

The emergence of localized vortex–wave interaction states in plane Couette flow

Kengo Deguchi¹, Philip Hall^{2,†} and Andrew Walton²

¹Department of Aeronautics and Astronautics, Graduate School of Engineering, Kyoto University, Kyoto 606-8501, Japan

²Department of Mathematics, Imperial College London, South Kensington Campus, London SW7 2AZ, UK

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The recently understood relationship between high-Reynolds-number vortex–wave interaction theory and computationally generated self-sustaining processes provides a possible route to an understanding of some of the underlying structures of fully turbulent flows. Here vortex–wave interaction (VWI) theory is used in the long streamwise wavelength limit to continue the development found at order-one wavelengths by Hall & Sherwin (*J. Fluid Mech.*, vol. 661, 2010, pp. 178–205). The asymptotic description given reduces the Navier–Stokes equations to the so-called boundary-region equations, for which we find equilibrium states describing the change in the VWI as the wavelength of the wave increases from $O(h)$ to $O(Rh)$, where R is the Reynolds number and $2h$ is the depth of the channel. The reduced equations do not include the streamwise pressure gradient of the perturbation or the effect of streamwise diffusion of the wave–vortex states. The solutions we calculate have an asymptotic error proportional to R^{-2} when compared to the full Navier–Stokes equations. The results found correspond to the minimum drag configuration for VWI states and might therefore be of relevance to the control of turbulent flows. The key feature of the new states discussed here is the thickening of the critical layer structure associated with the wave part of the flow to completely fill the channel, so that the roll part of the flow is driven throughout the flow rather than as in Hall & Sherwin as a stress discontinuity across the critical layer. We identify a critical streamwise wavenumber scaling, which, when approached, causes the flow to localize and take on similarities with computationally generated or experimentally observed turbulent spots. In effect, the identification of this critical wavenumber for a given value of the assumed high Reynolds number fixes a minimum box length necessary for the emergence of localized structures. Whereas nonlinear equilibrium states of the Navier–Stokes equations are thought to form a backbone on which turbulent flows hang, our results suggest that the localized states found here might play a related role for turbulent spots.

Key words: nonlinear instability, shear layer turbulence, transition to turbulence

† Email address for correspondence: philhall@ic.ac.uk

1. Introduction

Our concern is with the structure of large-amplitude modifications to shear flows. Here we concentrate on plane Couette flow, which of course is well known to be linearly stable for all Reynolds numbers, but the structures we describe are generic and are applicable to a wide range of shear flows, including those which are linearly unstable. In recent years, there has been much interest in the structures that underpin turbulent flows. These structures are often referred to as self-sustaining processes, coherent structures or vortex–wave interactions, and their common feature is that they apparently form at least part of the ‘backbone’ on which turbulent flows evolve. In fact, it turns out that vortex–wave interaction (VWI) theory involving inviscid waves is simply the high-Reynolds-number description of the computationally generated self-sustaining processes (SSPs) at finite Reynolds numbers – see, for example, Hall & Smith (1991), Hall & Sherwin (2010) and Hall (2012*a,b*) for a discussion of VWI theory and Waleffe (1997, 2001, 2003) for discussions of SSPs.

Although developed independently, the VWI theory of Hall & Smith (1989, 1990, 1991) can be viewed as a formal asymptotic approach to the type of mean flow–wave interaction proposed by Benney (1984), while the SSP is more closely related to the work of Nagata (1990) on finite-amplitude perturbations to plane Couette flow. The common key process identified independently in VWI and SSP is the nonlinear coupling between an inviscid wave instability of a streak and the roll flow. The streak is then reinforced indirectly through its modification of the roll flow, which then drives the streak. This three-point coupling was revealed explicitly in the asymptotic theory of Hall & Smith (1991), whereas its recognition by Waleffe (1997) was a result of the interpretation of numerical results. The initial work of Hall & Smith (1991) simply derived the interaction equations in the context of wave interactions in compressible boundary layers but no solutions of the associated nonlinear eigenvalue problem were found until Hall & Sherwin (2010) applied the theory to plane Couette flow. The latter authors showed that the high-Reynolds-number assumption on which VWI is based leads to results that compare well with the full numerical SSP calculations of lower branch states by Wang, Gibson & Waleffe (2007). Application of SSP at high Reynolds number is made increasingly difficult because it must resolve vanishingly small internal layers where the flow changes rapidly. Thus, if one is interested in VWI/SSP states at Reynolds numbers greater than, say, 10^5 in channel flows, only the VWI approach is as yet viable. Moreover, the remarkable agreement of the VWI results of Hall & Sherwin (2010) with the SSP results of Wang *et al.* (2007) down to Reynolds numbers quite close to the minimum one at which nonlinear states exist suggests that it might sensibly be used more widely.

Most of the numerical work has been in relation to channel flow simulations, but there exist many closely related results for pipe flow (see e.g. Faisst & Eckhardt 2003; Wedin & Kerswell 2004; Kerswell & Tutty 2007; Viswanath 2009). For channel flows, Waleffe and co-workers, guided by the early ideas of Benney and colleagues, looked into the possibility that the Navier–Stokes equations might support wave systems occurring as instabilities of streamwise vortex flows but sufficiently large to drive the vortex flows (see e.g. Waleffe 1995, 1997, 1998, 2001, 2003; Wang *et al.* 2007). The approach used was to find fully nonlinear solutions of the Navier–Stokes equations using various fictitious forces to identify equilibrium states and then continue them when the forcing was switched off. Earlier Nagata (1990) had performed related calculations with the nonlinear flow initially driven by rotation and then connected to a finite-amplitude state as the rotation decreased to zero. Much more detail of these structures was revealed subsequently in, for example, Clever & Busse (1997).

Nagata's study was at the time open to what turned out to be unfounded criticism because of the lack of sufficient computational power to demonstrate the convergence of his results. Note also that the interpretation of the processes taking place in the equilibrium structures identified by Nagata was made clear by the work of Itano & Toh (2001), who conducted a computational investigation into the role of the equilibrium states in the Navier–Stokes dynamics.

The VWI work of Hall & Smith (1991, hereinafter referred to as HS1) built on related work by Hall & Smith (1989, 1990) and laid down a formal asymptotic framework for inviscid or viscous (Tollmien–Schlichting) waves interacting with roll–streak flows. Hall & Sherwin (2010, henceforth referred to as HS2) gave the first results for VWI applied to inviscid waves and showed that structure was the high-Reynolds-number version of what were referred to as lower branch states in the SSP community. This terminology is at odds with the usual one for linear stability theory applied to shear flows where lower branch modes correspond to viscous waves, i.e. Tollmien–Schlichting waves, while upper branch modes refer to waves with critical layers away from any boundaries and which are therefore predominantly inviscid in nature. We note that numerical simulations of fully turbulent flow require a minimum box size for sustained turbulence and this size is consistent with the upper limit of the streamwise wavenumber from the VWI approach of HS2.

It turns out that the lower branch states play a crucial role in delineating between initial perturbations to an undisturbed flow that remain laminar or become turbulent. Thus, for example, if plane Couette flow is perturbed at time zero and the evolution of the flow is calculated numerically, it turns out that a critical size exists for the initial amplitude of a given form of disturbance. Below this size, the flow returns to the laminar flow state; whereas, above it, the flow ultimately becomes turbulent. The nature of these edge states, and their complicated structure, has been investigated for plane Couette flow by Skufca, Yorke & Eckhardt (2006) and Schneider *et al.* (2008) – see also Itano & Toh (2001) and Willis & Kerswell (2009) for other flow configurations.

More recently, Duguet, Schlatter & Henningson (2009) and Schneider, Marinic & Eckhardt (2010*a*) independently carried out time-dependent integrations of the Navier–Stokes equations corresponding to initial streamwise and spanwise spatially localized perturbations to plane Couette flow. The simulations produced results similar to the edge state calculations of, for example, Skufca *et al.* (2006), but the edge state was now found to be also spatially localized, whereas the latter authors found only edge states in small periodic cells. This result suggests that spatially localized lower branch states might be found from either SSP or VWI, and indeed Schneider, Gibson & Burke (2010*b*) found states spatially localized in the spanwise direction. Of interest here is the observation by Duguet *et al.* (2009) that turbulent patches formed during their calculations were elliptical in shape, with the largest axis increasing with Reynolds number.

In a related problem of natural convection, Hall (2012*a*) has found that VWI states there have a tendency to localize as the spanwise wavenumber is decreased. However, no VWI results for localization are available for plane Couette flow, and analytical progress with natural convection was only made possible because VWI there persists to small amplitudes because of the inviscid instability of the unperturbed shear flow.

A notable feature of the results found by Duguet *et al.* (2009) was that the localized states have many similarities with turbulent spots found experimentally or numerically, for example by Lundbladh & Johansson (1991). Thus, in the same way that Navier–Stokes simulations of turbulent plane Couette flow (Gibson, Halcrow &

Cvitanovic 2008) visit equilibrium states, it might be the case that the localized edge states found by Duguet *et al.* (2009) correspond to VWI states localized in the streamwise and spanwise directions and are in some sense attractors for flows exhibiting turbulent spots.

Since we concentrate on VWI theory in this paper, it is worth mentioning the key ideas that underpin the theory in the context of plane Couette flow. The basic idea is that across most of the channel the leading-order flow field consists of a nonlinear x -independent roll–streak flow that evolves over a slow $O(R)$ time scale. The dependence on a slow scale results in the streak component of the flow dominating over the roll. This nearly parallel flow is subjected to a small inviscid wave operating over $O(1)$ length and time scales. A critical layer is formed as a result and, provided the wave is of the appropriate amplitude, nonlinear wave interactions within this region lead to a jump in the roll flow across the critical layer. The roll–streak flow interacts back with the wave via the value of the streamwise shear stress of the streak at the critical layer, so that the system is fully interactive. More details of the process can be found in HS2 and in § 2 of this paper.

The main initial motivation for this work was to resolve a question left open in HS2. The question concerns the fate of inviscid waves participating in VWI in the limit of large streamwise wavelengths. The question is of some importance since the drag on the wall for plane Couette flow, say, has a minimum in the small-wavenumber limit and so, if turbulent flows are to be controlled, then that regime might be of interest. In fact, as we shall see in § 2, the HS2 theory fails for plane Couette flow when the streamwise wavenumber is of size R^{-1} , and at this stage the critical layer associated with the wave thickens and fills the whole channel. We shall derive the appropriate interaction equations for this limit and solve them. In fact, the interaction equations turn out to be the so-called ‘boundary-region equations’ or ‘reduced Navier–Stokes equations’, which are simply the Navier–Stokes equations without the perturbed streamwise pressure gradient and streamwise diffusion.

The interaction equations we derive in the long-wave limit arise in a number of contexts and were derived by Hall (1988) in the context of Görtler vortices but with the Görtler number set equal to zero. The equations are also relevant to high-Reynolds-number flows in microchannels (see e.g. Fletcher 1991; Chen & Kuo 2004). In the context of vortex–wave interactions, the equations were shown by Hall & Smith (1991) to govern the interaction of Tollmien–Schlichting waves and streaks. In the context of boundary-layer transition, the linearized versions of the equations are relevant to the transient growth of streaks (see Luchini 2000; Higuera & Vega 2009), while the full nonlinear equations were solved by Zuccher, Tumin & Reshotko (2006) to compute optimal nonlinear streaks in boundary layers. The linearized version of the equations are also relevant to the receptivity of boundary layers to free stream turbulence (see e.g. Goldstein & Wundrow 1998; Goldstein & Sescu 2008). The key property of these equations is that they are parabolic in the flow direction and elliptic in the spanwise direction.

The long-wave interaction equations we derive are solved numerically at different wavenumbers, and we find that a streamwise localization process appears at sufficiently small streamwise wavenumber. Likewise, for a fixed downstream wavenumber, localization in the spanwise direction occurs as the spanwise wavenumber is decreased. The relationship of this structure with the numerical simulations of Duguet *et al.* (2009) and Schneider *et al.* (2010a) will be discussed. At large values of the scaled streamwise wavenumber, the results are consistent with the HS1 theory. In the latter limit, the wave energy is confined to just the fundamental

mode and the wave amplitude decreases in size compared to the roll flow, thus recovering the asymptotic picture given in HS1.

The procedure adopted in the rest of this paper is as follows. In the following section, we derive the VWI equations in the limit of small streamwise wavenumber: more precisely, we investigate the case when the streamwise wavelength is comparable to the Reynolds number. In §3 we discuss the numerical method used to solve the interaction equations, while in §4 we present and discuss the results and draw some conclusions.

2. Formulation of the long-wave interaction equations

We consider viscous flow of kinematic viscosity ν^* in a channel occupying the region $-h < y^* < h$ in Cartesian coordinates (x^*, y^*, z^*) . The upper wall of the channel moves in the x^* direction with constant velocity U^* , and the lower wall moves with velocity $-U^*$. If we non-dimensionalize our length scales by writing $(x^*, y^*, z^*) = h(\hat{x}, \hat{y}, \hat{z})$ and employ the usual notation for the convective derivative, the Laplacian and the gradient operator, the required momentum and continuity equations take the form

$$\frac{D\hat{\mathbf{u}}}{Dt} = -\nabla\hat{p} + \frac{1}{R}\Delta\hat{\mathbf{u}}, \quad (2.1a)$$

$$\nabla \cdot \hat{\mathbf{u}} = 0. \quad (2.1b)$$

Here the dimensionless quantities $(\hat{u}, \hat{v}, \hat{w})$ represent the fluid velocity and \hat{p} the pressure. The Reynolds number $R = U^*h^*/\nu^*$ and the time t^* is written as $(h/U^*)\hat{t}$. The appropriate no-slip boundary conditions at the channel walls are given by

$$\hat{\mathbf{u}} = (\pm 1, 0, 0) \quad \text{on } \hat{y} = \pm 1. \quad (2.2)$$

In the absence of any disturbance, there exists a simple unidirectional basic solution $(u_b(\hat{y}), 0, 0)$ to (2.1) and (2.2), where

$$u_b(\hat{y}) = \hat{y} \quad (2.3)$$

is the well-known plane Couette flow.

Now we will write down the VWI equations from HS2 appropriate to waves with streamwise and spanwise wavelengths comparable with the channel half-depth h . We first note that the Navier–Stokes equations (2.1) admit a form corresponding to a roll–streak solution of the type discussed in HS2 with

$$\hat{\mathbf{u}} = (u(\hat{y}, \hat{z}, \tau), R^{-1}v(\hat{y}, \hat{z}, \tau), R^{-1}w(\hat{y}, \hat{z}, \tau)), \quad \hat{p} = R^{-2}p(\hat{y}, \hat{z}, \tau), \quad (2.4)$$

which satisfies the Navier–Stokes equations for all R . Here $\tau = R^{-1}\hat{t}$ is a slow time scale and the flow is taken to be periodic in the spanwise direction with wavelength $2\pi/\hat{\beta}$. We now follow the analysis of HS1 and HS2 and, assuming that the Reynolds number is large, we perturb the steady form of the roll–streak flow by writing

$$\hat{\mathbf{u}} = (u(\hat{y}, \hat{z}, \tau), R^{-1}v(\hat{y}, \hat{z}, \tau), R^{-1}w(\hat{y}, \hat{z}, \tau)) + \rho(\tau)R^{-7/6}U(\hat{y}, \hat{z}, \tau)E + \text{c.c.} + \dots, \quad (2.5a)$$

$$\hat{p} = R^{-2}p(\hat{y}, \hat{z}, \tau) + \rho(\tau)R^{-7/6}P(\hat{y}, \hat{z}, \tau)E + \text{c.c.} + \dots, \quad (2.5b)$$

$$E \equiv \exp\{i\hat{\alpha}(\hat{x} - R \int c(\tau) d\tau)\}, \quad (2.5c)$$

so that the wave propagates in the \hat{x} direction with instantaneous speed c and has wavenumber $\hat{\alpha}$. The wave pressure P satisfies a linear equation ((2.9) below) and is normalized at each value of τ such that $\iint |P|^2 d\hat{y}d\hat{z} = 1$. Here the integral is taken across the whole channel over one spanwise wavelength. The $O(1)$ quantity $\rho(\tau)$ represents the wave amplitude after normalization and is to be determined. Here we shall concentrate on waves with zero wave speed, but for completeness we shall initially show the form of the solution for travelling waves, which can exist in the present case or when the flow is driven by a pressure gradient. The size of the wave, $O(R^{-7/6})$, is fixed as in HS1 and HS2 in such a way that $O(1)$ jumps in the roll stresses are induced across the critical layer associated with the wave. The roll-streak field then satisfies

$$u_\tau + vu_{\hat{y}} + wu_{\hat{z}} = u_{\hat{y}\hat{y}} + u_{\hat{z}\hat{z}}, \tag{2.6a}$$

$$v_\tau + vv_{\hat{y}} + wv_{\hat{z}} + p_{\hat{y}} = v_{\hat{y}\hat{y}} + v_{\hat{z}\hat{z}}, \tag{2.6b}$$

$$w_\tau + vw_{\hat{y}} + ww_{\hat{z}} + p_{\hat{z}} = w_{\hat{y}\hat{y}} + w_{\hat{z}\hat{z}}, \tag{2.6c}$$

$$v_{\hat{y}} + w_{\hat{z}} = 0, \tag{2.6d}$$

subject to

$$u = \pm 1, \quad v = w = 0 \quad \text{on } \hat{y} = \pm 1. \tag{2.7}$$

Away from the critical layer, the wave satisfies the linearized system

$$i\hat{\alpha}(u - c)U + V u_{\hat{y}} + W u_{\hat{z}} = -i\hat{\alpha}P, \tag{2.8a}$$

$$i\hat{\alpha}(u - c)V = -P_{\hat{y}}, \tag{2.8b}$$

$$i\hat{\alpha}(u - c)W = -P_{\hat{z}}, \tag{2.8c}$$

$$i\hat{\alpha}U + V_{\hat{y}} + W_{\hat{z}} = 0, \tag{2.8d}$$

where $U = (U, V, W)$. We see that at high Reynolds numbers the wave depends only on the streak velocity field and, following Hall & Horseman (1991), elimination of the wave velocity field leads to the wave pressure eigenvalue problem

$$P_{\hat{y}\hat{y}} + P_{\hat{z}\hat{z}} - \hat{\alpha}^2 P - \frac{2u_{\hat{y}} P_{\hat{y}}}{u - c} - \frac{2u_{\hat{z}} P_{\hat{z}}}{u - c} = 0 \quad \text{with } P_{\hat{y}} = 0 \text{ on } \hat{y} = \pm 1. \tag{2.9}$$

In the spanwise direction we require that P is periodic. Throughout the interaction the wave is taken to be instantaneously neutral, so that the wave speed $c(\tau)$ remains real. The coupling between the wave and roll-streak flow is completed in the critical layer, which occupies a region of width $R^{-1/3}$ around the location $\hat{y} = f(\hat{z}, \tau)$ where $u = c$. Here we will restrict our attention to the case when the wave speed is zero, and we will see that the special symmetries of the solution we construct lead to a flat critical layer with $f = 0$. In the critical layer, we can deduce from (2.8) that the wave velocity components increase by $O(R^{1/3})$, and the wave interacts with itself to drive the roll flow. When the roll flow inside the critical layer is matched to the outer flow, we see that the result of the wave driving the roll is to produce jumps in the stresses associated with the roll for the flow outside the critical layer. The velocity and pressure fields associated with the wave, roll and streak within the critical layer are found in exactly the same manner as given in HS1 and HS2, so we do not repeat the analysis here. The roll velocity components (v, w) together with u and $u_{\hat{y}}$ are continuous across the critical layer. However, following HS1 and HS2, we find that the Reynolds stresses associated with the wave lead to the following jumps for the roll

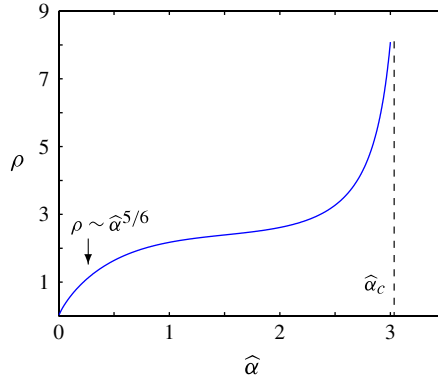


FIGURE 1. (Colour online) Wave amplitude ρ versus wavenumber $\hat{\alpha}$ from the VWI calculations of HS2, showing the critical wavenumber $\hat{\alpha}_c$.

flow across the layer:

$$[w_{\hat{y}}]_{-}^{+} = \frac{[v_{\hat{y}}]_{-}^{+}}{f_{\hat{z}}} = J(\hat{z}, \tau), \quad (2.10a)$$

$$J = \left(\frac{2}{3}\right)^{2/3} \Gamma\left(\frac{1}{3}\right) \frac{2\pi\rho^2}{a^{5/3}\Theta^5} \left\{ \left(-\frac{7\Theta_{\hat{z}}}{2\Theta} - \frac{5a_{\hat{z}}}{3a}\right) |P_{\hat{z}}|^2 + \frac{\partial}{\partial \hat{z}} |P_{\hat{z}}|^2 \right\}, \quad (2.10b)$$

$$[p]_{-}^{+} = -\left(\frac{2}{3}\right)^{2/3} \Gamma\left(\frac{1}{3}\right) \frac{2\pi\rho^2}{a^{5/3}\Theta^5} f_{\hat{z}\hat{z}} |P_{\hat{z}}|^2, \quad (2.10c)$$

where $\Theta = 1 + f_{\hat{z}}^2$, $a = \mu\hat{\alpha}/\Theta$ and μ is $\partial u/\partial \hat{y}$ evaluated at $\hat{y} = f(\hat{z}, \tau)$. The \pm signs denote values immediately above and below the critical layer, and the wave pressure P in (2.10) is to be evaluated on the critical layer. As explained in HS2, the easiest way to understand the origin of the jumps is by thinking of the flow in the critical layer as a steady streaming problem. Thus, in the critical layer, the wave field becomes relatively large and its interaction with itself generates Reynolds stresses that cause the roll velocity component parallel to the critical layer to have a jump in its normal derivative, while the momentum equation normal to the layer produces a jump in the pressure. Thus the Reynolds stresses basically produce jumps in the roll stresses normal and parallel to the critical layer. Finally, in this discussion of HS1 and HS2, we note that the wave has a viscous layer at the walls of the same thickness as the critical layer. No jumps are produced there at leading order because the streamwise momentum equation shows that the leading-order wave pressure is zero in these layers. Full details of the numerical procedure needed to solve the nonlinear eigenvalue problem $\rho = \rho(\hat{\alpha}, \hat{\beta})$ can be found in HS2. For given values of the streamwise and spanwise wavenumbers $\hat{\alpha}$ and $\hat{\beta}$, we can find ρ , and figure 1 shows the typical behaviour with equilibrium states possible for $0 < \hat{\alpha} < \hat{\alpha}_c$ for some critical wavenumber $\hat{\alpha}_c$. It was also found in HS2 that the mean drag on the walls increases monotonically with $\hat{\alpha}$, with the minimum drag therefore occurring at $\hat{\alpha} = 0$, according to this theory.

When $\hat{\alpha}$ is sufficiently small, however, the VWI scalings given in (2.5) break down. We can see this by first noting that, for small values of wavenumber, we can deduce

from the jump conditions (2.10) and the wave equations (2.8) that

$$\rho \sim \hat{\alpha}^{5/6}, \quad U \sim \hat{\alpha}^{-2}, \quad V \sim W \sim \hat{\alpha}^{-1} \quad \text{as } \hat{\alpha} \rightarrow 0. \tag{2.11}$$

The distinguished scaling for $\hat{\alpha}$ at which we enter a new regime can be most easily seen by comparing the contributions to the pressure from the roll and the wave in (2.5b). These terms become comparable in size when $R^{-2} \sim \rho R^{-7/6}$, and hence upon use of (2.11) we see that a new stage is encountered when

$$R^{-2} \sim \hat{\alpha}^{5/6} R^{-7/6} \implies \hat{\alpha} \sim R^{-1}. \tag{2.12}$$

At this same stage, the critical layer, formerly of thickness $O(\hat{\alpha}R)^{-1/3}$ under the HS1 scalings, enlarges to fill the whole channel, so that the wave is controlled by viscous effects everywhere. Note also that, away from the critical layer, the wave velocity components in the $\hat{y} - \hat{z}$ plane, which are of $O(R^{-7/6}\hat{\alpha}^{-1/6})$ from (2.5) and (2.11) as $\hat{\alpha} \rightarrow 0$ in the HS1 regime, increase to size $O(R^{-1})$, and are now the same magnitude as the roll velocity components. Similarly, an examination of (2.5a) shows that the streamwise component of the wave, which in the small-wavenumber limit is of order $(\hat{\alpha}R)^{-7/6}$ from (2.11), becomes $O(1)$ and is therefore now comparable with the streak velocity. Thus, when the wave has wavelength comparable with the Reynolds number, it becomes viscous throughout the channel and the VWI mechanism of HS1 fails. In the new regime, the separation of scales between the wave velocity components and the underlying roll-streak flow disappears and the wave now forces the roll across the whole of the channel. In fact, the new streamwise length scale is recognized as being the usual streamwise length scale over which Görtler vortices develop in curved flows (see Hall 1988). Hence the roll-streak flow that develops in the $\hat{\alpha} \sim R^{-1}$ case can be thought of as a Görtler vortex driven by the stresses associated with the wave part of the flow, rather than by curvature. Note, however, that, for flows where Tollmien-Schlichting waves are possible, an intermediate interaction problem occurs much along the lines of the viscous VWI case described in HS1.

In view of the above discussion, we now seek a solution of the Navier-Stokes equations in the form

$$\hat{\mathbf{u}} = (u(x, y, z, \tau), R^{-1}v(x, y, z, \tau), R^{-1}w(x, y, z, \tau))(1 + O(R^{-2})), \tag{2.13a}$$

$$\hat{p} = R^{-2}p(x, y, z, \tau) + O(R^{-4}), \tag{2.13b}$$

$$\tau = \hat{t}/R, \quad x = \hat{x}/R, \quad y = \hat{y}, \quad z = \hat{z}, \tag{2.13c}$$

and substitution into (2.1) yields

$$\frac{D\mathbf{u}}{D\tau} = -(0, p_y, p_z) + (\partial_{yy}^2 + \partial_{zz}^2)\mathbf{u}, \tag{2.14a}$$

$$\nabla \cdot \mathbf{u} = 0 \tag{2.14b}$$

for $\mathbf{u} = (u, v, w)$, where

$$\frac{D}{D\tau} \equiv \frac{\partial}{\partial \tau} + \mathbf{u} \cdot \nabla, \quad \nabla \equiv (\partial_x, \partial_y, \partial_z). \tag{2.15}$$

Thus the leading-order approximation to the VWI now reduces to the Navier-Stokes equations with the perturbed streamwise pressure gradient and streamwise diffusion negligible. However, the crucial and indeed remarkable simplification in this limit is that the effective Reynolds number is now scaled to unity.

In addition to the vortex–Rayleigh wave application emphasized earlier in this paper, the equations studied here arise in many other instability problems involving flow at high Reynolds numbers. The equations themselves are easily generalized to a cylindrical geometry using the scalings (2.13), together with an $O(1)$ azimuthal variation. They take the form

$$u_x + v_r + v/r + w_\theta/r = 0, \quad (2.16a)$$

$$u_\tau + uu_x + vu_r + wu_\theta/r = K + \nabla^2 u, \quad (2.16b)$$

$$v_\tau + uv_x + vv_r + wv_\theta/r - w^2/r = -p_r + \nabla^2 v - v/r^2 - 2w_\theta/r^2, \quad (2.16c)$$

$$w_\tau + uw_x + vw_r + ww_\theta/r + vw/r = -p_\theta/r + \nabla^2 w - w/r^2 + 2v_\theta/r^2, \quad (2.16d)$$

where $\nabla^2 \equiv \partial_{rr}^2 + r^{-1}\partial_r + r^{-2}\partial_{\theta\theta}^2$, together with the no-slip condition on the boundaries, and regularity conditions along the pipe axis. The constant K represents the axial pressure gradient associated with the basic flow. The above equations therefore control long-wavelength lower branch equilibrium structures in Hagen–Poiseuille flow. The linearized version of (2.16) was shown to govern the neutral stability in pipes of general cross-section with aspect ratios of $O(1)$ by Smith (1979), but no numerical solution was given, although a multi-region asymptotic solution of (2.16) is believed to exist in the limit of large aspect ratio. The stability of developing flows is another area where (2.16) is relevant. For example, the linear stability of spatially developing pipe and channel flows is governed by modes with an $O(R)$ wavelength, comparable to the length scale over which the basic flow is developing (Smith & Bodonyi 1980). The corresponding neutral modes in temporally developing pipe flow exhibit the same behaviour (Walton 2002). A further situation where the linearized version of (2.16) is relevant is the stability of the axial flow in a concentric annulus, where the equations describe the travelling-wave instability close to the cut-off in radius ratio or sliding velocity (Gittler 1993; Walton 2004).

Returning now to the planar case, the major simplifications of (2.14) compared to the Navier–Stokes equations are that they do not contain the Reynolds number, and the absence of a non-constant streamwise pressure gradient and streamwise diffusion renders them parabolic in x . Here we are interested in steady solutions of these equations that are periodic in x and z . For large values of the scaled streamwise wavenumber α , we also expect the solutions to match onto the small-wavenumber ($\hat{\alpha} \rightarrow 0$) limit of the VWI problem of HS2. The major difficulty associated with these equations compared to the VWI case at $O(1)$ streamwise wavenumbers is that the roll–streak flow and wave cannot be separated and linked through jump conditions, and indeed the wave can no longer be regarded as linear since all of its harmonics are now important.

3. Numerical method

Here we describe a numerical scheme to obtain travelling-wave solutions of (2.14). The scheme is based on the tried-and-tested method outlined in Nagata (1990), in which the potential decomposition

$$\hat{u} = \bar{u} + \partial_{\hat{x}\hat{y}}^2 \hat{\phi} + \partial_{\hat{z}} \hat{\psi}, \quad (3.1a)$$

$$\hat{v} = -\partial_{\hat{x}\hat{x}}^2 \hat{\phi} - \partial_{\hat{z}\hat{z}}^2 \hat{\phi}, \quad (3.1b)$$

$$\hat{w} = \bar{w} + \partial_{\hat{y}\hat{z}}^2 \hat{\phi} - \partial_{\hat{x}} \hat{\psi} \quad (3.1c)$$

is used for the full Navier–Stokes equations (2.1), and algebraic equations are calculated explicitly for their spectral coefficients and phase velocities. Here we seek the travelling-wave solution that has wavenumber $\hat{\alpha}$, phase velocity \hat{c} in the streamwise direction and wavenumber $\hat{\beta}$, phase velocity \hat{c}_z in the spanwise direction. The operator

$$\bar{*} \equiv \frac{\hat{\alpha}\hat{\beta}}{4\pi^2} \int_0^{2\pi/\hat{\alpha}} \int_0^{2\pi/\hat{\beta}} * \, d\hat{x} \, d\hat{z} \tag{3.2}$$

represents a spatial average over \hat{x} and \hat{z} . We generate solutions to our problem (2.14) by starting with known solutions of the full Navier–Stokes equations at finite Reynolds numbers, $(\hat{\phi}, \hat{\psi}, \bar{u}, \bar{w}, \hat{c}, \hat{c}_z)$, and using a homotopy to connect them to solutions of the required system. With that in mind, we consider the modified equations

$$\frac{D\mathbf{u}}{D\tau} = -(\epsilon p_x, p_y, p_z) + \Delta_\epsilon \mathbf{u}, \tag{3.3a}$$

$$\nabla \cdot \mathbf{u} = 0, \tag{3.3b}$$

$$\Delta_\epsilon \equiv \epsilon \partial_{xx}^2 + \partial_{yy}^2 + \partial_{zz}^2, \tag{3.3c}$$

with $\epsilon \equiv R^{-2}$. This augmented system is simply a rescaled form of the full Navier–Stokes equations (2.1), while upon setting $\epsilon = 0$ we recover the system (2.14). To solve (3.3), the most natural definition of scaled potentials and mean flows would be $(\hat{\phi}, \hat{\psi}, \bar{u}, \bar{w}) = (R^{-1}\phi, \psi, \bar{u}, R^{-1}\bar{w})$, which does not change the form of (3.1) except for some factors of ϵ after the scaling. However, a minor change is necessary because in the limit $\epsilon \rightarrow 0$ one of the spanwise-averaged vorticity components,

$$\Omega \equiv \frac{\hat{\beta}}{2\pi} \int_0^{2\pi/\hat{\beta}} \{ \partial_x v - \partial_y(u - \bar{u}) \} \, d\hat{z} = -\epsilon \frac{\hat{\beta}}{2\pi} \int_0^{2\pi/\hat{\beta}} (\partial_{xxx}^3 + \partial_{yyy}^3) \phi \, d\hat{z}, \tag{3.4}$$

vanishes identically and the Jacobian matrix appearing in our Newton procedure will therefore be singular. In order to account for this, we use a modified scaling for the spanwise-averaged poloidal potential and write

$$u = \bar{u} + \epsilon \partial_{xy}^2 \phi + \partial_z \psi + \partial_y \varphi, \tag{3.5a}$$

$$v = -\epsilon \partial_{xx}^2 \phi - \partial_{zz}^2 \phi - \partial_x \varphi, \tag{3.5b}$$

$$w = \bar{w} + \partial_{yz}^2 \phi - \partial_x \psi. \tag{3.5c}$$

The relationship between the solutions of the Navier–Stokes equations (2.1) and the augmented system (3.3) is

$$(\hat{\phi} - \bar{\phi}, \partial_{\hat{x}} \hat{\phi}, \hat{\psi}, \bar{u}, \bar{w}, \hat{c}, \hat{c}_z) = (R^{-1}\phi, \varphi, \psi, \bar{u}, R^{-1}\bar{w}, c, R^{-1}c_z), \tag{3.6}$$

where $\bar{\phi} = (\hat{\beta}/2\pi) \int_0^{2\pi/\hat{\beta}} \hat{\phi} \, d\hat{z}$. It is easy to confirm that $\Omega = -(\partial_{xx}^2 + \partial_{yy}^2)\varphi$ does not vanish at $\epsilon = 0$ and so the problem of the singular Jacobian has been overcome.

The governing equations in terms of potentials are found by considering the operations $\mathbf{e}_y \cdot \nabla_\epsilon \times \nabla \times$ (3.3a), $\mathbf{e}_y \cdot \nabla_\epsilon \times$ (3.3a), $\mathbf{e}_x \cdot$ (3.3a) and $\mathbf{e}_z \cdot$ (3.3a), and take the form

$$(c\partial_x + c_z\partial_z + \Delta_\epsilon)\Delta_\epsilon(\Delta_\epsilon^{2D}\phi + \partial_x\varphi) + \Delta_\epsilon((\mathbf{u} \cdot \nabla)v) - \partial_y(\nabla \cdot ((\mathbf{u} \cdot \nabla)\mathbf{u})) = 0, \tag{3.7a}$$

$$(c\partial_x + c_z\partial_z + \Delta_\epsilon)\Delta_\epsilon^{2D}\psi + \epsilon\partial_x((\mathbf{u} \cdot \nabla)w) - \partial_z((\mathbf{u} \cdot \nabla)u) = 0, \tag{3.7b}$$

$$\partial_{yy}^2 \bar{u} - \partial_y \bar{u} = 0, \tag{3.7c}$$

$$\partial_{yy}^2 \bar{w} - \partial_y \bar{v} \bar{w} = 0, \quad (3.7d)$$

$$\phi = \partial_y \phi = \varphi = \partial_y \varphi = \psi = \bar{w} = 0, \quad \bar{u} = \pm 1 \quad \text{on } y = \pm 1, \quad (3.7e)$$

where $\nabla_\epsilon \equiv (\epsilon \partial_x, \partial_y, \partial_z)$ and $\Delta_\epsilon^{2D} \equiv \epsilon \partial_{xx}^2 + \partial_{zz}^2$. Here a Galilean transformation has been applied in the x and z directions so that the problem becomes time-independent. For numerical purposes, the potentials and mean flows are approximated by the truncated spectral expansions

$$\phi = \sum_{m=-M}^M \sum_{\substack{n=-N \\ n \neq 0}}^N \sum_{l=0}^L X_{lmn}^{(1)} \Phi_l(y) e^{im\alpha x} e^{in\beta z}, \quad (3.8a)$$

$$\varphi = \sum_{\substack{m=-M \\ m \neq 0}}^M \sum_{l=0}^L \alpha^{-1} X_{lm0}^{(1)} \Phi_l(y) e^{im\alpha x}, \quad (3.8b)$$

$$\psi = \sum_{\substack{m=-M \\ (m,n) \neq (0,0)}}^M \sum_{n=-N}^N \sum_{l=0}^L \alpha^{-1} X_{lmn}^{(2)} \Psi_l(y) e^{im\alpha x} e^{in\beta z}, \quad (3.8c)$$

$$\bar{u} = y + \sum_{l=0}^L X_{l00}^{(1)} \Psi_l(y), \quad (3.8d)$$

$$\bar{w} = \sum_{l=0}^L X_{l00}^{(2)} \Psi_l(y), \quad (3.8e)$$

with α and β the scaled wavenumbers in the x and z directions, and where $\Phi_l(y) \equiv (1 - y^2)^2 T_l(y)$ and $\Psi_l(y) \equiv (1 - y^2) T_l(y)$ are modified l th Chebyshev polynomials of the first kind. Note also that, because of the rescaling in x given in (2.13), the wavelength in the streamwise direction is $2\pi/\alpha = 2\pi R/\hat{\alpha}$, whereas the spanwise scale is unchanged, i.e. we have

$$\hat{\alpha} = \alpha R, \quad \hat{\beta} = \beta. \quad (3.9)$$

Numerically we find that equilibrium solutions typically have $\alpha \sim O(10^2)$: since $O(\phi) \sim O(\alpha\varphi) \sim O(\alpha\psi)$ as $\epsilon \rightarrow 0$, we introduce the scaling factor α^{-1} in (3.8b) and (3.8c) in order to improve the condition number of the Jacobian matrix. If we now make a Galerkin projection $\langle * e^{-im_0\alpha x} e^{-in_0\beta z} \Phi_{l_0}(y) \varrho(y) \rangle$ for (3.7a) and $\langle * e^{-im_0\alpha x} e^{-in_0\beta z} \Psi_{l_0}(y) \varrho(y) \rangle$ for (3.7b), (3.7c) and (3.7d), with volumetric average operator $\langle * \rangle \equiv \frac{1}{2} \int_{-1}^1 * dy$ and weight function $\varrho(y) \equiv 2/(\pi\sqrt{1-y^2})$, we obtain a set of algebraic equations $F_i = D_{ij}X_j + H_{ijk}X_jX_k = 0$ as found in Nagata (1990).

Given a solution of (2.1) we can generate values of the solution vector X_j by using the relation (3.6). Therefore, in principle, we can obtain a finite-amplitude solution of (3.3c) for $\epsilon = 0$ by a homotopy method starting from some known solution. The solution is continued by Newton's method, i.e. X_j is updated as $X_j^{l+1} = X_j^l + q\delta X_j^l$ at each Newton iteration by solving the linear problem

$$G_{ij}^l \delta X_j^l = -F_i^l \quad (3.10)$$

by LAPACK package DGESVX. Here $G_{ij}^l = D_{ij} + 2H_{ijk}X_k^l$, the Jacobian matrix, and $F_i^l = D_{ij}X_j^l + H_{ijk}X_j^lX_k^l$ are computed by using double-precision FORTRAN code. The

coefficient q is usually chosen as 1, but if the prediction is overestimated, q is chosen as some smaller value. This scheme is iterated until the convergence criterion

$$r = \max_j r_j < 10^{-5} \tag{3.11}$$

is attained, where

$$r_j = \begin{cases} \left| \frac{X_j^{l+1} - X_j^l}{X_j^l} \right| & \text{if } |X_j^{l+1}| \text{ and } |X_j^l| > 10^{-10}, \\ 0 & \text{otherwise} \end{cases} \tag{3.12}$$

is satisfied.

The lower branch mirror-symmetric steady solution of (2.1) found by Gibson, Halcrow & Cvitanovic (2009) and Itano & Generalis (2009) is used as input to the scaled system (3.7) to provide the starting point of the homotopy. It is known that this type of solution is stationary, i.e. $c = c_z = 0$, and has the following symmetries:

$$[u, v, w](x, y, z) = [u, v, -w](x + \pi/\alpha, y, -z), \tag{3.13a}$$

$$[u, v, w](x, y, z) = [-u, -v, w](-x, -y, z + \pi/\beta), \tag{3.13b}$$

$$[u, v, w](x, y, z) = [u, v, w](x + \pi/\alpha, y, z + \pi/\beta). \tag{3.13c}$$

By using these symmetries, all the redundant unknowns are removed from the system to reduce the size of the calculation. For further cost reduction, we use an elliptic cut-off method, i.e. the coefficients and equations that satisfy $[l/(L + 1)]^2 + [m/(M + 1)]^2 + [n/(N + 1)]^2 > 1$ are omitted. We choose the solution at $R = 700$ with optimum wavenumber pair $(\hat{\alpha}, \hat{\beta}) = (0.75, 1.37)$ as the starting point of the homotopy of the augmented system (3.3) for $(\alpha, \beta) = (525, 1.37)$. Decreasing ϵ from $700^{-2} \simeq 2.04 \times 10^{-6}$, the branch successfully crosses $\epsilon = 0$ using a relatively low resolution. Then at $\epsilon = 0$, spectral accuracy is refined.

We set $\epsilon = 0$ henceforth. We consider the mean shear stress on the wall,

$$\Delta = \partial_y \bar{u}|_{y=1} = 1 - 2 \sum_{l=0}^L X_{l00}^{(1)}, \tag{3.14}$$

and the L_2 -norm of the spectral coefficients,

$$\|X_j\|_2 = \sqrt{\sum_{j=1}^{\dim} |X_j|^2}, \tag{3.15}$$

as measures of the nonlinear solutions.

The result of resolution tests for $\alpha = 500$ are shown in table 1. Since our convergence criterion does not involve the value of $\|F_j\|_2$, we show this together with its linear part $\|D_{jk}X_k\|_2$ in the table. Readers who are concerned about the signal-to-noise-floor ratio of the solution can easily confirm that our convergence criterion assures a reasonably small normalized residual $\|F_j\|_2 / \|D_{jk}X_k\|_2$. The results presented in table 1 suggest that an increase in resolution cannot remove fluctuation in the third digit in terms of Δ and in the fourth digit in terms of $\|X_j\|$. In order to convince the reader about the accuracy of our calculations, we make a couple of observations. The first concern is that of the ill-conditioned Jacobian matrix associated with the Newton iteration. It is found that the typical condition number of this problem is around 10^{13} – 10^{15} . However, the use of quadruple precision and/or a stricter convergence

(L, M, N)	dim	$\Delta - 1$	$\ X_j\ _2$	$\ D_{jk}X_k\ _2$	$\ F_j\ _2$
(60, 30, 34)	35 066	4.145×10^{-2}	12.9422	1.24×10^3	8.08×10^{-12}
(70, 16, 30)	19 766	4.168×10^{-2}	12.9699	1.26×10^3	1.04×10^{-11}
(70, 18, 32)	23 495	4.116×10^{-2}	12.9072	1.21×10^3	1.77×10^{-8}
(70, 18, 34)	24 955	4.113×10^{-2}	12.9029	1.21×10^3	8.81×10^{-12}
(85, 16, 30)	23 904	4.135×10^{-2}	12.9292	1.23×10^3	8.77×10^{-11}
(85, 18, 32)	28 402	4.131×10^{-2}	12.9243	1.22×10^3	1.60×10^{-11}
(85, 18, 34)	30 148	4.128×10^{-2}	12.9217	1.22×10^3	2.52×10^{-11}
(95, 16, 30)	26 653	4.125×10^{-2}	12.9179	1.22×10^3	3.19×10^{-8}
(95, 18, 32)	31 687	4.116×10^{-2}	12.9585	1.25×10^3	5.23×10^{-9}
(95, 18, 34)	33 613	4.136×10^{-2}	12.9306	1.23×10^3	2.86×10^{-9}

TABLE 1. The convergence of the long-wave high- R limit solution. Here $(\alpha, \beta) = (500, 1.37)$.

(L, M, N)	dim	$\Delta - 1$	$\ X_j\ _2$	$\ D_{jk}X_k\ _2$	$\ F_j\ _2$
(95, 18, 34)	33 613	1.396×10^{-2}	4.6817	1.73×10^3	7.04×10^{-9}
(80, 24, 34)	37 417	1.108×10^{-2}	4.2844	1.08×10^3	1.67×10^{-8}
(70, 30, 34)	40 754	1.128×10^{-2}	4.3108	1.12×10^3	4.00×10^{-9}
(70, 34, 28)	38 125	1.113×10^{-2}	4.3090	1.12×10^3	1.24×10^{-11}

TABLE 2. The convergence of the long-wave high- R limit solution. Here $(\alpha, \beta) = (100, 1.37)$.

criterion for the Newton iterations did not alter the leading digits of the results. The other concern is the convergence property of the spectral expansion, because of the lack of dissipation in the x direction. To study the spectral convergence against the truncation level L, M, N , the following partial sums are introduced:

$$S_l = \sqrt{\sum_{m=-M}^M \sum_{n=-N}^N |X_{lmn}^{(1)}|^2 + |X_{lmn}^{(2)}|^2}, \tag{3.16a}$$

$$S_m = \sqrt{\sum_{l=0}^L \sum_{n=-N}^N |X_{lmn}^{(1)}|^2 + |X_{lmn}^{(2)}|^2}, \tag{3.16b}$$

$$S_n = \sqrt{\sum_{l=0}^L \sum_{m=-M}^M |X_{lmn}^{(1)}|^2 + |X_{lmn}^{(2)}|^2}. \tag{3.16c}$$

Note that $S_m = S_{-m}$ and $S_n = S_{-n}$. We can see from figure 2 that, as the truncation level is increased, the magnitude of the coefficients is almost monotonically decreasing, whereas the leading-order terms are unchanged except for a tiny variation. Hence we can conclude that the current computation is reliable within the realm of the indicated error levels. The truncation level $(L, M, N) = (70, 16, 30)$ is considered to be reliable for $\alpha = 500$. It is found, however, that we need more harmonics in the x direction as α is decreased. We exhibit another resolution test at $\alpha = 100$ (table 2) and set $(L, M, N) = (70, 34, 28)$ for $\alpha < 100$. We can see that the quantity $\|F_j\|_2 / \|D_{jk}X_k\|_2$ also stays sufficiently small in this case. However, as α is decreased

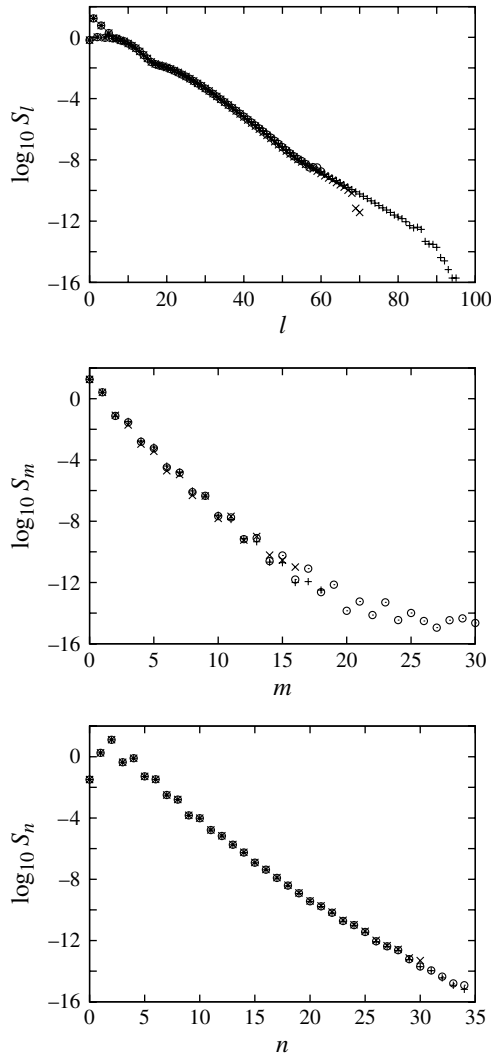


FIGURE 2. The partial sums of the spectral coefficients. Here $(\alpha, \beta) = (500, 1.37)$. The spectral convergence: points \times , $+$ and \circ are calculated by $(L, M, N) = (70, 16, 30)$, $(L, M, N) = (95, 18, 34)$ and $(L, M, N) = (60, 30, 34)$, respectively.

and M increased, the condition number becomes worse and some coefficients never achieve the convergence criterion. For this reason, at small α , we have to switch to a convergence criterion expressed in terms of $\|F_j\|_2 / \|D_{jk}X_k\|_2$.

The use of the perturbed Newton’s method introduced recently by Peris, Marquina & Candela (2011) helps to achieve a decrease in $\|F_j\|_2$. In this method, the correction δX_j^I for $I \neq 0$ is solved by using the perturbed Jacobian matrix $\tilde{G}_{ij}^I = D_{ij} + 2H_{ijk}\tilde{X}_k^I$ instead of the exact Jacobian matrix G_{ij}^I , where \tilde{X}_j^I is the perturbed solution obtained by $G_{ij}^0(\tilde{X}_j^I - X_j^I) = -F_j^I$. Obviously, the success of this method depends on the proximity of the initial condition to the desired solution. We perturbed X_j^0 by an $O(10^{-7})$ random disturbance during the tracing of the bifurcation curves, and confirmed that the

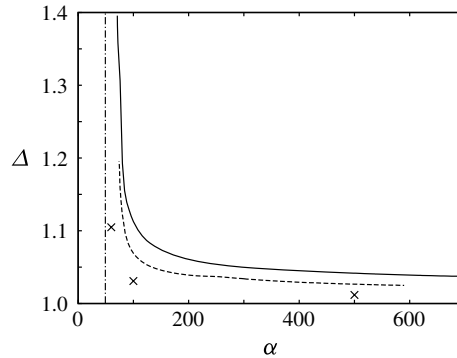


FIGURE 3. The mean wall shear Δ . The solid and dashed curves are $\beta = 1.37$ and 1, respectively. The vertical dash-dotted line is the two-dimensional energy threshold. The crosses represent the results for $\beta = 0.5$, where resolution is chosen as $(L, M, N) = (70, 40, 40)$ for $\alpha = 60$ and 100, whereas $(L, M, N) = (75, 24, 50)$ for $\alpha = 500$.

fluctuation during the iterations did not affect the present accuracy of Δ . The quantity $\|F_j\|_2 / \|D_{jk}X_k\|_2$ is $O(10^{-6})$ at worst for the results we present in the next section.

4. Results and discussion

Once the results for $\beta = 1.37$ were found, then α was fixed and β varied until $\beta = 1$. For this new value of β , we then varied α . We then varied β with $\alpha = 100$ until $\beta = 0.5$, but it is difficult to vary α with this spanwise wavenumber because (i) we need to increase N to 40 and the computation then takes an inordinate amount of time and (ii) the ill-conditioned nature of the Newton method described in the previous section becomes worse and it becomes difficult to obtain reliable solutions for small α . In view of this difficulty, we show only three results for this value of β . The results of these calculations for different β are shown in figure 3, where the mean wall shear Δ is plotted as a function of the scaled wavenumber α . For large values of α , the shear approaches a constant value greater than the basic flow value of unity, consistent with the prediction of the VWI approach described in HS2. An examination of the velocity field showed that, as in HS2, only the fundamental mode in the streamwise direction was significant. We conclude, then, that the minimum drag for lower branch states is predicted by the zero streamwise wavenumber limit of HS2. For smaller values of the scaled wavenumber α , we see that the shear increases rapidly, and the results suggest that it becomes infinite at a finite value of $\alpha = \alpha_c \sim 70$ for the case $\beta = 1.37$. This suggests that, as well as the minimum box size for equilibrium states already predicted by HS2, there is also a critical size given by $2\pi R/\alpha_c$ at which a significant change in the nature of the equilibrium states occurs; the nature of that change will become clear in the following discussion.

We expect the result described above for $\beta = 1.37$ to be true for all values of the spanwise wavenumber for which the HS2 theory applies. We certainly expect that the two curves shown are representative and that, for each β , the shear approaches the HS2 value at large α and blows up at some value of α in the region 50–70. In figure 3 we also show as a vertical line the value of α below which energy theory predicts stability (see the Appendix for more details); our solutions can therefore not cross this line.

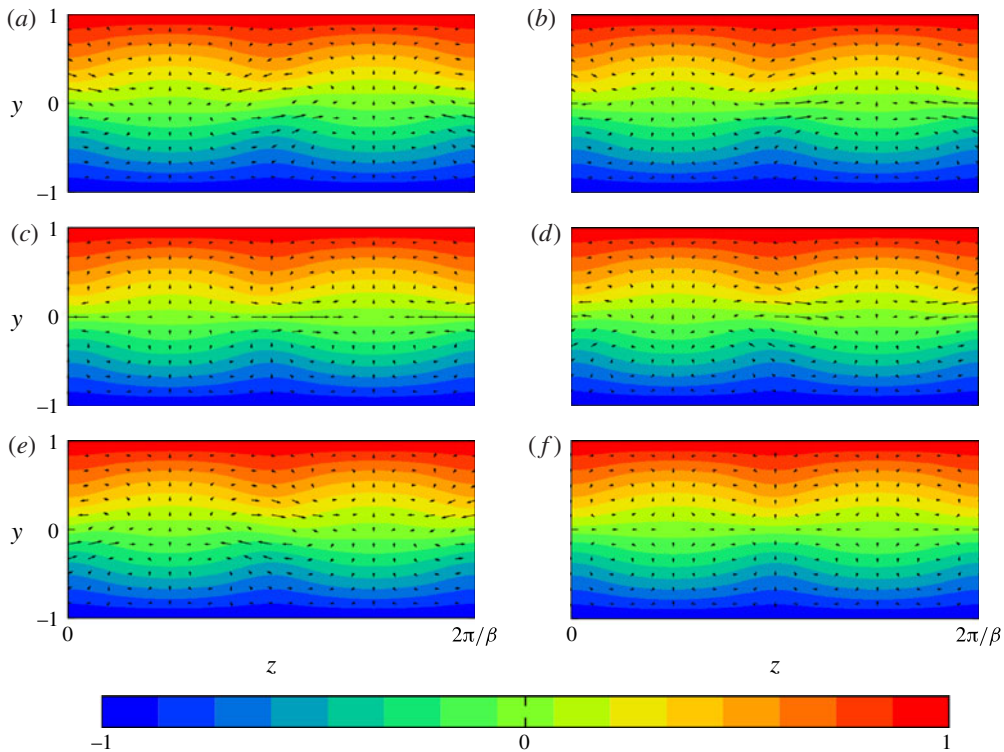


FIGURE 4. The velocity field \mathbf{u} for $(\alpha, \beta) = (500, 1.37)$. The colours represent the streamwise component u , whereas the arrows are cross-sectional components v and w . (a–e) Streamwise cross-sections at $x = (n/8)(2\pi/\alpha)$, with $n = 0, 1, 2, 3$ and 4 , respectively. (f) The streamwise-averaged field.

Figures 4, 5 and 6 show contour plots of the velocity components for the cases $\alpha = 500$ and 71 with $\beta = 1.37$. Figure 6(b) offers a clue to the change in the nature of the equilibrium state as the limiting wavenumber α_c is approached. We refer to the fact that there appears to be a localization developing in the streamwise direction of the disturbance structure. This suggests that the energy of the disturbance is moving into the higher streamwise harmonics as the limiting solution is approached. In order to demonstrate that this is indeed the case, we show in figure 7 plots of the energy in the different modes as α is decreased. For the highest value of α , we see that the energy is concentrated in the fundamental as predicted by the HS2 theory. At the lower value of the wavenumber we now see that the energy is spreading to the higher harmonics: this accounts for the increasing localization of the flow in the streamwise direction.

In order to see the localization more clearly, we have in figure 8 plotted contours of the streamwise velocity component in the mid-plane $y = 0$ for $\alpha = 500, 300, 100$ and 71 with $\beta = 1.37$. Note that the unperturbed flow is zero here so the figure shows how the equilibrium x -velocity component develops a localization in the streamwise direction. Note also that in our discussion it should be remembered that the streamwise scale should be multiplied by R so that figure 8 represents elongated structures that develop small regions where the activity becomes concentrated as α decreases. Figure 9 shows similar results for the case $\alpha = 100$, but now for two smaller values of spanwise wavenumber $\beta = 1.0$ and 0.5 . We now observe that the localization is also

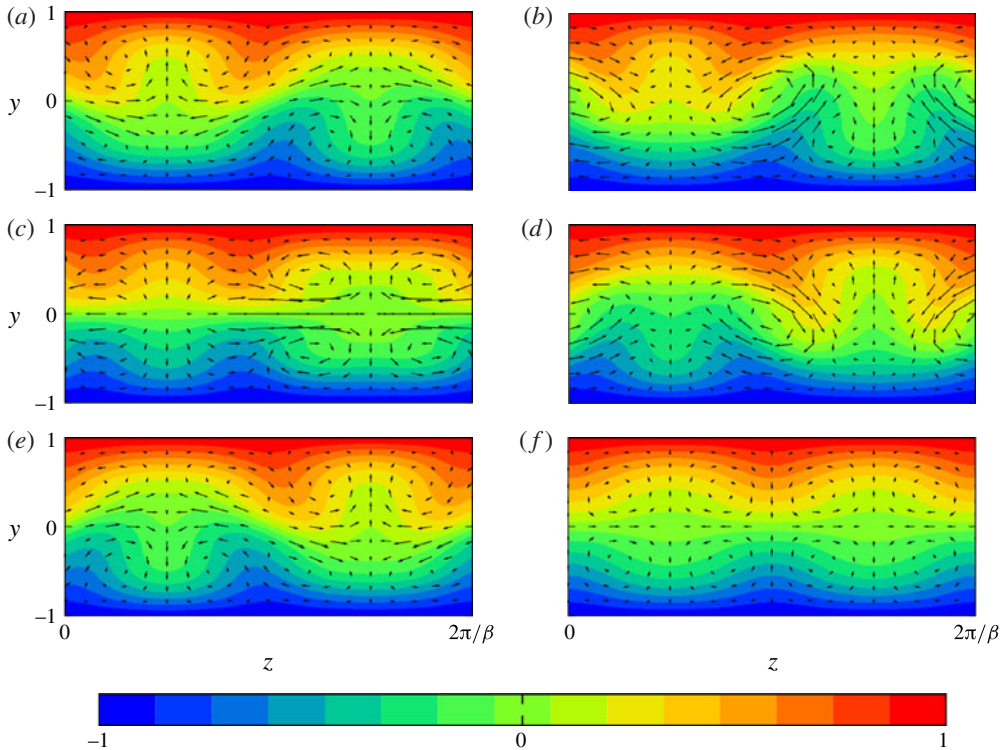


FIGURE 5. The velocity field \mathbf{u} for $(\alpha, \beta) = (71, 1.37)$. The colours represent the streamwise component u , whereas the arrows are cross-sectional components v and w . (a–e) Streamwise cross-sections at $x = (n/8)(2\pi/\alpha)$, with $n = 0, 1, 2, 3$ and 4 , respectively. (f) The streamwise-averaged field.

taking place in the spanwise direction as β is decreased. This localization can perhaps be even more clearly seen in figure 10 where, for a range of values of α and β , we show the regions where the total streamwise vorticity is greater than 80 per cent of its maximum absolute value.

The short-scale structures have similar shapes to turbulent spots that have been observed experimentally in plane Couette flow (see e.g. Tillmark & Alfredsson 1992; Tillmark 1995). Turbulent spots have been simulated computationally by, for example, Lundbladh & Johansson (1991). Thus it is natural to expect some connection between these spots and our solutions.

Further evidence for that possibility can be found in recent papers by Duguet *et al.* (2009) and Schneider *et al.* (2010a). The latter authors conducted direct numerical simulations of plane Couette flow, restricting flow dynamics in some symmetric subspace. The initial state was taken to be localized in the spanwise and streamwise directions, and by carefully altering the initial states the authors were able to generate ‘edge states’, which have similarities with turbulent spots. In the same way as the edge state calculations of Schneider *et al.* (2008), who did not impose symmetry, generated states similar to the lower branch state found by Wang *et al.* (2007) numerically and by HS2 asymptotically, we might conclude that the localized states found in this paper are related to the states tracked by Duguet *et al.* (2009) and Schneider *et al.* (2010a). We note, however, that the latter two publications concentrated on

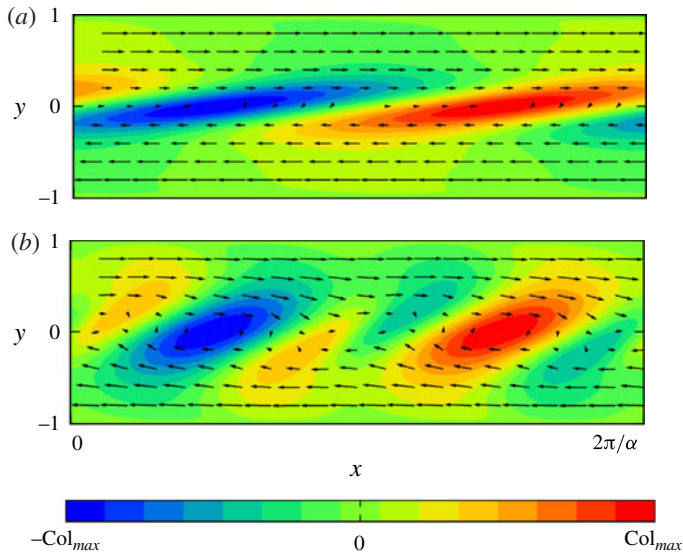


FIGURE 6. The velocity field \mathbf{u} at $z = 0$. The colours represent the spanwise component w , whereas the arrows are cross-sectional components u and v/α : (a) $(\alpha, \beta) = (500, 1.37)$, $\text{Col}_{\max} = 18.9$; (b) $(\alpha, \beta) = (71, 1.37)$, $\text{Col}_{\max} = 36.7$.

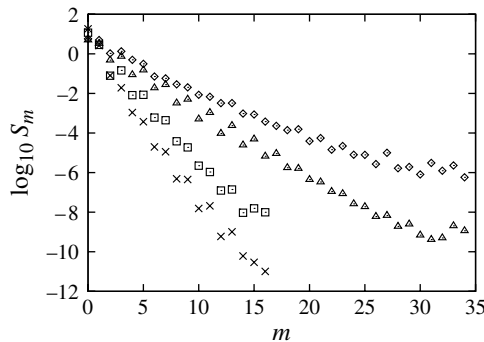


FIGURE 7. The partial sums of the spectral coefficients. Here $(\alpha, \beta) = (500, 1.37)$. The spectral convergence: points \times , \square , \triangle and \diamond are for $\alpha = 500, 300, 100$ and 71 , respectively. $(L, M, N) = (70, 16, 30)$ is used for $\alpha = 500$ and 300 , whereas $(L, M, N) = (70, 34, 28)$ is used for $\alpha = 100$ and 71 .

initial perturbations localized in both the streamwise and spanwise directions, whereas our results for $\beta = 1.37$ and 1.0 correspond to states that are just localized in the streamwise direction, but certainly the result shown for $\beta = 0.5$ demonstrates the localization beginning to occur in the spanwise direction as β decreases.

For plane Couette flow with downstream wavenumbers of $O(1)$, we know that the lower branch states of HS2 or equivalently those of Wang *et al.* (2007) are visited in unsteady Navier–Stokes simulations of turbulence (see Gibson *et al.* 2008). The nature of the localized solutions we have uncovered suggests that the interaction problem we have formulated has streamwise-localized states, which might well be weakly unstable

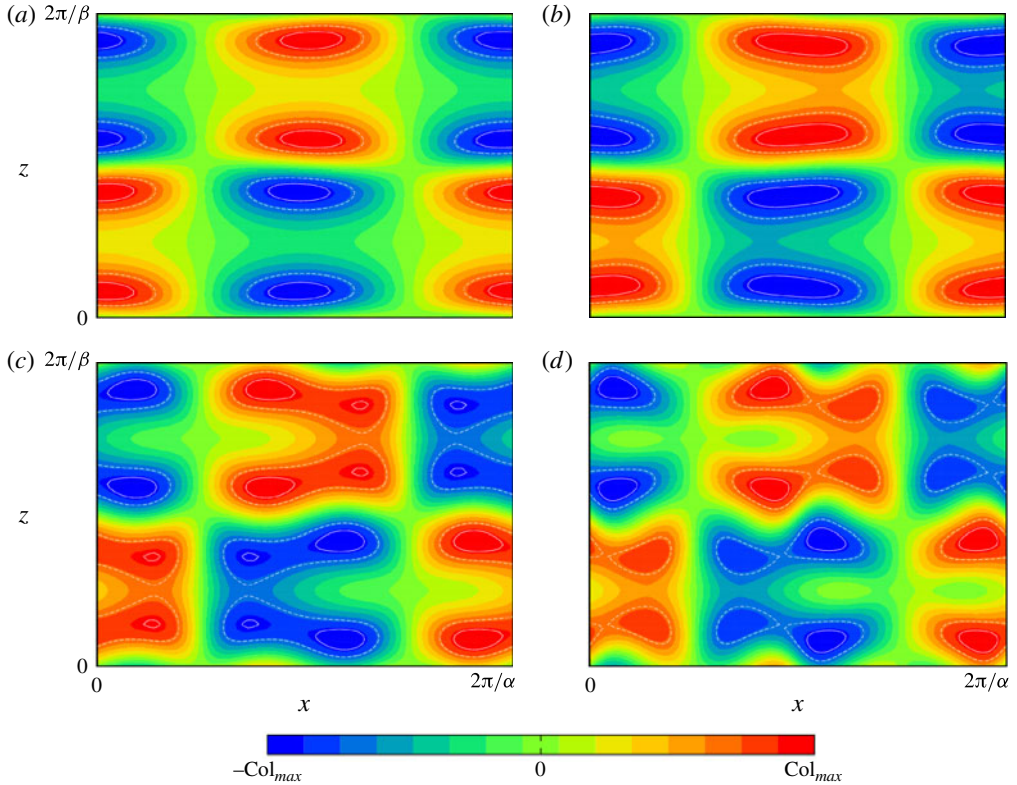


FIGURE 8. The streamwise velocity field u at the mid-gap $y = 0$. The solid and dashed white curves are iso-contours of $\pm 90\%$ Col_{max} and $\pm 70\%$ Col_{max} , respectively. (a) $(\alpha, \beta) = (500, 1.37)$, $Col_{max} = 0.045$; (b) $(\alpha, \beta) = (300, 1.37)$, $Col_{max} = 0.059$; (c) $(\alpha, \beta) = (100, 1.37)$, $Col_{max} = 0.165$; (d) $(\alpha, \beta) = (71, 1.37)$, $Col_{max} = 0.332$.

saddle-like points for trajectories of the unsteady Navier–Stokes equations. What we are speculating, of course, is that, in the same way as fully turbulent flows apparently need some backbone of equilibrium structures on which to hang, flows that exhibit turbulent spots similarly need a backbone of solutions exhibiting localization on which to hang.

We note that periodic solutions (see e.g. Kawahara & Kida 2001) are also of huge importance in the turbulent dynamics of an evolving flow. Similarly, the localized solutions we have found are likely to be just part of a number of solutions that may play some role in the evolution of turbulent spots.

For the Reynolds numbers at which Navier–Stokes simulations are possible, upper branch solutions are also visited and are thought to relate to upper bounds for the turbulence and dissipation (Gibson *et al.* 2008). It is not yet known whether upper branch modes occur at high values of R , since no equivalent theory to HS2 has been developed for them, and finite-Reynolds-number calculations fail to follow them when R is large. Thus one might argue about the relevance of our solutions to turbulent spots. However, by analogy with the recent discovery of homoclinic orbits of van Veen & Kawahara (2011) in plane Couette flow, we can expect orbits homoclinic to our lower branch state, which can mimic the localized burst of a turbulent spot.

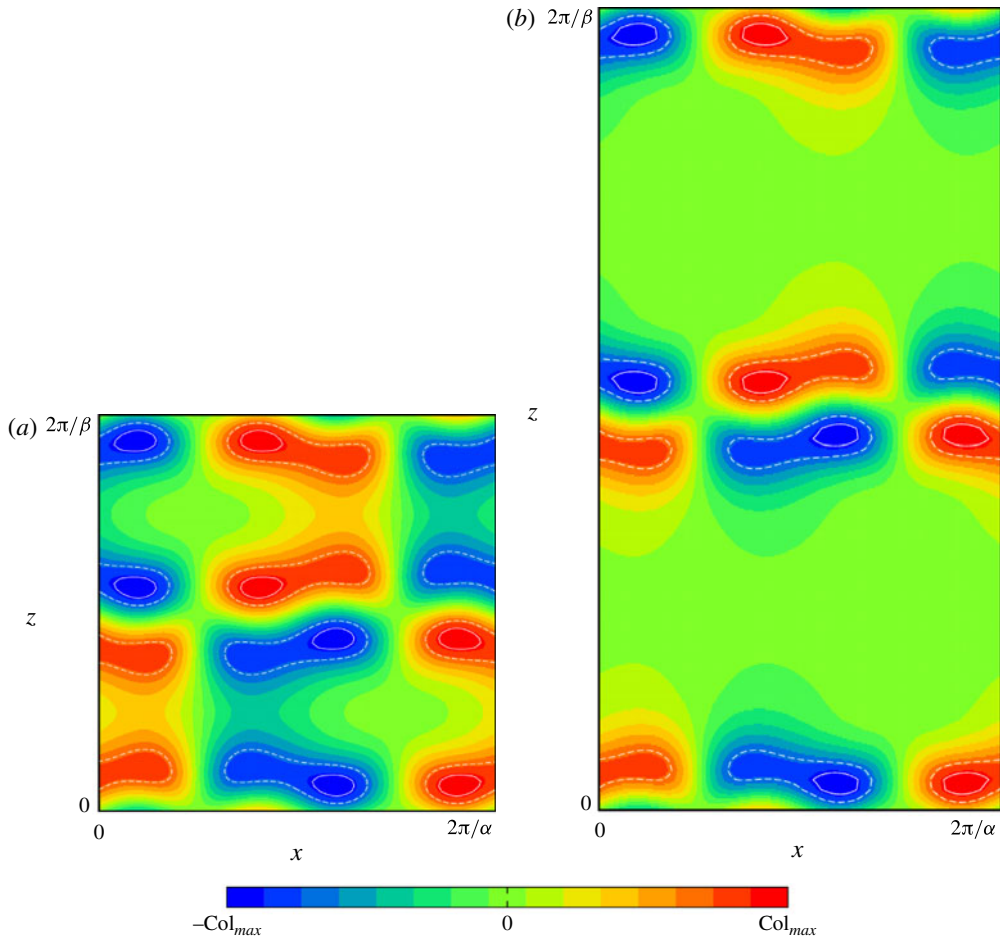


FIGURE 9. The streamwise velocity field u at the mid-gap $y = 0$. The solid and dashed white curves are iso-contours of $\pm 90\%$ Col_{max} and $\pm 70\%$ Col_{max} respectively. (a) $(\alpha, \beta) = (100, 1)$, $Col_{max} = 0.148$; (b) $(\alpha, \beta) = (100, 0.5)$, $Col_{max} = 0.148$.

Figure 11 shows further properties of the localized solutions, but this time using the normal velocity component in the plane $y = 0$. The parameter values in figure 11(a–c) correspond to those of figure 10, and once again demonstrate the simultaneous spatial localization of the velocity field in both directions. If we concentrate on the most active regions, we see alternating spots (denoted by dark red or blue) aligned along the centre of the channel. Given that the length of the channel has been scaled with the Reynolds number, our results are quite consistent with the description given by Duguet *et al.* (2009, see figure 2) of elliptically shaped spots of length increasing with Reynolds number. Thus it is possible that the edge states possibly associated with turbulent spots found by Duguet *et al.* (2009) are related to the localized equilibrium structures that we have found.

Figure 12 shows similar results using a slightly different representation and now showing the effect at Reynolds numbers of 500 and 1000. The flow structure illustrated here is not unlike that shown in, for example, Schneider *et al.* (2010a, figure 5) and Duguet *et al.* (2009, figure 3). Further evidence that the equilibrium

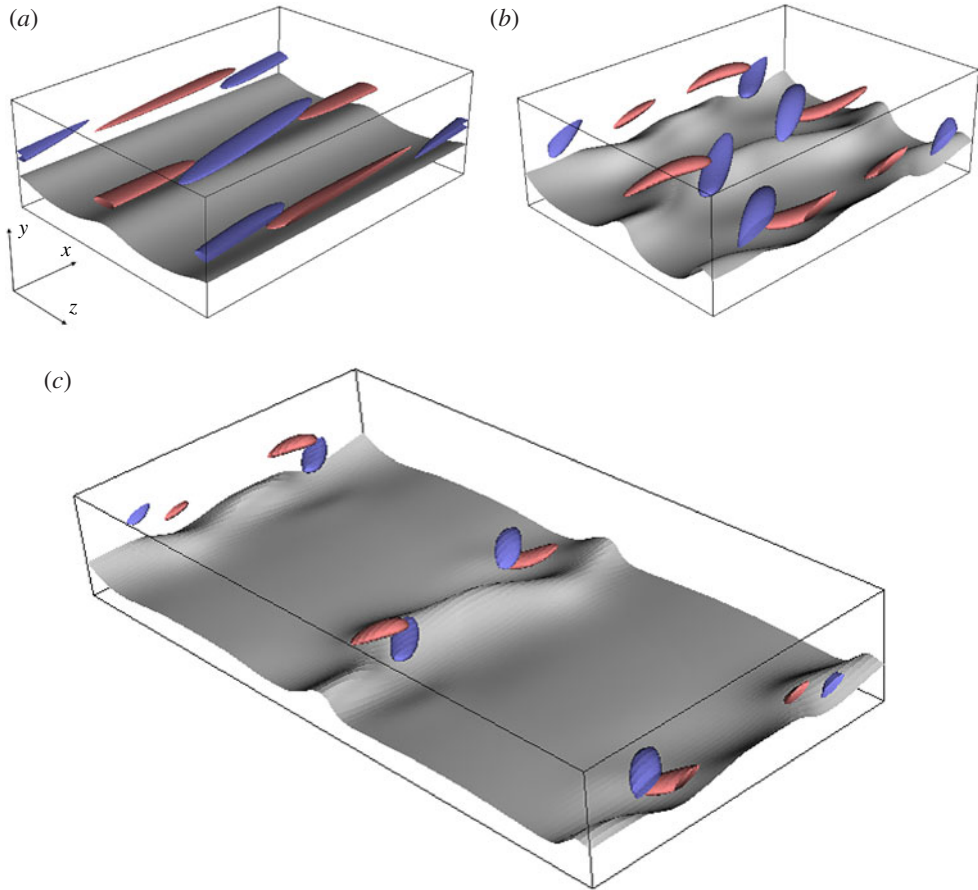


FIGURE 10. The grey surface represents an isosurface of 50% of the maximum magnitude of the streamwise velocity in the lower half of the channel. The red/blue bubbles are the isosurface of 80% of the maximum/minimum of the streamwise vorticity. (a) $(\alpha, \beta) = (500, 1.37)$; (b) $(\alpha, \beta) = (71, 1.37)$; (c) $(\alpha, \beta) = (60, 0.5)$.

structures that we have found are related to the edge states computed from the Navier–Stokes equations can be found by consideration of the energy of the flow. Schneider *et al.* (2010a, figure 5) plots the average kinetic energy of the flow. The averages are taken over the y – z or x – y planes, and it was found that the localization in z corresponds to ‘double-exponential’ decay while the streamwise localization corresponds to just exponential decay. In figure 13 we plot the average in x – y of the energy in the streamwise velocity component $(u - u_b)^2$, and the energy associated with the flow in the y – z plane, $v^2 + w^2$. Note that, since the velocity components in the y – z plane have been scaled with $1/R$, the energy of the flow is dominated by the streamwise velocity component (in the energy plots, these velocity components are divided by α instead). We see that, when β decreases from 1.0 to 0.5, the energy in the streamwise direction becomes localized in a manner consistent with Schneider *et al.* (2010a, figure 5(c)). Interestingly, this localization is already present when $\beta = 1.0$ for the v – w flow in the y – z plane. Figure 14 shows similar results when the averages are taken in the y – z plane. Once again, we observe localization

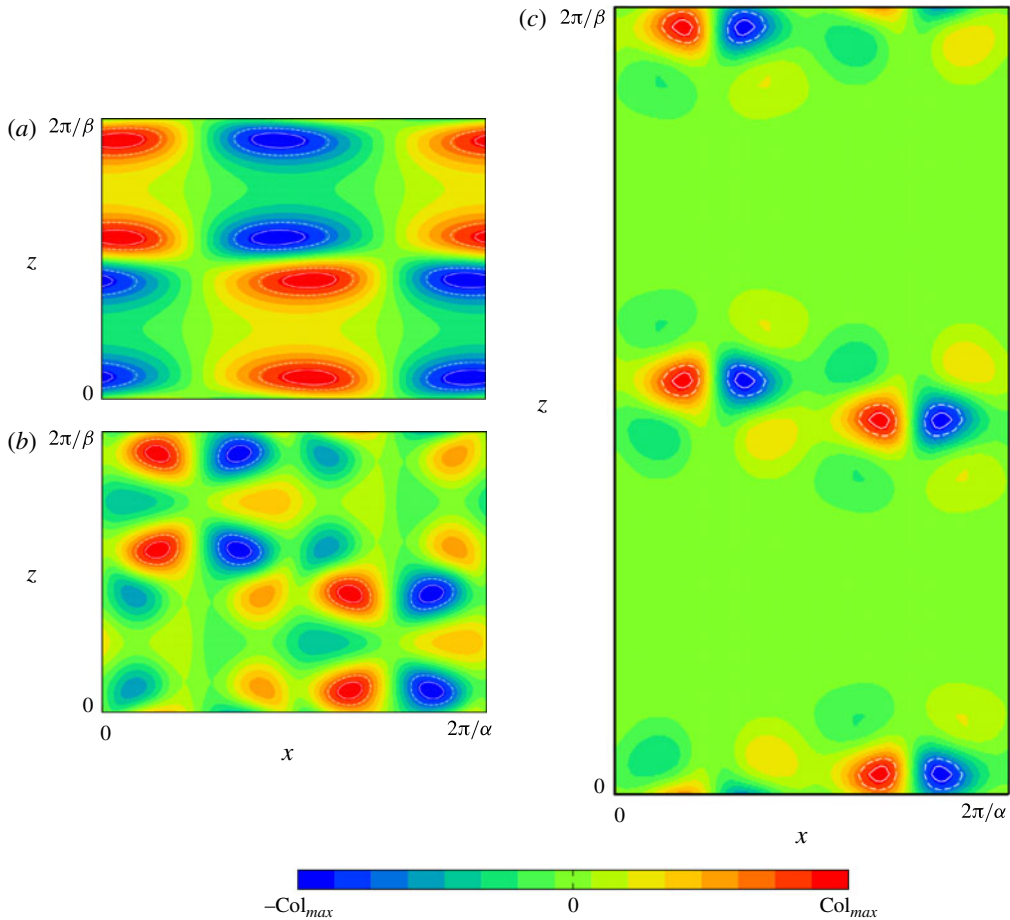


FIGURE 11. The wall-normal velocity field v at the mid-gap $y = 0$. The solid and dashed white curves are iso-contours of $\pm 90\%$ Col_{max} and $\pm 70\%$ Col_{max} , respectively. (a) $(\alpha, \beta) = (500, 1.37)$, $Col_{max} = 2.10$; (b) $(\alpha, \beta) = (71, 1.37)$, $Col_{max} = 17.09$; (c) $(\alpha, \beta) = (60, 0.5)$, $Col_{max} = 15.19$.

when α changes from 500 to 71. The first of these plots shows similar behaviour to Schneider *et al.* (2010a, figure 5(b)), with the localization in x not as pronounced as that observed in the spanwise direction. These results suggest very strongly that the edge states found from the unsteady Navier–Stokes equations have a very similar form to the equilibrium states found here.

Figure 15(a) shows the mean (over both x and z) streamwise velocity component at different values of α with $\beta = 1.37$. We observe that, for the higher values of α , the mean-flow correction changes sign within $[0, 1]$ whereas, as the localization develops, the mean-flow correction is always of one sign. However, as the streamwise wavenumber decreases, the mean-flow correction becomes S-shaped. Figure 15(b) shows the mean-flow correction for a fixed $\alpha = 100$ at different values of β . The results suggest strongly that the mean-flow correction in the limit $\beta \rightarrow 0$ is proportional to $\beta S(y)$, where $S(y)$ is close to the solid line in that figure. For any β , this limiting profile, when added to the basic state, has an inflexion point at

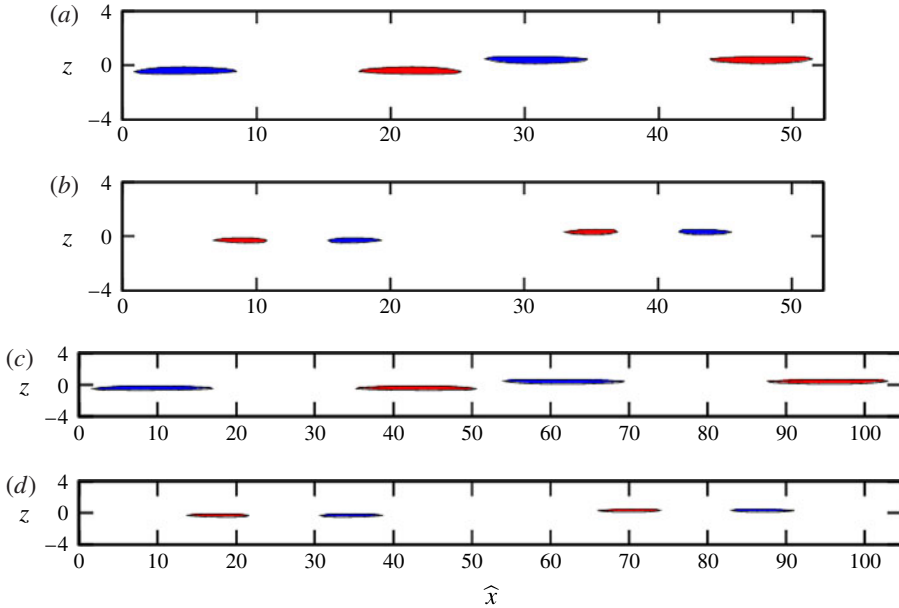


FIGURE 12. Red/blue represent the regions where the velocity exceeds 80 % of its in-plane maximum/minimum. (a) Streamwise component u on $y = 0$. (b) Wall-normal component v on $y = 0$. The solution at $(\alpha, \beta) = (60, 0.5)$ is used. The streamwise coordinate is rescaled to the original variable \hat{x} by using $R = 500$. Panels (c) and (d) show the same as (a) and (b) but for $R = 1000$.

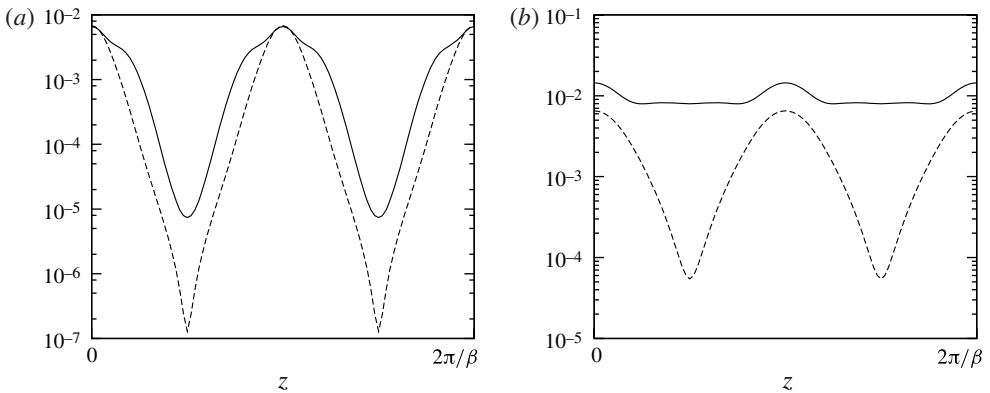


FIGURE 13. Averages in x - y of the energy associated with the perturbation: (a) $(\alpha, \beta) = (100, 0.5)$; (b) $(\alpha, \beta) = (100, 1.0)$. The solid line shows the energy associated with the streamwise velocity, $(u - u_b)^2$, while the dashed line shows the energy associated with the flow in the y - z plane, $v^2 + w^2$, divided by α^2 .

the origin so is itself unstable to inviscid waves of wavelength comparable with the channel half-depth h . Thus it is not surprising that the localization develops when the higher harmonics corresponding to wavelengths comparable with h in the streamwise direction grow. However, the implication of the canonical scaling for the mean-flow

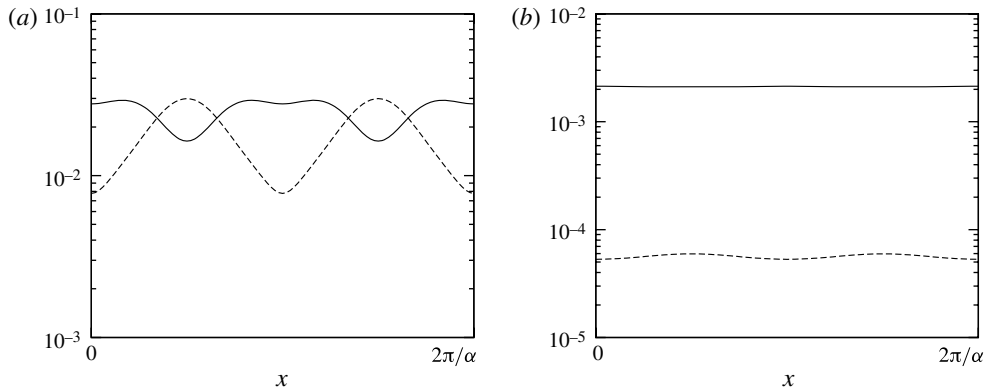


FIGURE 14. Averages in y - z of the energy associated with the perturbation: (a) $(\alpha, \beta) = (71, 1.37)$; (b) $(\alpha, \beta) = (500, 1.37)$. The solid line shows the energy associated with the streamwise velocity, $(u - u_b)^2$, while the dashed line shows the energy associated with the flow in the y - z plane, $v^2 + w^2$, divided by α^2 .

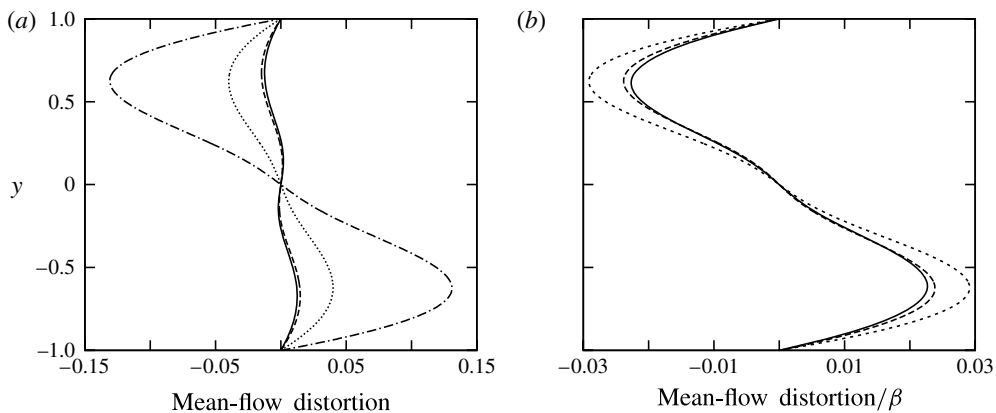


FIGURE 15. Mean-flow distortion $(\bar{u} - u_b)$: (a) the dash-dotted, dotted, dashed and solid curves are $(\alpha, \beta) = (71, 1.37)$, $(100, 1.37)$, $(300, 1.37)$ and $(500, 1.37)$, respectively; (b) the dotted, dashed and solid curves are $(\alpha, \beta) = (100, 1.37)$, $(100, 1)$ and $(100, 0.5)$, respectively.

correction and the relation to fully turbulent mean profiles, while intriguing, is not pursued here. In fact, the limiting form of the mean-flow correction agrees well with turbulent mean profiles for plane Couette flow (see e.g. Komminaho, Lundblad & Johansson 1996, figure 2). It appears, then, that the basic shape for the mean-flow correction in a turbulent boundary layer is driven by the large-scale structures described here.

Now let us turn to a discussion of the significance of the critical value of α at which the equilibrium structures that we calculate develop spatial localization. We assume that the value of $\alpha \simeq 70$, where the $\beta = 1.0$ and 1.37 solutions localize, is in fact the limit as $\beta \rightarrow 0$ of the values of α where localization develops for a given β . We saw earlier that, as this develops, the energy cascades down into the higher harmonics, and formally our analysis will fail when there is $O(1)$ energy

in the harmonics of order R . At that stage, our original assumption used to derive our interaction equations fails and, at the very least locally, the full Navier–Stokes equations must be solved, and the full detailed spot structures found by Duguet *et al.* (2009) and Schneider *et al.* (2010a) cannot be found without doing so. However, the existence of a critical wavenumber for a given spanwise wavelength at which short-scale structures can develop predicts a critical minimum box length needed for localized structures possibly related to turbulent spots to form. The existence of such a minimum length for spot formation has been known by the computational community for some time (see e.g. Lundbladh & Johansson 1991; Komminaho *et al.* 1996). If the result that we find is indeed a predictor of the box length needed to allow spot formation to begin, it is significant because for any high Reynolds number it tells us how long the computational box used must be in order to capture turbulent spots. In that case it can be argued that VWI predicts successfully two of the key results found by computational experimentation regarding turbulent flows. Firstly, the upper $O(1)$ streamwise wavenumber limit predicted in HS2 for VWI states to exist at high R apparently is related to the minimum box size needed to sustain turbulent flow. This would infer that fully turbulent flows can occur only if a backbone of equilibrium states such as those described by VWI exist. Secondly, the present calculation clearly shows that equilibrium localized states only develop when the streamwise wavenumber decreases to a critical value. This corresponds to the box length increasing to a critical value proportional to the Reynolds number before localization can occur. If the localization is indeed the embryonic stage of a turbulent spot, then we have predicted the minimum box length needed for spot formation. The discussion in Lundbladh & Johansson (1991) and indeed in Duguet *et al.* (2009) is certainly consistent with that notion.

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Appendix. The energy stability limit for the long-wave interaction equations

Here we apply standard energy analysis arguments (e.g. Joseph & Carmi 1969) to the long-wave interaction equations (2.14) in order to derive a bound on the scaled wavenumber α , below which all disturbances must monotonically decay.

First, we write $\mathbf{u} = (y + \tilde{u}, \tilde{v}, \tilde{w})$, $p = \tilde{p}$ and substitute into (2.14). The fluctuations are then governed by

$$\frac{D\tilde{\mathbf{u}}}{D\tau} + (y\tilde{u}_x + \tilde{v}, y\tilde{v}_x, y\tilde{w}_x) = -(0, \tilde{p}_y, \tilde{p}_z) + (\partial_{yy}^2 + \partial_{zz}^2)\tilde{\mathbf{u}}, \quad (\text{A } 1)$$

$$\nabla \cdot \tilde{\mathbf{u}} = 0, \quad (\text{A } 2)$$

and are subject to zero boundary conditions on $y = \pm 1$ and periodic boundary conditions in x and z with wavenumbers α and β , respectively. If we consider a periodic box, we can derive the equivalent of the Reynolds–Orr energy equation in the form

$$\frac{DE}{D\tau} = I_1 - I_2, \quad (\text{A } 3)$$

where we have defined

$$E = \frac{1}{2} \langle \tilde{u}^2 \rangle, \quad I_1 = -\langle \tilde{u}\tilde{v} \rangle, \quad I_2 = \langle \tilde{u}_y^2 + \tilde{u}_z^2 \rangle > 0, \quad (A 4)$$

with $\langle * \rangle$ the volumetric average operator defined in § 3. It follows that, if $DE/D\tau > 0$ over some interval in τ , then we require $M = \max m > 1$, where $m = I_1/I_2$. Suppose that the solution that maximizes m is $\tilde{\mathbf{u}}_0 = (\tilde{u}_0, \tilde{v}_0)$. We then consider all other solutions in the form $(\tilde{u}, \tilde{v}) = \tilde{\mathbf{u}}_0 + \varepsilon(\tilde{u}_1, \tilde{v}_1)$ with corresponding energy ratio $m(\varepsilon)$. By definition, $M = m(0)$ and $m'(0) = 0$. Since $m(\varepsilon)I_2(\varepsilon) = I_1(\varepsilon)$, we have

$$M = I_1'(0)/I_2'(0). \quad (A 5)$$

It is readily established that

$$I_1'(0) = -\langle \tilde{\mathbf{u}}_1 \cdot (\tilde{v}_0 \mathbf{e}_x + \tilde{u}_0 \mathbf{e}_y) \rangle, \quad I_2'(0) = -2\langle \tilde{\mathbf{u}}_1 \cdot (\tilde{u}_{0yy} + \tilde{u}_{0zz}) \mathbf{e}_x \rangle, \quad (A 6)$$

and hence

$$\langle \tilde{\mathbf{u}}_1 \cdot \{2M(\tilde{u}_{0yy} + \tilde{u}_{0zz})\mathbf{e}_x - (\tilde{v}_0 \mathbf{e}_x + \tilde{u}_0 \mathbf{e}_y)\} \rangle = 0 \quad (A 7)$$

for arbitrary solenoidal fields $\tilde{\mathbf{u}}_1$. It follows that the expression in $\{ \}$ can be written in potential form $\nabla\phi$, and thus the maximizing state is governed by

$$-\tilde{v}_0 + 2M(\tilde{u}_{0yy} + \tilde{u}_{0zz}) = \phi_x, \quad -\tilde{u}_0 = \phi_y, \quad 0 = \phi_z, \quad (A 8)$$

together with $\nabla \cdot \tilde{\mathbf{u}}_0 = 0$. The solution of the energy problem that maximizes m is therefore two-dimensional (in contrast to our computed solutions of (2.14), which are, of course, strongly three-dimensional).

Introducing a streamfunction $\psi(y) \exp(i\alpha x)$ to satisfy continuity, we find that

$$\psi'''' + \frac{i\alpha}{M}\psi' = 0, \quad \psi = \psi' = 0 \quad \text{on } y = \pm 1. \quad (A 9)$$

Integration and application of the boundary conditions leads to the eigenvalue equation

$$(1 + e^{-2q\sqrt{3}}) \cos(q) - 2e^{-q\sqrt{3}} \cos(2q) - \sqrt{3}(1 - e^{-2q\sqrt{3}}) \sin(q) = 0 \quad (A 10)$$

for q , where $q^3 = \alpha/M$. Since we require $M > 1$ for energy growth, we need $\alpha > q_0^3$, where q_0 is the smallest positive root of (A 10). From computation, $q_0 \simeq 3.666$, and hence the lower bound on α is given by

$$\alpha > q_0^3 \simeq 49.272. \quad (A 11)$$

As remarked in the main text, the full numerical simulations of (2.14) generate results for values of α that are only slightly in excess of this number, indicating that this bound is surprisingly sharp.

REFERENCES

- BENNEY, D. 1984 The evolution of disturbances in shear flows at high Reynolds numbers. *Stud. Appl. Maths* **70**, 1–19.
- CHEN, C. S. & KUO, W. J. 2004 Heat transfer characteristics of gaseous flow in long mini- and microtubes. *Numer. Heat Transfer A* **46**, 497–514.
- CLEVER, R. M. & BUSSE, F. H. 1997 Tertiary and quaternary solutions for plane Couette flow. *J. Fluid Mech.* **344**, 137–153.
- DUGUET, Y., SCHLATTER, P. & HENNINGSON, D. S. 2009 Localized edge states in plane Couette flow. *Phys. Fluids* **21**, 111701.
- FAISST, H. & ECKHARDT, B. 2003 Traveling waves in pipe flow. *Phys. Rev. Lett.* **91**, 224502.

- FLETCHER, C. A. J. 1991 *Computational Techniques for Fluid Dynamics*, 2. Springer.
- GIBSON, J. F., HALCROW, J. & CVITANOVIC, P. 2008 Visualizing the geometry of state space in plane Couette flow. *J. Fluid Mech.* **611**, 107–130.
- GIBSON, J. F., HALCROW, J. & CVITANOVIC, P. 2009 Equilibrium and travelling-wave solutions of plane Couette flow. *J. Fluid Mech.* **638**, 1–24.
- GITTLER, P. 1993 Stability of axial Poiseuille–Couette flow between concentric cylinders. *Acta Mech.* **101**, 1–13.
- GOLDSTEIN, M. E. & WUNDROW, D. W. 1998 On the environmental realizability of algebraically growing disturbances and their relation to Klebanoff modes. *Theor. Comput. Fluid Dyn.* **10**, 171–186.
- GOLDSTEIN, M. E. & SESCU, A. 2008 Boundary-layer transition at high free stream disturbance levels – beyond Klebanoff modes. *J. Fluid Mech.* **613**, 95–124.
- HALL, P. 1988 The nonlinear development of Görtler vortices in growing boundary layers. *J. Fluid Mech.* **193**, 243–266.
- HALL, P. 2012a Vortex–wave interactions/self-sustained processes in high Prandtl number natural convection in a vertical channel with moving sidewalls. *Stud. Appl. Maths* **129**, 1–25.
- HALL, P. 2012b Vortex–wave interactions: long-wavelength streaks and spatial localization in natural convection. *J. Fluid Mech.* **703**, 99–110.
- HALL, P. & HORSEMAN, N. J. 1991 The linear inviscid secondary instability of longitudinal vortex structures in boundary layers. *J. Fluid Mech.* **232**, 357–375.
- HALL, P. & SHERWIN, S. 2010 Streamwise vortices in shear flows: harbingers of transition and the skeleton of coherent structures. *J. Fluid Mech.* **661**, 178–205.
- HALL, P. & SMITH, F. T. 1989 Nonlinear Tollmien–Schlichting/vortex interaction in boundary layers. *Eur. J. Mech. (B/Fluids)* **8** (3), 179–205.
- HALL, P. & SMITH, F. T. 1990 *Near Planar TS Waves and Longitudinal Vortices in Channel Flow: Nonlinear Interaction and Focussing*, pp. 5–39 Springer.
- HALL, P. & SMITH, F. T. 1991 On strongly nonlinear vortex/wave interactions in boundary-layer transition. *J. Fluid Mech.* **227**, 641–666.
- HIGUERA, M. & VEGA, J. 2009 Modal description of internal optimal streaks. *J. Fluid Mech.* **626**, 21–31.
- ITANO, T. & GENERALIS, S. C. 2009 Hairpin vortex solution in planar Couette flow: a tapestry of knotted vortices. *Phys. Rev. Lett* **102**, 114501.
- ITANO, T. & TOH, S. 2001 The dynamics of bursting process in wall turbulence. *J. Phys. Soc. Japan* **70**, 703–716.
- JOSEPH, D. D. & CARMÍ, S. 1969 Stability of Poiseuille flow in pipes, annuli, and channels. *Q. Appl. Maths* **26**, 575–599.
- KAWAHARA, G. & KIDA, S. 2001 Periodic motion embedded in plane Couette turbulence: regeneration cycle and burst. *J. Fluid Mech.* **449**, 291–300.
- KERSWELL, R. R. & TUTTY, O. R. 2007 Recurrence of travelling waves in transitional pipe flow. *J. Fluid Mech.* **584**, 69–102.
- KOMMINAHO, K., LUNDBLADH, A. & JOHANSSON, A. V. 1996 Very large structures in plane turbulent Couette flow. *J. Fluid Mech.* **320**, 259–285.
- LUCHINI, P. 2000 Reynolds-number-independent instability of the boundary layer over a flat surface: optimal perturbations. *J. Fluid Mech.* **404**, 289–309.
- LUNDBLADH, A. & JOHANSSON, A. V. 1991 Direct simulation of turbulent spots in plane Couette flow. *J. Fluid Mech.* **229**, 499–516.
- NAGATA, M. 1990 Three-dimensional finite-amplitude solutions in plane Couette flow: bifurcation from infinity. *J. Fluid Mech.* **217**, 519–527.
- PERIS, R., MARQUINA, A. & CANDELA, V. 2011 The convergence of the perturbed Newton method and its application for ill-conditioned problems. *Appl. Maths Comput.* **218**, 2988–3001.
- SCHNEIDER, T. M., MARINC, D. & ECKHARDT, B. 2010a Localized edge states nucleate turbulence in extended plane Couette cells. *J. Fluid Mech.* **646**, 441–451.
- SCHNEIDER, T. M., GIBSON, J. F. & BURKE, J. 2010b Snakes and ladders: localized solutions of plane Couette flow. *Phys. Rev. Lett.* **104**, 104501.

- SCHNEIDER, T. M., GIBSON, J. F., LAGHA, M., DE LILLO, F. & ECKHARDT, B. 2008 Laminar-turbulent boundary in plane Couette flow. *Phys. Rev. E* **78**, 037301.
- SKUFCA, J. D., YORKE, J. A. & ECKHARDT, B. 2006 Edge of chaos in a parallel shear flow. *Phys. Rev. Lett.* **96**, 174101.
- SMITH, F. T. 1979 Instability of flow through pipes of general cross-section. Part 1. *Mathematika* **26**, 187–210.
- SMITH, F. T. & BODONYI, R. J. 1980 On the stability of the developing flow in a channel or circular pipe. *Q. J. Mech. Appl. Maths* **33**, 293–320.
- TILLMARK, N. 1995 On the spreading mechanisms of a turbulent spot in plane Couette flow. *Europhys. Lett.* **32**, 481.
- TILLMARK, N. & ALFREDSSON, P. H. 1992 Experiments on transition in plane Couette flow. *J. Fluid Mech.* **235**, 89–102.
- VAN VEEN, L. & KAWAHARA, G. 2011 Homoclinic tangle on the edge of shear turbulence. *Phys. Rev. Lett.* **107**, 114501.
- VISWANATH, D. 2009 The critical layer in pipe flow at high Reynolds number. *Phil. Trans. R. Soc. A* **367**, 561–576.
- WALEFFE, F. 1995 Hydrodynamic stability and turbulence: beyond transients to a self-sustaining process. *Stud. Appl. Maths* **95**, 319–343.
- WALEFFE, F. 1997 On a self-sustaining process in shear flows. *Phys. Fluids* **9**, 883–900.
- WALEFFE, F. 1998 Three-dimensional coherent states in plane shear flows. *Phys. Rev. Lett.* **81**, 4140–4143.
- WALEFFE, F. 2001 Exact coherent structures in channel flow. *J. Fluid Mech.* **435**, 93–102.
- WALEFFE, F. 2003 Homotopy of exact coherent structures in plane shear flows. *Phys. Fluids* **15**, 1517–1534.
- WALTON, A. G. 2002 The temporal evolution of neutral modes in the impulsively started flow through a circular pipe and their connection to the nonlinear stability of Hagen-Poiseuille flow. *J. Fluid Mech.* **457**, 339–376.
- WALTON, A. G. 2004 Stability of circular Poiseuille-Couette flow to axisymmetric disturbances. *J. Fluid Mech.* **500**, 169–210.
- WANG, J., GIBSON, J. & WALEFFE, F. 2007 Lower branch coherent states in shear flows: transition and control. *Phys. Rev. Lett.* **98**, 204501.
- WEDIN, H. & KERSWELL, R. R. 2004 Exact coherent structures in pipe flow: travelling wave solutions. *J. Fluid Mech.* **508**, 333–371.
- WILLIS, A. P. & KERSWELL, R. R. 2009 Turbulent dynamics of pipe flow captured in a reduced model: puff relaminarization and localized ‘edge’ states. *J. Fluid Mech.* **619**, 213–233.
- ZUCCHER, S., TUMIN, A. & RESHOTKO, E. 2006 Parabolic approach to optimal perturbations in compressible boundary layers. *J. Fluid Mech.* **556**, 189–216.