Viscous boundary layer properties in turbulent thermal convection in a cylindrical cell: the effect of cell tilting

Ping Wei and Ke-Qing Xia[†]

Department of Physics, The Chinese University of Hong Kong, Shatin, Hong Kong, China

(Received 25 September 2012; revised 3 December 2012; accepted 3 January 2013; first published online 27 February 2013)

We report an experimental study of the properties of the velocity boundary layer in turbulent Rayleigh-Bénard convection in a cylindrical cell. The measurements were made at Rayleigh numbers Ra in the range $2.4 \times 10^8 < Ra < 5.6 \times 10^9$ and were conducted with the convection cell tilted with an angle θ relative to gravity, at $\theta = 0.5, 1.0, 2.0$ and 3.4° , respectively. The fluid was water with Prandtl number Pr = 5.3. It is found that at small tilt angles ($\theta \leq 1^{\circ}$), the measured viscous boundary layer thickness δ_v scales with the Reynolds number Re with an exponent close to that for a Prandtl–Blasius (PB) laminar boundary layer, i.e. $\delta_v \sim Re^{-0.46\pm0.03}$. For larger tilt angles, the scaling exponent of δ_v with *Re* decreases with θ . The normalized mean horizontal velocity profiles measured at the same tilt angle but with different Ra are found to have an invariant shape. However, for different tilt angles, the shape of the normalized profiles is different. It is also found that the Reynolds number Re based on the maximum mean horizontal velocity scales with Ra as $Re \sim Ra^{0.43}$ and the Reynolds number Re_{σ} based on the maximum root mean square velocity scales with Ra as $Re_{\sigma} \sim Ra^{0.55}$. Within the measurement resolution neither exponent depends on the tilt angle θ . Several wall quantities are also measured directly and their dependencies on Re are found to agree well with those predicted for a classical laminar boundary layer. These are the wall shear stress τ ($\sim Re^{1.46}$), the viscous sublayer δ_w ($\sim Re^{0.75}$), the friction velocity u_{τ} (~ $Re^{-0.86}$) and the skin friction coefficient c_f (~ $Re^{-0.46}$). Again, all of these near-wall quantities do not exhibit a dependence on the tilt angle within the measurement resolution. We also examined the dynamical scaling method proposed by Zhou and Xia (Phys. Rev. Lett., vol. 104, 2010, p. 104301) and found that in both the laboratory and the dynamical frames the mean velocity profiles show deviations from the theoretical PB profile, with the deviations increasing with Ra. However, profiles obtained from dynamical scaling in general have better agreement with the theoretical profile. It is also found that the effectiveness of this method appears to be independent of Ra.

Key words: buoyant boundary layers, turbulent flows, turbulent convection

1. Introduction

1.1. Rayleigh-Bénard convection

Rayleigh–Bénard (RB) convection, a fluid layer heated from below and cooled from the top, is an idealized model to study turbulent flows involving heat transport and has attracted much attention during the past few decades (Siggia 1994; Kadanoff 2001; Ahlers, Grossmann & Lohse 2009; Lohse & Xia 2010). The system is characterized by two control parameters: the Rayleigh number Ra and Prandtl number Pr, which are defined as

$$Ra = \frac{\alpha g \Delta T H^3}{\nu \kappa},\tag{1.1}$$

and

$$Pr = \frac{\nu}{\kappa},\tag{1.2}$$

respectively. Here α is the thermal expansion coefficient, ν the kinematic viscosity, κ the thermal diffusivity of the convecting fluid, g the gravitational acceleration, ΔT the temperature difference between the bottom and the top plates and H the height of the fluid layer between the plates. In addition, the aspect ratio $\Gamma = D/H$ (D is the lateral dimension of the system) also plays an important role in the structures and dynamics of the flow.

In a fully developed RB turbulent flow, most of the imposed temperature difference is localized in the thermal boundary layers (BLs) near the surface of the top and bottom plates, within which heat is transported via conduction (Wu & Libcharber 1991; Belmonte, Tilgner & Libchaber 1994; Lui & Xia 1998). The velocity field has the same character: velocity gradient is localized in a thin layer near the plates, which is called the viscous boundary layer. Turbulent flow in the central region of the RB cell is approximately homogenous and isotropic (Sun, Zhou & Xia 2006; Zhou, Sun & Xia 2008; Ni, Huang & Xia 2011a, 2012). As the top and bottom boundary layers contribute the main resistance to heat transfer through the cell and thus dominantly determine the Nusselt number, they deserve special attention. Indeed, nearly all theories in RB convection are in essence BL theories. For example, a turbulent BL was assumed in the models by Shraiman & Siggia (1990), Siggia (1994) and Dubrulle (2001, 2002). On the other hand, a Prandtl-Blasius (PB)-type laminar BL was assumption in the Grossmann and Lohse (GL) theory (Grossmann & Lohse 2000, 2001, 2002, 2004). Therefore, direct characterization of the BL properties is essential for testing and differentiating the various theoretical models, and will also provide insight into the physical nature of turbulent heat transfer.

1.2. BL measurements in turbulent thermal convection

One of the earlier measurements of temperature and also of velocity profiles in turbulent RB convection was taken by Tilgner, Belmonte & Libchaber (1993) in water (Pr = 6.6) at the fixed $Ra = 1.1 \times 10^9$ and at a fixed lateral position. Belmonte, Tilgner & Libchaber (1993) extended these measurements over the range $5 \times 10^5 \le Ra \le 10^{11}$ in compressed gas (air) at room temperature (Pr = 0.7), but still at fixed lateral position. Lui & Xia (1998) measured the mean temperature profiles at various horizontal positions on the lower plate of a cylindrical convection cell, the result shows that the thermal BL thickness δ_{th} varied over the plate for the same Ra, and the thinnest BL is closest to the centre of the plate. Wang & Xia (2003) found similar results for a cubic cell. du Puits *et al.* (2007*b*) measured high-resolution

temperature profiles in RB convection near the top plate of a cylindrical container with air (Pr = 0.7) as the working fluid. Their result shows that the thermal BL thickness $\delta_{th} \sim Ra^{-0.25}$ in the cell with $\Gamma = 1.13$. Sun, Cheung & Xia (2008) found that the thermal BL thickness scales with $Ra^{-0.32}$ in a rectangular cell, at Pr = 4.3 and Ra ranging from 10^8 to 10^{10} .

For the velocity measurement, the methods for determining the velocity profiles near the solid walls of the cell are developed in recent years. Since strong temperature fluctuations exist in RB convection, the well-established hot-wire anemometry could not be applied to this system. For the viscous BL, the large temperature fluctuations make conventional laser Doppler velocimetry ineffective because temperature fluctuations cause fluctuations in the refractive index of the fluid that in turn make it difficult to steadily focus two laser beams to cross each other in the fluid (Xia, Xin & Tong 1995). Tilgner et al. (1993) introduced an electrochemical labelling method and measured the velocity profile and BL thickness near the top plate of a cubic cell filled with water, but only at a single value of Ra. In a later study, Belmonte et al. (1993, 1994) developed an indirect method, the correspondence between the peak position of the cutoff frequency profile of the temperature power spectrum and the peak position of the velocity, to infer the viscous BL thickness in gaseous convection. This method has subsequently been used to infer the viscous layer in thermal convection in mercury (Naert, Segawa & Sano 1997). It may just be that the method works in certain situations, but there is no theoretical basis for it. Xin, Xia & Tong (1996), using a novel light-scattering technique developed by Xia et al. (1995), conducted the first direct systematic measurement of velocity profiles in RB convection in a cylindrical cell as a function of Ra. They found $\delta_v \sim Ra^{-0.16}$ from the velocity profile above the centre of the lower plate. Qiu & Xia (1998a,b) extended these measurements to convection in cubic cells, finding the same scaling exponent -0.16 at the bottom plate, but at the sidewall a different result $\delta_v \sim Ra^{-0.26}$. Using various organic liquids, Lam et al. (2002) explored the Pr dependence, finding $\delta_{\nu} \sim Pr^{0.24}Ra^{-0.16}$. With the measured Ra-Re scaling relationship $Re \sim Ra^{0.5}$ obtained in these studies (in fact, it was the Péclet number $Pe = \nu L/\kappa$ rather than Re in some of these studies; see Sun & Xia (2005) for more detailed discussions), the above results imply a scaling relation $\delta_v \sim Re^{-0.32}$.

In recent years, the technique of particle image velocimetry (PIV) has been introduced to the experimental studies of thermal convection (Xia, Sun & Zhou 2003; Sun & Xia 2005; Sun, Xi & Xia 2005*a*; Sun, Xia & Tong 2005*b*). Sun *et al.* (2008) further applied the PIV technique to measure the viscous BL in a rectangular cell. Their results show that $\delta_v \sim Ra^{-0.27}$ and $\delta_v \sim Re^{-0.50}$, which showed that the viscous BL in thermal turbulence has the same Re-scaling as a PB laminar BL. This result validates the laminar BL assumption made in the GL model in a scaling sense. Thus it appears there is a discrepancy in the measured scaling exponent of δ_{ij} with respect to Ra (Re) between those obtained in cylindrical and cubic cells and that obtained in rectangular cells. Sun et al. (2008) argued that because of the more complicated flow dynamics of the large-scale circulation (LSC), such as the azimuthal motion in the cylindrical cell (Brown, Nikolaenko & Ahlers 2005; Sun et al. 2005b; Xi, Zhou & Xia 2006) and the secondary flows in the cubic cell (Qiu & Xia 1998a), the shear flow near the plates is less steady as compared with that in the rectangular cell which is closer to quasi-two-dimensional (quasi-2D). As the viscous boundary is created by the shear of the LSC, this may plausibly change the BL properties, resulting in a different exponent. However, the above argument has not been substantiated experimentally. Part of the motivation of the present work is to determine how the three-dimensional

LSC dynamics will affect the BL properties. It is known that tilting the cell by a small angle will 'lock' the LSC in a fixed azimuthal plane in the sense that it will limit the range of the LSC's azimuthal meandering (Sun et al. 2005a) and reduce its azimuthal oscillation amplitude near the top and bottom plates of the cell (Ahlers, Brown & Nikolaenko 2006). In the present work, we present measurements of BL properties in a cylindrical cell with the cell tilted over a range of angles. For small tilting angles, we measure a BL under a more steady shear compared with the 'levelled' case when the LSC can freely meander in the azimuthal direction but presumably the BL is otherwise unperturbed under such a small tilting angle ($\leq 1^{\circ}$). We also examine how the BL scaling exponent and other BL properties behave when the tilting angle becomes not so small $(>1^{\circ})$, which would amount to a perturbation to the BLs. BLs play such an important role in turbulent thermal convection, it is therefore important to examine how BLs respond to external perturbations. Understanding the stability or instability of BLs is also relevant to the search for the so-called ultimate state of thermal convection, as the transition from the 'classical state' to the ultimate state is essentially an instability transition of the BL from being laminar to being turbulent.

In addition to the scaling of the BL thickness, the shape of the velocity profile near the top and bottom plates has attracted a lot of attention recently. Although the BL has been found scaling wise to be of PB type (at least in the quasi-2D case), the time-averaged velocity profiles are found to differ from the theoretically predicted one (du Puits, Resagk & Thess 2007a; Sun et al. 2008), especially in the region around the thermal BL. Recently, Zhou & Xia (2010) have proposed a dynamic scaling method that shows that the mean velocity profile measured in the laboratory frame can be brought into coincidence with the theoretical PB laminar BL profile, if it is resampled relative to the time-dependent frame that fluctuates with the instantaneous BL thickness. This method was tested initially for the case of velocity profile in turbulent convection in a quasi-2D rectangular cell with water as working fluid (Pr = 4.3). In a follow-up study using 2D direct numerical simulation (DNS) data, Zhou et al. (2010) found that the method is also valid for thermal BLs and for the case of Pr = 0.7. More recently, these authors further showed, again using numerical data, that the method works also in other positions in the horizontal plate other than the central axis (Zhou et al. 2011) and in a three-dimensional (3D) cylindrical cell for moderate values of Ra (Stevens et al. 2012). On the other hand, Scheel, Kim & White (2012) and Shi, Emran & Schumacher (2012), both using numerical approaches, have found that the dynamic scaling works less well in the 3D cylindrical geometry than in the quasi-2D case (see also Chillà & Schumacher (2012)). Note that the method has not been tested experimentally so far in a 3D system. Here we would like to examine the dynamical scaling method using the experimentally obtained instantaneous velocity profiles in our three-dimensional cylindrical cell. We caution, however, that the PB theory rely on a two-dimensional and steady flow, and the temperature is assumed to be passively advected and pressure gradients are zero. So there is no a prior justification for why the PB profile should necessarily be applicable to the present case.

1.3. Organization of the paper

The remainder of this paper is organized as follows. We give detailed descriptions of the experimental set-up and measurement instrumentation in § 2 and present and analyse experimental results in § 3, which is divided into six subsections. In § 3.1 we present the measured temperature profiles and corresponding position-dependent fluid properties, which will be used to calculate the viscous and Reynolds stresses. In § 3.2,



FIGURE 1. (Colour online) Sketch of the convection cell and the Cartesian coordinates used in temperature and velocity measurements: (a) side view of the set-up (shaded region indicates the PIV measurement area) and (b) the top view.

the measured velocity profiles and their characterizations are presented. In § 3.3, the scaling properties, with both *Ra* and *Re*, of the thickness δ_v obtained from the mean velocity profiles and δ_σ obtained from root mean square (r.m.s.) velocity profiles, are presented and discussed. We also discuss the influence of the cell tilting angle θ on the BL scaling. In § 3.4 statistical properties (r.m.s. and skewness) of the velocity field in the BL region are discussed. In § 3.5 we present results of the viscous and Reynolds shear stress distributions in the BL, and discuss the scaling of the wall quantities. In § 3.6 we test the dynamic scaling method with respect to the measured instantaneous velocity profiles. We summarize our findings and conclude in § 4.

2. Experimental apparatus

2.1. Convection cell

The measurements were made in a cylindrical RB convection cell, which has been described in detail previously (Zhou & Xia 2002; Sun et al. 2005b; Ni, Zhou & Xia 2011b). Here we give only its essential features. The top and bottom conducting plates are made of pure copper and coated with a thin layer of nickel to avoid oxidation. The sidewall is made of Plexiglas. To avoid distortions in the images viewed by the camera, a square-shaped jacket is fitted around the sidewall of the convection cell. As shown in figure 1(b), the jacket is filled with water. The diameter and height of the cell is D = 19.6 cm and H = 18.6 cm, respectively. The aspect ratio $\Gamma = D/H$ is thus close to one. Two (three) thermistors are embedded in the top (bottom) plate. The top plate temperature is maintained constant by a refrigerated circulator (Polyscience Model 9702) that has a temperature stability of 0.01 °C. A NiChrome wire (26 Gauge, Aerocon Systems) surrounded by fibreglass sleeving and Teflon tape is distributed inside the grooves carved under the bottom plate. The wire is connected with five DC power supplies (GE Model GPS-3030) in series to provide constant and uniform heating. During the measurement, the whole cell is placed in a thermostat box that is kept at the same temperature (30 °C) as that of the fluid at the centre of the cell. During the experiment the cell was tilted by an angle θ such that the circulation plane of the LSC was parallel to the image plane of the camera (the x-z plane, see figure 1).

2.2. PIV measurement

The application of PIV to thermal turbulence has been described in detail in several previous publications (Xia *et al.* 2003; Sun *et al.* 2005*b*, 2008). Here we only provide

details concerning the particular features of the present experiment. The PIV system consists of one CCD camera with resolution 2048 pixels \times 2048 pixels, a dual pulse Nd-YAG laser with 135 mJ per pulse, a synchronizer and software (Dantec Dynamics, Inc.). As the cell was tilted, both the CCD and the laser light-sheet were tilted accordingly with the same angle. A 105 mm focal-length macro lens was attached to the CCD to achieve a measuring area with size varying from $18 \text{ mm} \times 18 \text{ mm}$ to 30 mm \times 30 mm. Each 2D velocity vector is calculated from a subwindow $(32 \text{ pixels} \times 32 \text{ pixels})$ that has 50% overlap with its neighbouring subwindows, so each vector corresponds to a region of 16 pixels \times 16 pixels and each velocity map contains 127×127 velocity vectors in the x-z plane (see figure 1). This corresponds to spatial resolutions of $\sim 0.135 \text{ mm} \times 0.135 \text{ mm}$ to 0.236 mm $\times 0.236 \text{ mm}$ for velocities u and w measured in the horizontal x and vertical z directions, respectively. For the measurement at $\theta = 3.4^{\circ}$ particles with diameter 2 µm were used, while particles with diameter of 10 um were used for measurements with other three tilted angles. For each run, typically about 25 200 image pairs were acquired with frame rate of 2 Hz. It is known that particles with finite sizes and with densities unmatched to that of carrier fluid have inertia. The inertial effect is characterized by the Stokes number $St = \tau_p / \tau_\eta$, which is the ratio between the time scale of the Stokes drag due to the interaction between particle and fluid and the Kolmogorov time scale. Here, $\tau_p = d^2 (2\rho_p + \rho_f)/(48\nu\rho_f)$, d is the diameter of particles, ρ_p is the density of particles, ρ_f is the density of fluid, ν is the kinematic viscosity of fluid. According to the measurement by Ni *et al.* (2012), the local Kolmogorov time scale is $\tau_{\eta} = 0.5 \sim 3$ s in our range of *Ra*. So in the present case, $St = 2 \times 10^{-8} \sim 2 \times 10^{-6}$ for 10 µm diameter particles, $St = 7 \times 10^{-9} \sim 4 \times 10^{-8}$ for 2 µm diameter particles. As St is much smaller than one in both cases, these particles can be safely treated as tracer particles. We also note that within the duration of the measurements (typically lasting for 3.5 h) most of the particles remain suspended in the fluid. To calibrate the positions of particles in the measured image to the real positions and also to correct the difference between the apparent positions captured by the camera and the real position due to refractive index (temperature) gradient in the BL, we placed a ruler inside the cell with an imposed temperature difference that is the same as that used in the experiment and this process is repeated for every Ra used in the experiment. The effect of refractive index (temperature) fluctuations may be estimated from the magnitude of the measured vertical velocity. As the vertical velocity should be zero at or near the plate, the magnitude of a non-zero vertical velocity should provide an indication of the influence of temperature effect on the PIV measurement and provide an estimate of the measurement error. We find that the magnitude of the vertical velocity W is indeed very small (see figure 8), especially inside the BL region. Compared with the magnitude of the horizontal velocity, this is indeed negligible.

It perhaps should be noted that in general it is more difficult to measure the velocity accurately in the cylindrical cell than it is in the rectangular cell. There are several reasons for this. One is that the curved surface of the sidewall in the cylindrical cell. Although the square jacket reduces the distortion to the optical image seen by the CCD camera, it cannot completely eliminate this effect. The distance between the camera and the measuring plane in the present case is also larger than that in the rectangular case, which affects the optical quality of the measured image. There is also the azimuthal motion of the LSC. Although the cell has been titled, the random meandering of the LSC is not suppressed completely. All of these will contribute negatively to the quality of the measured velocity, which makes the measurement accuracy in a cylindrical geometry less than that in a rectangular geometry.

P. Wei and K.-Q. Xia

θ (deg.)	Ra	Pr	U_{max} (mm s ⁻¹)	δ_v (mm)	θ (deg.)	Ra	Pr	U_{max} (mm s ⁻¹)	δ_v (mm)
0.5	5.77×10^{8}	5.45	4.77	2.80	2.0	1.34×10^{9}	5.42	7.47	2.57
	2.79×10^{8}	5.41	3.57	3.40		5.85×10^{8}	5.42	5.21	3.25
	1.55×10^{9}	5.39	7.38	2.44		5.53×10^{9}	5.32	13.49	1.97
	2.93×10^{9}	5.39	9.11	2.13	3.4	2.74×10^{8}	5.67	4.02	4.54
	4.26×10^{9}	5.38	10.88	2.06		4.19×10^{8}	5.66	4.70	4.23
1.0	1.68×10^{9}	5.45	7.53	3.46		6.69×10^{8}	5.66	6.00	3.58
	3.19×10^{9}	5.39	9.96	2.87		1.38×10^{9}	5.68	7.59	2.70
	5.54×10^{9}	5.42	11.60	2.66		2.19×10^{9}	5.64	9.40	2.50
	9.46×10^{8}	5.45	5.72	3.82		2.66×10^{9}	5.66	10.20	2.11
	2.40×10^{8}	5.42	3.06	5.07		9.89×10^{8}	5.66	6.47	3.00
	6.00×10^{8}	5.43	4.71	3.94		3.20×10^{9}	5.65	10.84	2.14
2.0	2.76×10^{8}	5.41	3.76	4.12		4.28×10^{9}	5.63	12.65	1.89
	2.78×10^{9}	5.42	10.09	2.15		5.19×10^{9}	5.56	13.49	1.84

TABLE 1. Control parameters of the experiment (cell tilt angle θ , the Rayleigh number Ra and the Prandtl number Pr) and the measured maximum horizontal velocity U_{max} and viscous BL thickness δ_v . The data are listed in chronological order within a data set of given θ .

3. Results and discussion

PIV measurements were made at four values of the tilting angle $\theta = 0.5, 1, 2$ and 3.4°. For each θ , measurements over a range of *Ra* were made. Table 1 lists the parameters (θ , *Ra* and *Pr*) of each measurement, which typically lasted for ~3.5 h. As already mentioned, tilting the cell by a small angle has the effect of 'locking' the LSC's circulation plane at a fixed azimuthal angle (in reality it restricts the angular range of the LSC's azimuthal meandering). Thus, measurements made with small $\theta (\leq 1^\circ)$ are aimed at studying BL properties under more steady shear, but the BL itself is assumed to be unperturbed otherwise. For large values of $\theta (> 1^\circ)$ we wish to examine how the BL responds to relatively large perturbations.

3.1. Temperature profile and fluid properties

The local values of fluid properties are needed in calculating the viscous and Reynolds shear stress, which requires measurement of the local temperature. Temperature profiles for the levelled case have been measured systematically by Lui & Xia (1998) in a similar cylindrical cell. To check whether tilting the cell by a relatively large angle will change the temperature profile, we measured one mean temperature along the central axis (x = y = 0) of the cell at a tilting angle $\theta = 3.4^{\circ}$ $(Ra = 6.8 \times 10^{\circ})$. The result is shown in figure 2(a) and the dynamic viscosity corresponding to the local temperature is shown in figure 2(b). In the inset of figure 2(a) we also show the profile of the temperature r.m.s. σ_T . From this mean temperature profile, we obtain the thermal BL thickness $\delta_{th} = 1.9$ mm using the 'slope' method, while the peak position of the r.m.s. profile is 1.6 mm. As these results are similar to those obtained in previous studies by Lui & Xia (1998) and Sun et al. (2008), we will use results from those studies at similar Ra in the calculations of Reynolds stress (§ 3.5) and other wall quantities that require position-dependent viscosity and density. For reference, in the range of Ra of the present experiment, δ_{th} varies from 0.5 to 2.0 mm (Lui & Xia 1998).



FIGURE 2. (a) A profile of mean temperature $\langle T \rangle$ measured at $Ra = 6.8 \times 10^8$ and $\theta = 3.4^\circ$. The inset is the profile of the corresponding r.m.s. temperature σ_T . (b) A profile of dynamic viscosity μ obtained from the mean temperature profile in (a).



FIGURE 3. (Colour online) Coarse-grained vector maps of the instantaneous (a) and timeaveraged (b) velocity field measured near the centre of the bottom plate ($Ra = 4.2 \times 10^8$ with $\theta = 3.4^\circ$), the velocity scale bar is in unit of mm s⁻¹.

3.2. Velocity profiles and the Reynolds number scaling

Figure 3(a) shows an example of measured instantaneous velocity map and (b) time-averaged velocity field taken over a period of 3.5 h (corresponding to 25 200 velocity frames), with the cell tilted at $\theta = 3.4^{\circ}$ and at $Ra = 4.2 \times 10^8$. In the present measurement, x spans from -8.75 to 8.75 mm, and z spans from 0 to 17.5 mm. From the velocity scale in figure 3(a) and (b), it is seen that there exist velocity bursts with values much larger than the maximum velocity in the time-averaged velocity field. It is found that velocity maps measured at other tilt angles have similar features. As the mean velocity and the velocity fluctuations do not exhibit any obvious dependence on the horizontal position x over the small measurement area, the quantities presented below are based on values averaged along the x-direction over the width of the measuring area.

https://doi.org/10.1017/jfm.2013.17 Published online by Cambridge University Press



FIGURE 4. (Colour online) Time-averaged horizontal velocity profiles measured at tilt angles $\theta = 0.5^{\circ}$ (*a*), 1.0° (*b*), 2.0° (*c*) and 3.4° (*d*). In each plot the corresponding value of *Ra* decreases from top to bottom (see table 1 for exact values).

Figure 4 plots the velocity profiles for different tilt angles and various values of Ra, which shows that the shapes of the profiles are rather similar at this level of detail. Figure 5 plots normalized profiles in which U(z) is normalized by the maximum horizontal velocity $U_{max}(Ra)$ (for ease of reference the values of U_{max} are also listed in table 1) and the distance z from the wall by the viscous BL thickness $\delta_v(Ra)$ (to be defined below). The figure shows that up to $2\delta_v$ profiles for different Ra and for the same tilt angle collapse on to a single curve quite well (except perhaps those correspond to the largest Ra for $\theta = 2.0$ and 3.4°). Note that $z \simeq 2\delta_v$ is around where U reaches its maximum value and beyond this position it decays toward cell centre. So this position may be taken as the separation between the BL region and the bulk. The above results suggest that for the same tilt angle the profiles in the BL region have an invariant shape with respect to different values of Ra. This result is consistent with the finding by \dot{X} in *et al.* (1996), Qiu & Xia (1998*a*,*b*) and Sun *et al.* (2008). In figure 5 we also plot the theoretical PB profile. It is seen that within the BL $(z \leq \delta_v)$ the profiles match the theoretical solution very well, whereas in the region just outside the BL where plume emissions occur, all measured profiles are generally less steep than the PB profile. This feature is also similar to that observed by Zhou & Xia (2010) and will be further discussed in \S 3.6. On the other hand, it is seen from figure 5 that profiles obtained at different θ seem to have different degrees of deviation from the PB profile. This can be seen more clearly in figure 6 where we show two examples in which profiles for different θ but with values of Ra close to each other are plotted together along with the theoretical PB profile. This result suggests that the shape of



FIGURE 5. (Colour online) Profiles normalized by their respective maximum velocity $U_{max}(Ra)$ and the corresponding viscous BL thickness $\delta_v(Ra)$ with tilt angles $\theta = 0.5^\circ$ (a), 1.0° (b), 2.0° (c) and 3.4° (d). The solid line in each plot represents the theoretical PB profile.

the velocity profile near the plume-emission region is modified by the tilting angle. It is also seen from the figure that at small z/δ_v some of the data (especially those for $\theta = 1^\circ$) deviate from a straight line passing through the origin. This is probably caused by the strong reflection of light near the plate that can give false velocity vectors with zero or very small values. In this case, we ignore the few points very close to z = 0when determining δ_v . As there are still quite number of points with $z/\delta_v < 1$ that lie on a straight line in such profiles, the determined δ_v should not be significantly affected by the exclusion of these few points. We also note that the profile for $\theta = 1.0^\circ$ shown in figure 6(b) is a rather extreme case and there are only two or three such profiles among all of the measured ones.

Taking U_{max} as the characteristic velocity of LSC, we define the Reynolds number $Re = U_{max}H/\nu$ and plot Re as a function of Ra and for different θ in figure 7(*a*). When fitting a power law to the data for different θ separately, they all produce an exponent close to 0.43. To better compare the amplitude of Re for different θ , we fix the scaling exponent at 0.43 and fit power laws to the different data sets again. This gives $Re = (0.185 \pm 0.002, 0.182 \pm 0.003, 0.206 \pm 0.001, 0.203 \pm 0.001) \times Ra^{0.43}$, where the amplitudes in the brackets are for $\theta = 0.5$, 1.0, 2.0 and 3.4°, respectively. These results show that in general the values of Re with larger θ are larger than those with smaller θ . In an earlier study of the effect of cell tilting, Ahlers *et al.* (2006) have found that Re obtained indirectly from temperature measurement increases with the tilted angle, which is consistent with the trend observed here. We also note that the value of the scaling exponent of Re obtained from many previous studies and sometimes under



FIGURE 6. (Colour online) Normalized profiles measured at different tilt angles θ but with approximately the same value of *Ra*. Profiles in (*a*) have a nominal value of $Ra = 5 \times 10^8$ and in (*b*) have a nominal value of $Ra = 1.5 \times 10^9$. In both figures the symbols are: inverted triangles ($\theta = 0.5^\circ$); squares ($\theta = 1.0^\circ$); triangles ($\theta = 2.0^\circ$); and circles ($\theta = 3.4^\circ$). The solid line in each plot represents the theoretical PB profile.



FIGURE 7. (Colour online) (a) Reynolds number Re based on the maximum horizontal velocity U_{max} and (b) Re_{σ} based on the maximum fluctuating velocity σ_{max} as a function of Ra for different tilting angles. Inverted triangles: $\theta = 0.5^{\circ}$; squares: 1°; triangles: 2°; circles: 3.4°. The dashed lines in (a) represent power-law fits to the individual data sets all with a scaling exponent 0.43 (see the text for the fitting results). The solid line in (b) is a power law fit to all data sets in the plot, which gives $Re_{\sigma} = 0.0072Ra^{0.55\pm0.01}$.

nominally similar conditions, varies over a rather wide range from 0.43 to 0.55 (see, for example, Xin *et al.* (1996), Xin & Xia (1997), Qiu & Xia (1998*a,b*), Ashkenazi & Steinberg (1999), Chavanne *et al.* (2001), Lam *et al.* (2002), Brown, Funfschilling & Ahlers (2007), Sun *et al.* (2008) and Xie, Wei & Xia (2012)). The reason for such variations is not completely clear at present. A detailed study on this issue is beyond the scope of this paper. For interested readers, we refer to Sun & Xia (2005) who offered an explanation that can account for most of these differences in the exponent.

From the measured profile of the r.m.s. velocity (see figure 8), we can define another Reynolds number $Re_{\sigma} = \sigma_{max}H/\nu$, which is shown in figure 7(b) as a function of Rain a log-log scale for the four tilt angles. Here it is seen that within the measurement resolution Re_{σ} does not have an obvious dependence on θ . We therefore fitted a single power law to all four data sets on the plot, which gave $Re_{\sigma} = 0.0072Ra^{0.55\pm0.01}$. The value of the exponent is somewhat larger than 0.50 that was obtained from several previous studies (Xin *et al.* 1996; Xin & Xia 1997; Qiu & Xia 1998*a*,*b*; Sun *et al.* 2008).

3.3. The viscous BL and its scaling with Ra and Re

We define the thickness δ_v of the viscous BL through the 'slope method' as shown in figure 8 where a mean velocity U(z) (circles) profile and the corresponding standard deviation profile $\sigma_u(z)$ (crosses) are shown, which are measured at $Ra = 4.2 \times 10^8$ with $\theta = 3.4^\circ$. It is seen that δ_v is defined as the distance at which the extrapolation of the linear part of U(z) equals its maximum value U_{max} , i.e. $\delta_v = U_{max} [dU/dz]_{z=0}]^{-1}$. A length scale δ_σ can also be defined from the profile of $\sigma_u(z)$ where σ_u reaches its maximum value. For the present example, the values for the two BL length scales δ_v and δ_σ are found to be 4.20 and 6.05 mm, respectively. For ease of reference, the values of δ_v are listed in table 1. Note that if we define δ_m as the position of the maximum horizontal velocity, the exponents obtained from the fitting results $\delta_m - Ra$ and $\delta_m - Re$ have similar trend as those for δ_v . However, because the velocity profiles near U_{max} are rather flat, the determined δ_m have relatively larger uncertainties as compared with δ_v .

We now examine the scalings of the BL thickness with both the Rayleigh number Ra and the Reynolds number Re. In figure 9(a) and (b) we plot the measured viscous BL thickness δ_v versus Ra and Re, respectively, for the four tilt angles. The lines in the figures represent the best power-law fits to the respective data sets, as listed in the caption of figure 9. The obtained fitting parameters for the normalized thicknesses $\delta_v/H = A_1 R a^{\beta_1}$ and $\delta_v/H = A_2 R e^{\beta_2}$ are listed in table 2. Also shown in the table for comparison are results obtained in cells with different geometries and using different methods. It is seen from the table that for small tilt angles ($\theta = 0.5$ and 1.0°), the exponents are essentially the same and within the experimental uncertainties the Re-scaling exponent may be taken the same as that predicted for a PB BL, i.e. $\delta_v \sim Re^{-1/2}$. For larger tilting angles, there appears to be a trend for both β_1 and β_2 to decrease (absolute value increases) with increasing θ . It thus appears that tilting the cell by over 1° is a rather strong perturbation to the BL, at least for its scaling. The situation for the amplitude of viscous BL thickness δ_v is a bit more complicated. From both figure 9(a) and (b) it seems that at lower values of Ra (Re) the BL thickness increases with increasing tilting angle, except for $\theta = 1^{\circ}$. For this latter tilting angle, δ_v appears to have an overall upward shift from the rest data sets. While we do not know the exact reason(s) for this, we note from figure 6 that the profiles for this tilt angle seem to have a non-zero intercept on the horizontal axis. This appears to suggest that the origin of the z-axis for this tilt angle was somehow shifted. But even if this is

lantity	рV		OCULIER D	ľ	14	7	- <i>P</i> 2	v (uucg.)	Source	
H_{\prime}	$10^8\sim 10^{10}$	$L\sim$	Cylin.	0.51	0.16 ± 0.02		0.32	0	а	
	$10^8\sim 10^{10}$	$6\sim 1027$	Cylin.	$0.65 Pr^{0.24}$	0.16 ± 0.02		0.32	0	p	
	$10^8\sim 10^{10}$	$L\sim$	Cubic	0.69	0.18 ± 0.04		0.36	0	C	
				3.6	$0.26 {\pm} 0.03$		0.52	0	d	
	$10^9\sim 10^{10}$	4.3	Rectan.	4.95	0.27 ± 0.01	0.64	0.50	0	в	
	$10^8 \sim 10^9$	5.4	Cylin.	0.745	0.19 ± 0.01	0.369	0.45 ± 0.04	0.5	f	
			•	1.41	$0.20 {\pm} 0.01$	0.564	0.46 ± 0.03	1.0	f	
				5.86	0.29 ± 0.01	1.41	0.61 ± 0.04	2.0	, t	
				13.3	$0.32 {\pm} 0.01$	6.26	$0.81 {\pm} 0.01$	3.4	Ĵ	
				A_3	$-\beta_3$	A_4	$-eta_4$			
H/	$10^7 \sim 10^{11}$	$L\sim$	Cylin.	1.02	0.25 ± 0.02		0.5	0	а	
	$10^8\sim 10^{10}$	$L\sim$	Cubic	0.95	0.25 ± 0.04		0.5	0	c	
				43	$0.38 {\pm} 0.03$		1.0	0	d	
	$10^9\sim 10^{10}$	4.3	Rectan.	16.5	$0.37 {\pm} 0.10$	0.69	0.72 ± 0.14	0	в	
	$10^8 \sim 10^9$	5.4	Cylin.	0.58	0.15 ± 0.02	0.14	0.26 ± 0.03	0.5	f	
			•	1.77	0.20 ± 0.02	0.27	0.37 ± 0.04	1.0	,t	
				2.68	0.23 ± 0.02	0.32	0.41 ± 0.04	2.0	f	
				9.6	0.29 ± 0.02	0.75	0.54 ± 0.04	3.4	f	

determined from the r.m.s. horizontal velocity profile. The fitting parameters A_i and β_i (i = 1, 2, 3, 4) are defined through the power laws: $\delta_{\sigma}/H = A_1 R a^{\beta_1}$, $\delta_{\sigma}/H = A_2 R e^{\beta_2}$, $\delta_{\sigma}/H = A_3 R a^{\beta_3}$ and $\delta_{\sigma}/H = A_4 R e^{\beta_4}$. The control parameters Ra and Pr and cell geometry of measurements are also listed. Also shown in the table are results from some previous experiments. The sources are: (a) Xin et al. (1996); (b) Lam et al. (2002); (c) Qiu & Xia (1998a) (bottom); (d) Qiu & Xia (1998b) (sidewall); (e) Sun et al. (2008); and (f) present work. (Note: the cell tilt angle θ is indicated as zero when it was not mentioned in the respective papers and we assume the cell was nominally levelled in those TABLE 2. Fi cases.)

https://doi.org/10.1017/jfm.2013.17 Published online by Cambridge University Press



FIGURE 8. (Colour online) Determination of the viscous BL thickness δ_v through the slope method from the mean horizontal velocity profile U(z) (circles) and the thickness δ_σ from the standard deviation profile σ_u (crosses). Also shown is the mean vertical velocity profile W(z)(squares). The measurement was made near the bottom plate with tilt angle $\theta = 3.4^{\circ}$ and at $Ra = 4.2 \times 10^8$.

the case, the relatively small 'shift' cannot account for the large 'deviation' of this δ_v from the rest data sets (assuming there is indeed something 'wrong' with this data set). Aside from the amplitude, the behaviour of the *Ra*- and *Re*-scaling exponents may be summarized as follows. For small tilting angle ($\theta \leq 1^\circ$), the effect of tilting is to lock the azimuthal plane of the LSC (or restrict its azimuthal meandering range) but the BL is otherwise not strongly perturbed and scaling wise the BL is approximately PB type. For relatively large tilting angle ($\theta > 1^\circ$), the BL appears to be strongly perturbed as far as scaling is concerned and the absolute value of the scaling exponent increases with tilting angle, i.e. the BL thickness δ_v decays with increasing *Ra* (*Re*) with a steeper slope. The situation is illustrated in figure 10 where β_2 is plotted as a function of the tilt angle θ .

In addition to the BL thickness δ_v determined from the mean horizontal velocity profile, another length scale can also be defined based on the profile of the horizontal r.m.s. velocity σ_u , which may be called the r.m.s. velocity BL thickness, as defined in figure 8. In figure 11(*a*) and (*b*) we plot δ_σ versus *Ra* and Re_σ , respectively. The *Ra*-scaling exponent varies from -0.15 to -0.29, which appears to follow a similar trend to that of δ_v , i.e. its absolute value increases with increasing θ . But there are significant differences between this and the Re_σ -scaling exponent and those obtained in previous studies. Table 2 shows the fitting results of $\delta_\sigma = A_3 Ra^{\beta_3}$ and $\delta_\sigma = A_4 Re_{\sigma}^{\beta_4}$.

Now we compare our result with previous experimental results obtained in the cells with different geometries. Using a cylindrical geometry with $\Gamma = 1$ and with air as the working fluid, Li *et al.* (2012) found experimentally that $\beta_1 = -0.24 \pm 0.03$ and $\beta_2 = -0.54 \pm 0.09$. In a numerical study also in a $\Gamma = 1$ cylindrical geometry, Verzicco & Camussi (1999) found for the scaling of viscous BL that $\beta_1 = -0.18$ for Pr = 0.022 and $\beta_1 = -0.23$ for Pr = 0.7. For the $\Gamma = 1/2$ geometry, Verzicco & Camussi (2003) obtained $\beta_1 = -0.309$ for Pr = 0.7. Here we compare our results to those from previous studies with similar or larger Prandtl numbers. As shown in table 2, the value of β_1 obtained in both cylindrical and cubic geometries and measured near the bottom plate of the cell is -0.16. In all of these previous measurements, the Reynolds number



FIGURE 9. (Colour online) (a) Measured viscous BL thickness δ_v versus Ra and power-law fits (dashed lines) to the respective data sets for the four tilt angles: $\delta_v = 137Ra^{-0.19\pm0.01}$ mm (inverted triangles, $\theta = 0.5^{\circ}$); $\delta_v = 260Ra^{-0.20\pm0.01}$ mm (squares, 1.0°); $\delta_v = 1078Ra^{-0.29\pm0.01}$ mm (triangles, 2.0°); and $\delta_v = 2442Ra^{-0.32\pm0.01}$ mm (circles, 3.4°). (b) Plot of δ_v versus Re and power-law fits to the respective data sets for the four tilt angles: $\delta_v = 68Re^{-0.45\pm0.04}$ mm (inverted triangles, $\theta = 0.5^{\circ}$); $\delta_v = 104Re^{-0.46\pm0.03}$ mm (squares, 1.0°); $\delta_v = 260Re^{-0.61\pm0.04}$ mm (triangles, 2.0°); and $\delta_v = 115Re^{-0.81\pm0.01}$ mm (circles, 3.4°).

based on the maximum horizontal velocity near the plate was also obtained and they gave a scaling exponent $\gamma = 0.50$ via $Re \sim Ra^{\gamma}$. From this we obtain $\beta_2 = -0.32$. In these studies, the convection cell was nominally levelled, i.e. not intentionally tilted. In the present study, for the small tilting angle cases, where we assume the BL is not strongly perturbed, the measured $\beta_1 \simeq -0.19$ when combined with $\gamma = 0.43$ give a $\beta_2 \simeq -0.45 \pm 0.04$ (note that the actual value of β_2 are obtained from fitting the δ_v versus Re data, not from the relationship between the exponents). If we take these values to be close to the PB result, then scaling wise the viscous BL in a cylindrical geometry is also of a PB type, as was already found in a rectangular cell (Sun *et al.* 2008). For the relatively large deviations found in the untilted case, it may be attributed to the random azimuthal motion of the LSC.

Finally we remark that as far as the scaling of the viscous BL is concerned, there is no theoretical prediction for the dependence of δ_v on Ra, only that on Re (for example, $\delta_v \sim Re^{-1/2}$ for the PB BL). In the literature, it is sometimes stated that δ_v should scale as $Ra^{-1/4}$ for the PB BL. This is based on the assumption that $Re \sim Ra^{1/2}$. From above we have seen that the scaling exponent of Re with Ra varies over a rather wide range. It is therefore more meaningful to talk about the scaling of δ_v with Re, rather than with Ra. We further note that in Sun *et al.* (2008) it was found



FIGURE 10. (Colour online) The scaling exponent β_2 versus cell tilt angle θ , where β_2 is obtained from the power-law fit $\delta_v/H \sim Re^{\beta_2}$. The dashed line indicates $\beta_2 = -0.5$ for a PB laminar BL.



FIGURE 11. (Colour online) (a) Scalings of the BL scale δ_{σ} determined from the measured r.m.s. velocity profiles versus Ra with power-law fits (dashed lines) to the respective data sets for the four tilt angles: $\delta_{\sigma} = 106Ra^{-0.15\pm0.02}$ (inverted triangles, $\theta = 0.5^{\circ}$); $\delta_{\sigma} = 327Ra^{-0.20\pm0.02}$ (squares, 1.0°); $\delta_{\sigma} = 493Ra^{-0.23\pm0.02}$ (triangles, 2.0°); and $\delta_{\sigma} = 1822Ra^{-0.29\pm0.02}$ (circles, 3.4°). (b) Plot of δ_{σ} versus Re_{σ} , and the dashed lines are power-law fits to the respective data sets for the four tilt angles: $\delta_{\sigma} = 25Re_{\sigma}^{-0.26\pm0.03}$ (inverted triangles, $\theta = 0.5^{\circ}$); $\delta_{\sigma} = 50Re_{\sigma}^{-0.37\pm0.04}$ (squares, 1.0°); $\delta_{\sigma} = 59Re_{\sigma}^{-0.41\pm0.04}$ (triangles, 2.0°); and $\delta_{\sigma} = 139Re_{\sigma}^{-0.54\pm0.04}$ (circles, 3.4°).

that $\delta_v \sim Ra^{-0.27}$ and $Re \sim Ra^{0.55}$, which together give $\delta_v \sim Re^{-0.50}$. In the present case, we have $\delta_v \sim Ra^{-0.20}$ and $Re \sim Ra^{0.43}$, which together give $\delta_v \sim Re^{-0.46\pm0.03}$. Whether this is fortuitous or there is something deep here remains to be explored.

3.4. Fluctuations and statistical properties of the velocity field in the BL

In previous BL measurements in the cylindrical cell, owing to the nature of the dual-beam incoherent cross-correlation technique employed (Xin *et al.* 1996; Lam *et al.* 2002), only time-averaged velocity profiles are measured and no time-dependent quantities are obtained. It is therefore interesting to examine these quantities and



FIGURE 12. Time traces of horizontal u(t) (a-e) and vertical w(t) (f-j) velocity components measured at $Ra = 2.4 \times 10^8$ and $\theta = 1^\circ$, at x = 0 and at different distances z from the bottom plate.

compare them with similar quantities obtained in other types of turbulent flows. Figure 12 shows the time series of both the horizontal component u(t) (*a*-*e*) and the vertical component w(t) (*f*-*j*) of the velocity, measured at various positions from the plate. The corresponding velocity histograms are shown in figure 13. The measurements were made at $Ra = 2.4 \times 10^8$ and $\theta = 1^\circ$. We show the velocity trace at several typical positions: (i) inside the thermal BL; (ii) around the thermal BL; (iii) around the viscous BL; (iv) at the position of the maximum velocity; and (v) far



FIGURE 13. (Colour online) Histograms of (a) the horizontal velocity u(t) and (b) vertical velocity w(t) measured at various distances from the bottom plate with $Ra = 2.4 \times 10^8$ and $\theta = 1^\circ$.

away from the BLs. The figures show that the absolute horizontal velocity is much higher than vertical velocity at each position. One general feature we observe is that velocity time series and histograms look similar for different tilting angles. For this reason, we show here results for only one tilting angle.

At $Ra = 2.4 \times 10^8$ and $\theta = 1.0^\circ$, the viscous BL thickness is $\delta_v = 5.07$ mm. It is seen from figure 12(a,b) that at positions inside the BL, the horizontal velocity u(t)skews toward the positive side, i.e. the velocity is skewed toward the mean flow direction. This may be understood by the fact that close to the plate the mean velocity is almost zero and a fluctuation resulting a negative instantaneous horizontal velocity is a rather rare event. Once outside of the BL, one observes more symmetric fluctuations around the mean velocity. For the vertical velocity w(t), its mean velocity is very small at most positions. But the fluctuation increases significantly when the position is outside of the BL, which are signatures of plume emissions at these positions. These properties can also be seen from the velocity histograms shown in figure 13. A notable difference between the present results and those observed by Sun *et al.* (2008) is that for positions outside of the BL the horizontal velocity fluctuates more or less symmetrically around the mean, rather than skewed toward the negative as seen in the rectangular cell.

The statistical properties of the velocity may be characterized more quantitatively by its r.m.s. value and its skewness, which are shown in figure 14. Figure 14(*a*) plots the velocity r.m.s. σ_u and σ_w normalized by maximum horizontal velocity U_{max} versus the normalized distance z/δ_v . Figure 14(*b*) shows the skewness profiles $S_u = \langle (u - \langle u \rangle)^3 \rangle / (\langle (u - \langle u \rangle)^2 \rangle)^{3/2}$ and $S_w = \langle (w - \langle w \rangle)^3 \rangle / (\langle (w - \langle w \rangle)^2 \rangle)^{3/2}$ for the horizontal and vertical velocities, respectively. Similar to Sun *et al.* (2008), our result could not tell whether σ_w favours a power law or a logarithmic scaling with the distance *z*, even though our measurement had a much higher spatial resolution. This is partly due to the limited size of the measurement area.

3.5. Properties of shear stresses and near-wall quantities

One of the advantages of PIV measurement is that it enables one to measure the horizontal and vertical velocities at the same time, so that one can calculate the Reynolds shear stress $\tau_R = -\rho(z)\langle u'w' \rangle$. Here u' and w' are, respectively, the fluctuations of the horizontal and vertical velocity components away from their



FIGURE 14. (Colour online) Profiles of (a) the normalized r.m.s. velocity σ_u (σ_w) and of (b) the skewness S_u (S_w) measured at $Ra = 2.4 \times 10^8$ and $\theta = 1^\circ$. The vertical distance z is normalized by the velocity BL thickness δ_v . In both plots the circles represent those for the horizontal velocity component u and the crosses represent those for the vertical velocity component w.



FIGURE 15. (Colour online) Viscous stress τ_v (solid squares, green online), the Reynolds stress τ_R (open circles, red online) and the total stress τ (open diamonds, blue online) as functions of the normalized distance from the plate for (a) $Ra = 4.19 \times 10^8$ and (b) $Ra = 4.28 \times 10^9$, both with $\theta = 3.4^\circ$.

mean value, $u' = u(t) - \langle u \rangle$ and $w' = w(t) - \langle w \rangle$. Viscous shear stress is defined as $\tau_v = \mu(z) du/dz$, Note that the dynamic viscosity $\mu(z)$ and fluid density $\rho(z)$, because of their temperature dependence, are dependent on position z in the thermal BL. The Reynolds stress represents the transport of momentum by turbulent fluctuations, whereas the viscous stress describes the momentum transfer by viscosity. The total shear stress is then $\tau = \tau_v + \tau_R$.

Figure 15 plots the profiles of the viscous shear stress, Reynolds stress and total stress for (a) $Ra = 4.19 \times 10^8$, and (b) 4.28×10^9 . It is seen that both Ra have the same qualitative features. Here the examples are for $\theta = 3.4^\circ$ and results for other tilting angles are similar. Near the plate, it is seen that the Reynolds stress τ_R is close to zero, while the viscous shear stress τ_v is maximum because of the large velocity gradient du/dz at the wall. So the total stress at the wall $\tau_w(=\tau(0))$ comes almost entirely from the contribution of the viscous shear stress. Moving away from the plate, the velocity gradient becomes smaller and the viscous shear stress decreases to zero. The



FIGURE 16. (Colour online) The *Ra*-dependence of (*a*) the wall shear stress τ_w , (*b*) the friction velocity u_{τ} , (*c*) the wall thickness (viscous sublayer) δ_w and (*d*) the friction coefficient c_f , with power-law fits shown as solid lines. The symbols represent data for different tilting angles: $\theta = 0.5^{\circ}$ (inverted triangles), 1° (squares), 2° (triangles) and 3.4° (circles).

Reynolds stress τ_R increases and attains its maximum at $z \approx 1.5\delta_v$. It then decreases to around zero $z \approx 2\delta_v$ and becomes negative in the bulk flow. It is also seen clearly from the figure that τ_R and τ_v cross at $z \approx 1.5\delta_v$, where τ_R is close to its maximum value. This suggests that the momentum transfer in the outer region is dominated by turbulent fluctuations. But in the viscous BL, the momentum transfer is still dominated by the viscous diffusion, which implies that the viscous BL is still laminar in this range of *Ra*.

With the velocity field obtained from high-resolution measurements, we are now in a position to check the dynamic wall properties in turbulent thermal convection. We first consider the scaling of four basic wall quantities with both Ra and Re. These are the wall shear stress τ_w , the skin-friction velocity $u_{\tau} = (\tau/\rho_0)^{1/2}$, the viscous sublayer length scale $\delta_w = v_0/u_{\tau}$ and the skin-friction coefficient $c_f = \tau/\rho_0 U_{max}^2$. Here $\rho_0 \equiv \rho(z=0)$ and $v_0 \equiv v(z=0)$. Figure 16 shows the scaling of these quantities with Ra. It is seen that within experimental uncertainties there is no difference between data with different θ . This suggests that tilting the cell does not have any appreciable effect on BL properties near the wall. Without differentiating the different data sets, power-law fits to all data yield $\tau_w \sim Ra^{0.63\pm0.03}$, $u_{\tau} \sim Ra^{0.32\pm0.01}$, $\delta_w \sim Ra^{-0.37\pm0.01}$ and $c_f \sim Ra^{-0.19\pm0.02}$. In a rectangular cell, Sun *et al.* (2008) found for the same quantities the fitted power-law exponents 0.86, 0.44, -0.50 and -0.28 respectively. It is seen that the absolute values of these exponents are all larger than those obtained in the present experiment. There is no theoretical prediction for the *Ra*-scaling of these quantities in turbulent thermal convection, so we do not know what the differences mean.



FIGURE 17. (Colour online) The *Re*-dependence of (*a*) the wall shear stress τ_w , (*b*) the friction velocity u_{τ} , (*c*) the wall thickness (viscous sublayer) δ_w and (*d*) the friction coefficient c_f . The symbols represent data for different tilting angles: $\theta = 0.5^{\circ}$ (inverted triangles), 1° (squares), 2° (triangles) and 3.4° (circles). Power-law fits are indicated in the figure.

It will be more useful perhaps to examine the scaling of these quantities with the Reynolds number Re, since theoretical predictions exist for such scalings for wall-bounded shear flows (Schlichting & Gersten 2000). Figure 17 plots these quantities as a function of Re, the symbols are the same as in figure 16. For the quantities τ_w , u_τ , δ_w and c_f our results give the exponents 1.46 ± 0.05 , 0.75 ± 0.02 , -0.86 ± 0.03 and -0.46 ± 0.05 , respectively. For a laminar BL over a flat plate, the theoretically predicted 'classical' exponents for these quantities are 3/2, 3/4, -1and -1/2, respectively. One sees that within the experimental uncertainties there is an excellent agreement between the present experiment and the theoretical predictions for all of the wall quantities except for δ_w , which is a bit smaller. For reference, the previous measurement in a rectangular cell gives 1.55, 0.8, -0.91 and -0.34 for the corresponding quantities (Sun *et al.* 2008). For the skin-friction coefficient c_f , Verzicco & Camussi (2003) did not give a fitting for $c_f - Re$, but based on Re_σ the exponent for c_f is approximately -1/2.

To further compare the present system with classical BLs, we examine velocity profiles in terms of the wall units. Figure 18 shows the normalized mean horizontal velocity profiles for four different values of Ra taken at $\theta = 3.4^{\circ}$ in a semi-log plot, here $u^+ = u(z)/u_{\tau}$ and $z^+ = z/\delta_w$. The linear scaling of u^+ over z^+ in the viscous sublayer below $z^+ < 5$ is reflected quite well by the measured profiles confirming that the BL is not turbulent in the present range of Ra and Pr. The velocity normalized by wall unit decrease after reaching the maximum value in $z^+ \sim 10$. Compared with the



FIGURE 18. (Colour online) Measured horizontal velocity profiles normalized by wall units for four different *Ra* all taken at tilting angle $\theta = 3.4^{\circ}$.

same quantity measured in the rectangular cell (Sun *et al.* 2008), however, our result shows some deviation from the theoretical profile.

3.6. Dynamical scaling and the shape of velocity profiles in the BL

The dynamic scaling method of Zhou & Xia (2010) has been found to work well when tested in quasi-2D experiment and 2D numerical simulations (Zhou *et al.* 2010, 2011), but it has not been examined in 3D experiments. Here we investigate how effective this method is in our cylindrical geometry. As the method has been well documented elsewhere (Zhou & Xia 2010; Zhou *et al.* 2010, 2011), we will only give a brief description of it here. From the measured instantaneous velocity profile u(z, t) one can obtain an instantaneous viscous BL thickness $\delta_v(t)$ using the same 'slope' method as used for the mean velocity profiles. A local dynamical BL frame can then be constructed by defining the time-dependent rescaled distance $z^*(t)$ from the plate as

$$z^*(t) \equiv z/\delta_v(t). \tag{3.1}$$

The dynamically time-averaged mean velocity profile $u^*(z^*)$ in the dynamical BL frame is then obtained by averaging over all values of u(z, t) that were measured at different discrete times t but at the same relative position z^* , i.e.

$$u^*(z^*) \equiv \langle u(z,t) \mid z = z^* \delta_v(t) \rangle.$$
(3.2)

Figure 19 shows the mean velocity profiles measured in the laboratory and the dynamical frames respectively at the four tilting angles and for comparable values of Ra (as indicated in the figure caption). These results show that the dynamical scaling method is more effective for larger values of θ , i.e. for larger θ profiles obtained in the dynamic frame have faster approach to the asymptotic (maximum) velocity than the corresponding lab frame profiles. This may be understood based on the fact that a larger tilt angle will place stronger restriction on the azimuthal meandering of the LSC so that it has less fluctuations in the horizontal direction perpendicular to the mean flow. We note, however, regardless of the tilt angle, the method works less effectively than it is in quasi-2D experiment and 2D simulations.

A more quantitative approach to characterize the shape of the mean velocity profiles is to investigate their shape factor $H = \delta_d / \delta_m$ defined as the ratio between



FIGURE 19. (Colour online) Comparison between profiles obtained in the dynamical frame $(u^*(z^*), \text{ circles}, \text{ red online})$ and the laboratory frame (u(z), squares, blue online), measured at different tilt angles θ but with comparable values of Ra: (a) $\theta = 0.5^\circ$, $Ra = 5.77 \times 10^8$; (b) $\theta = 1.0^\circ$, $Ra = 6.00 \times 10^8$; (c) $\theta = 2.0^\circ$, $Ra = 5.85 \times 10^8$; and (d) $\theta = 3.4^\circ$, $Ra = 6.69 \times 10^8$. Also shown for comparison is the theoretical PB laminar velocity profile (solid line).

the displacement thickness δ_d and the momentum thickness δ_m , where

$$\delta_d = \int_0^\infty \left[1 - \frac{u(z)}{u_{max}} \right] dz, \quad \text{and} \quad \delta_m = \int_0^\infty \left[1 - \frac{u(z)}{u_{max}} \right] \frac{u(z)}{u_{max}} dz. \tag{3.3}$$

Since u(z) decays after reaching its maximum value, the above integrations are evaluated only over the range from z = 0 to where $u(z) = u_{max}$. For our profiles the obtained shape factors range between 1.9 to 2.3, which are smaller than H = 2.59, the value for a laminar PB BL. A shape factor smaller than the theoretical value means the corresponding profile will approach its asymptotic value (the maximum velocity) slower than the theoretical profile does.

In figure 20 we show the shape factor H for mean velocity profiles obtained in the laboratory and dynamical frames respectively for the four tilting angles and for all Ra measured. The dashed lines in the figure indicate the PB value of 2.59. It is seen that, despite the data scatter, there is a general trend that for both lab- and dynamical-frame profiles the deviation from the PB profile increases with Ra. This is no surprise, since, as the convective flow above the BL becomes more turbulent with increasing Ra, the BL itself will experience stronger fluctuations and hence larger deviations from the same geometry by Stevens *et al.* (2012) for the temperature profile and by Shi *et al.* (2012) and by Scheel *et al.* (2012) for the velocity profile. The second feature is that for



FIGURE 20. (Colour online) The shape factor $H = \delta_d / \delta_m$ of profiles $u^*(z^*)$ (circles, red online) obtained in the dynamical frame and of u(z) (squares, blue online) obtained in the laboratory frame as a function of Ra and for different tilting angles: (a) $\theta = 0.5^\circ$, (b) 1.0°, (c) 2.0° and (d) 3.4°. The dashed line represents the value of 2.59 for the theoretical PB laminar BL.

all θ and Ra the profiles obtained in the dynamical frame in general show some degree of improvement towards that of PB value as compared with those obtained in the laboratory frame. We also note that the degree of improvement does not have an obvious dependence on Ra within the resolution of the measurement, which is also consistent with the findings of Zhou & Xia (2010) and Zhou *et al.* (2010).

Some insight can be obtained by examining the rescaled instantaneous velocity profiles. Figure 21 show examples of rescaled instantaneous velocity profiles, where the distance from the plate has been normalized by the instantaneous BL thickness corresponding to that moment and the velocity has been normalized by the instantaneous maximum horizontal velocity. It is seen that there are quite few cases where the rescaled instantaneous velocity profile is rather close to the theoretical PB profile (up to the point of the maximum velocity) and deviations of the instantaneous shape are likely caused by disturbances such as plume emissions. Also shown in the figure are the shape factor H(t) of these instantaneous profiles. To quantify how the instantaneous profiles are distributed with respect to the PB profile, we examine the probability density function (p.d.f.) of the shape factor difference $\delta H(t) = H(t) - H^{PB}$ where $H^{PB} = 2.59$. Figure 22 plots the p.d.f.s of $\delta H(t)$ for the four tilting angles and for all measured Ra, respectively. Despite the seemingly large variations among them, these p.d.f.s show the general trend that the rescaled instantaneous profiles measured at lower values of Ra ($\leq 1 \times 10^9$) are most of the time having a shape closer to that of the PB profile and that for higher values of Ra the peaks of the p.d.f.s shift to



FIGURE 21. (Colour online) Examples of instantaneous horizontal velocity profiles scaled by the instantaneous viscous BL thickness and the instantaneous maximum velocity (measured at $Ra = 6.00 \times 10^8$ and $\theta = 1^\circ$). The corresponding instantaneous shape factor is also indicated on the plot. The solid curves are the PB velocity profiles.

smaller values of *H*. This indicates that with increasing *Ra* the profiles around the BL thickness becomes more rounded, i.e. the approach to the maximum velocity becomes slower and slower. We further note that these general trends are true across all tilt angles. Another feature observed in the present 3D case is that we did not find any strong correlation between the instantaneous BL thickness δ_v and the velocity u(t) just above the BL. This is in contrast to the finding in the quasi-2D experiment where δ_v and u(t) are found to have a strong negative correlation, i.e. a large velocity above would exert a stronger shear and therefore thins the BL thickness (Zhou & Xia 2010). This result suggests that in certain aspect the BLs in the 3D and in the 2D/quasi-2D cases are dynamically different.

4. Summary and conclusions

We have conducted an experimental study of velocity BL properties in turbulent thermal convection. Two-dimensional high-resolution measurements of the velocity field were made using the PIV technique in a cylindrical cell of height H = 18.6 cm and aspect ratio close to unity, with the Rayleigh number *Ra* varying from 2.4×10^8 to 5.6×10^9 and the Prandtl number *Pr* fixed at ~5.4, with the convection cell tilted with respect to gravity at angles $\theta = 0.5$, 1, 2 and 3.4° , respectively. Measurements made with small θ are aimed at studying BL properties under more steady shear, but the BL itself is assumed to be unperturbed otherwise. For large values of θ we wish



FIGURE 22. (Colour online) Probability density functions of the shape-factor difference δH between those of the rescaled instantaneous profiles and that of the PB profile, measured at (a) $\theta = 0.5^{\circ}$, (b) 1.0° , (c) 2.0° , and (d) 3.4° .

to examine how the BL responds to relatively large perturbations. We also examined effectiveness of the dynamical BL scaling method in a 3D system.

It is found that the Reynolds number $Re \ (=U_{max}H/\nu)$ based on the maximum mean horizontal velocity scales with Ra as $Re \sim Ra^{0.43}$ and the Reynolds number Re_{σ} $(=\sigma_{max}H/\nu)$ based on the maximum r.m.s. velocity scales with Ra as $Re_{\sigma} \sim Ra^{0.55}$. Both exponents do not seem to have a dependence on the tilt angle. On the other hand, the amplitude of Re seem to show a weak increasing trend with θ .

With the measured horizontal velocity, we obtain two length scales, i.e. the viscous BL thickness δ_v based on the mean horizontal velocity profile and the length scale δ_σ based on the r.m.s. horizontal velocity profile. It is found that as far as scaling with the Reynolds number *Re* is concerned, the behaviour of δ_v can be divided into two regimes according to the tilting angle of the cell. For $\theta \leq 1^\circ$, it is found that $\delta_v \sim Re^{-0.46\pm0.03}$, which within experimental uncertainty may be considered to be consistent with that of the Prandtl–Blasius BL. For $\theta > 1^\circ$, the absolute value of the exponent is found to increase with θ and in this case the BL may be considered to be strongly perturbed. It is found that similar to δ_v , the absolute value of the scaling exponent of δ_σ increases with increasing θ .

It is also found that tilting the cell modifies the velocity profile in the BL region, i.e. for different tilt angles the shape of profiles is different. But for the same tilting angle the velocity profiles measured at different Ra can be brought to collapse on a single curve when the mean velocity is normalized by the maximum velocity U_{max} and the distance from the plate by the viscous BL thickness δ_v .

P. Wei and K.-Q. Xia

With simultaneously measured horizontal and vertical velocity components, we also obtain the Reynolds stress τ_R in the velocity BL. It is found that τ_R is stronger in the mixing zone comparing with the rectangular cell. The wall quantities such as the wall shear stress τ_w , the viscous sublayer δ_w , the friction velocity u_{τ} are also measured. Their scaling exponents with the Reynolds number are found to be in excellent agreement with those predicted for classical laminar BLs, which is also consistent with the measurement in rectangular cell.

Regarding the dynamical scaling method, we found that in general it leads to profiles that are closer to the theoretical PB profile when the cell is tilted at a larger angle. With respect to the influence of Ra, it is found that in general the mean velocity profile sampled in both the laboratory and dynamical frames are closer to the PB profile at smaller values of Ra than they are at larger Ra, which is consistent with findings from previous DNS studies. Moreover, it is found that for smaller values of $Ra \ (\leq 1 \times 10^9)$ the p.d.f.s of the shape factor H for the rescaled instantaneous profiles exhibit a peak close to that for the PB profile, whereas for larger values of Ra the peaks shift to smaller values of H, indicating the profile's approach to the maximum velocity becomes slower and slower with increasing Ra. Another finding is that the effectiveness of the dynamical scaling method, in terms of its ability of bringing the mean velocity profile closer to that of PB profile, does not have any apparent dependence on Ra. Our general conclusion is that as far as the effectiveness of the dynamical scaling method is concerned the influence of tilting angle is much smaller than that of the Rayleigh number Ra. We note that the Prandtl–Blasius BL theory is a 2D model, so it is perhaps no surprise that the dynamic method works less well in three dimensions than in two dimensions.

Acknowledgements

We would like to thank X.-D. Shang and S.-Q. Zhou for kindly making their PIV facility available to us and L. Qu, Y.-C. Xie and S.-D. Huang for helping with the experiment. This work was supported in part by the Hong Kong Research Grants Council (RGC) under Project Numbers CUHK404409 and CUHK403811, and in part by a RGC Direct Grant (project code 2060441).

REFERENCES

- AHLERS, G., BROWN, E. & NIKOLAENKO, A. 2006 The search for slow transients, and the effect of imperfect vertical alignment, in turbulent Rayleigh–Bénard convection. J. Fluid Mech. 557, 347–367.
- AHLERS, G., GROSSMANN, S. & LOHSE, D. 2009 Heat transfer and large-scale dynamics in turbulent Rayleigh–Bénard convection. *Rev. Mod. Phys.* 81, 503–537.
- ASHKENAZI, S. & STEINBERG, V. 1999 High Rayleigh number turbulent convection in a gas near the gas-liquid critical point. *Phys. Rev. Lett.* **83**, 3641–3644.
- BELMONTE, A., TILGNER, A. & LIBCHABER, A. 1993 Boundary layer length scales in thermal turbulence. *Phys. Rev. Lett.* **70**, 4067–4070.
- BELMONTE, A., TILGNER, A. & LIBCHABER, A. 1994 Temperature and velocity boundary layers in turbulent convection. *Phys. Rev.* E 50, 269–279.
- BROWN, E., FUNFSCHILLING, D. & AHLERS, G. 2007 Anomalous Reynolds-number scaling in turbulent Rayleigh–Bénard convection. J. Stat. Mech. Theory E. 2007 (10), P10005.
- BROWN, E., NIKOLAENKO, A. & AHLERS, G. 2005 Reorientation of the large-scale circulation in turbulent Rayleigh–Bénard convection. *Phys. Rev. Lett.* 95, 084503.
- CHAVANNE, X., CHILLÀ, F., CHABAUD, B., CASTAING, B. & HÉBRAL, B. 2001 Turbulent Rayleigh–Bénard convection in gaseous and liquid He. *Phys. Fluids* **13** (5), 1300–1320.

- CHILLÀ, F. & SCHUMACHER, J. 2012 New perspectives in turbulent Rayleigh–Bénard convection. *Eur. Phys. J.* E 35, 1–25.
- DUBRULLE, B. 2001 Momentum transport and torque scaling in Taylor–Couette flow from an analogy with turbulent convection. *Eur. Phys. J. B* 21, 295.
- DUBRULLE, B. 2002 Scaling in large Prandtl number turbulent thermal convection. *Eur. Phys. J.* B **28**, 361–367.
- DU PUITS, R., RESAGK, C. & THESS, A. 2007*a* Mean velocity profile in confined turbulent convection. *Phys. Rev. Lett.* **99**, 234504.
- DU PUITS, R., RESAGK, C., TILGNER, A., BUSSE, F. H. & THESS, A. 2007b Structure of thermal boundary layers in turbulent Rayleigh–Bénard convection. J. Fluid Mech. 572, 231–254.
- GROSSMANN, S. & LOHSE, D. 2000 Scaling in thermal convection: a unifying theory. J. Fluid Mech. 407, 27–56.
- GROSSMANN, S. & LOHSE, D. 2001 Thermal convection for large Prandtl number. *Phys. Rev. Lett.* **86**, 3316–3319.
- GROSSMANN, S. & LOHSE, D. 2002 Prandtl and Rayleigh number dependence of the Reynolds number in turbulent thermal convection. *Phys. Rev.* E 66, 016305.
- GROSSMANN, S. & LOHSE, D. 2004 Fluctuations in turbulent Rayleigh–Bénard convection: the role of plumes. *Phys. Fluids* **16**, 4462–4472.
- KADANOFF, L. P. 2001 Turbulent heat flow: structures and scaling. Phys. Today 54 (8), 34-39.
- LAM, S., SHANG, X.-D., ZHOU, S.-Q. & XIA, K.-Q. 2002 Prandtl number dependence of the viscous boundary layer and the Reynolds numbers in Rayleigh–Bénard convection. *Phys. Rev.* E 65, 066306.
- LI, L., SHI, N., DU PUITS, R., RESAGK, C., SCHUMACHER, J. & THESS, A. 2012 Bounday layer analysis in turbulent Rayleigh–Bénard convection in air: experiment versus simulation. *Phys. Rev.* E 86, 026315.
- LOHSE, D. & XIA, K.-Q. 2010 Small-scale properties of turbulent Rayleigh–Bénard convection. *Annu. Rev. Fluid Mech.* **42**, 335–364.
- LUI, S. L. & XIA, K.-Q. 1998 Spatial structure of the thermal boundary layer in turbulent convection. *Phys. Rev.* E 57, 5494–5503.
- NAERT, A., SEGAWA, T. & SANO, M. 1997 High-Reynolds-number thermal turbulence in mercury. *Phys. Rev.* E 56, 1302–1305.
- NI, R., HUANG, S.-D. & XIA, K.-Q. 2011*a* Local energy dissipation rate balances local heat flux in the centre of turbulent thermal convection. *Phys. Rev. Lett.* **107**, 174503.
- NI, R., HUANG, S.-D. & XIA, K.-Q. 2012 Lagrangian acceleration measurements in convective thermal turbulence. J. Fluid Mech. 692, 395–419.
- NI, R., ZHOU, S.-Q. & XIA, K.-Q. 2011b An experimental investigation of turbulent thermal convection in water-based alumina nanofluid. *Phys. Fluids* 23, 022005.
- QIU, X.-L. & XIA, K.-Q. 1998a Spatial structure of the viscous boundary layer in turbulent convection. *Phys. Rev.* E 58, 5816.
- QIU, X.-L. & XIA, K.-Q. 1998b Viscous boundary layers at the sidewall of a convection cell. Phys. Rev. E 58, 486.
- SCHEEL, J. D., KIM, E. & WHITE, K. R. 2012 Thermal and viscous boundary layers in turbulent Rayleigh–Bénard convection. J. Fluid Mech. **711**, 281–305.
- SCHLICHTING, H. & GERSTEN, K. 2000 Boundary Layer Theory. Springer.
- SHI, N., EMRAN, M. S. & SCHUMACHER, J. 2012 Boundary layer structure in turbulent Rayleigh-Bénard convection. J. Fluid Mech. 706, 5-33.
- SHRAIMAN, B. I. & SIGGIA, E. D. 1990 Heat transport in high-Rayleigh number convection. *Phys. Rev.* A **42**, 3650–3653.
- SIGGIA, E. D. 1994 High Rayleigh number convection. Annu. Rev. Fluid Mech. 26, 137-168.
- STEVENS, R. J. A. M., ZHOU, Q., GROSSMANN, S., VERZICCO, R., XIA, K.-Q. & LOHSE, D. 2012 Thermal boundary layer profiles in turbulent Rayleigh–Bénard convection in a cylindrical sample. *Phys. Rev.* E 85, 027301.
- SUN, C., CHEUNG, Y.-H. & XIA, K.-Q. 2008 Experimental studies of the viscous boundary layer properties in turbulent Rayleigh–Bénard convection. J. Fluid Mech. 605, 79–113.

- SUN, C., XI, H.-D. & XIA, K.-Q. 2005a Azimuthal symmetry, flow dynamics, and heat flux in turbulent thermal convection in a cylinder with aspect ratio one-half. *Phys. Rev. Lett.* 95, 074502.
- SUN, C. & XIA, K.-Q. 2005 Scaling of the Reynolds number in turbulent thermal convection. *Phys. Rev.* E **72**, 067302.
- SUN, C., XIA, K.-Q. & TONG, P. 2005b Three-dimensional flow structures and dynamics of turbulent thermal convection in a cylindrical cell. *Phys. Rev.* E **72**, 026302.
- SUN, C., ZHOU, Q. & XIA, K.-Q. 2006 Cascades of velocity and temperature fluctuations in buoyancy-driven turbulence. *Phys. Rev. Lett.* 97, 144504.
- TILGNER, A., BELMONTE, A. & LIBCHABER, A. 1993 Temperature and velocity profiles of turbulence convection in water. *Phys. Rev.* E 47, 2253–2256.
- VERZICCO, R. & CAMUSSI, R. 1999 Prandtl number effects in convective turbulence. J. Fluid Mech. 383, 55-73.
- VERZICCO, R. & CAMUSSI, R. 2003 Numerical experiments on strongly turbulent thermal convection in a slender cylindrical cell. J. Fluid Mech. 447, 19–49.
- WANG, J. & XIA, K.-Q. 2003 Spatial variations of the mean and statistical quantities in the thermal boundary layers of turbulent convection. *Eur. Phys. J. B* 32, 127–136.
- WU, X.-Z. & LIBCHARBER, A. 1991 Non-Boussinesq effects in free thermal convection. *Phys. Rev.* A 43, 2833–2839.
- XI, H.-D., ZHOU, Q. & XIA, K.-Q. 2006 Azimuthal motion of the mean wind in turbulent thermal convection. *Phys. Rev.* E **73**, 056312.
- XIA, K.-Q., SUN, C. & ZHOU, S.-Q. 2003 Particle image velocimetry measurement of the velocity field in turbulent thermal convection. *Phys. Rev.* E 68, 066303.
- XIA, K.-Q., XIN, Y.-B. & TONG, P. 1995 Dual-beam incoherent cross-correlation spectroscopy. J. Opt. Soc. Am. A 12, 1571.
- XIE, Y.-C., WEI, P. & XIA, K.-Q. 2013 Dynamics of the large-scale circulation in high Prandtl number turbulent thermal convection. J. Fluid Mech. 717, 322–346.
- XIN, Y.-B. & XIA, K.-Q. 1997 Boundary layer length scales in convective turbulence. *Phys. Rev.* E 56, 3010.
- XIN, Y.-B., XIA, K.-Q. & TONG, P. 1996 Boundary layers in turbulent convection. *Phys. Rev. Lett.* 77, 1266.
- ZHOU, Q., STEVENS, R. J. A. M., SUGIYAMA, K., GROSSMANN, S., LOHSE, D. & XIA, K.-Q. 2010 Prandtl-Blasius temperature and velocity boundary layer profiles in turbulent Rayleigh-Bénard convection. J. Fluid Mech. 664, 297–312.
- ZHOU, Q., SUGIYAMA, K., STEVENS, R. J. A. M., GROSSMANN, S., LOHSE, D. & XIA, K.-Q. 2011 Horizontal structures of velocity and temperature boundary layers in two-dimensional numerical turbulent Rayleigh–Bénard convection. *Phys. Fluids* 23, 125104.
- ZHOU, Q., SUN, C. & XIA, K.-Q. 2008 Experimental investigation of homogeneity, isotropy, and circulation of the velocity field in buoyancy-driven turbulence. J. Fluid. Mech. 598, 361–372.
- ZHOU, S.-Q. & XIA, K.-Q. 2002 Plume statistics in thermal turbulence: mixing of an active scalar. *Phys. Rev. Lett.* **89**, 184502.
- ZHOU, Q. & XIA, K.-Q. 2010 Measured instantaneous viscous boundary layer in turbulent Rayleigh–Bénard convection. *Phys. Rev. Lett.* **104**, 104301.