



A hybrid artificial intelligence control of a turbulent jet: Reynolds number effect and scaling

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This work aims to investigate experimentally the effect of the Reynolds number Re , based on the nozzle diameter D , on jet mixing manipulation using an unsteady radial minijet. A novel artificial intelligence (AI) control system has been developed to manipulate the jet over $Re = 5800\text{--}40\,000$. The system may optimize simultaneously the control law and a time-independent parameter, which dictate the actuation ON/OFF states and amplitude, respectively. The control parameters include the mass flow rate, excitation frequency and diameter ratios (C_m , f_e/f_0 and d/D) of the minijet to the main jet as well as the duty cycle (α) of minijet injection. Jet mixing is quantified using K_e and K_0 , where K is the decay rate of the jet centreline mean velocity, and subscripts e and 0 denote the manipulated and unforced jets, respectively. It has been found that the maximum K_e achievable does not vary with Re . Scaling analysis of the huge volume of experimental data obtained from the AI system reveals that the relationship $K_e = g_1(C_m, f_e/f_0, \alpha, d/D, Re, K_0)$ may be reduced to $K_e/K_0 = g_2(\zeta)$, where g_1 and g_2 are different functions and the scaling factor $\zeta = (C_m/\alpha)(D/d)^{1-n}(1/Re)(f_e/f_{e,opt})^m$, m and $1-n$ are the power indices, and subscript opt denotes the value at which K_e is maximum. The scaling law is discussed in detail, along with the physical meanings of the dimensionless parameters K_e/K_0 , ζ , $(C_m/\alpha)(D/d)^{1-n}(1/Re)$ and $(f_e/f_{e,opt})^m$.

Key words: mixing enhancement, control theory, jets, machine learning

1. Introduction

As one of classical shear flows, the turbulent jet has been extensively studied in the past due to its vast range of applications in, e.g. aero and automobile engines, combustion, heat transfer and chemical reactors. Therefore, it is of both fundamental and practical significance to investigate how to manipulate a turbulent jet to achieve the desired

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entrainment and mixing. Recent review articles by Reynolds *et al.* (2003), Knowles & Saddington (2006) and Henderson (2010) provide excellent compendiums for published papers on jet control.

Jet control can be passive and active. The former requires no power input, such as changing the geometric shape of nozzles (Mi & Nathan 2010), which is efficient and reliable. The latter, such as speaker- or plasma-based and fluidic control (e.g. Arbey & Williams 1984; Samimy *et al.* 2007; Yang & Zhou 2016), needs additional power input but may achieve more flexible and drastic flow modifications than the former (Perumal & Rathakrishnan 2022). Active methods are further divided into open- and closed-loop control, depending on whether sensor signals are used to drive actuation. Closed-loop control provides an opportunity to achieve a better performance than open-loop techniques. As such, various closed-loop control schemes have been developed for diversified applications, such as suppressing vortex shedding in wakes (Zhang, Cheng & Zhou 2004; Beaudoin *et al.* 2006; Pastoor *et al.* 2008), flow separation control (Gautier & Aider 2013) and jet mixing enhancement (Wu *et al.* 2016). Based on the time scale of the control loop and whether a model is required in a closed-loop system, Brunton & Noack (2015) classified the most used closed-loop systems into three categories: (i) stabilizing laminar flow, (ii) adaptive control of turbulence and (iii) model-free tuning of control laws. Category (i) is an in-time closed-loop control, usually based on white-box, grey-box or black-box model (Kim & Bewley 2007), where the ‘in-time’ means the time scale of the actuation response is much smaller than that of a natural flow. The white-box model, such as the direct solutions to Navier–Stokes equations, may resolve all flow features, while the grey-box model, e.g. the proper orthogonal decomposition model, captures merely limited flow features. Some successful applications of using a reduced-order model for the in-time control of fluid systems can be found in Noack, Morzynski & Tadmor (2011). The black-box model (e.g. transfer functions) only represents the input–output relationship. Category (ii) is based on the steady-state system response, and can be used only to optimize the given design of control laws in a slow manner as compared with the time scale of physical processes (Brunton & Noack 2015), e.g. extremum-seeking and slope-seeking control (Beaudoin *et al.* 2006). Category (iii) may realize rapid in-time control but needs a thorough understanding of the flow physics; examples include the proportional–integral–derivative (Zhang *et al.* 2004) and opposition control (Choi, Moin & Kim 1994) methods. However, a fluid system such as turbulent jet mixing enhancement involves a large range of temporal and spatial scales with complex nonlinear interactions. This makes model-based control difficult to implement. These challenges motivate the search for model-free control laws using machine learning methods, such as artificial neural networks and genetic programming (GP).

GP was developed in the 1990s and was rediscovered in fluid mechanics as machine learning control (MLC) by Gautier *et al.* (2015). Recently, the MLC method has been applied in numerous experiments and numerical simulations for control law optimization (Noack 2018). For example, MLC shows an extraordinary ability to find the optimal control law when deployed to enhance turbulent jet mixing using six independent unsteady minijets (Zhou *et al.* 2020). In the experimental systems of MLC developed so far, however, the discovered control law either regulates slowly, often less than 2 Hz, the actuation strength (Gautier *et al.* 2015) or determines rapidly only the ON/OFF states of actuators, with the actuation strength or amplitude and the Reynolds number Re fixed (e.g. Li *et al.* 2017; Wu *et al.* 2018a). Given varying flow conditions such as changing Re , the actuation strength may need to be optimized as well in order to obtain the best control performance. Then one issue naturally arises. Can we develop an MLC or artificial

intelligence (AI) control system that may optimize both actuation states and strength simultaneously? Furthermore, an AI control system may produce hundreds or thousands of control laws (e.g. Zhou *et al.* 2020) and accordingly a huge amount of data where all the control variables, along with the cost function or control goal, are physically connected. Can we extract an in-depth physical insight into the underlying control mechanism and inter-connection between the control variables based on a careful analysis of the data?

The effect of Re on an unforced jet has been investigated by e.g. Ricou & Spalding (1961), Panchapakesan & Lumley (1993), Hussein, Capp & George (1994) and Mi, Xu & Zhou (2013). These investigations unveiled that the centreline velocity, jet decay, ratio of mass entrainment to the mass flow rate at the jet exit and mean velocity profiles are independent of Re for $10\,000 \leq Re \leq 20\,000$. There have also been several studies on the Re effect of jet control. For example, Parekh, Leonard & Reynolds (1988) studied a jet controlled by an axial acoustic actuator at one end of the upstream plenum chamber and 4 equidistantly placed radial acoustic actuators around the nozzle exit with phases of 0° , 90° , 180° and 270° at Re from 10^4 to 10^5 . The bifurcating jet could be produced at a frequency ratio between the axial and radial excitations of 2, and the excitation amplitude required to produce bifurcating jets was found to increase with Re . Wu, Wong & Zhou (2018b) manipulated a turbulent round jet based on single-frequency radial fluidic injection using a dual-input–single-output extremum-seeking technique. With the actuation frequency and mass flow rate ratio optimized simultaneously, the maximum jet entrainment/mixing rate was found to be unchanged from $Re = 5700$ to 13 300. The same conclusion was reached when this technique was further extended to three control parameters, with the duty cycle α of an unsteady minijet simultaneously optimized, by Fan *et al.* (2020). However, there is a lack of information on the dependence on Re of the inter-relationship between the jet entrainment or mixing rate and the control parameters. The possible important impact of Re on the control performance has yet to be thoroughly documented in the literature.

Unsteady fluidic injection may have a number of control parameters, including the mass flow rate, excitation frequency and diameter ratios, $C_m, f_e/f_0$ and d/D , of minijet to main jet as well as the duty cycle α of minijet injection, where f_0 is the preferred-mode frequency of an unforced main jet. Perumal & Zhou (2018) conducted an empirical scaling analysis on a jet manipulated by single unsteady minijet at $Re = 8000$. Their study reveals that the jet centreline mean velocity decay rate $K_e = f_1(C_m, d/D, \alpha)$ may be reduced to $K_e = f_2(\xi)$ and the scaling factor $\xi = (\sqrt{MR}/\alpha)(d/D)^n$, where f_1 and f_2 are different functions, $\sqrt{MR} = C_m(D/d)$ is physically the effective momentum ratio per pulse and n is a function of α . Here MR is the momentum ratio of the minijet to the main jet. However, Re and f_e/f_0 have not been considered in this scaling law. A variation in Re may influence jet mixing (e.g. New, Lim & Luo 2006; Mi *et al.* 2013) and hence the centreline mean velocity decay rate K_0 for an unforced jet. As such, under manipulation, K_e may also depend on K_0 given a varying Re . One important question naturally arises: Can we find a physically meaningful scaling factor ζ so that $K_e = g_1(C_m, f_e/f_0, \alpha, d/D, Re, K_0)$ may be reduced to $K_e = g_2(\zeta)$? Such a scaling factor ζ is of great significance, and may be used to design unsteady fluidic injectors for full-scale practical applications (Wickersham 2007).

This work aims to address the issues raised above. An extended AI control system, termed a hybrid AI control system hereinafter, is developed to manipulate a turbulent jet based on a pulsed minijet, with a view to maximizing its mixing. The system is capable of searching simultaneously a near-optimal control law and a time-independent parameter in spite of a variation in Re . The Re effect on control performance and the optimal control parameters are investigated over $Re = 5800$ – $40\,000$, covering the regimes of $Re < Re_{cr}$ and $Re \geq Re_{cr}$ to document the effect of Re on jet mixing, where Re_{cr} is the critical Reynolds

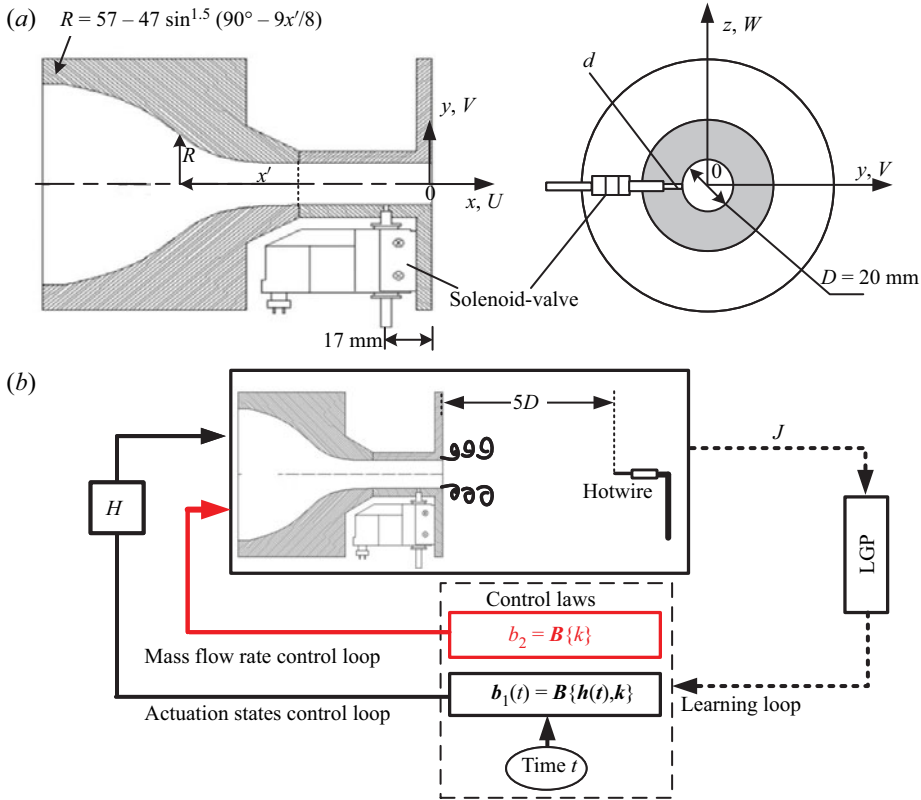


Figure 1. (a) Schematic of experimental set-up; (b) sketch of principle of the hybrid AI system.

number, $\sim 10\,000$, at which turbulent ‘mixing transition’ starts (Mi *et al.* 2013). A scaling analysis is then performed based on the massive data produced from the AI system. The manuscript is organized as follows. Section 2 describes the experimental set-up. Section 3 introduces the hybrid AI system. The Re effect on control performance is presented in § 4. A scaling law is extracted from experimental data in § 5, along with a detailed discussion of involved dimensionless parameters. This work is concluded in § 6.

2. Experimental details and unforced jet

2.1. Jet facility and actuator system

The jet rig to produce an axisymmetric main jet, along with the assembly to generate a pulsed minijet, is the same as used in Wu *et al.* (2018a) and is briefly introduced here. As shown in figure 1(a), the nozzle is extended with a 47 mm long smooth tube with the same inner diameter $D = 20$ mm as the nozzle exit. A radial pinhole of diameter d is made 17 mm upstream of the main jet exit. Two pinhole diameters, i.e. 0.5 mm and 1.0 mm, are chosen, resulting in $d/D = 1/20$ and $1/40$, respectively. Seven different Reynolds numbers $Re \equiv \bar{U}_j D / \nu$, varying from 5800 to 40 000, are examined, where U_j is the velocity measured at the centre of the jet exit ($x^* = 0.05$), the overbar denotes time averaging and ν is the kinematic viscosity of air. In this paper, an asterisk superscript denotes normalization by D or/and \bar{U}_j . The coordinate system (x, y, z) is defined in figure 1(a), with its origin at the centre of the jet exit.

The mass flow rate of the minijet through the pinhole is measured using a mass flow controller (FLOWMETHOD FL-802) with a measurement range of 0–7 l min⁻¹, whose experimental uncertainty is no more than 1 %. The duty cycle and frequency of the minijet injection are controlled using an electromagnetic valve that is operated on an ON/OFF mode with a maximum operating frequency of 500 Hz.

2.2. Flow measurement and control instruments

The instantaneous velocity in the jet is measured using a single hot-wire. The sensor is a 1 mm long tungsten wire of 5 μm diameter operated, at an overheat ratio of 1.8, on a constant temperature mode (Dantec Streamline). The hot-wire is mounted on a three-dimensional traverse system to measure the jet centreline velocities, U_j and $U_{5D,0}$, at the jet exit and 5D downstream, respectively, without control and also $U_{5D,e}$ under control. The output signal is offset, amplified and filtered by a low-pass filter at a cutoff frequency of 500 Hz before being digitized at a sampling frequency F_{RT} of 10 kHz. The hot-wire is calibrated at the jet exit using a Pitot tube and a micromanometer (Furness Controls FCO510). The experimental uncertainty in measured \bar{U}_j , $\bar{U}_{5D,e}$ or $\bar{U}_{5D,0}$ is estimated to be within 2 %.

The real-time control is implemented via a National Instrument PXI system, which consists of a chassis, a multi-function I/O Device (PXIe-6356) and a controller (PXIe-8821). A LabVIEW Real-Time module is used for digitizing the analogue signal and providing control commands for the mass flow controller and electromagnetic valve at a loop time of 100 μs ($F_{RT} = 10$ kHz). The same sampling rate is used for the velocity data acquisition and control command generation. As discussed in Wu et al (2018a), the available duty cycles α are determined by F_{RT} and periodic excitation frequencies f_e . For instance, there are 48 possible α values available for use given $F_{RT} = 10$ kHz and $f_e = 200$ Hz.

2.3. Unforced jet characteristics

In the absence of control, \bar{U}^* exhibits a ‘top-hat’ profile (figure 2a) and the corresponding root-mean-squared (r.m.s.) velocity u_{rms}^* is less than 0.06 (figure 2b) at $Re = 20\ 000$. Both \bar{U}^* and u_{rms}^* collapse reasonably well with Mi et al.’s (2013) data at $Re = 20\ 100$. The displacement thickness δ and momentum thickness θ estimated from the velocity profile are 0.57 mm and 0.21 mm, respectively, at the nozzle exit for $Re = 8000$ (Perumal & Zhou 2021), which are very close to their counterparts (0.56 mm and 0.24 mm) measured at approximately the same Re (=8050) by Mi et al (2013). The shape factor $H = \delta/\theta$ is 2.71, slightly higher than the Blasius flat plate value ($H = 2.59$), suggesting that the boundary layer at the nozzle exit is close to laminar. With increasing Re , both δ and θ decrease, as noted by Mi et al. (2013), and H drops to 2.49 and 2.41 for $Re = 13\ 300$ and 20 000, respectively, suggesting a turbulent boundary layer at the nozzle exit. The streamwise variations of \bar{U}^* and u_{rms}^* also agree with each other between the two studies (figure 2c,d).

The flow visualization images for an unforced jet at various Re are presented in figure 3. At $Re = 5800$ and 8000 (figure 3a,b), the quasi-periodical ring vortices exhibit laminar features during the initial rollup process and then gradually transition to turbulence. Once Re exceeds 10 000 (figure 3c,d), the vortices appear to be turbulent from the beginning and the jet spreads out more rapidly than $Re < 10\ 000$. This is also supported by the streamwise fluctuating velocity signals U^* measured along the jet axis for the natural jet at $Re = 8000$ –20 000 (figure 4). The flow is clearly laminar near the nozzle exit ($x^* \leq 3.0$) at $Re = 8000$ but displays random fluctuations, a feature of turbulence, starting from $Re = 10$

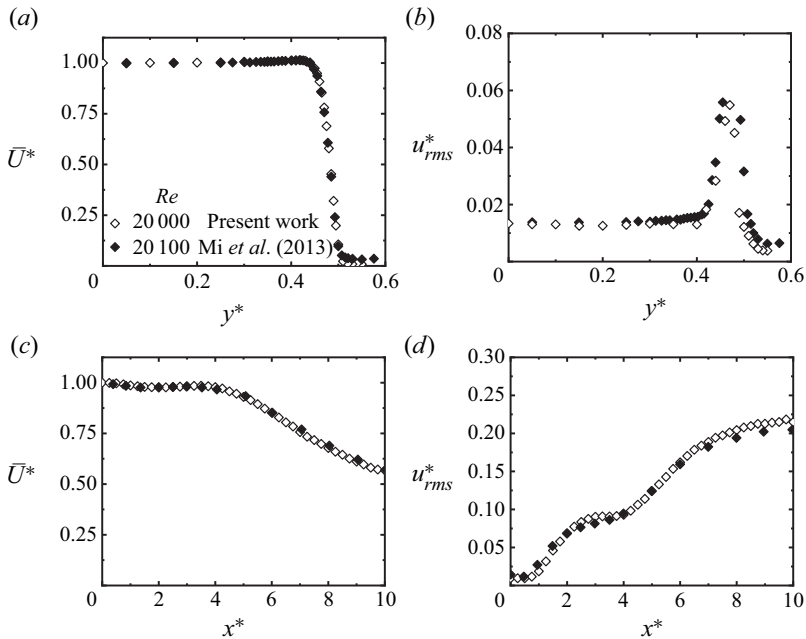


Figure 2. (a,b) Radial distributions of \bar{U}^* and u_{rms}^* measured at $x^*=0.05$ in the $(x-y)$ plane and (c,d) streamwise variations of centreline mean and r.m.s. velocities for $Re=20\,000$. Mi *et al.* (2013) is included for comparison.

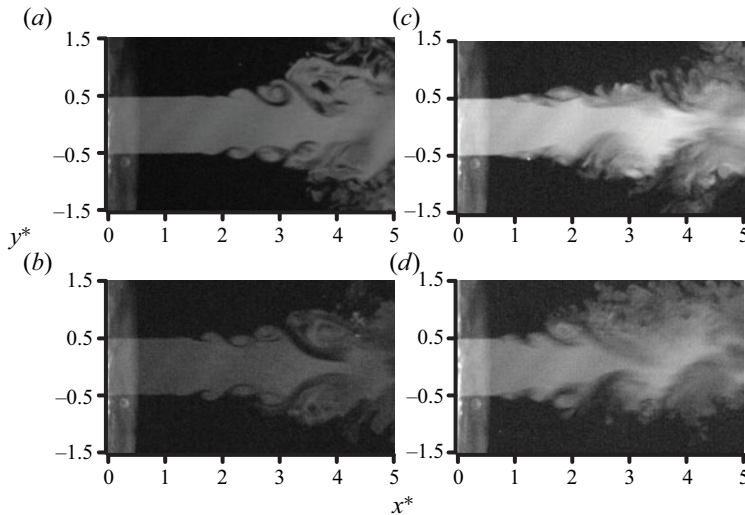


Figure 3. Flow visualization of unforced jet in the $(x-y)$ plane for (a) $Re=5800$, (b) 8000 , (c) $10\,600$ and (d) $13\,300$.

600 and becoming more evident at $Re=20\,000$. Mi *et al.*'s (2013) extensive hot-wire measurements over $Re=4000-20\,000$ indicated that the mean flow decay rate and spread vary with Re given $Re < 10\,000$ and tend to become Re -independent for $Re > 10\,000$. Furthermore, the small-scale turbulence properties, such as the mean dissipation rate of kinetic energy and the Kolmogorov and Taylor microscales, vary differently between the two Re ranges. It seems plausible from both the present and Mi *et al.*'s measurements

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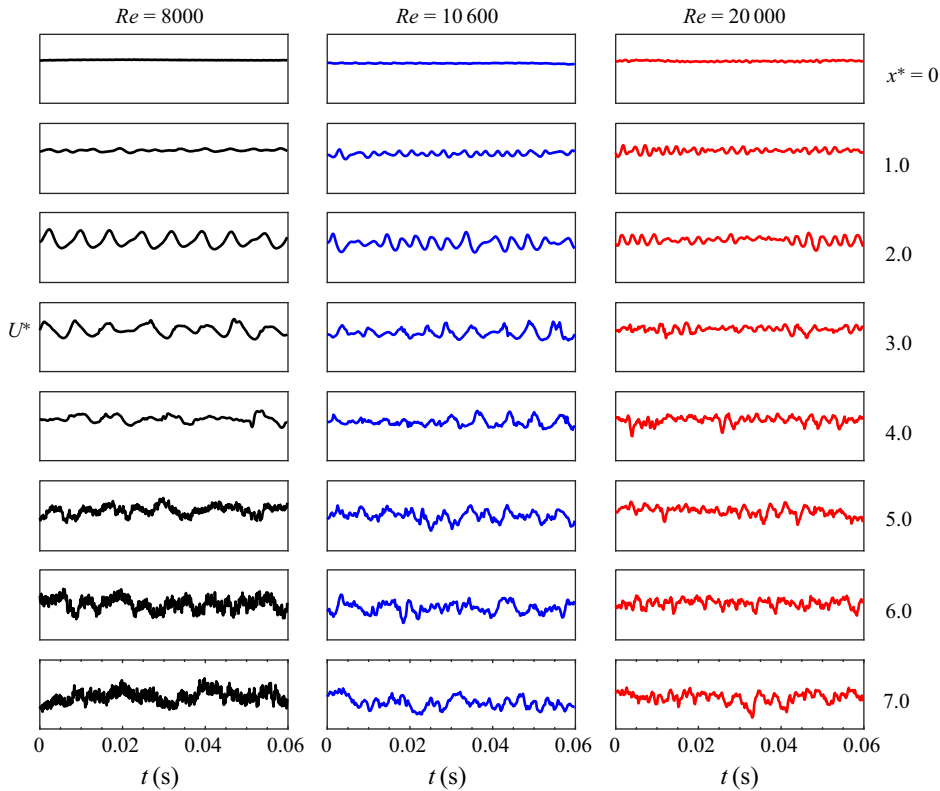


Figure 4. Typical signals of instantaneous streamwise U^* measured along jet centreline (y^* and $z^* = 0$) for the natural jet at $Re = 8000$ – $20\,000$. The same scale is applied for all signals.

that $Re = 10\,000$ is a critical Reynolds number across which the jet turbulence behaves distinctly.

The power spectral density functions of streamwise velocity U measured along the centreline for $x^* = 2$ – 6 show a pronounced peak at $f_0 = 100$ – 680 Hz for $Re = 5800$ – $40\,000$, as shown by Fan *et al.* (2017) for $Re = 8000$ – $16\,000$, suggesting the occurrence of the preferred-mode structures. The corresponding Strouhal number $St(\equiv f_0 D / \bar{U}_j)$ varies between 0.45 and 0.50, falling in the range of 0.24–0.64 for a natural jet (Crow & Champagne 1971). The unforced jet develops slowly at $x^* \leq 5$, and the Re -related variation of jet centreline mean velocity is small, within 2.5%, for $4050 \leq Re \leq 20\,100$ (Mi *et al.* 2013).

2.4. Evaluation of jet decay rate and mixing

Following Zhou *et al.* (2012) and Perumal & Zhou (2018), the decay rate K_e of the jet centreline mean velocity is used to evaluate jet entrainment rate, defined by

$$K_e = \frac{\bar{U}_j - \bar{U}_{5D,e}}{\bar{U}_j}, \quad (2.1)$$

where \bar{U}_{5D} and \bar{U}_j denote the jet centreline mean velocities at $x^* = 5$ and $x^* \approx 0$, respectively. This quantity may also provide a measure for the mixing efficacy of the jet based on the following considerations. Firstly, as documented by Fan *et al.* (2017), the

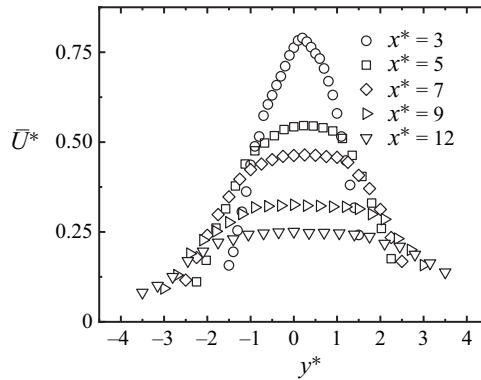


Figure 5. Normalized streamwise mean velocities measured along the y axis at different axial locations for $C_m = 5.0\%$, $f_e/f_0 = 0.5$, $\alpha = 0.1$ and $d/D = 1/20$ at $Re = 13\,300$.

difference ΔK between the K_e values with and without control reaches a maximum at $x^* \approx 5$ (their figure 7), that is, K_e estimated based on the streamwise velocity measured at $x^* = 5$ is most sensitive to control. Secondly, K_e is correlated with both the jet half-width and potential core length. Zhou *et al.* (2012) demonstrated that K_e is related approximately linearly to an equivalent jet half-width $Re_q = [R_H R_V]^{1/2}$, where R_H and R_V are the jet half-widths in two orthogonal planes, that is, K_e is directly connected to the entrainment rate of the manipulated jet. Furthermore, an increase in K_e corresponds to a decrease in the potential core length of the jet (Perumal & Zhou 2018), implying that K_e may provide a measure for jet mixing. Finally, this one-point criterion for jet mixing has also been used by other researchers (e.g. Breidenthal *et al.* 1985; Wickersham 2007). Breidenthal *et al.* (1985) used an aspirating probe to measure the concentration fluctuation c of two mixing streams, i.e. a rectangular duct flow and a transverse jet, and noted that the decay rate of the r.m.s. value c_{rms} of c was almost linearly related to the momentum ratio of the transverse jet to the rectangular duct flow. This relationship persisted over $x^* = 1 \sim 10$ (see their figure 5). As such, they used the single decay rate of c_{rms} , measured at $x^* = 8.9$, to characterize the mixing of the two streams and argued that this decay rate provided a measure for the mixing efficacy of turbulence.

The main jet may deflect when manipulated asymmetrically using a single minijet. One question naturally arises as to whether the jet decay rate may be estimated correctly from the centreline mean velocities. Perumal & Zhou (2018) manipulated a round jet at $Re = 8000$ using a single minijet injection and presented the normalized mean velocity profiles in two orthogonal planes (see their figure 6) for the manipulated jet at $C_m = 1.0\%$, $f_e/f_0 = 0.5$, $\alpha = 0.1$ and $d/D = 1/20$. The manipulated jet exhibited a reasonable symmetry about the geometric centre ($y^* = 0$ or $z^* = 0$), showing a slight deviation in the location of the maximum velocity from this centre and causing an error of no more than 1% in K_e . This deviation apparently depends on C_m which is presently up to 5.0%. As such, figure 5 presents the normalized streamwise mean velocities measured along the injection direction (y axis) at different axial locations for $C_m = 5.0\%$, $f_e/f_0 = 0.5$, $\alpha = 0.1$ and $d/D = 1/20$ ($Re = 13\,300$). Evidently, the difference between the maximum velocity and the velocity at the geometric centre is very small, resulting in an error of no more than 1% in K_e at $x^* = 3$. This difference diminishes with increasing x^* . It may be concluded that the effect of the jet deflection, due to a single minijet injection, is negligibly small in the present experimental conditions. Note that this difference may grow appreciably for a quasi-steady control, i.e. at large α and f_e/f_0 , when the jet column deflects away from the geometric centreline, producing an increased artificial mixing (Perumal & Zhou 2018).

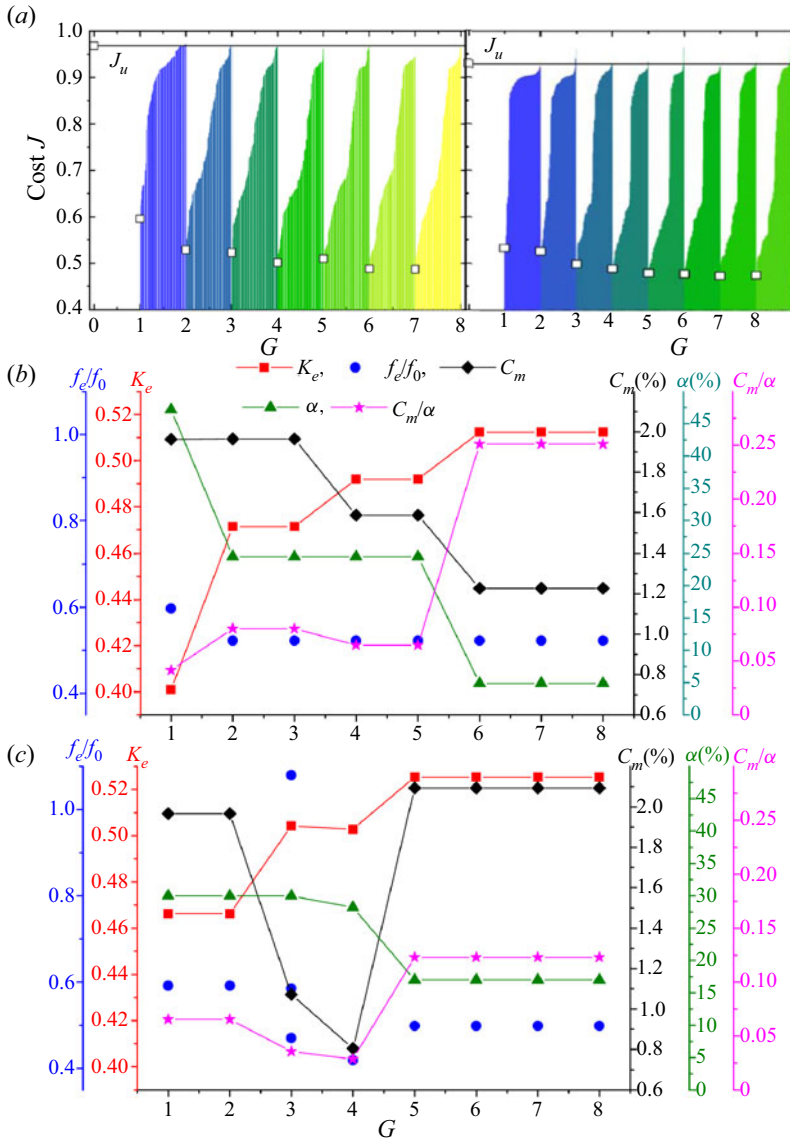


Figure 6. (a) Learning curves of AI control (left: $Re = 8000$; right: $Re = 20000$). Here, J_u is the cost value corresponding to the benchmark of an unforced jet. The open square symbol denotes the best individual for each generation. (b,c) Evolution of the control parameters, associated with the best individual in each generation, and the corresponding K_e : (b) $Re = 8000$, (c) 20000 .

3. Controller design: hybrid AI system

Following our previous work (Wu *et al.* 2018a; Zhou *et al.* 2020), the cost function J is defined as $\bar{U}_{5D,e}$ normalized with \bar{U}_j

$$J = \frac{\bar{U}_{5D,e}}{\bar{U}_j} = 1 - K_e. \quad (3.1)$$

Apparently, minimizing J is equivalent to maximizing K_e .

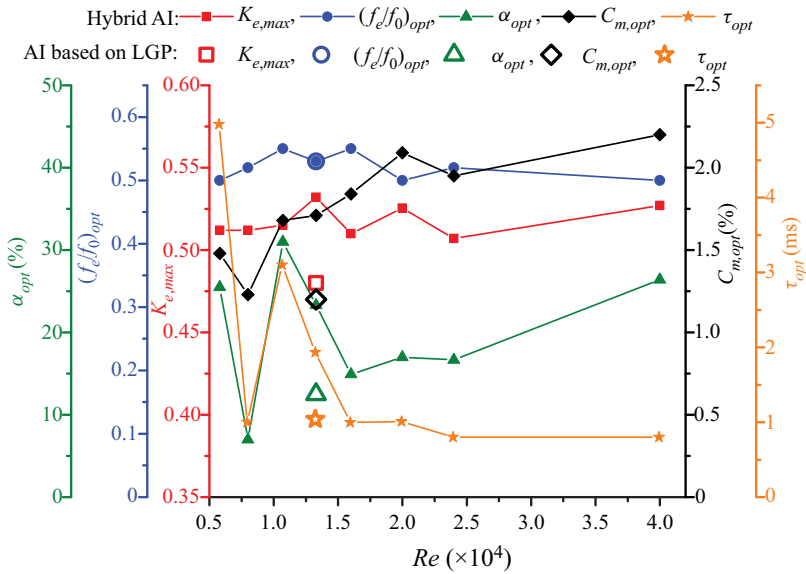


Figure 7. The optimum control parameters obtained at different Re from AI control. The open symbols correspond to the data obtained at a fixed $C_m = 1.2\%$.

The major difference between the presently developed hybrid AI and previously reported AI/MLC systems (e.g. Parezanović *et al.* 2015; Li *et al.* 2017; Wu *et al.* 2018a; Zhou *et al.* 2020) is that the latter searches only for the control laws, while the former optimizes an independent control parameter (C_m), along with the control laws. The GP-based systems, developed by Wu *et al.* (2018a) and Zhou *et al.* (2020), may work on a multi-frequency forcing mode and/or a sensor-based feedback mode. The former, characterized by α , f_e and a number of distinct frequencies in the control law, is found to overwhelm the latter in performance. As such, only is the former presently examined, and the control law assumes following form:

$$b(t) = B(h(t), k), \tag{3.2}$$

where B is a vector consisting of functions generating actuation commands, argument k is a set of random constants and $h(t)$ is a set of input harmonic functions of time t , viz.

$$h(t) = [h_1 h_2 \dots h_{11}]^T, \quad h_i = \cos(2\pi f_i t), \quad i = 1, 2, \dots, 11. \tag{3.3}$$

The choice of eleven harmonic functions follows Wu *et al.* (2018a). Here, B can be linear, quadratic or any other nonlinear function, and a large range of frequencies can be generated in the control signals. The control law in previously reported AI systems yields in general a time-variant binary signal for multi-frequency forcing (e.g. Li *et al.* 2017; Wu *et al.* 2018a; Zhou *et al.* 2020), with a fixed actuation strength (C_m). Yet, it may be desirable to optimize, other than the control law, a time-independent parameter, say C_m , to maintain the best control performance with varying flow conditions, such as Re . Note that a genetic algorithm (GA) works only on a population of constants and is thus suited for optimizing the parameters, such as C_m , of a preset open-loop control law (e.g. Qiao *et al.* 2021; Yu *et al.* 2022), while GP works on a population of hierarchical computer programs of varying sizes and shapes (Koza 1990), which may cover the open-loop control laws of different forms (such as single- or multi-frequency forcing laws) or the function of real-time feedback/feedforward sensor signals in a closed-loop system. As such, the latter

is characterized by a larger search space than GA and often outperforms the former, as illustrated by Li *et al.* (2017) for reducing the drag of a car model. We may modify GP by replacing the time-dependent functions with time-independent functions in order to optimize the time-independent parameters, as a GA does. Combining this modified GP with a ‘conventional’ GP forms a so-called hybrid AI system, which may optimize not only the parameters of a preset open-loop control law but also the control laws, be they open or closed loop.

Figure 1(b) presents a schematic for the hybrid AI control, where the control law $\mathbf{b}(t)$ contains two independent actuation commands, i.e. time-dependent binary signal $\mathbf{b}_1(t)$ and time-independent parameter b_2 , viz.

$$\mathbf{b}(t) = [\mathbf{b}_1(t), b_2]^T = [\mathbf{B}(\mathbf{h}(t), \mathbf{k}), \mathbf{B}(\mathbf{k})]^T. \quad (3.4)$$

Control law $\mathbf{b}_1(t)$ is in the form of a Heaviside function H to drive the electro-magnetic valve, producing and varying multiple excitation frequencies (f_e) and duty cycle (α), while b_2 is a function of 17 random constants ranging from -1 to 1 , and dictates C_m . The ensuing actuation produces a pulsed minijet whose maximum velocity is proportional to C_m and $1/\alpha$ before the minijet is choked (Perumal & Zhou 2018).

The optimization process is the same as in Wu *et al.* (2018a) and Zhou *et al.* (2020). Briefly, linear genetic programming (LGP) acts as a regression solver to search for a law in the form of (3.4) that minimizes the cost function J involving four major components: (i) creating a population of $N_i = 100$ control laws, (ii) evaluating the performance of each control law, (iii) checking whether the minimum J is converged and (iv) evolving to the next generation where 100 new control laws are generated based on the performance of the previous generation. Please refer to Li *et al.* (2017) for a detailed description of LGP.

4. Reynolds number effect on control performance

The robustness of the hybrid AI control is examined by performing experiments with Re varying from 5800 to 40 000. Figure 6(a) shows the learning curves of the AI control at $Re = 8000$ and 20 000, where a colour bar consists of the costs J corresponding to the 100 control laws of each generation. The square symbol marks the first and best cost of this generation and the remaining costs grow monotonically with their indices. The curve formed by the square symbols unveils the best performance from generation 1 to 7. The parameters associated with the best control law in each generation vary with increasing generation number G ; so does the corresponding K_e (figure 6b). The f_e/f_0 of the best performing individual converges to 0.52 at $G = 2$. The effective excitation at the subharmonic of f_0 has been previously reported by e.g. Freund & Moin (2000) and Wu *et al.* (2018a). With G increasing from 1 to 6, C_m and α gradually decline from 2.0% and 50% to 1.2% and 7%, respectively, the resulting K_e grows from 0.40 to 0.51. Interestingly, the ratio C_m/α correlates well with K_e ; in general, a larger C_m/α leads to a higher K_e . The converged f_e/f_0 , α and C_m are almost identical to those found from the conventional GP-based AI control (Wu *et al.* 2018a) where the optimal C_m is predetermined from conventional open-loop control with f_e/f_0 and α fixed at 0.5 and 0.15, respectively. This comparison demonstrates that the present hybrid AI system is capable of optimizing simultaneously C_m and the multi-frequency forcing law in the context of a more complex control landscape.

At other Re values, the learning curve is qualitatively the same as shown in figure 6(a). Nevertheless, the parameters associated with the best control law in each generation may exhibit different variations. The best control law of each generation is always associated

with a single f_e/f_0 , approximately 0.52, at $Re < 10\,000$ but may be characterized by multiple frequencies, as seen from the power spectra (not shown) of the control signals, at $Re > 10\,000$, e.g. at $G = 3$ in figure 6(c), which presents the evolution of the control parameters associated with the best individual in each generation, and the corresponding K_e . Nevertheless, the multiple frequencies always reduce to a single frequency $f_e/f_0 \approx 0.5$ once the learning process is converged.

Figure 7 presents the dependence on Re of the maximum K_e , i.e. $K_{e,max}$, along with the corresponding control parameters under the optimal control law. Interestingly, $K_{e,max}$ is found to be approximately constant, at between 0.51 and 0.53, for the Re range examined, that is, the best control performance is essentially unchanged and so is the optimal f_e/f_0 or $(f_e/f_0)_{opt}$ (≈ 0.5). This result reconfirms Wu *et al.*'s (2018b) finding from an adaptive control technique over $Re = 5800\text{--}13\,300$. However, the other control parameters exhibit a significant variation with Re . The optimum C_m or $C_{m,opt}$, the optimum duty cycle α_{opt} and the optimum pulse width τ_{opt} ($= \alpha_{opt} f_{e,opt}$) of minijet injection decrease from 1.5%, 25% and 5.0 ms to 1.2%, 7% and 1.0 ms, respectively, from $Re = 5800$ to 8000 ($Re < Re_{cr}$). The value of $C_{m,opt}$ rises gradually from 1.2% to 2.2% from $Re = 8000$ to 40 000, suggesting a larger minijet momentum to maintain the penetration depth or the best control performance at a higher Re , which complies with our instinct. The observed distinct behaviours of the control parameters between the Re ranges of 5800–8000 and 10 000–40 000 may have a link to the occurrence of the jet transition from laminar to turbulence in the range of $Re = 8000\text{--}10\,000$ (Mi *et al.* 2013). As expected, the variations in α_{opt} and τ_{opt} are similar to each other. Given a fixed C_m , a smaller α or pulse width τ of minijet injection should lead to a higher injection velocity or momentum per pulse, thus catering the need for a higher momentum at larger Re . For the presently used electromagnetic valve, the nominal smallest τ achievable is 0.8 ms. This may explain why $\tau_{opt} \approx 0.8$ ms at $Re \geq 16\,000$. When τ_{opt} reaches its minimum for $Re = 24\,000$, a further increase in Re will lead to a larger α_{opt} as f_0 and hence $f_{e,opt}$ rises with Re . Nevertheless, the inability to reduce τ further is compensated by the increased C_m so that $K_{e,max}$ does not fall appreciably. There is a considerable increase in τ_{opt} , reaching approximately 3.1 ms, from $Re = 8000$ to 10 700, but there is accordingly an appreciable rise in C_m that acts to maintain the penetration depth and hence $K_{e,max}$. The observation points to that the hybrid AI control may find an optimum combination of f_e/f_0 , α and C_m to achieve $K_{e,max}$ for given Re .

For the purpose of comparison, figure 7 also includes the optimal parameters $(f_e/f_0)_{opt}$ (≈ 0.5), α_{opt} ($= 12.5\%$) and τ_{opt} (≈ 1.0 ms) obtained at $Re = 13\,300$ from the same experimental rig using a conventional AI control, with C_m fixed at 1.2%, as developed by Wu *et al.* (2018a). Their $(f_e/f_0)_{opt}$ is the same but α_{opt} and τ_{opt} are smaller than the present results; however, their $K_{e,max}$ ($= 0.48$) is 10% lower than the present result, apparently resulting from the fixed C_m ($= 1.2\%$), which is inadequate to achieve the maximum penetration depth at $Re = 13\,300$.

The inter-relation between the control laws and associated evolution may be presented via a proximity map (Duriez, Brunton & Noack 2016). The idea is to represent control laws $\mathbf{b}(t)$ as points in a two-dimensional feature plane $\boldsymbol{\gamma}_j = (\gamma_{j,1}, \gamma_{j,2})$, where $j = 1, 2, \dots, N_i \times G$, so that the distance between feature vectors best manifests the difference between the control laws. How to define a distance matrix D_{jk} ($j, k = 1, 2, \dots, N_i \times G$) between the j th and k th $\mathbf{b}(t)$ is crucial in forming this feature plane. For the present actuation, this matrix is the averaged squared Euclidean difference between the actuation command vectors, given by

$$D_{jk}^2 = ||PSD(\mathbf{b}_{1,j}(t)) - PSD(\mathbf{b}_{1,k}(t))||^2 + \beta |b_{2,j} - b_{2,k}| + \delta |J_j - J_k|. \quad (4.1)$$

The first term on the right side of (4.1) is the square of the averaged Euclidean distance between $\mathbf{b}_{1,j}(t)$ and $\mathbf{b}_{1,k}(t)$ in the frequency domain, and thus the power spectral density functions (PSD) of $\mathbf{b}_{1,j}(t)$ and $\mathbf{b}_{1,k}(t)$ are used. The second term is the difference between $b_{2,j}$ and $b_{2,k}$ weighted by a factor β . The control performance J weighted by a factor δ is also incorporated in (4.1). The parameters β and δ may act to smooth the control landscape and are chosen so that the maximum differences in the three terms are the same, viz.

$$\begin{aligned} \max_{j,k=1,2,\dots,N_i \times G} ||PSD(\mathbf{b}_{1,j}(t)) - PSD(\mathbf{b}_{1,k}(t))||^2 &= \beta \max_{j,k=1,2,\dots,N_i \times G} |b_{2,j} - b_{2,k}| \\ &= \delta \max_{j,k=1,2,\dots,N_i \times G} |J_j - J_k|. \end{aligned} \quad (4.2)$$

Given D_{jk} , feature vectors $\boldsymbol{\gamma}_j$ ($j = 1, 2, \dots, N_i \times G$) can be obtained by a classical multi-dimensional scaling (Cox & Cox 2000) so that the distances are optimally preserved

$$\sum_{j=1}^{N_i \times G} \sum_{k=1}^{N_i \times G} (||\boldsymbol{\gamma}_j - \boldsymbol{\gamma}_k|| - D_{jk})^2 = \min. \quad (4.3)$$

The feature vectors are sorted and rotated so that the first coordinate is characterized by the largest variance, the second by the second largest, etc. Finally, control laws can be visualized on a scatter plot or proximity map, as done by Kaiser *et al.* (2017) and Wu *et al.* (2018a). The map provides an overall picture of the control landscape, and the feature coordinates may correlate with some features of control, as shown by Zhou *et al.* (2020).

Figure 8(a) shows the proximity map of all control laws obtained in the learning process at $Re = 8000$ in a three-dimensional plane ($\gamma_1, \gamma_2, C_m/\alpha$). The addition of the third coordinate is due to the fact that C_m/α is well correlated with K_e (figure 6b). Each circular symbol represents one control law and its colour corresponds to the value of J . Given γ_1 (or γ_2), we randomly select the control laws at different γ_2 (or γ_1) in figure 8(a) and examine carefully the control parameters, which unveils that γ_1 tends to be positively correlated with C_m and γ_2 is adversely correlated with α for given γ_1 . Furthermore, C_m/α is adversely correlated with J or positively with K_e (figure 8a), which is fully consistent with the fact that C_m/α corresponds physically to the penetration depth of the minijet into the main jet (Perumal & Zhou 2018). Similar observations are made at $Re = 20\,000$ (figure 8b).

5. Scaling Analysis

5.1. Dependence of K_0 and f_0 on Re

It is well known that the boundary-layer thickness at the jet exit diminishes with increasing Re , producing a significant influence on the flow structure development and accelerating the mixing rate (e.g. New *et al.* 2006; Mi *et al.* 2013) or centreline mean velocity decay rate of an unforced jet. Naturally, a manipulated jet is also sensitive to this change in the boundary-layer thickness. As such, $K_e = g_1(C_m, f_0, \alpha, d/D, Re, K_0)$, where K_0 is calculated by $K_0 = (\bar{U}_j - \bar{U}_{5D,0})/\bar{U}_j$. That is, the effect of boundary-layer thickness of the unforced jet on jet mixing is considered via K_0 . Both K_0 and f_0 are dependent on Re .

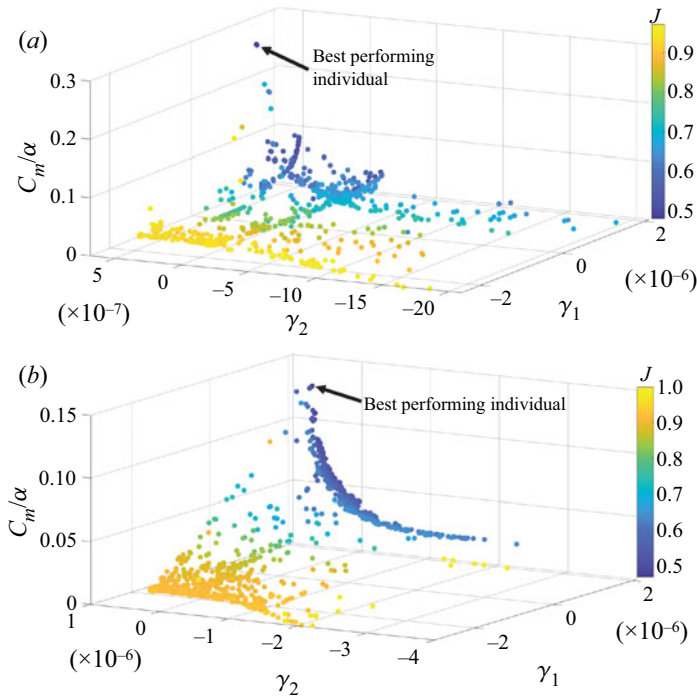


Figure 8. Control landscape based on a proximity map of all tested control laws at (a) $Re = 8000$ and (b) $20\,000$. Each circle represents an individual control law and the distance between two control laws indicates their dissimilarity.

As shown in figure 9,

$$K_0 = -103Re^{-0.83} + 0.1, \tag{5.1}$$

which rises rapidly with Re for $Re \leq 10\,000$ but less so for $Re > 10\,000$. On the other hand, f_0 varies from 100 to 680 Hz (figure 9) and is linearly correlated with Re , viz.

$$f_0 = 0.0167Re + 16. \tag{5.2}$$

Both (5.1) and (5.2) are valid for $Re = 5800\text{--}40\,000$.

5.2. Dependence of K_e/K_0 on control parameters

A careful analysis of experimental data from conventional open-loop control (Perumal & Zhou 2018) shows reasonably well collapsed K_e/K_0 for different d/D provided that C_m is re-scaled as $C_m(D/d)^{1-n}$. This collapse is illustrated for $Re = 13\,000$, where $1 - n = 0.55$ for $\alpha = 0.1$ (figure 10a1), 0.67 for $\alpha = 0.5$ (figure 10b1) and 1.00 for $\alpha = 0.9$ (figure 10c1). As a matter of fact, $n = -0.56\alpha + 0.50$, distinctly different from $n = -0.31\alpha + 0.28$ at $Re = 8000$ (Perumal & Zhou 2018) for the same $f_e/f_0 (=0.5)$. Interestingly, the same collapse is observed at $f_e/f_0 = 0.3$ and 1.0 (figure 10a2–c2) for $Re = 13\,000$ and n is unchanged, although the magnitude of K_e/K_0 is different. The result suggests that n is independent of f_e/f_0 , that is, n depends only on α for a given Re . Similar analysis has also been performed for other Re values to determine the dependence of n on α (figure 11a,b). In general,

$$n = C_1\alpha + C_2, \tag{5.3a}$$

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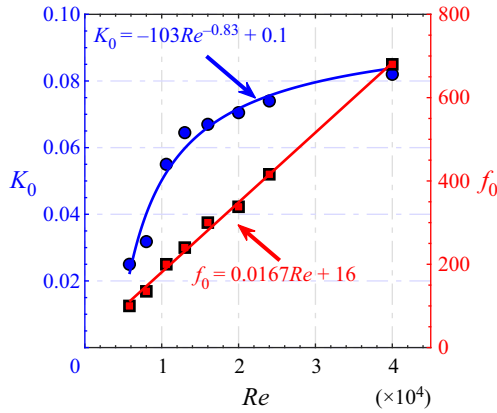


Figure 9. Dependence of K_0 and f_0 on Re for unforced jet.

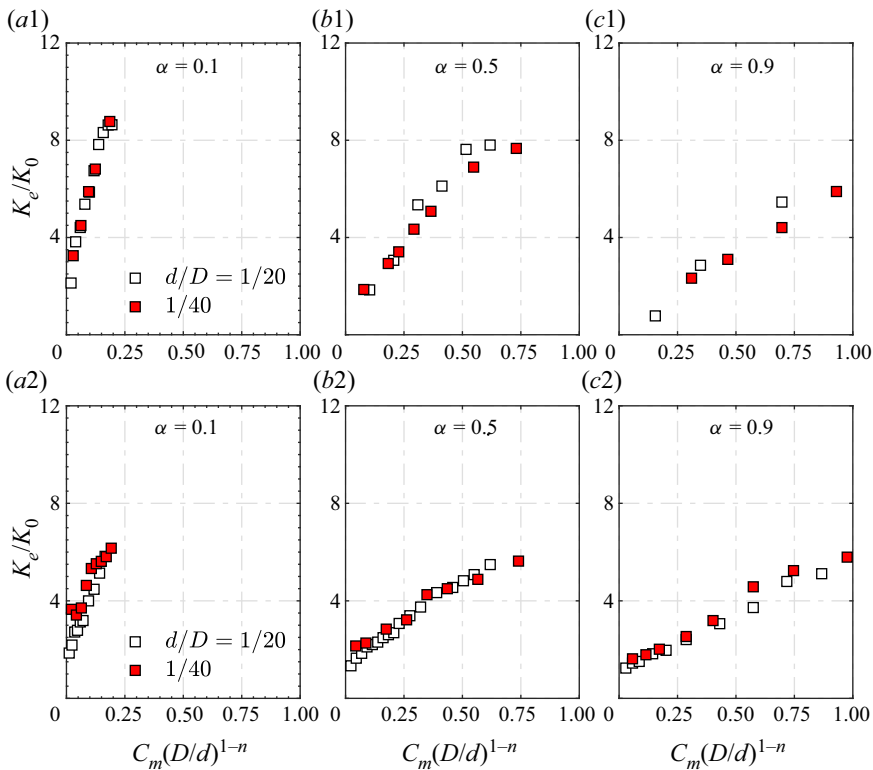


Figure 10. Dependence of K_e/K_0 on $(C_m/\alpha)(D/d)^{1-n}$ for $Re = 13\,000$ at (a1–c1) $f_e/f_0 = 0.5$ and (a2–c2) 1.0.

where the coefficients C_1 and C_2 depend on Re , as shown in figure 11, given by

$$C_1 = \begin{cases} -9.64 \times 10^{-5} Re + 0.46, & 5800 \leq Re < 10\,600 \\ -0.56, & 10\,600 \leq Re \leq 40\,000 \end{cases}, \quad (5.3b)$$

$$C_2 = \begin{cases} 8.67 \times 10^{-5} Re - 0.42, & 5800 \leq Re < 10\,600 \\ 0.51, & 10\,600 \leq Re \leq 40\,000 \end{cases}. \quad (5.3c)$$

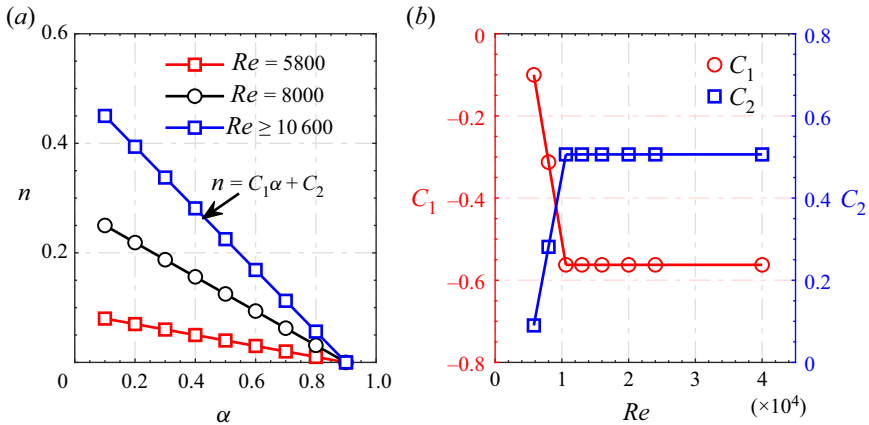


Figure 11. Dependence of (a) n on α ; (b) C_1 and C_2 for $n = C_1\alpha + C_2$ on Re .

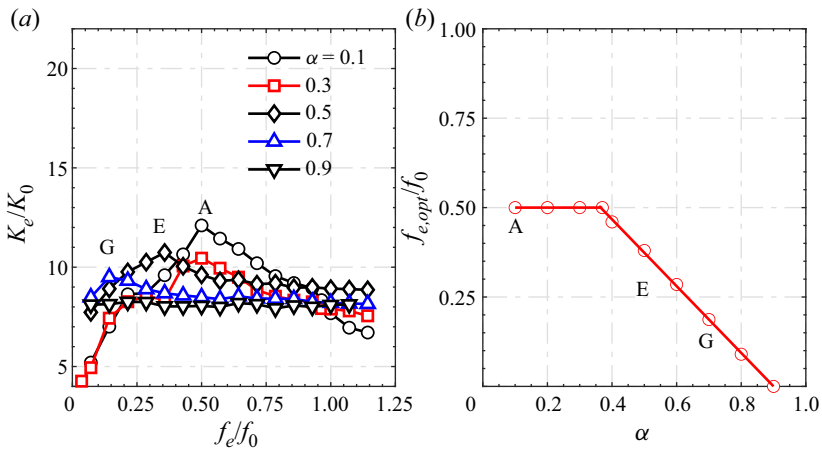


Figure 12. (a) Dependence of K_e/K_0 on f_e/f_0 ; (b) dependence of $f_{e,opt}/f_0$ on α ($Re = 8000$, $C_m = 1.0\%$, $d/D = 1/20$).

Apparently, n depends only on α for $Re \geq 10\,600$ and is physically meaningful (to be discussed later).

The dependence of K_e/K_0 on f_e/f_0 is presented in figure 12(a) for $\alpha = 0.1-0.7$ ($Re = 8000$, $C_m = 1.0\%$, $d/D = 1/20$). We may extract the dependence on α of $f_{e,opt}/f_0$ at which the local maximum K_e/K_0 occurs, as shown in figure 12(b), viz.

$$\frac{f_{e,opt}}{f_0} = \begin{cases} 0.5, & \alpha \leq 0.37 \\ -0.93\alpha + 0.84, & \alpha > 0.37 \end{cases} \quad (5.4)$$

It is worth mentioning that the K_e/K_0 data are not shown for $\alpha = 0.2, 0.4, 0.6$ and 0.8 to avoid overcrowding in figure 12(a), although they are used to obtain figure 12(b). Similar analysis has been performed for other Re . Interestingly, $f_{e,opt}/f_0$ is independent of Re , as found by Wu *et al.* (2018b) for $Re = 5700-13\,300$ (their figure 10a).

It would be difficult to develop a scaling law with f_e/f_0 incorporated since the dependence of K_e/K_0 on f_e/f_0 displays a distinct behaviour as α varies (figure 12a). Thus, we introduce $f_e/f_{e,opt} \equiv (f_e/f_0)/(f_{e,opt}/f_0)$, and $f_e/f_{e,opt} = 1$ corresponds to the local

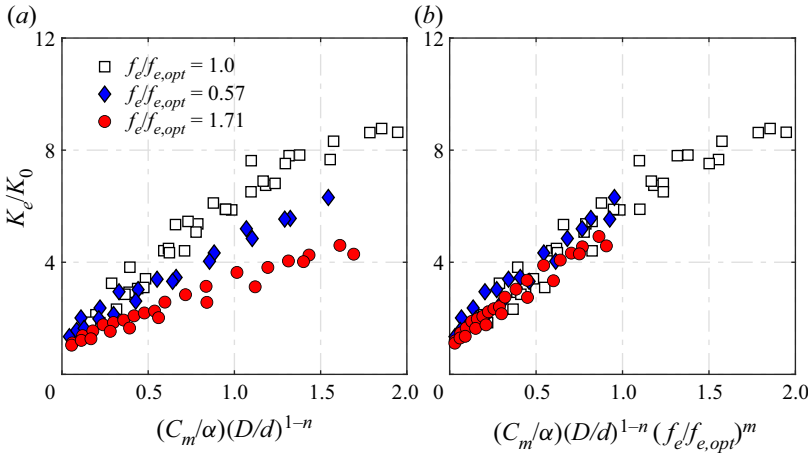


Figure 13. Dependence of K_e/K_0 on (a) $(C_m/\alpha)(D/d)^{1-n}$ for different $f_e/f_{e,opt}$ and (b) $(C_m/\alpha)(D/d)^{1-n}(f_e/f_{e,opt})^m$ ($\alpha = 0.1-0.9$; $Re = 13\ 000$).

maximum K_e/K_0 , irrespective of α . Replotting the data in figure 10(a2–b2) in terms of the dependence of K_e/K_0 on $(C_m/\alpha)(D/d)^{1-n}$ for $f_e/f_{e,opt} = 1.00, 0.57$ and 1.71 , we see a large departure from one $f_e/f_{e,opt}$ to another (figure 13a) but a reasonably good collapse if $(C_m/\alpha)(D/d)^{1-n}$ is replaced by $(C_m/\alpha)(D/d)^{1-n}(f_e/f_{e,opt})^m$ (figure 13b). The choice of power index m is not so straightforward. As shown in figure 12(a), with increasing f_e/f_0 , K_e/K_0 climbs for $f_e/f_0 \leq f_{e,opt}/f_0$ or $f_e/f_{e,opt} = (f_e/f_0)/(f_{e,opt}/f_0) \leq 1$ but declines for $f_e/f_0 > f_{e,opt}/f_0$ or $f_e/f_{e,opt} > 1$. One may surmise that the corresponding m should be positive and negative, respectively. After numerous trial-and-error analyses, m is found to be 0.7 and -0.9 for $f_e/f_0 \leq f_{e,opt}/f_0$ and $f_e/f_0 > f_{e,opt}/f_0$, respectively, which reflects the sensitivity of K_e/K_0 to the variation in f_e/f_0 . It is further found that the m values are unchanged as Re varies from 5800 to $40\ 000$, that is, m is independent of Re . Nevertheless, the dependence of K_e/K_0 on $(C_m/\alpha)(D/d)^{1-n}(f_e/f_{e,opt})^m$ exhibits a large scatter as Re varies from 5800 to $40\ 000$ (figure 14). However, as shown in figure 15, it is, surprisingly, found that all the data of K_e/K_0 , from more than 7000 AI-generated control laws or from conventional control (Perumal & Zhou 2018), collapse well together, with a small scatter, once a weighting factor $1/Re$ is introduced in the abscissa, i.e.

$$\zeta = \frac{C_m}{\alpha} \left(\frac{D}{d}\right)^{1-n} \frac{1}{Re} \left(\frac{f_e}{f_{e,opt}}\right)^m. \tag{5.5}$$

Here, $K_e = g_1(C_m, f_e/f_0, \alpha, d/D, Re, K_0)$ is now reduced to $K_e/K_0 = g_2(\zeta)$. The data may be least-square fitted to a cubic function, viz.

$$K_e/K_0 = 2.2 \times 10^{12} \zeta^3 - 1.3 \times 10^9 \zeta^2 + 2.6 \times 10^5 \zeta + 1.0. \tag{5.6}$$

The measured K_e/K_0 may deviate from (5.6) by no more than 10% , with a 95% confidence. Apparently, K_e/K_0 is the jet entrainment ratio, indicating the enhancement of entrainment with respect to the unforced jet.

One may wonder whether the same scaling law (5.6) could be obtained from a small amount of data, say generated by 200 control laws instead of 7000 . One test is then performed. To ensure various control performances are covered, all the control laws for a given Re are re-numbered based on their J values, and 25 control laws are selected with

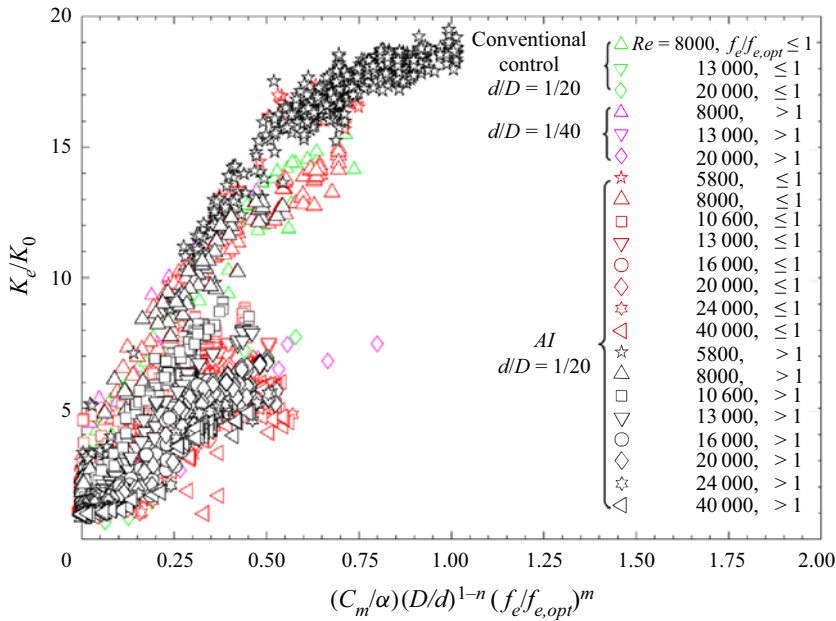


Figure 14. Dependence of K_e/K_0 on $(C_m/\alpha)(D/d)^{1-n}(f_e/f_{e,opt})^m$ for $Re = 5800\text{--}40\,000$ ($\alpha = 0.07\text{--}0.90$, $f_e/f_0 = 0.1\text{--}1.5$).

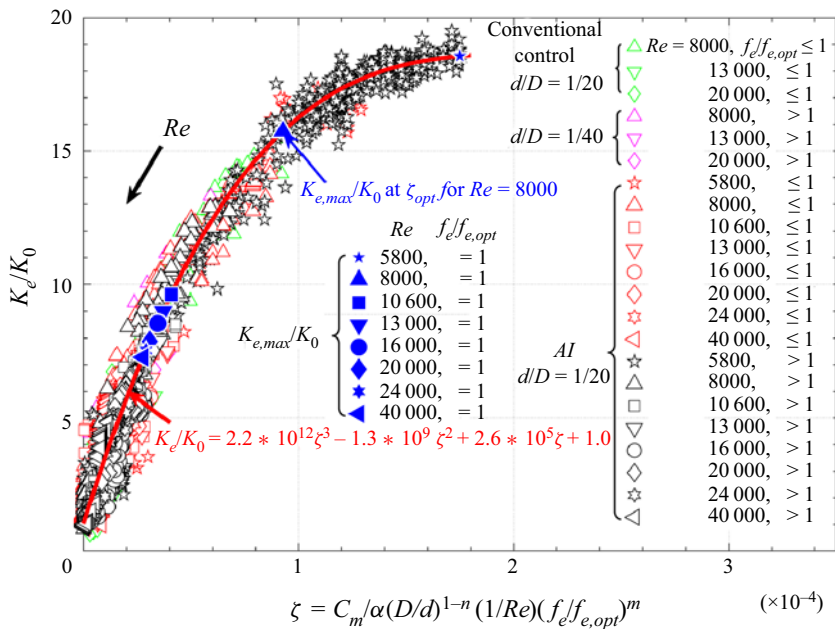


Figure 15. Dependence of K_e/K_0 on $(C_m/\alpha)(D/d)^{1-n}(1/Re)(f_e/f_{e,opt})^m$ for $Re = 5800\text{--}40\,000$, $\alpha = 0.07\text{--}0.9$, $f_e/f_0 = 0.1\text{--}1.5$. The red curve is the least-squares fit to experimental data. Blue filled symbols correspond to $K_{e,max}/K_0$, predicted from (5.6), at a given Re .

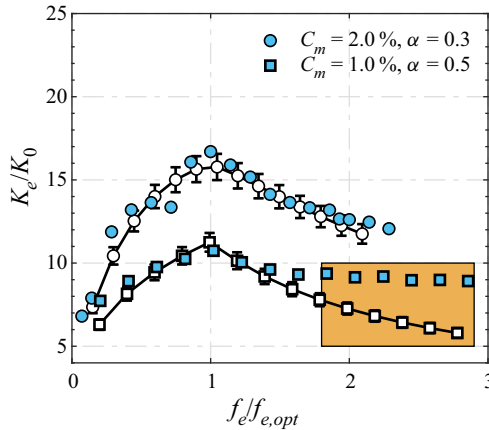


Figure 16. Comparison of the dependence of K_e/K_0 on $f_e/f_{e,opt}$ for $Re = 8000$ between prediction from (5.6) (open symbol) and measurement (filled symbol) for $(C_m, \alpha, d/D) = (2.0\%, 0.3, 1/20)$ and $(1.0\%, 0.5, 1/20)$. The rectangle highlights the deviation between prediction and measurement.

the same difference in J . The dependence of K_e/K_0 on $(C_m/\alpha)(D/d)^{1-n}(1/Re)(f_e/f_{e,opt})^m$ (figure not shown) can be least-squares fitted to a curve almost identical to that shown in figure 15, albeit with very few data falling between $\zeta = (1.0-1.9) \times 10^{-4}$, where Re is 5800. One may conclude that the confidence level of the scaling law would not drop appreciably in spite of a drop in the control laws from the order of 1000 to 25 for each Re , given $Re \geq 8000$.

The way m is determined may imply a limited range of $f_e/f_{e,opt}$ over which (5.6) is valid. As illustrated in figure 16, K_e/K_0 calculated from (5.6) agrees with measurement for $Re = 8000$ at $(C_m, \alpha, d/D) = (1.0\%, 0.5, 1/20)$ and $(2.0\%, 0.3, 1/20)$, thus providing a validity for the choice of m . Note an appreciable deviation for $f_e/f_{e,opt} \geq 1.8$ in K_e/K_0 between calculation and measurement at $\alpha = 0.5$. The measured K_e/K_0 appears to change little for $f_e/f_{e,opt} \geq 1.8$. Perumal & Zhou (2018) demonstrated that an unsteady minijet at large α (≥ 0.5) and f_e/f_0 (≥ 1.0) behaves like a quasi-steady blowing, and jet mixing is independent of both α and f_e/f_0 , which accounts for the deviation. The observation suggests a critical $f_e/f_{e,opt}$ beyond which the scaling law (5.6) is invalid, and this critical $f_e/f_{e,opt}$ is the $f_{e,steady}/f_{e,opt}$ value at which K_e/K_0 becomes independent of $f_e/f_{e,opt}$ for a given α . Figure 17 presents the dependence on α of $f_{e,steady}/f_0$, which is extracted from the data illustrated in figure 12(a) and least-square fitted to

$$f_{e,steady}/f_0 = -1.55\alpha^{1.1} + 1.4. \tag{5.7}$$

Then, the critical $f_e/f_{e,opt}$ or $f_{e,steady}/f_{e,opt}$ at which (5.6) may deviate appreciably from the measured data can be obtained from (5.4) and (5.7), viz.

$$\frac{f_{e,steady}}{f_{e,opt}} = \frac{f_{e,steady}/f_0}{f_{e,opt}/f_0}. \tag{5.8}$$

Substitution of $\alpha = 0.5$ into (5.4) and (5.7) yields $f_{e,steady}/f_{e,opt} = 1.8$, which is internally consistent with the observation from figure 16.

There is a need to understand the physical meaning of n , which may offer valuable insight into the flow physics behind control. The physical interpretation of n is given by Perumal & Zhou (2018) for a fixed Re . As d/D is decreased, say by half, one might have

expected, given $\alpha = 0.1$ and $f_e/f_0 = 0.5$, that the required C_m would also reduce by 50 % to achieve the same K_e/K_0 ($Re = 8000$); yet, C_m was measured to drop by 60 % (their figure 23). The additional 10 % drop results from the retardation effect of minijet injection with contracting d/D . That is, a non-zero n reflects the need for additional C_m to achieve the same K_e/K_0 for given α . Apparently, the retardation effect depends on α and Re (5.3). Perumal & Zhou (2018) have documented in detail the dependence of the retardation effect on α . Thus, we present only its dependence on Re . In order to determine the dependence of required C_m on Re to achieve the same K_e/K_0 at given $f_e/f_{e,opt}$, α and d/D , we rewrite (5.6) as

$$K_e/K_0 \approx \frac{C_m}{\alpha} \left(\frac{D}{d}\right)^{1-n} \frac{1}{Re} \left(\frac{f_e}{f_{e,opt}}\right)^m. \tag{5.9}$$

Rearranging (5.9) yields

$$C_m \approx Re K_e/K_0 \alpha \left(\frac{d}{D}\right)^{1-n} \left(\frac{f_{e,opt}}{f_e}\right)^m. \tag{5.10}$$

Equation (5.10) can be used to predict the required C_m with varying Re for a pre-specified K_e/K_0 at given $f_e/f_{e,opt}$, α and d/D . It is of interest to compare the variation in C_m with Re for a given K_e/K_0 at the same $f_e/f_{e,opt}$, α and d/D with and without the retardation effect. On substitution of $n = 0$ in (5.10), we may obtain C_m in the absence of the retardation effect

$$C_m \approx Re K_e/K_0 \alpha \left(\frac{d}{D}\right) \left(\frac{f_{e,opt}}{f_e}\right)^m. \tag{5.11}$$

Figure 18 compares the dependence of C_m on Re ($= 5800\text{--}40\ 000$) for a pre-specified $K_e/K_0 = 6$ ($f_e/f_{e,opt} = 1.0$, $\alpha = 0.1$, $d/D = 1/20$) calculated from (5.11) with that from (5.10) where n is determined from (5.3a–c). Clearly, given the same Re , C_m calculated from (5.10) is substantially larger, as a result of the retardation effect, than that from (5.11). For example, at $Re = 20\ 000$, the C_m values from (5.10) and (5.11) are approximately 0.85 % and 0.22 %, respectively. The retardation effect may depend on Re , which is defined by the percentage increase in required C_m calculated from (5.10) as compared with (5.11), viz.

$$\Delta C_m = ([C_m]_{(5.10)} - [C_m]_{(5.11)})/[C_m]_{(5.11)}. \tag{5.12}$$

The value of ΔC_m increases with Re for $Re < 10\ 600$ and becomes independent of Re for $Re \geq 10\ 600$ (figure 18), as n does (figure 11b). At $Re = 20\ 000$, the required C_m to achieve a pre-specified $K_e/K_0 = 6$ ($f_e/f_{e,opt} = 1.0$, $\alpha = 0.1$, $d/D = 1/20$) from (5.10) is 2.8 times higher than from (5.11). Evidently, n plays a significant role in characterizing the retardation effect of the minijet, which contracts with increasing α and reaches zero at $\alpha = 0.9$, where $n \approx 0$ (Perumal & Zhou 2018).

5.3. Physical interpretation of similarity parameters

Some interesting inferences can be made from the scaling law (figure 15 or (5.6)). Firstly, it is important to understand physically ξ/Re , where $\xi = (C_m/\alpha)(D/d)^{1-n}$ is interpreted as the effective penetration depth at a given $Re = 8000$ (Perumal & Zhou 2018). Yang *et al.* (2016) and Perumal & Zhou (2018) demonstrated that the distribution of u_{rms}^* at the nozzle exit and the minijet penetration depth into the main jet are correlated. Figure 19 shows the radial distributions of $u_{rms}^* = u_{rms}/\bar{U}_j$ measured at $x^* = 0.05$ ($f_e/f_{e,opt} = 1$) along the injection (x – y) plane for $\xi/Re = 0.3 \times 10^{-4}$ at $Re = 8000$, 13 000 and 20 000.

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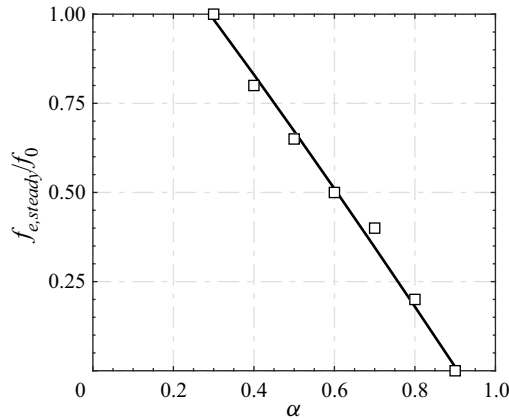


Figure 17. Dependence of $f_{e,steady}/f_0$ on α extracted from the dependence of K_e/K_0 on f_e/f_0 , as illustrated in figure 12(a).

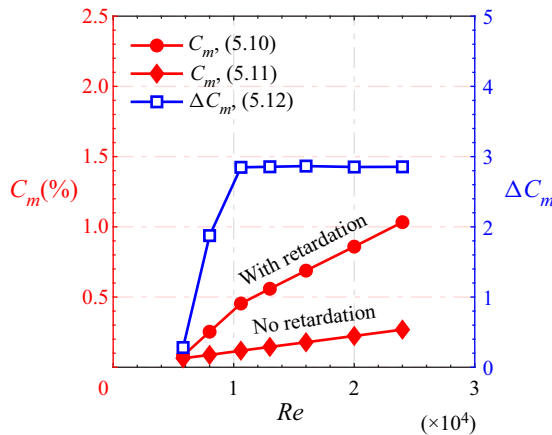


Figure 18. Dependence of C_m and ΔC_m on Re ($f_e/f_{e,opt} = 1.0$, $\alpha = 0.1$, $d/D = 1/20$) calculated from ((5.10)–(5.12)) for a pre-specified $K_e/K_0 = 6$.

Interestingly, given the same ξ/Re , the u_{rms}^* distributions are qualitatively the same despite different combinations of C_m , α , d/D and Re , suggesting that the penetration depth, as marked in figure 19, is the same given the same ξ/Re , that is, there is a correspondence between ξ/Re and the penetration depth. Perumal & Zhou (2018) pointed out that ξ is physically the effective momentum ratio of the minijet to the main jet per pulse of injection or the effective penetration depth for a fixed Re . The present finding points to the fact that ξ/Re is a more general definition for the effective momentum ratio of the minijet to the main jet per pulse of injection, which is valid even in the context of varying Re .

Secondly, to gain insight into the physics behind $f_e/f_{e,opt}$, we present in figure 20(a1–c2) typical images from flow visualization ($Re = 8000$) in the injection plane ($x-z$) for $f_e/f_{e,opt} < 1$, $f_e/f_{e,opt} = 1$ and $f_e/f_{e,opt} > 1$ at $\xi/Re = 0.3 \times 10^{-4}$, along with the signals of instantaneous streamwise U^* measured at $(x^*, y^*, z^*) = (1.5, 0, 0.45)$. The sharp peaks in the signals represent the perturbation induced by the injection. Once manipulated, the shear layer rolls up early on the injection side (cf. unforced flow shown in figure 3b) and the vortex dynamics is very different, depending on $f_e/f_{e,opt}$. There exist three different

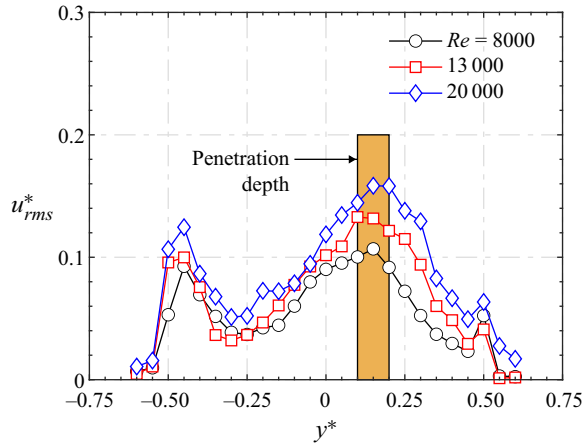


Figure 19. Radial distributions of the streamwise fluctuating velocity $u_{rms}^* = u_{rms}/\bar{U}_j$ measured at $x^* = 0.05$ ($f_{el}f_{e,opt} = 1$) for $\xi/Re = 0.3 \cdot 10^{-4}$.

states of the manipulated jet depending on $f_{el}f_{e,opt}$. State 1 corresponds to $f_{el}f_{e,opt} = 1$, at which the optimal separation takes place between incomplete ring vortices. As shown in figure 20(b1), an incomplete ring vortex V_2 induced by minijet injection is advected downstream without direct interaction with a following vortex V_1 induced by the next cycle of injection. This assertion is complemented by the velocity signal in figure 20(a1), where the peaks are distinctly separated. It seems plausible that the ring vortices with the minimum but clear separation produce the maximized jet decay rate (figure 16). State 2 corresponds to a smaller $f_{el}f_{e,opt}$ ($= 0.57$) or an increased gap between the successive ring vortices, as shown in figure 20(a2,b2). Evidently, the incomplete ring vortex V_4 needs more time to grow before interacting with the next one V_3 . This increased gap seems to facilitate the establishment of less anti-symmetrically arranged vortices about the centreline in the (x - y) plane (figure 20a2, cf. figure 20a1), resembling more a natural jet and resulting in a reduced jet decay rate (figure 16). Under state 3, where $f_{el}f_{e,opt} > 1$, the spatial separation between the vortices V_5 and V_6 contracts, as illustrated in figure 20c2 ($f_{el}f_{e,opt} = 1.71$), and the interaction between the vortices is intensified. This is corroborated by the hot-wire signal (figure 20c1), which shows the peaks closely separated from each other. This interaction may incur the occurrence of turbulent puff-like structures, causing a retreat of penetration (Hermanson, Wahba & Johari 1998; Johari 2006) and weakened mixing. Johari, Pacheco-Tougas & Hermanson (1999) made a similar observation for jet in cross-flow and pointed out that a decrease in spacing between the vortical structures resulted in increased interaction and reduced penetration. Thus, the jet decay rate under state 3 also drops compared with state 1 (figure 16). The three states observed at $Re = 8000$ are also evident at other Re (not shown) given an identical ξ/Re . The similarity parameter $f_{el}f_{e,opt}$ on which states 1–3 strongly depend physically corresponds to the spatial separation between successive vortices formed during minijet injection (figure 20a2–c2). Thus, the jet mixing depends on the penetration depth of the minijet into the main jet (ξ/Re) and the interaction between vortices ($f_{el}f_{e,opt}$) in the manipulated jet, and ζ is physically the momentum ratio (the momentum per pulse of injection to the inertia momentum of main jet) times the frequency ratio.

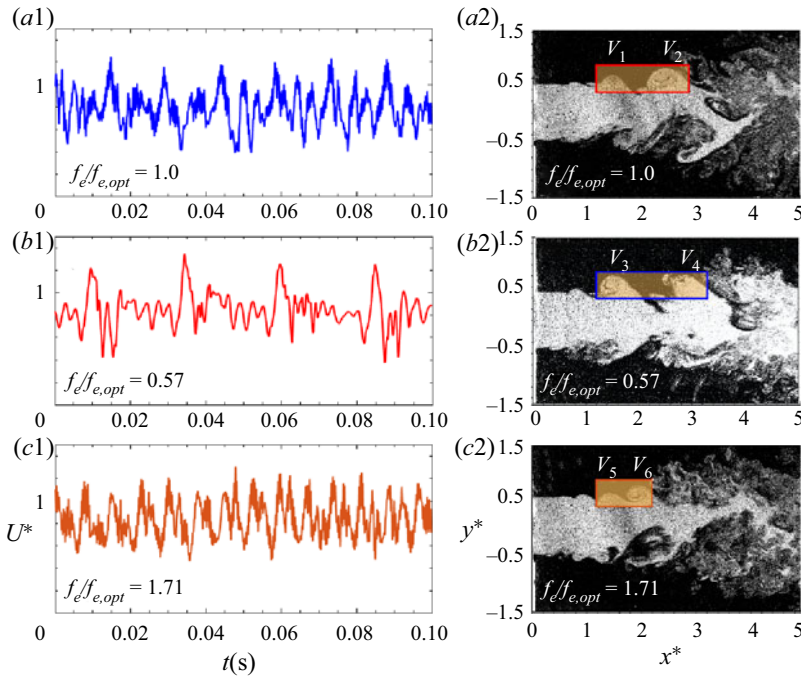


Figure 20. (a1–c1) Typical U^* signals measured at $(x^*, y^*, z^*) = (1.5, 0, 0.45)$ for $f_e/f_{e,opt} = 0.57, 1$ and 1.71 at $\xi/Re = 0.3 \times 10^{-4}$ for $Re = 8000$. (a2–c2) Typical flow structures from flow visualization in the injection plane $(x-z)$ at representative $f_e/f_{e,opt}$, where the vortices V_1 – V_6 result from minijet injection at $\xi/Re = 0.3 \times 10^{-4}$ for $Re = 8000$.

Thirdly, the ratio $K_{e,max}/K_0$ drops rapidly with increasing Re for $Re < 10\,600$ but very slowly for $Re \geq 10\,600$ (figure 21), which may be described by

$$K_{e,max}/K_0 \approx 2.24 * 10^{15} Re^{-3.7} + 6.8. \quad (5.13)$$

This feature is primarily due to the variation in K_0 with increasing Re (figure 9) as $K_{e,max}$ is unchanged (figure 7). As such, $\zeta = (\xi/Re)(f_e/f_{e,opt})^m$ and $\xi_{opt} = ((\sqrt{MR}/\alpha)(d/D)^n)_{opt}$ at which $K_{e,max}/K_0$ occurs. Then, at $f_e/f_{e,opt} = 1$, (5.9) may be rewritten as

$$K_{e,max}/K_0 \approx \left(\frac{\sqrt{MR}}{\alpha} \left(\frac{d}{D} \right)^n \right)_{opt} \frac{1}{Re} \left(\frac{f_{e,opt}}{f_{e,opt}} \right)^{0.7} = \frac{\xi_{opt}}{Re} = \zeta_{opt}. \quad (5.14)$$

Thus, given Re , $K_{e,max}/K_0$ may be estimated from (5.13). As $\zeta_{opt} = \xi_{opt}/Re$ is connected to the effective penetration depth of the minijet into main jet and hence K_e/K_0 , the drop in $K_{e,max}/K_0$ with Re can be directly correlated to the reduced effective minijet penetration into main jet measured in terms of ξ_{opt}/Re . One may wonder why the effective penetration depth or ξ_{opt}/Re contracts with increasing Re . Figure 21 presents a variation with Re in the normalized stroke length L/d determined at $K_{e,max}/K_0$. Physically, L represents the slug of jet fluid ejected during each pulse of injection or the amount of fluid injected during one injection (Steinfurth & Weiss 2020) and, following Johari (2006), is estimated by $L = (1/A_m) \int_0^{\tau_{opt}} \int_A U_m \, dA_m \, dt = \bar{U}_m \tau_{opt}$, where $\tau_{opt} = \alpha/f_{e,opt}$, the minijet velocity $\bar{U}_m = C_m (D/d)^2 \bar{U}_j$ (Perumal & Zhou 2018) and A_m is the minijet exit area. Johari (2006) found that the minijet-generated flow structure depended on L/d , characterized by

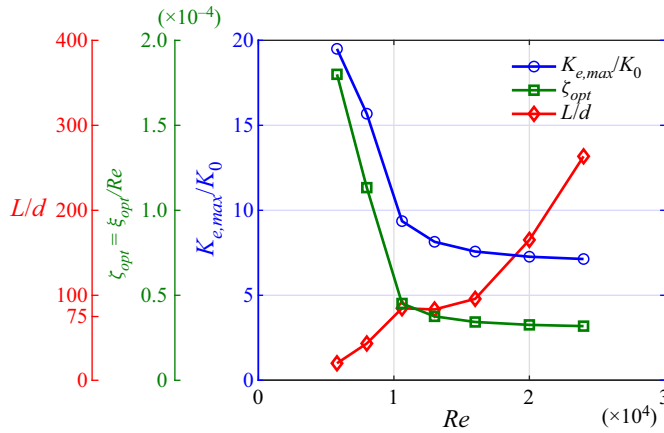


Figure 21. Dependence of $K_{e,max}/K_0$ on Re extracted from figure 7, and $\zeta_{opt}(= \xi_{opt}/Re)$ on Re at $K_{e,max}/K_0$ ($f_e/f_{e,opt} = 1.0$) obtained from (5.14) and the variation of stroke length ratio L/d with increasing Re when the control performance is optimal ($f_e/f_{e,opt} = 1.0$).

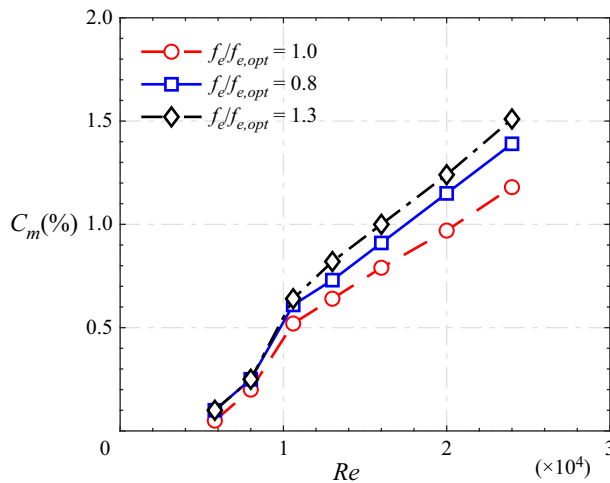


Figure 22. Dependence of C_m on Re for $f_e/f_{e,opt} = 1.0, <1.0$ and >1.0 ($\alpha = 0.1, d/D = 1/20$), predicted from (5.10) for a pre-defined $K_e/K_0 = 6$.

turbulent puffs for $25 < L/d < 75$ and an elongated steady jet structure for $L/d > 75$, and the penetration depth of the latter was significantly smaller than that of the former. Figure 21 shows that L/d grows with Re and indeed exceeds 75 for $Re \geq 10\ 600$. This provides an explanation for the observation that $K_{e,max}/K_0$ or ξ_{opt}/Re drops greatly from $Re = 8000$ to $10\ 600$ (figure 21).

Fourthly, given a pre-specified $K_e/K_0 = 6$ and $(\alpha, d/D) = (0.1, 1/20)$, we may determine the relation between C_m and Re from (5.10) for $f_e/f_{e,opt} = 0.57, 1.0$ and 1.71 . Figure 22 shows that C_m is always least at $f_e/f_{e,opt} = 1.0$, regardless of Re .

Finally, Wickersham (2007) investigated the effect of excitation frequency of pulsed injection on a jet of $Re = 71\ 000\text{--}355\ 000$ manipulated by two pulsed injections. They argued that, under the optimal pulsing condition, the minijet penetrated farther into the jet and the induced vortices were able to persist longer downstream, thus enhancing mixing.

In the study of a pulsed jet in cross-flow, Johari (2006) also reported that the maximum mixing can be achieved with pulsed jets if the pulsing parameters are chosen specifically to create compact vortex rings and subsequent interaction among vortical structures. Nevertheless, to the best of our knowledge, there has been no report in the literature on the parameters that determine the optimal pulsing condition. This condition is evident from (5.14), determined by ξ_{opt}/Re and $f_e/f_{e,opt} = 1$, and hence ζ_{opt} , that is, $K_{e,max}/K_0$ is achieved at the maximum penetration ξ_{opt}/Re and meanwhile the vortices thus formed under minijet injection are optimally separated from each other, as dictated by $f_e/f_{e,opt} = 1$.

6. Conclusions

A novel hybrid AI system has been developed to manipulate a turbulent jet using a single pulsed radial minijet over a range of Re ($= 5800\text{--}40\,000$). Four control parameters are investigated, i.e. C_m ($= 0.1\% \text{--} 8.0\%$), f_e/f_0 ($= 0\text{--}1.2$), α ($= 0.1\text{--}0.9$) and d/D ($= 1/20, 1/40$). The following conclusions can be drawn out of this work:

- (i) The developed hybrid AI algorithm may optimize simultaneously a control law and a time-independent variable C_m , overcoming the drawback of the previous AI system based on the LGP algorithm that can be used to optimize only the control laws (e.g. Zhou *et al.* 2020). At $Re = 8000$, for example, the system successfully finds the optimal control law, which consists of two sub-control laws that govern a time-dependent pulse signal $b_1(t)$ ($\alpha = 7\%$, $f_e/f_0 = 0.52$) and a time-independent mass flow rate signal b_2 ($C_m = 1.2\%$), respectively. Proximity map analysis performed based on all the control laws reveals that the ratio C_m/α , corresponding physically to the penetration depth of the pulsed minijet, dictates the control performance (figure 8). The system exhibits good robustness as Re varies from 5800 to 40 000, finding essentially the same maximum K_e and the same optimal f_e/f_0 .
- (ii) The AI system produces more than 7000 control laws for eight Re values examined and subsequently a tremendous amount of data involving six intrinsically related variables K_e , K_0 , C_m , f_e/f_0 , α and Re . Careful analysis of the data, along with those produced from a conventional open-loop control technique, which include one more variable, d/D , unveils, surprisingly, a scaling law, that is, the relationship $K_e = g_1(C_m, f_e/f_0, \alpha, d/D, Re, K_0)$ may be reduced to $K_e/K_0 = g_2(\zeta)$, where the scaling factor $\zeta = (C_m/\alpha)(D/d)^{1-n}(1/Re)(f_e/f_{e,opt})^m$. This law governs the effect of Re on jet manipulation using an unsteady minijet. Here, K_e/K_0 accounts for the effect of Re on the control performance, while ζ is physically the momentum ratio $(C_m/\alpha)(D/d)^{1-n}(1/Re)$ (the momentum per pulse of minijet injection to the inertia momentum of the main jet) times the frequency ratio $(f_e/f_{e,opt})^m$, the latter providing a measure for the spatial separation between successive vortices (figure 20a2–c2). The manipulated jet may exhibit three states, i.e. state 1 – $f_e/f_{e,opt} = 1$, state 2 – $f_e/f_{e,opt} > 1$ and state 3 – $f_e/f_{e,opt} < 1$, dictated by $(f_e/f_{e,opt})^m$. State 1 is associated with the optimally separated vortices, resulting in $K_{e,max}/K_0$, while states 2 and 3 yield closely and over-separated vortices, respectively, both causing a decline in K_e/K_0 (figure 16).
- (iii) Several inferences can be made from the scaling law. Firstly, K_e/K_0 increases with ζ . Secondly, given that $\xi = (C_m/\alpha)(D/d)^{1-n}$ may be interpreted as the effective momentum ratio of the minijet per pulse of injection to the main jet at a given Re (Perumal & Zhou 2018), ξ/Re carries the same physical meaning, although with a more general sense, valid even in the context of varying Re . Thirdly, with

$\zeta = (\xi/Re)(f_e/f_{e,opt})^m$, ζ_{opt} at which $K_{e,max}/K_0$ occurs for any given Re depends on ξ_{opt} and $(f_e/f_0)_{opt}^m$. The value of ξ_{opt} drops from $Re = 8000$ to $10\,600$ and then rises for a further increase in Re (figure 21). Evidently, jet mixing depends on both the penetration depth and the interaction between vortices or ξ/Re and $(f_e/f_{e,opt})^m$, whose optimum values lead to the maximized mixing. The optimal pulsing condition at which the maximum effective penetration of the minijet into the main jet occurs corresponds to ξ_{opt} (Wickersham 2007). At $f_e/f_{e,opt} = 1$, the minijet creates compact vortex rings (Johari 2006), which may persist farther downstream (Wickersham 2007), thus maximizing jet mixing. The similarity parameter ζ unveils that, for given α , the required C_m to achieve a pre-defined jet mixing or a constant K_e/K_0 is the smallest at $f_e/f_{e,opt} = 1$ or the most efficient, irrespective of Re (figure 22). Finally, with increasing Re , $K_{e,max}/K_0$ diminishes rapidly for $Re < 10\,600$ but very slowly for $Re \geq 10\,600$ (figure 21). The stroke length ratio L/d rises with Re (figure 21). Once L/d exceeds 75, the minijet flow structure is characterized by turbulent puffs, associated with the distributed vorticity and greater entrainment rate (Johari 2006), instead of vortex rings, resulting in a reduced effective penetration depth and hence a smaller $K_{e,max}/K_0$.

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