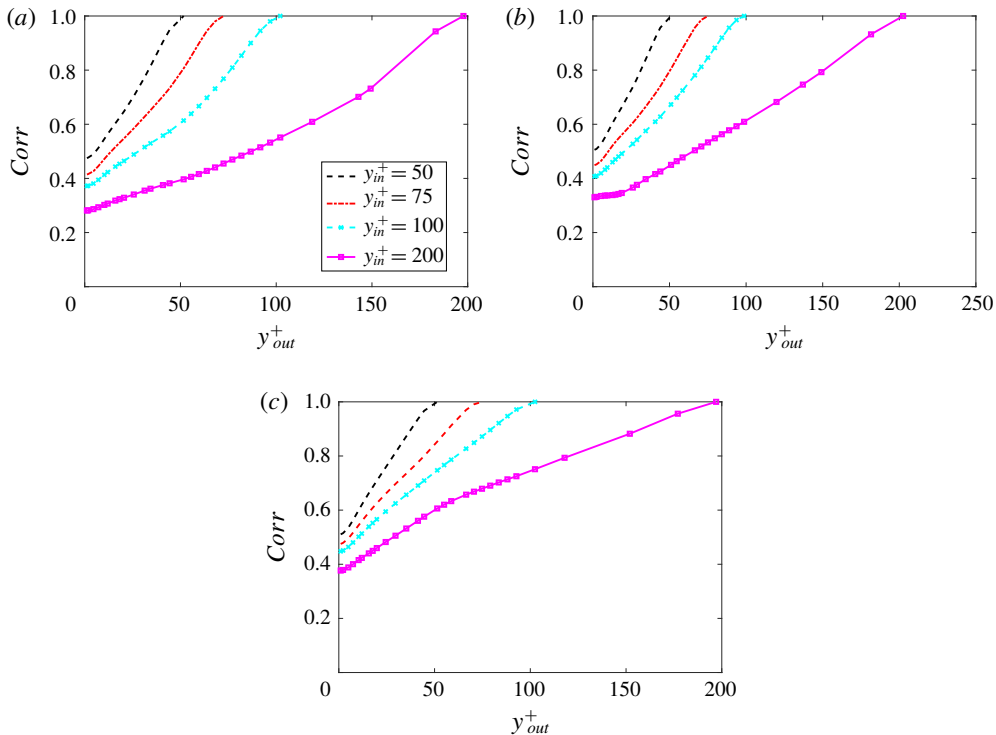


FIGURE 12. (Colour online) Correlation between estimated and LES fields for different positions along the wall-normal direction at three Reynolds numbers. Each curve corresponds to a fixed input position. (a) $Re_\theta = 2240$. (b) $Re_\theta = 4430$. (c) $Re_\theta = 8200$.

which is present when the input is considered on the wall. This is the main motivation for this section, which explores the flexibility of these empirically derived transfer functions to estimate higher wall-normal positions via a quantity potentially available in an experiment, namely the instantaneous wall-shear stress (see, for instance, Vinuesa & Örlü 2017). As a sample of the prediction, the streamwise velocity fluctuation will be considered as the output variable. As was the case for the previous models, instantaneous measurements of both input/output quantities are necessary only for the construction of the model.

Furthermore, the predictions considered here are based on non-causal transfer functions. This implies that they can be used in their present form only for data reconstruction. Their use for on-line (real-time) predictions would require the input measurement to be moved upstream of the output sensor. By doing so, a similar performance to the one shown here is expected.

The behaviour of the variance of the predicted fluctuations in comparison to the actual variance of the LES data and the correlations, calculated for the time-domain data (5.1) are shown in figure 14, using the wall-shear stress as the input quantity. A good performance is observed until $y^+ \approx 10$; above that position the variances start to be rapidly underestimated. A similar trend is also observed for the correlations, however they remain above 0.4 until $y_{out}^+ = 250$, which indicates that the footprint of the large-scale fluctuations present at this position is still felt on the near-wall region. Only the phase of the fluctuations can still be fairly well predicted, their amplitude,

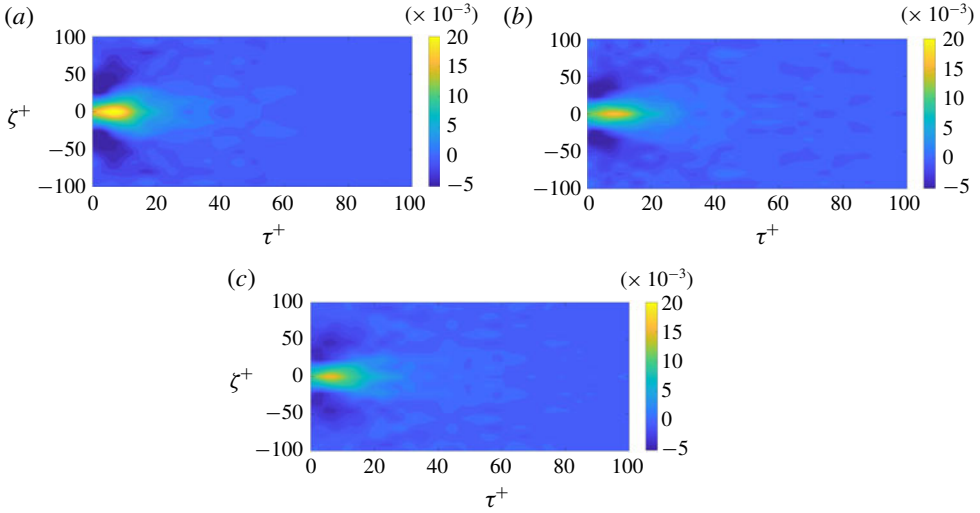


FIGURE 13. (Colour online) Behaviour of the linear transfer function for the three evaluated Reynolds numbers, for $y_{in}^+ = 50$ and $y_{out}^+ = 15$. (a) $Re_\theta = 2240$. (b) $Re_\theta = 4430$. (c) $Re_\theta = 8200$.

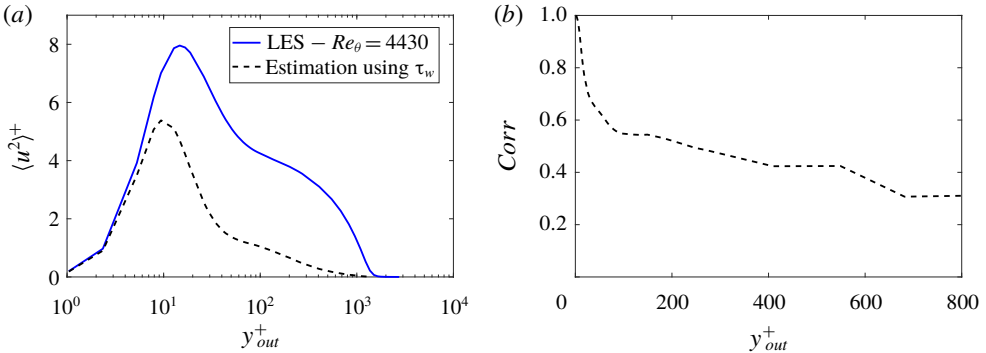


FIGURE 14. (Colour online) Performance metrics for the predicted field in comparison to the LES data, for the $Re_\theta = 4430$ case, using wall shear as input and the linear TF. (a) Variance of predicted and LES fields. (b) Correlation between predicted and LES fields.

on the other hand, is highly underestimated, which account for the large discrepancy observed in the variances at this region.

The resulting pre-multiplied spectrum at position $y^+ = 15$ is shown below in figure 19, and the corresponding error in comparison to the actual LES data in figure 20. It may be observed that the location of the peak in the spectrum is well determined, although its magnitude is underestimated. The better performance obtained when the input y_{in}^+ is considered as the wall-shear stress rather than the streamwise velocity in the outer layer is related to the fact that the fluctuations are coherent within the near-wall region.

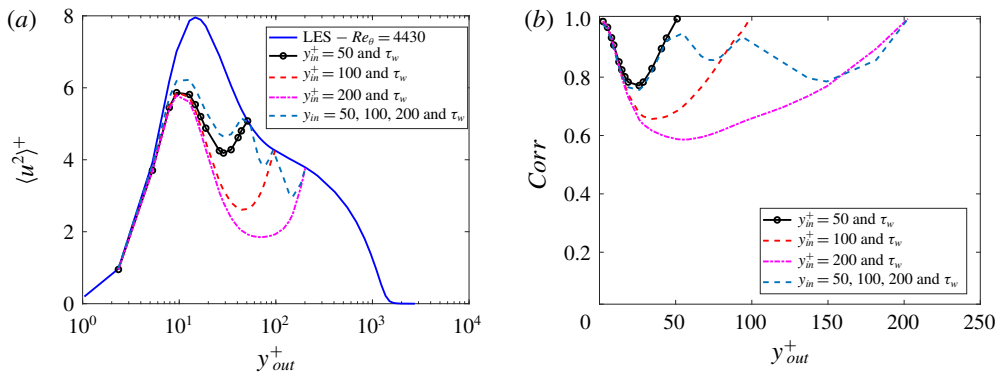


FIGURE 15. (Colour online) Performance metrics for the predicted field, using the MISO transfer functions, in comparison to the LES data. The inputs in the outer layer were $y^+ = 50$, $y^+ = 100$ and $y^+ = 150$. (a) Variance of predicted and LES fields. (b) Correlation between predicted and LES fields.

5.2. Multiple-input transfer function

As shown in § 5.1, the performance of the predictions decays as the distance between input and output positions is increased. In this subsection, we try to diminish such effect by considering the multiple-input–single-output (MISO) transfer function, where one input is located at the wall, given in terms of shear measurements (i.e. wall-shear stress), and the other is at $y^+ = 50$, $y^+ = 100$ or $y^+ = 200$. We also consider the case with four inputs, at the wall and at $y^+ = 50$, 100 and 200. As before, the transfer functions present non-causal values when the input is closer to the wall than the output, which would have to be compensated for real-time implementations by means of offsets in the streamwise separation.

Two metrics will be used to evaluate the performance of the transfer functions: the streamwise velocity variance and the correlation between estimated and LES data, which are given in figure 15, for the case of $Re_\theta = 4430$. Analogous results are also obtained for $Re_\theta = 2240$ and 8200, (not shown). The use of a multiple-input TF with different flow quantities (e.g. streamwise and wall-normal velocity fluctuations) was also assessed, however it did not improve the performance of the predictions and will therefore not be shown here, for the sake of brevity. The use of multiple inputs considerably increases the performance of the predictive model, indicating that inputs on and above the wall are complementary in terms of the information they add to the resulting field. In particular, for the case with four inputs, the correlations between prediction and reference data remain above 70% from $y_{out}^+ = 180$ until the wall. The pre-multiplied spectra of the predicted and LES fields are shown below in figure 19, at the output position of $y^+ = 15$, using a MISO TF with two inputs (namely the velocity fluctuations at $y_{in}^+ = 50$ and the wall-shear stress). The performance of the MISO prediction is significantly improved from the single-input cases, with a good prediction of the main features of the pre-multiplied spectrum.

5.3. Nonlinear transfer functions

In this section, the method of nonlinear spectral analysis will be considered in order to try to further improve the performance of the estimations. A single input will be

considered, located at the wall, using wall-shear stress. The results will concern the $Re_\theta = 4430$ case, however similar observations were also made for the other Reynolds numbers. The output to be estimated was taken as the streamwise velocity fluctuation. Polynomial nonlinearities will be considered as follows:

$$O(z, t) = \sum_{i=1}^{N_{TF}} g_i(I^i(z, t)), \tag{5.2}$$

such that for $N_{TF} = 1$, the linear case is recovered, and for this particular case, N_{TF} will coincide with the highest polynomial order of the transfer function. The evaluation of the number of terms to be used in the series is made in terms of error metrics defined in the time domain, correlations between estimated and LES fields and relative mean values of the error,

$$e_{MS}[\%] = \frac{\int_{-z_{max}/2}^{z_{max}/2} \int_0^{t_{max}} (u_{est}^+ - u_{LES}^+)^2 dt dz}{\int_{-z_{max}/2}^{z_{max}/2} \int_0^{t_{max}} (u_{est}^+)^2 dt dz} \times 100. \tag{5.3}$$

Error metrics in the spectral domain will be obtained by calculating the two-dimensional spectrum of the error, $E_{ee}(\omega, \beta) = \langle \hat{e}\hat{e}^* \rangle$, where

$$e(z, t) = u_{est}^+(z, t) - u_{LES}^+(z, t) \tag{5.4}$$

and determining the relative error at the near-wall peak (err_{peak}) and the relative mean error,

$$err_{mean}[\%] = \frac{\int_{\beta_1}^{\beta_2} \int_{\omega_1}^{\omega_2} E_{ee}(\omega, \beta) d\omega d\beta}{\int_{\beta_1}^{\beta_2} \int_{\omega_1}^{\omega_2} E_{uu}(\omega, \beta) d\omega d\beta} \times 100 \tag{5.5}$$

all of which are indices relevant for closed-loop control applications. The variance of the prediction will also be observed, as a complementary metric.

Figure 16 shows the defined error metrics as a function of the number of terms used in the definition of the nonlinear TF, until $N_{TF} = 20$. The output is located at $y_{out}^+ = 15$. There is a slight improvement when a quadratic term is used for the prediction. The correlations are increased from 0.79 to 0.84, the error decreases from 36 to 31 % and the value of the inner wall peak improves from 4.9 to 5.9. The peak error in the spectral domain decreases from 27 to 20 % and the mean error from 42 to 36 %. However, the addition of more terms, beyond the quadratic estimation, causes the predicted field to be less accurate in the time domain, as the correlation decreases and the error worsens. Although not shown, the same trend is observed for other output positions within the buffer layer, an improvement in the predicted field when a quadratic term is considered in the series. We therefore proceed by truncating the nonlinear analysis at the quadratic term.

The behaviour of the prediction and the residual in the (t, z) domain is shown in figure 17 and demonstrates an improvement with respect to the linear case, particularly regarding the amplitude of the predicted field. The relative mean error, defined as per (5.3) diminishes from 27 to 20 %, from the linear to the nonlinear case, respectively.

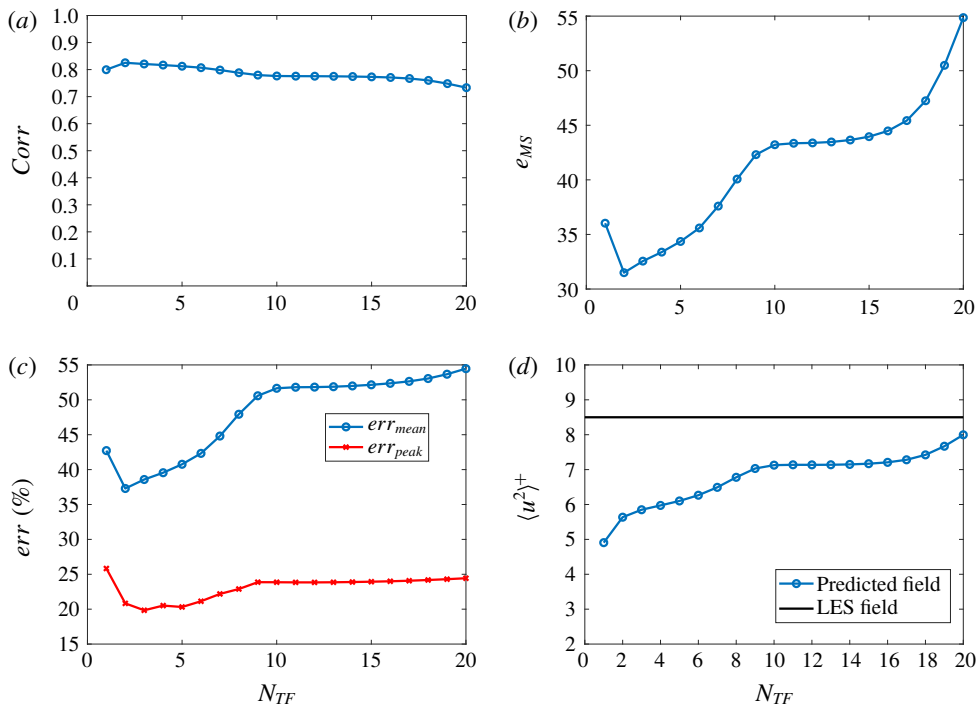


FIGURE 16. (Colour online) Behaviour of the error calculated in the time and frequency domains and variance of the predicted field as a function of the number of terms used in the nonlinear transfer function. (a) Correlation between prediction and LES. (b) Mean square value of the error. (c) Spectral error metrics. (d) Variance of the predicted field.

Analogously to the linear case, in order to better assess the method, the accuracy of the prediction was evaluated as a function of the wall-normal distance. This is shown in figure 18, in terms of the correlation between predicted and LES fields and the corresponding variances. The result is overlaid to the linear case, to better highlight the differences. Up to $y_{out}^+ \approx 30$, the nonlinear method exhibits slightly higher correlations with the actual LES data, with a significant improvement of the variances, indicating that the amplitude is better captured by this strategy. As the output is moved above the buffer layer, the nonlinear strategy does not lead to improvements of the predicted field.

Application of the nonlinear spectral analysis with an input measurement above the buffer layer did not improve the prediction in the time domain. This indicates that the footprint of the larger scales on the near-wall region is predominantly a linear superposition phenomenon, which is being captured by the linear transfer function approach. Although not shown, the quadratic term of the shear-stress input presents a mild coherence with the output signal, which justifies for its improvement in the prediction. Such improvement with a quadratic term had first been observed by Naguib *et al.* (2001), who attributed such characteristic to a turbulent–turbulent interaction, and observed that the quadratic term tends to scale in outer units.

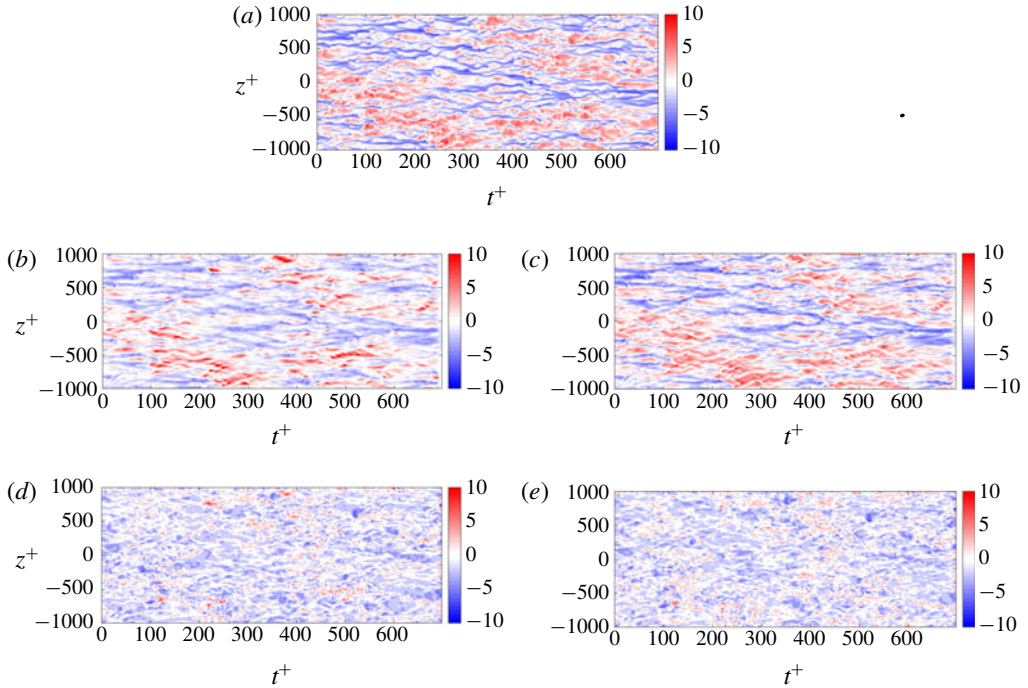


FIGURE 17. (Colour online) Behaviour of the prediction in the (t, z) domain, input at wall using wall-shear stress and output at $y_{out} = 15$. (a) LES data. (b) Linear prediction. (c) Nonlinear prediction with the linear and quadratic terms. (d) Residual, $u_{est}^+ - u_{LES}^+$ with the linear method. (e) Residual, $u_{est}^+ - u_{LES}^+$ with the nonlinear method.

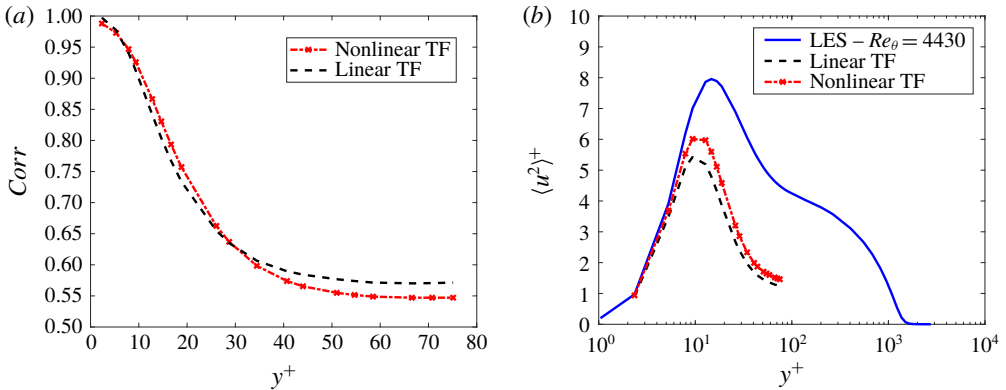


FIGURE 18. (Colour online) Performance of the nonlinear prediction as a function of the wall-normal direction, results are compared to the linear case. (a) Correlation between prediction and LES. (b) Variances of the predicted and LES streamwise velocity fluctuation.

5.4. Comparison of the methods and discussion

We start by computing the two-dimensional pre-multiplied spectra of predictions obtained by the different approaches. The output is located at $y_{out}^+ = 15$, corresponding to the near-wall peak. One, two or four inputs are considered: at the wall, i.e. using

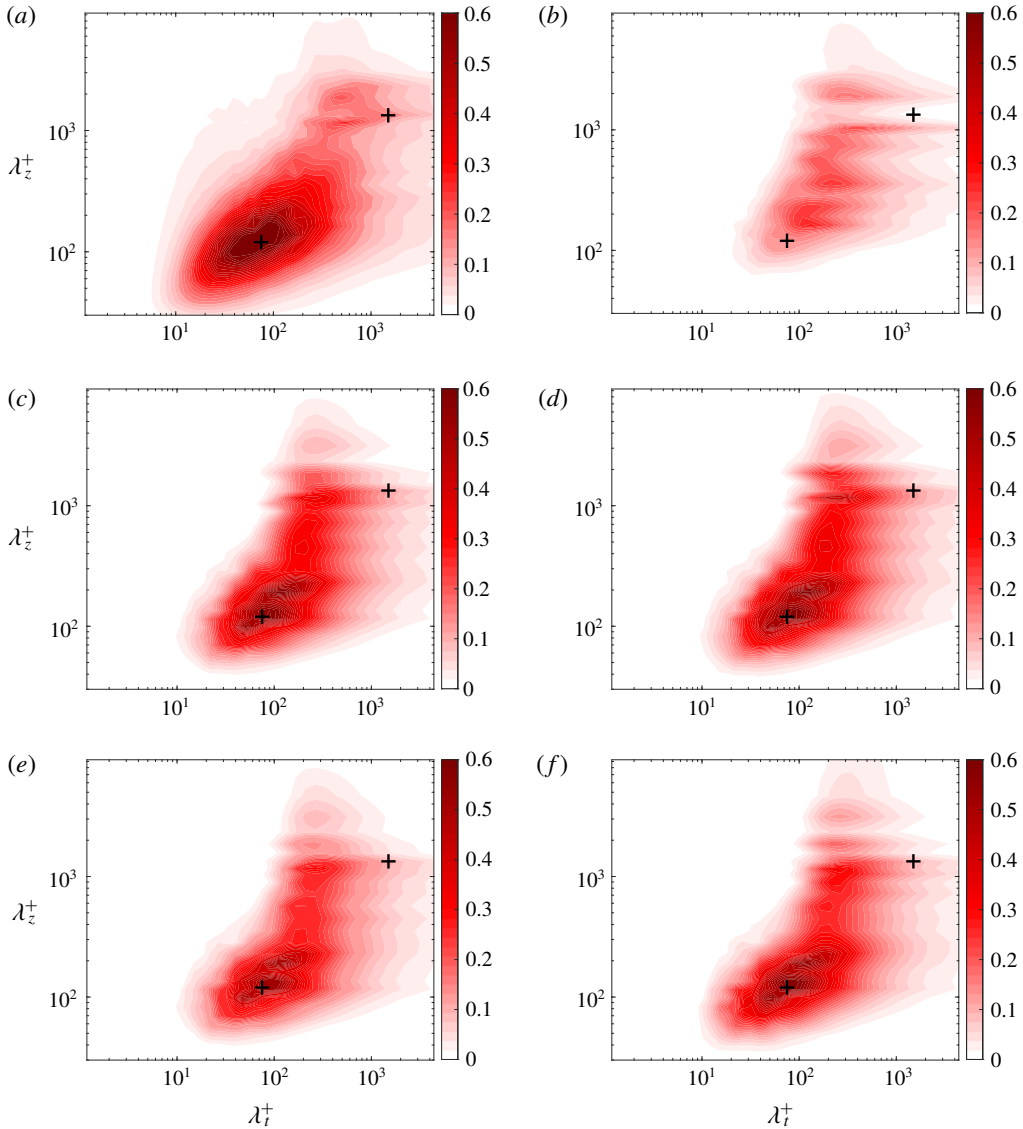


FIGURE 19. (Colour online) Comparison of inner-scaled pre-multiplied two-dimensional spectra from the linear and nonlinear predictions with the LES field at $y^+ = 15$, for the $Re_\theta = 4430$. The crosses correspond to reference locations for the inner $((\lambda_r^+, \lambda_z^+) \approx (100, 100))$ and outer peaks $((\lambda_r^+, \lambda_z^+) \approx (10\delta_{99}/U_\infty, \delta_{99}), (\lambda_r^+, \lambda_z^+) \approx (1400, 1300))$. (a) LES data. (b) Linear transfer function using $y_{in}^+ = 50$ as input. (c) MISO transfer function using $y_{in}^+ = 50$ and wall-shear stress. (d) MISO transfer function using $y_{in}^+ = 50$, $y_{in}^+ = 100$, $y_{in}^+ = 200$ and wall-shear stress. (e) Linear transfer function using wall-shear stress. (f) Nonlinear transfer function using wall-shear stress.

the wall-shear stress, and at $y_{in}^+ = 50$, $y_{in}^+ = 100$ and $y_{in}^+ = 200$. The resulting spectra are shown in figure 19 for the different approaches considered in this work, in comparison to the actual LES data.

Method	err_{peak} (%)	err_{mean} (%)	$Corr$	e_{MS} (%)	TFs/convolutions
Linear SISO, $y_{in}^+ = 50$	84	73	0.62	55	1
Linear SISO using wall-shear stress	27	42	0.79	36	1
Linear MISO, $y_{in}^+ = 50$ and wall-shear stress	25	40	0.82	33	3
Linear MISO, with four inputs	25	40	0.82	33	10
Nonlinear TF using wall-shear stress	20	36	0.84	31	3

TABLE 1. Summary of the error metrics and computational cost for the cases evaluated.

In order to better assess the results, we will consider the two-dimensional pre-multiplied spectra of the residual error. This metric supplies information concerning the instantaneous accuracy of the prediction, an important information for active control applications. The result is shown in figure 20. The linear TF based on an input on the outer layer leads to a significant error in the prediction of the near-wall peak, as only the large-scale structures are coherent between these positions. The accuracy of the determination of the near-wall data is significantly improved when wall-shear data are used, either in the SISO or MISO applications, as the near-wall cycle maintains significant coherence down to the wall.

Four additional metrics will be considered to compare the predictions of the three methods at $y_{out}^+ = 15$: the correlation coefficient (5.1), the relative mean error e_{MS} between predicted and reference fields (5.3), the relative error in the determination of the value of the peak and the relative mean error in the determination of the spectra, both of which can be obtained by the two-dimensional pre-multiplied spectra of the error in figure 20. It should be noted that all of these metrics are related to the ability of the methods to capture the unsteady behaviour of the fluctuations, rather than only the turbulence statistics.

In order to evaluate the computational cost of each method, a suitable parameter is the number of constructed transfer functions (which corresponds to around 20 s per transfer function on a typical workstation, without an optimized algorithm) and of performed convolutions (around 90 s each), which are the most expensive operations of the methods, accounting for more than 90% of the total computational time. For the current approach, the number of convolutions and transfer functions which has to be calculated is the same. All these results are summarized in table 1.

For this particular application, the nonlinear TF outperformed the MISO TF with two or four inputs, with the two MISO approaches presenting similar performances, indicating that the addition of more inputs in the log layer does not add information to the prediction of the near-wall peak. However, when one considers the accuracy variation along the wall-normal direction, it is observed that the MISO prediction exhibits better results, with the addition of measurements improving the predictions, such that the exact method of choice is dependent on the application and complexity of the experimental set-up to be used.

The linear TF using $y_{in}^+ = 50$ exhibits the worst behaviour, since it does not account for the fine-scale fluctuations, where most of the energy is contained in the near-wall region. Its spectrum is mainly due to larger fluctuations, indicating that such large structures behave linearly between the different wall heights, leaving a footprint on the near-wall structures. This trend is also observed in figure 20(a), where the error for this method is considerable at $\lambda_z^+ \approx 100$, leading to a high error in the determination of the near-wall spectral peak.

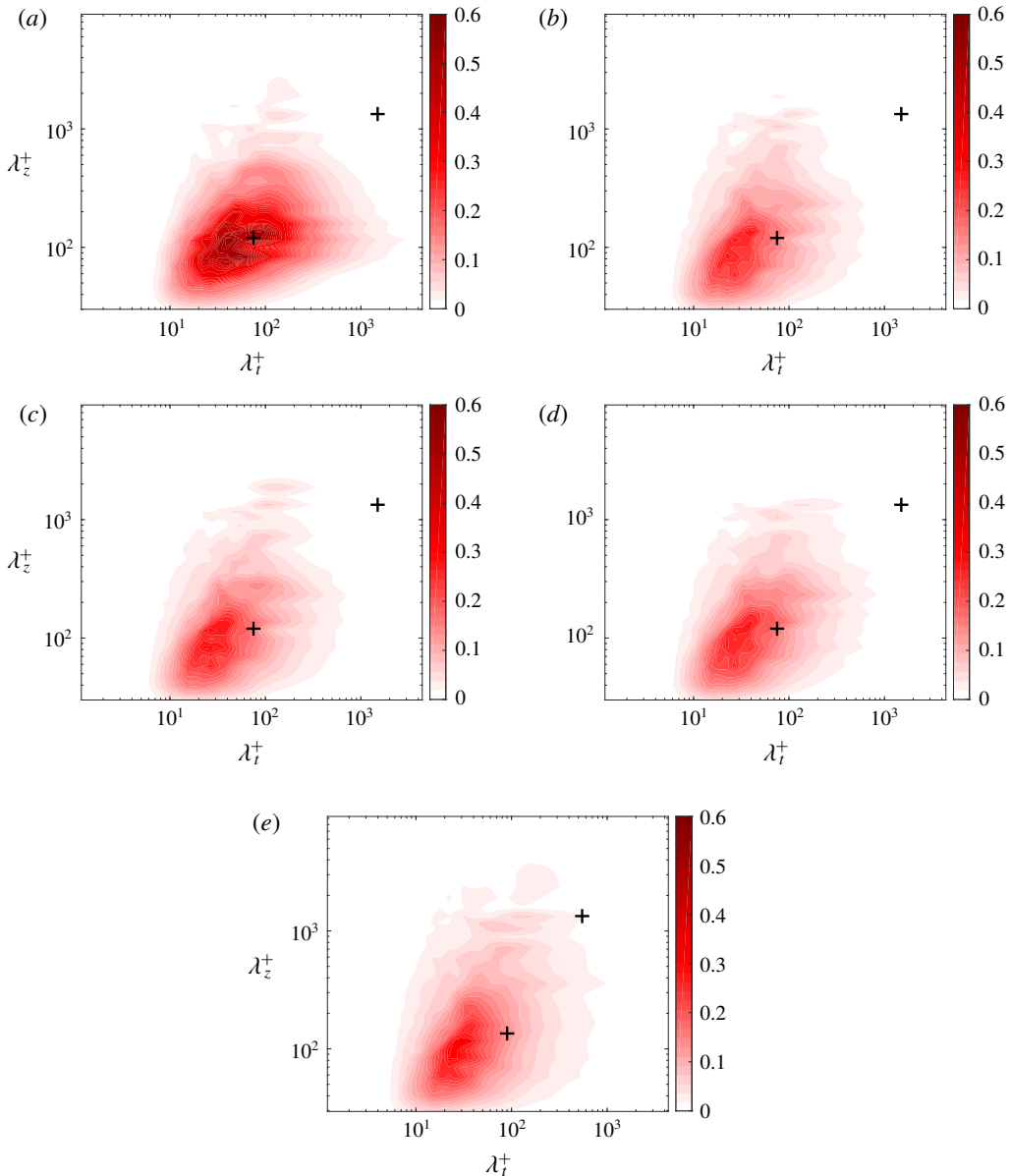


FIGURE 20. (Colour online) Pre-multiplied two-dimensional spectra of the error calculated between predicted and LES streamwise velocity fields. The limits of the colour bar were kept as those of the reference data, in order to facilitate the comparison. The crosses correspond to reference locations for the inner $((\lambda_t^+, \lambda_z^+) \approx (100, 100))$ and outer peaks $((\lambda_t^+, \lambda_z^+) \approx (10\delta_{99}/U_\infty, \delta_{99}), (\lambda_t^+, \lambda_z^+) \approx (1400, 1300))$. (a) Linear transfer function using $y^+ = 50$ as input. (b) MISO transfer function using $y_{in}^+ = 50$ and wall-shear stress. (c) MISO transfer function using $y_{in}^+ = 50, y_{in}^+ = 100, y_{in}^+ = 200$ and wall-shear stress. (d) Linear transfer function using wall-shear stress as input. (e) Nonlinear transfer function using wall-shear stress.

The linear and nonlinear SISO and linear MISO predictions using wall-shear-stress velocity fluctuations above the wall, respectively, considerably improve the accuracy of the models and allow a reasonable determination of the inner peak, such that only the amplitude is slightly off in the predicted field. The main improvement comes from the use of wall-shear stress, which is highly correlated with the structures responsible for the near-wall dynamics.

The three methods evaluated in this work vary considerably in terms of their accuracy, with the linear SISO TF being significantly outperformed by the other approaches, particularly when an input in the log layer ($y_{in}^+ = 50$ or 100) is used to estimate the near-wall behaviour. There could be two explanations for such an observation: either the method to identify the transfer function is not efficiently capturing the linear mechanisms that occur, or there is not enough coherence between these two separated regions.

The method proposed in § 5.1 could potentially exhibit errors with respect to the exact model either due to the presence of noise in the sensors that acquire the data for the system identification, or due to spectral leakage. The former is not present in this work, since the raw LES signals are taken directly for the construction of the model. As for the latter, it is avoided by means of using an adequately long time series, an assumption which was assessed by considering different lengths for the time series. Therefore, the implementation employed in this work can be regarded as the best linear approximation between the two positions separated in the wall-normal direction (Schoukens *et al.* 1997).

From $y_{in}^+ = 50$ or 100 one can only predict the footprint of the large scales on the near-wall region. This statement is supported by the two-dimensional coherence plots of figure 3, which show that only the long wavelengths are coherent, a behaviour further confirmed by the two-dimensional pre-multiplied spectra of the linear SISO prediction (figure 19), which contains only the large-scale structures. This also justifies for the large mismatch in the predicted variance. Furthermore, the attempts to use a nonlinear method with an input outside the buffer layer were unsuccessful. By considering the two-dimensional coherence plots (figure 3) it is evident that in order to capture the small-scale structures it is necessary to have access to data in the near-wall region (note that in this work we considered the wall-shear stress). An interesting method arises when data at the wall and $y_{in}^+ = 50$ or 100 are used as prediction inputs, since these are complementary in terms of the coherence of the structures.

Other ways to capture this effect would be by directly adding the statistics of the remaining unpredicted field, as it is done with the universal signal in Marusic *et al.* (2010), or attempting a state estimator using the linearized Navier–Stokes system including an eddy-viscosity profile, as in Illingworth *et al.* (2018), which is not straightforward for the linear SISO TF dealt with in this work. Comparable results to the two works cited above were obtained by means of the current approaches, where no extra modelling parameters are needed in order to perform the predictions. These comparisons will be discussed in further detail in § 7.

The computational cost of the MISO and nonlinear approaches evaluated here is about three times higher than that of the SISO linear case. However, we do not consider this difference to increase the cost such that it would be prohibitively expensive to implement these methods.

6. Off-design predictions

As outlined in the previous sections of this work, the on-line prediction of the designed transfer functions is made in two steps. The convolution kernels are

firstly built using simultaneous time- and spanwise-resolved signals at the positions corresponding to inputs and output of the system, which are separated along the wall-normal direction. Once this system has been built, the prediction is made exclusively from measurements of the inputs.

This therefore raises questions accounting for the robustness of the methods for mild Reynolds-number variations, which would certainly be present in experimental applications. Furthermore, the fact that simultaneous measurements have to be performed as an initial step may lead to constraints in practical applications, where the correspondingly high Reynolds number would require a very fine resolution in the wall-normal direction, which may be difficult to obtain. Such constraints are shared by other methods available in the literature which require an initial set of data to build the model (also referred to as a training dataset in the context of machine learning methods).

One solution would therefore be to perform a large-eddy or direct numerical simulation of the geometry to be studied experimentally, and build the transfer functions *a priori*, which would only require the capability of measuring the inputs in the experiment (i.e. wall-shear stress and the fluctuations outside the buffer layer). Since high-fidelity simulations can typically only be performed at Reynolds numbers lower than those in certain experimental studies, it is of interest to design a model using a moderate Reynolds number, obtained from a high-fidelity simulation, and apply it to an experimental implementation at higher Re . This possibility is explored in this section, where the transfer functions will be designed at $Re_\theta = 880$ and tested at $Re_\theta = 8200$, spanning approximately one order of magnitude in Reynolds number. Three cases will be considered here: linear SISO and MISO using wall shear and data outside the buffer layer, and the quadratic TF using wall-shear stress, which exhibits mild improvements in comparison to the linear case. The same input/output positions for the two Reynolds numbers will be considered in terms of the corresponding inner variables, which showed better performance than if the outer non-dimensionalization was utilized. Figure 21 shows the resulting correlations and variances for these three methods, for the original and off-design cases. For all the cases, the correlations in the off-design configurations are degraded, where the SISO implementation is the most affected. It should also be noted, however, that the prediction of the near-wall peak is not much affected, where compelling correlations are seen for the SISO using wall-shear stress and MISO methods. The prediction of the MISO transfer function is also less affected by the Reynolds-number variation, where it should be noted that for the case with four inputs, the off-design prediction remains with compelling correlations (above 60%) throughout the evaluated range of wall-normal positions. The fact that the near-wall peak prediction from wall-shear data was practically unaffected supplies evidence that the near-wall region is directly correlated to the wall. The increase in Re has an effect on the large-scale modulation, modifying the footprint of the large-scale structures near the wall which in turn leads to the high degradation of the prediction of the near-wall peak when SISO transfer functions using data above the wall are used. We have therefore demonstrated the feasibility of designing SISO and MISO transfer functions at lower Reynolds numbers and implementing them at higher Re (one order of magnitude higher in this particular case), along with a considerable robustness of the methodology.

Although not shown for the sake of brevity, variations of up to 50% in Reynolds number had no significant impact on the accuracy of the prediction. This feature is expected to be appealing for the application of the developed methods for experimental implementations, where uncertainties in the measured parameters are expected to be present.

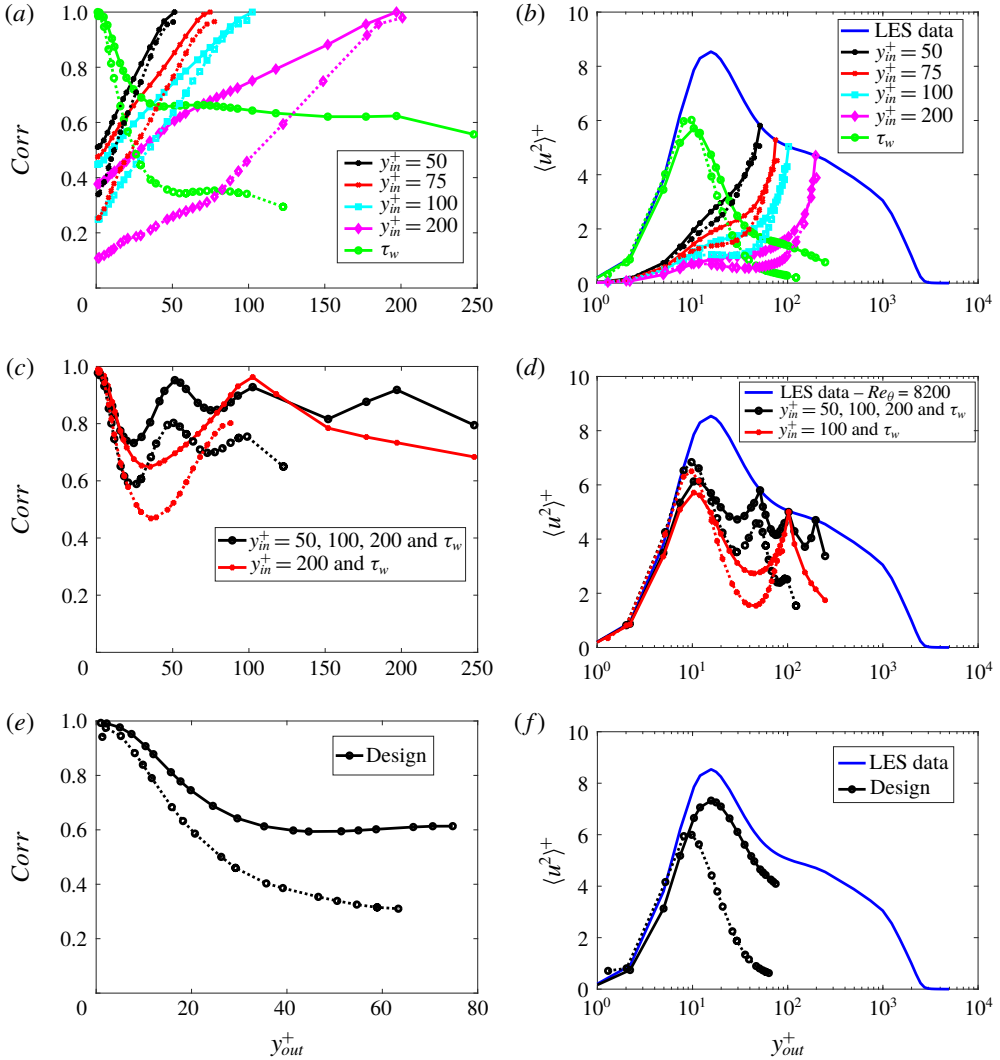


FIGURE 21. (Colour online) (a,c,e) Correlations between prediction and LES data and (b,d,f) comparison of the resulting variances using the SISO, MISO and nonlinear methods. Solid lines correspond to the same Reynolds number being used for design and test, whereas dotted lines are used for off-design cases, i.e. the model is built at $Re_\theta = 880$ and tested at $Re_\theta = 8200$. (a) Correlations using linear SISO. (b) Variances using linear SISO. (c) Correlations using the linear MISO. (d) Variances using the linear MISO. (e) Correlations using quadratic SISO. (f) Variances using quadratic SISO.

7. Comparison with other methods

The methodologies developed in this work bear similarities with the recent studies by Baars *et al.* (2016) and Illingworth *et al.* (2018). The different focus of the first, which is on the characterization of the statistics, and application of the second, a channel at $Re_\tau = 1000$, prevent quantitative comparisons of performance between these methods and the one proposed here. The objective of the current section is therefore

to highlight differences in the approach, applicability, limitations and computational cost of the methods.

Illingworth *et al.* (2018) considered a turbulent channel flow of $Re_\tau = 1000$, obtained from DNS. Their approach is based on a Reynolds decomposition of the Navier–Stokes operator and gathering the nonlinear terms from the fluctuations into a forcing to an otherwise linear system. A Fourier transform is then applied to the homogeneous coordinates, corresponding to stream- and spanwise directions. The resulting inhomogeneous Orr–Sommerfeld squire (OSS) system is expressed in the form of an input/output state-space system, where the input corresponds to the forcing (nonlinear terms) and the output to the three velocity components. The idea is then to design a state observer, i.e. a linear operator that permits the estimation of the quantities based on a measurement of the three velocity components at a given height. This is made by means of an optimization problem, which enables the consideration of noise in the measurement sensors. The size of this problem is of the same order as the discretized Orr–Sommerfeld squire (OSS) operator, which could correspond to a square matrix of a few hundred lines. The computation of the model is made ‘off line’, without access to the simulated/experimental quantities. An eddy viscosity is added to the linear operator, and therefore a model is used for the determination of its behaviour along the wall-normal direction. Once the observer has been designed, the prediction is made by measuring the three velocity components at a given wall-normal plane and integrating the observer equations. The Fourier transform applied to the homogeneous directions implies that such measurements correspond to a whole plane in the span and streamwise directions, which is of course more difficult to accomplish experimentally than the proposal of the current work, in which we consider a spanwise line at a fixed streamwise position.

Baars *et al.* (2016) proposed a modification of the inner/outer interaction model by Marusic *et al.* (2010), where the superposition term, which was originally calculated by a linear operation over the input signal, is replaced by a linear stochastic estimator (LSE). Their flow case is a ZPG TBL of Re_τ up to 19 000. The LSE is an entity calculated in the frequency domain and it requires measurements of the input and output quantities which are Fourier transformed to build a frequency-domain transfer function. The model is built using time measurements only, which is a configuration simpler to implement experimentally than the one proposed in this study. However, note that while we aim at predicting instantaneous quantities, the work of Baars *et al.* (2016) is only focused on turbulence statistics. Once the LSE is available, a prediction is performed from Fourier-transformed measurements of the input. Since the predictions are obtained in the frequency domain and then inverse Fourier transformed, certain characteristics such as the causality of the prediction cannot be evaluated. The second part of the inner/outer interaction model is unchanged and its calculation involves an iterative procedure to find a demodulated ‘universal signal’ which ensures that the statistics of the resulting predicted signal are recovered. This also means that the full model, which exhibits very good agreement with experimental data, cannot be used for unsteady (time-domain) predictions, such as those necessary for control.

Finally, the method proposed here could be considered as an alternative to these two methods, depending on the desired application. Our results in terms of the predicted statistics are expected to be inferior to those of Baars *et al.* (2016), with a moderately higher computational cost, since the method presented in that work avoids the use of a two-dimensional convolution. However, for time-domain predictions, the method outlined here is expected to outperform the one of Baars *et al.* (2016), particularly when it is enhanced with multiple inputs which are complementary in terms of

their coherent content with the output, a feature which, to the best of the authors' knowledge, is not available in any other model in the literature for this particular application. The current method produces correlation coefficients on the same level as those of Illingworth *et al.* (2018), with an expected lower computational cost and a simpler scheme to implement in an experimental application. The characteristics of the methods by Baars *et al.* (2016) and Illingworth *et al.* (2018), together with those of the method proposed in this study, are summarized in table 2.

8. Conclusions

Three different techniques were evaluated to predict the behaviour of the near-wall streamwise velocity fluctuations in ZPG turbulent boundary layers. The methods vary in terms of their inherent complexity and their potential difficulties in an experimental implementation.

The linear, single-input–single-output (SISO) transfer function is the simplest and allows for predictions of the output signals from a single input measurement, here considered as space–time series of the streamwise velocity, $u(z, t)$, at a given station x and wall-normal position y . Reasonable performances were obtained for small separations along the wall-normal direction, leading to an eduction of the footprint of the large-scale fluctuations, a linear superposition of such structures on the near-wall region. Such footprint remained for wall-normal distances as high as $y^+ = 250$, as depicted in the results of figure 14. Note that although the amplitudes for such high separations were under-predicted, the phase of the predicted field still led to correlations of the order of 40%. The implementation of this method in experimental applications would depend on the availability of two simultaneous measurements for constructing the TFs, one of them possibly corresponding to wall-shear stress. The availability of the SISO transfer function also allows for the derivation of linear control laws, as explored in other works by this group, for transitional flows (Sasaki *et al.* 2018a). Furthermore, as long as the wall-normal separation between input and output measurements is kept small, a reasonable prediction is obtained by this method.

Addition of a quadratic term to the SISO prediction using wall-shear stress leads to a significant improvement of the prediction, particularly on what concerns the amplitude of the predicted signal. The mean square relative error corresponds to 31% in the near-wall peak, with an error of 20% in the determination of the near-wall spectral peak. The nonlinear prediction has the same experimental requirements as the linear one and would therefore be recommendable when a single wall-shear-stress measurement is available.

Finally, the multiple-input transfer function requires a more complex set-up, where several simultaneous measurements are required both for the construction of the model and the prediction. Nevertheless, improved performances were observed for this method, with the correlations between prediction and reference simulation data remaining above 70%, from $y_{out}^+ = 180$ until the wall, when four input measurements were considered. Since this method corresponds to a linear approach, the use of standard linear control techniques may be carried out directly within this framework (Bendat & Piersol 2011). This is the most interesting approach for data reconstruction from a limited set of measurements, since it produces velocity fields in compelling agreement with the reference data.

The differences between the resulting accuracy of the methods were understood in terms of the coherence between the considered wall-normal positions and it was shown

Characteristic	Present SISO/MISO TFs	Illingworth <i>et al.</i> (2018)	Baars <i>et al.</i> (2016)
Flow case	ZPG turbulent boundary layer	Turbulent channel flow	ZPG turbulent boundary layer
Approach	Compute identified transfer functions from the data, prediction is made on the time domain	Design a state observer on the OSS system, prediction on the time domain	Compute a LSE from the data and modify the inner/outer interaction model; prediction is made on the frequency domain
Degree of modelling	Parameters of the ensemble average, window and length of the time series (low)	Model for eddy viscosity (high)	Parameters of the ensemble average, window and length of the time series (low)
Measurement requirements to build the model	Simultaneous time and span measurements in the input/output locations	None	Time measurements in two locations in the boundary layer
Measurement requirements to perform predictions	Simultaneous measurements of the inputs in time and spanwise direction	Measurements of the three velocity components, time, streamwise and spanwise	One-point measurements of the input in time
Computational cost of building the model	Number of convolutions and ensemble averages grows with the factorial of the number of inputs/nonlinear terms (intermediate)	Solution of an optimal problem for a system of the order of a few hundred points (high)	Only one ensemble average for input/output plus the de-modulation (low)
Computational cost of performing predictions	One convolution per considered input (intermediate)	Solution of a set of differential equations for the observer (high)	One inverse Fourier transform and computation of the modulation term (low)
Does it consider experimental restrictions?	Yes, it minimizes measurement noise	Maybe: there is the possibility to consider a given shape for the noise in the measurement	Yes, the LSE part minimizes measurement noise. Only needs data readily available in experiments
Application to control	Same models have been previously used by this group for the control of transitional flows	Observer can be coupled to a control problem (similar to the classic linear quadratic Gaussian regulator)	The full model cannot be used for control, only the LSE, although not straightforward
Other characteristics	Data-driven method but allows the evaluation of certain physical characteristics such as causality and the tilting angle of the structures	Developed from a theoretical approach but relies on eddy viscosity model	Data driven, does not consider causality, the model could be non-causal

TABLE 2. Summary of the characteristics of available models for the prediction of turbulence fluctuations in the time domain.

that the footprint of the large scales in the outer layer can be connected to a linear mechanism, deduced by the linear SISO transfer function. This footprint is considered as a linear superposition in the model of Marusic *et al.* (2010) and is here directly extracted from the data via the linear transfer function approach.

In summary, when considering the reference output position at $y_{out}^+ = 15$, it is observed that the linear SISO with a measurement outside the buffer layer can only capture the linear footprint of the large scales, which leads to considerable errors in the spectral content and variances of the prediction, as only approximately 40% of the energy in this position can be reproduced by the linear method. The use of nonlinear terms was ineffective for this case, indicating mostly a linear correlation between the large scales and the near-wall region. By measuring wall-shear stress, the prediction is considerably improved, leading to errors of 36% and 31%, depending on whether the linear or nonlinear methodology is performed. The use of a quadratic term here improved the predictions, a trend which had been previously observed (Naguib *et al.* 2001) and which is related to turbulent–turbulent interactions of the fluctuations. As for the MISO prediction, even though it does not outperform the nonlinear analysis for the output at $y_{out}^+ = 15$, given that it deals with inputs of complementary coherence, it permits to obtain low error values throughout the range of wall-normal positions, depending on the number of inputs considered, presenting therefore overall better performances than the SISO methods.

Although not shown here, the same analysis with the different methods evaluated here was also performed for other input/output quantities. Very similar trends and levels of accuracy were observed when the same quantity was used as input and output (i.e. wall-normal and spanwise velocity or pressure fluctuations both as the measured and estimated quantities, for example).

The present work also shows a significant robustness of the various methods concerning Reynolds-number variations between the data used to construct the transfer functions and to perform the flow predictions. It is proposed that the present methodology could be used in experimental studies at higher Reynolds numbers, where it is very difficult to accurately measure close to the wall. A test spanning a one order of magnitude variation in Reynolds number (between the data used to build the model and for the predictions) led to very satisfactory results, providing evidence of the feasibility of this approach.

Comparison between the method outlined in this article with the recent work of Baars *et al.* (2016) demonstrates an expected higher accuracy in the time-domain predictions along with the possibility of using our method for in the design of control laws; furthermore, a simpler experimental set-up would be required compared to that of the method by Illingworth *et al.* (2018) along with a lower computational demand. To summarize, the strategy proposed here demonstrates a high flexibility in terms of input/output measurements which could be advantageous for implementations in flow control and at high Re , where the exact choice of method (SISO/MISO, linear/nonlinear) would depend on the available experimental setting, the desired accuracy and the particular application.

Acknowledgements

Part of this work has been developed during an exchange programme to KTH, when the first author appreciated a scholarship from Capes, project number 88881.132008/2016-01. K.S. has obtained a scholarship from FAPESP, grant no. 2016/25187-4. A.V.G.C. was supported by a CNPq grant 310523/2017-6. The authors would

also like to acknowledge the funding from PreLaFloDes, Swedemo and CISB (Swedish-Brazilian Research and Innovation Centre). R.V. and P.S. acknowledge the funding provided by the Swedish Research Council (VR) and from the Knut and Alice Wallenberg Foundation. This research is also supported by the ERC grant no. ‘2015-AdG-694452, TRANSEP’ to D.S.H.

Appendix A. Calculation of the transfer functions

This section outlines the procedure to calculate the power spectral densities used in the calculation of the (ω, β) transfer functions as per (4.3). The case of the auto power spectral density will be considered, however the method is easily adapted for the cross power spectral density case.

Consider the signal $I(z, t)$, measured along the spanwise direction and time, where z is a periodic direction. The auto spectral density of such signal is then defined as,

$$S_{II} = \hat{\hat{I}}(\omega, \beta) \hat{\hat{I}}^*(\omega, \beta), \tag{A 1}$$

where the double hat indicates a double Fourier transform from z to β and from t to ω . The first transform is taken along the z direction,

$$\hat{I}(t, \beta) = \int_{-z_{max}/2}^{z_{max}/2} I(t, \zeta) e^{-i\beta\zeta} d\zeta. \tag{A 2}$$

Upon discretization in z , the continuous Fourier transform becomes a discrete Fourier transform (DFT). Since this direction is periodic, the DFT may be regarded directly as the discretization of the coefficients of the corresponding Fourier series.

Consider now the discretization in time, with a total number of N snapshots, such that the signal may be regarded as:

$$\hat{I}(\beta) = [\hat{I}_1(\beta) \quad \hat{I}_2(\beta) \quad \dots \quad \hat{I}_N(\beta)], \tag{A 3}$$

where $\hat{I}(\beta)$ is $N_\beta \times N$, N_β corresponding to the total number of transverse wavenumbers considered. It is a known result (Bendat & Piersol 2011) that the direct application of the DFT on the lines of $\hat{I}(t, \beta)$ should not be performed, as the result will not converge with the increasing number of snapshots N . The error in the estimate of each frequency could be as high as the calculated magnitude of the corresponding auto spectrum. In order to obtain converged estimates it is necessary to appropriately average the spectra over multiple ensembles of the flow, which may be accomplished by Welch’s method (Welch 1967). Start by partitioning the full signal into N_b blocks with N_f elements, such that the n th block is given as:

$$\hat{I}^{(n)}(\beta) = [\hat{I}_1^{(n)}(\beta) \quad \hat{I}_2^{(n)}(\beta) \quad \dots \quad \hat{I}_{N_f}^{(n)}(\beta)], \tag{A 4}$$

with each block referred to a realization of the flow. The blocks may also present an overlap region with the adjacent elements, which allows for an increased number of blocks for the same length of the original signal, permitting a faster convergence of the algorithm. The k th entry in the n th block is then given as $\hat{I}_k^{(n)}(\beta) = \hat{I}_{k+(n-1)(N_f-N_o)}(\beta)$, where N_o is the number of overlapping snapshots. The DFT is then calculated at each block,

$$\hat{I}^{(n)}(\beta) = [\hat{I}_1^{(n)}(\beta) \quad \hat{I}_2^{(n)}(\beta) \quad \dots \quad \hat{I}_{N_f}^{(n)}(\beta)], \tag{A 5}$$

such that:

$$\hat{I}_k^{(n)}(\beta) = \frac{1}{\sqrt{N_f}} \sum_{j=1}^{N_f} w_j \hat{I}_j^{(n)}(\beta) e^{-2\pi i(k-1)[(j-1)/N_f]}, \tag{A 6}$$

for $k = 1, \dots, N_f$ and $n = 1, \dots, N_b$. This represents the DFT of the block, adding the weights w_j to determine the application of a window function which is used to reduce spectral leakage due to each block being non-periodic. The normalization factor $1/N_f$ ensures that the DFT is unitary for a square window. Therefore, $\hat{I}_k(\beta)$ is the DFT at a frequency ω_k , for each block, and

$$\omega_k = 2\pi \frac{k-1}{n\Delta T}, \quad k \leq n/2 \tag{A 7}$$

or

$$\omega_k = 2\pi \frac{k-1-n}{n\Delta T}, \quad k > n/2. \tag{A 8}$$

To simplify the notation, the subscript k is dropped from the frequency components in the remaining of this section and in the main body of the paper.

Finally, S_{II} at a pair (ω, β) is calculated by averaging the blocks,

$$S_{II}(\omega, \beta) = \frac{\Delta t}{N_f} \frac{\sum_{n=1}^{N_b} \hat{I}(\omega, \beta) [\hat{I}(\omega, \beta)]^*}{\sum_{j=1} w_j^2 N_b} \tag{A 9}$$

which converges as the number of snapshots and blocks is increased together. The final value obtained from this average is referred to as the expected value of the quantity $\hat{I}(\omega, \beta) [\hat{I}(\omega, \beta)]^*$ and the process to obtain it is referred to as an ensemble average.

Appendix B. Decorrelation via spectral conditioning

The spectral conditioning technique consists in removing all the linear influence that exists between the inputs, hence decorrelating them (Bendat & Piersol 2011). The technique is performed iteratively, the first input remains unchanged and all the others have to be computed. In order to spectrally conditionally remove the influence of input 1 over input 2, for example, the following operation is performed in the frequency domain

$$\hat{I}_{2-1}(\beta, \omega) = \hat{I}_2(\beta, \omega) - \frac{S_{I_1 I_2}}{S_{I_1 I_1}} \hat{I}_1, \tag{B 1}$$

which, in the time domain is equivalent to

$$I_{2-1}(t, z) = I_2(t, z) - g_{I_1 I_2}(t, z) \otimes I_1(t, z), \tag{B 2}$$

where \otimes indicates a double convolution and the subscript $2-1$ indicates that input 1 is linearly decorrelated from input 2, via the transfer function $g_{I_1 I_2}$, which relates the two inputs. For the sake of brevity, the following operations will be shown only in the time domain, where the definition of the linear transfer function is implicit. The third input has to be decorrelated from $I_1(t, z)$ and $I_{2-1}(t, z)$, such that

$$I_{3-2!}(t, z) = I_3(t, z) - g_{I_2-1 I_3}(t, z) \otimes I_{2-1}(t, z) - g_{I_1 I_3}(t, z) \otimes I_1(t, z), \tag{B 3}$$

where the subscript 3 – 2! indicates the decorrelation of input 3 from inputs 2 and 1. For the N th input, the decorrelation results in

$$I_{N-(N-1)!}(t, z) = I_N(t, z) - \sum_{j=1}^{N-1} g_{I_{[N-j]-[N-j-1]!}} I_N(t, z) \otimes I_{[N-j]-[N-j-1]!}. \quad (\text{B } 4)$$

REFERENCES

- ABREU, L. I., CAVALIERI, A. V. & WOLF, W. 2017 Coherent hydrodynamic waves and trailing-edge noise. In *23rd AIAA/CEAS Aeroacoustics Conference*, p. 3173. AIAA.
- DEL ÁLAMO, J. C. & JIMÉNEZ, J. 2006 Linear energy amplification in turbulent channels. *J. Fluid Mech.* **559**, 205–213.
- BAARS, W. J., HUTCHINS, N. & MARUSIC, I. 2016 Spectral stochastic estimation of high-Reynolds-number wall-bounded turbulence for a refined inner–outer interaction model. *Phys. Rev. Fluids* **1** (5), 054406.
- BENDAT, J. S. 1993 Spectral techniques for nonlinear system analysis and identification. *Shock Vib.* **1** (1), 21–31.
- BENDAT, J. S. & PIERSOL, A. G. 2011 *Random Data: Analysis and Measurement Procedures*. vol. 729. Wiley.
- BERNARDINI, M. & PIROZZOLI, S. 2011 Inner/outer layer interactions in turbulent boundary layers: a refined measure for the large-scale amplitude modulation mechanism. *Phys. Fluids* **23** (6), 061701.
- BLACKWELDER, R. F. & KOVASZNY, L. S. G. 1972 Time scales and correlations in a turbulent boundary layer. *Phys. Fluids* **15** (9), 1545–1554.
- BROWN, G. L. & THOMAS, A. S. W. 1977 Large structure in a turbulent boundary layer. *Phys. Fluids* **20** (10), S243–S252.
- CHEVALIER, M., LUNDBLADH, A. & HENNINGSON, D. S. 2007 Simson – a pseudo-spectral solver for incompressible boundary layer flow. *Tech. Rep.* TRITA-MEK, KTH Mechanics, Stockholm, Sweden.
- COSSU, C., PUJALS, G. & DEPARDON, S. 2009 Optimal transient growth and very large-scale structures in turbulent boundary layers. *J. Fluid Mech.* **619**, 79–94.
- CRIGHTON, D. G. & GASTER, M. 1976 Stability of slowly diverging jet flow. *J. Fluid Mech.* **77** (2), 387–413.
- CUVIER, C., SRINATH, S., STANISLAS, M., FOUCAUT, J. M., LAVAL, J. P., KÄHLER, C. J., HAIN, R., SCHARNOWSKI, S., SCHRÖDER, A., GEISLER, R., AGOCS, J., RÖSE, A., WILLERT, C., KLINNER, J., AMILI, O., ATKINSON, C. & SORIA, J. 2017 Extensive characterisation of a high Reynolds number decelerating boundary layer using advanced optical metrology. *J. Turbul.* **18** (10), 929–972.
- DOGAN, E., ÖRLÜ, R., GATTI, D., VINUESA, R. & SCHLATTER, P. 2019 Quantification of amplitude modulation in wall-bounded turbulence. *Fluid Dyn. Res.* **51** (1), 011408.
- EITEL-AMOR, G., ÖRLÜ, R. & SCHLATTER, P. 2014 Simulation and validation of a spatially evolving turbulent boundary layer up to $Re_\theta = 8300$. *Intl J. Heat Fluid Flow* **47**, 57–69.
- FARRELL, B. F. & IOANNOU, P. J. 1996 Generalized stability theory. Part I. Autonomous operators. *J. Atmos. Sci.* **53** (14), 2025–2040.
- FAVRE, A., GAVIGLIO, J. & DUMAS, R. 1967 Structure of velocity space–time correlations in a boundary layer. *Phys. Fluids* **10** (9), S138–S145.
- FLORES, O. & JIMÉNEZ, J. 2010 Hierarchy of minimal flow units in the logarithmic layer. *Phys. Fluids* **22** (7), 071704.
- GUALA, M., HOMMEMA, S. E. & ADRIAN, R. J. 2006 Large-scale and very-large-scale motions in turbulent pipe flow. *J. Fluid Mech.* **554**, 521–542.

- GUILLAUME, P., PINTELON, R. & SCHOUKENS, J. 1992 Nonparametric frequency response function estimators based on nonlinear averaging techniques. In *Instrumentation and Measurement Technology Conference, 1992. IMTC'92, 9th IEEE*, pp. 3–9. IEEE.
- HOYAS, S. & JIMÉNEZ, J. 2006 Scaling of the velocity fluctuations in turbulent channels up to $Re_\tau = 2003$. *Phys. Fluids* **18** (1), 011702.
- HUTCHINS, N. & MARUSIC, I. 2007 Evidence of very long meandering features in the logarithmic region of turbulent boundary layers. *J. Fluid Mech.* **579**, 1–28.
- HWANG, Y. & COSSU, C. 2010 Linear non-normal energy amplification of harmonic and stochastic forcing in the turbulent channel flow. *J. Fluid Mech.* **664**, 51–73.
- ILLINGWORTH, S. J., MONTY, J. P. & MARUSIC, I. 2018 Estimating large-scale structures in wall turbulence using linear models. *J. Fluid Mech.* **842**, 146–162.
- JIMÉNEZ, J. 2012 Cascades in wall-bounded turbulence. *Annu. Rev. Fluid Mech.* **44**, 27–45.
- JIMÉNEZ, J. 2013 Near-wall turbulence. *Phys. Fluids* **25** (10), 101302.
- JIMÉNEZ, J. & PINELLI, A. 1999 The autonomous cycle of near-wall turbulence. *J. Fluid Mech.* **389**, 335–359.
- KIM, K. C. & ADRIAN, R. J. 1999 Very large-scale motion in the outer layer. *Phys. Fluids* **11** (2), 417–422.
- KOMMINAHO, J., LUNDBLADH, A. & JOHANSSON, A. V. 1996 Very large structures in plane turbulent Couette flow. *J. Fluid Mech.* **320**, 259–285.
- LANG, Z. & BILLINGS, S. A. 1996 Output frequency characteristics of nonlinear systems. *Intl J. Control* **64** (6), 1049–1067.
- LUNDELL, F. 2007 Reactive control of transition induced by free-stream turbulence: an experimental demonstration. *J. Fluid Mech.* **585**, 41–71.
- MARUSIC, I. & HEUER, W. D. C. 2007 Reynolds number invariance of the structure inclination angle in wall turbulence. *Phys. Rev. Lett.* **99** (11), 114504.
- MARUSIC, I., MATHIS, R. & HUTCHINS, N. 2010 Predictive model for wall-bounded turbulent flow. *Science* **329** (5988), 193–196.
- MATHIS, R., HUTCHINS, N. & MARUSIC, I. 2009 Large-scale amplitude modulation of the small-scale structures in turbulent boundary layers. *J. Fluid Mech.* **628**, 311–337.
- MATHIS, R., HUTCHINS, N. & MARUSIC, I. 2011 A predictive inner–outer model for streamwise turbulence statistics in wall-bounded flows. *J. Fluid Mech.* **681**, 537–566.
- MCKEON, B. J. 2017 The engine behind (wall) turbulence: perspectives on scale interactions. *J. Fluid Mech.* **817**, P1.
- MCKEON, B. J. & SHARMA, A. S. 2010 A critical-layer framework for turbulent pipe flow. *J. Fluid Mech.* **658**, 336–382.
- MCKEON, B. J., SHARMA, A. S. & JACOBI, I. 2013 Experimental manipulation of wall turbulence: a systems approach. *Phys. Fluids* **25** (3), 031301.
- MOEHLIS, J., FAISST, H. & ECKHARDT, B. 2004 A low-dimensional model for turbulent shear flows. *New J. Phys.* **6** (1), 56.
- NAGUIB, A. M., WARK, C. E. & JUCKENHÖFEL, O. 2001 Stochastic estimation and flow sources associated with surface pressure events in a turbulent boundary layer. *Phys. Fluids* **13** (9), 2611–2626.
- ÖRLÜ, R. & SCHLATTER, P. 2011 On the fluctuating wall-shear stress in zero pressure-gradient turbulent boundary layer flows. *Phys. Fluids* **23** (2), 021704.
- PANTON, R. L. 2001 Overview of the self-sustaining mechanisms of wall turbulence. *Prog. Aerosp. Sci.* **37** (4), 341–383.
- PENG, Z. K., LANG, Z. Q. & BILLINGS, S. A. 2007 Crack detection using nonlinear output frequency response functions. *J. Sound Vib.* **301** (3–5), 777–788.
- PUJALS, G., GARCÍA-VILLALBA, M., COSSU, C. & DEPARDON, S. 2009 A note on optimal transient growth in turbulent channel flows. *Phys. Fluids* **21** (1), 015109.
- RICE, H. J. & FITZPATRICK, J. A. 1988 A generalised technique for spectral analysis of non-linear systems. *Mech. Syst. Signal Process.* **2** (2), 195–207.
- RICE, H. J. & FITZPATRICK, J. A. 1991 A procedure for the identification of linear and non-linear multi-degree-of-freedom systems. *J. Sound Vib.* **149** (3), 397–411.

- ROCKLIN, G. T., CROWLEY, J. & VOLD, H. 1985 A comparison of H_1 , H_2 , and H_v frequency response functions. In *Proceedings of the 3rd International Modal Analysis Conference*, pp. 272–278. Union College.
- SASAKI, K., MORRA, P., FABBIANE, N., CAVALIERI, A. V. G., HANIFI, A. & HENNINGSON, D. S. 2018a On the wave-cancelling nature of boundary layer flow control. *Theor. Comput. Fluid Mech.* **32** (5), 1–24.
- SASAKI, K., PIANTANIDA, S., CAVALIERI, A. V. G. & JORDAN, P. 2017 Real-time modelling of wavepackets in turbulent jets. *J. Fluid Mech.* **821**, 458–481.
- SASAKI, K., TISSOT, G., CAVALIERI, A. V. G., SILVESTRE, F. J., JORDAN, P. & BIAU, D. 2018b Closed-loop control of a free shear flow: a framework using the parabolized stability equations. *Theoret. Comput. Fluid Dyn.* **32** (6), 1–24.
- SCHLATTER, P. & ÖRLÜ, R. 2010 Assessment of direct numerical simulation data of turbulent boundary layers. *J. Fluid Mech.* **659**, 116–126.
- SCHLATTER, P. & ÖRLÜ, R. 2012 Turbulent boundary layers at moderate Reynolds numbers: inflow length and tripping effects. *J. Fluid Mech.* **710**, 5–34.
- SCHLATTER, P., ÖRLÜ, R., LI, Q., BRETHOUWER, G., FRANSSON, J. H. M., JOHANSSON, A. V., ALFREDSSON, P. H. & HENNINGSON, D. S. 2009 Turbulent boundary layers up to $Re_\theta = 2500$ studied through simulation and experiment. *Phys. Fluids* **21** (5), 051702.
- SCHMID, P. J. & HENNINGSON, D. S. 2012 *Stability and Transition in Shear Flows*. Springer Science & Business Media.
- SCHOUKENS, J., ROLAIN, Y. & PINTELO, R. 1997 Improved frequency response function measurements for random noise excitations. In *Instrumentation and Measurement Technology Conference*, pp. 749–753. IEEE.
- SCHRAUF, G. 2005 Status and perspectives of laminar flow. *Aeronaut. J.* **109** (1102), 639–644.
- SMITS, A. J., MCKEON, B. J. & MARUSIC, I. 2011 High-Reynolds number wall turbulence. *Annu. Rev. Fluid Mech.* **43** (1), 353–375.
- TOWNE, A., SCHMIDT, O. T. & COLONIUS, T. 2018 Spectral proper orthogonal decomposition and its relationship to dynamic mode decomposition and resolvent analysis. *J. Fluid Mech.* **847**, 821–867.
- TREFETHEN, L. N., TREFETHEN, A. E., REDDY, S. C. & DRISCOLL, T. A. 1993 Hydrodynamic stability without eigenvalues. *Science* **261** (5121), 578–584.
- VINUESA, R., HITES, M. H., WARK, C. E. & NAGIB, H. M. 2015 Documentation of the role of large-scale structures in the bursting process in turbulent boundary layers. *Phys. Fluids* **27** (10), 105107.
- VINUESA, R. & ÖRLÜ, R. 2017 Measurement of wall shear stress. In *Experimental Aerodynamics* (ed. S. Discetti & A. Ianiro), pp. 393–428. CRC Taylor Press.
- WALEFFE, F. 1995 Hydrodynamic stability and turbulence: beyond transients to a self-sustaining process. *Stud. Appl. Math.* **95** (3), 319–343.
- WALEFFE, F. 1997 On a self-sustaining process in shear flows. *Phys. Fluids* **9** (4), 883–900.
- WARK, C. E. & NAGIB, H. M. 1991 Experimental investigation of coherent structures in turbulent boundary layers. *J. Fluid Mech.* **230**, 183–208.
- WELCH, P. 1967 The use of fast Fourier transform for the estimation of power spectra: a method based on time averaging over short, modified periodograms. *IEEE Trans. Audio Electroacoust.* **15** (2), 70–73.