

The dynamical states of a prolate spheroidal particle suspended in shear flow as a consequence of particle and fluid inertia

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The rotational motion of a prolate spheroidal particle suspended in shear flow is studied by a lattice Boltzmann method with external boundary forcing (LB-EBF). It has previously been shown that the case of a single neutrally buoyant particle is a surprisingly rich dynamical system that exhibits several bifurcations between rotational states due to inertial effects. It was observed that the rotational states were associated with either fluid inertia effects or particle inertia effects, which are always in competition. The effects of fluid inertia are characterized by the particle Reynolds number $Re_p = 4Ga^2/\nu$, where G is the shear rate, a is the length of the particle major semi-axis and ν is the kinematic viscosity. Particle inertia is associated with the Stokes number $St = \alpha Re_p$, where α is the solid-to-fluid density ratio. Previously, the neutrally buoyant case ($St = Re_p$) was studied extensively. However, little is known about how these results are affected when $St \neq Re_p$, and how the aspect ratio r_p (major axis/minor axis) influences the competition between fluid and particle inertia in the absence of gravity. This work gives a full description of how prolate spheroidal particles in the range $2 \leq r_p \leq 6$ behave depending on the chosen St and Re_p . Furthermore, consequences for the rheology of a dilute suspension containing such particles are discussed. Finally, grid resolution close to the particle is shown to affect the quantitative results considerably. It is suggested that this resolution is a major cause of quantitative discrepancies between different studies. Thus, the results of this work and previous direct numerical simulations of this problem should be regarded as qualitative descriptions of the physics involved, and more refined methods must be used to quantitatively pinpoint the transitions between rotational states.

Key words: bifurcation, complex fluids, multiphase and particle-laden flows

1. Introduction

Understanding the flow of particle suspensions is of great importance in many industrial, biological and geophysical applications. Such dispersed particle flows

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have certain properties that are of interest. For the quality of coatings (Li, Zhu & Zhang 2005), the capture rate of plankton (Pésceli, Trulsen & Fiksen 2012) and the formation of cloud droplets (Balkovsky, Falkovich & Fouxon 2001), the spatial distribution of particles within the suspending fluid is of great importance. Particle suspensions usually also have a shear-dependent effective viscosity. In many cases, such as in determining the texture and pumping requirements of food (Bayod & Willers 2002), understanding the rheology of the suspensions is necessary for design and optimization.

A common way of modelling such properties in suspensions involves assuming that the particles are perfectly spherical, due to the accessibility of models for particle–fluid interactions (Crowe *et al.* 2012). However, assuming the particles to be spherical is insufficient for numerous applications, such as in predicting particle motion of fibres in paper pulp (Lundell, Söderberg & Alfredsson 2011) and composite moulds (Le *et al.* 2008), the motion of asbestos fibres in the respiratory system (Miserocchi *et al.* 2008) or the motion and growth of snow crystals in the atmosphere (Gavze, Pinsky & Khain 2012). Having a suspension of non-spherical particles presents an additional challenge, as properties become dependent not only on the spatial distribution of particles but also on the orientation distribution; in order to understand the physics of the collective orientation distribution of non-spherical particles, one needs to know how a single particle rotates due to local velocity gradients in the flow, and how this affects the rheology.

In the present work, we consider a single prolate spheroidal particle in a simple shear flow, which is ideally created through the induced flow of two parallel, opposite moving walls (figure 1). The flow will affect the particle through forces on the particle surface and give rise to a torque on the particle. This in turn yields a rotational motion. In the absence of fluid and particle inertia, analytical expressions for this rotational motion were derived by Jeffery (1922). The particle was found to move periodically in one of an infinite number of closed kayaking orbits around the vorticity axis (figure 2c; the kayaking state can be described as a precession and nutation around the vorticity axis). It was later seen in experiments (Taylor 1923; Binder 1939; Karnis, Goldsmith & Mason 1963) that upon including fluid inertia, the particle has a preferred orbit of tumbling (rotation around the minor axis, which is aligned with the vorticity axis; see figure 2a) or log-rolling (rotation around the major axis, which is aligned with the vorticity axis; see figure 2b), depending on the aspect ratio r_p and the particle Reynolds number Re_p .

Theoretical studies have shown that the leading effect of fluid inertia is a drift towards log-rolling for nearly spherical particles (Saffman 1956; Subramanian & Koch 2006), whereas slender fibers drift towards tumbling (Subramanian & Koch 2005). Numerical results have also shown that there exists a critical Reynolds number $Re_p = Re_c$, above which a motionless (steady) state exists (figure 2f), such that the particle major axis is perpendicular to the vorticity axis and makes a small angle ϕ_c to the flow direction (Aidun, Lu & Ding 1998; Ding & Aidun 2000; Zettner & Yoda 2001; Subramanian & Koch 2005; Huang, Wu & Lu 2012a; Huang *et al.* 2012b; Rosén, Lundell & Aidun 2014).

Particle inertia, associated with the Stokes number St , leads to a drift towards tumbling (Subramanian & Koch 2006; Lundell & Carlsson 2010) if fluid inertia is negligible. Furthermore, very heavy particles have been observed to undergo a transition from tumbling (where angular velocity is dependent on orientation) to rotating (where angular velocity is constant) (Lundell & Carlsson 2010; Nilsen & Andersson 2013). Computer simulations that include fluid and particle dynamics have

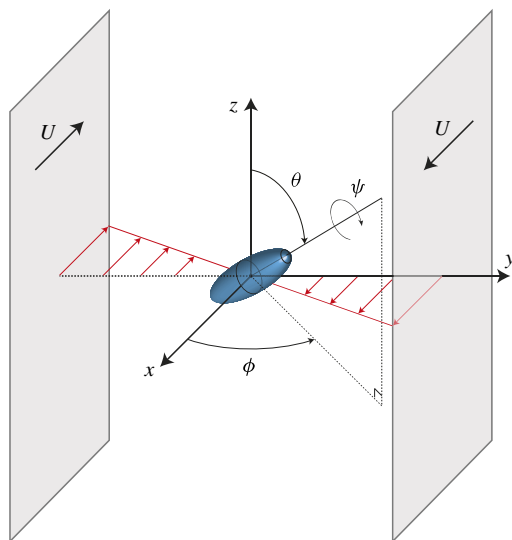


FIGURE 1. (Colour online) A spheroidal particle is placed in a linear shear flow generated by two parallel, opposite moving walls; the orientation of the particle is described by the angles ϕ and θ ; the angle ψ is of less interest for spheroids due to the rotational symmetry of the particle.

proven to be a very useful tool for studying the full three-dimensional rotational behaviour of particles. Qi & Luo (2002, 2003) found that a neutrally buoyant spheroid (with the particle having the same density as the fluid) of aspect ratio (major axis/minor axis) $r_p = 2$ was tumbling at low Re_p but log-rolling at high Re_p . However, their study considered only two initial orientations, and from this they claimed that the final rotational state was independent of initial orientation. Recent numerical work has shown that there exist several other equilibrium rotational states when Re_p is increased, including inclined rolling, inclined kayaking and kayaking (Yu, Phan-Thien & Tanner 2007; Huang *et al.* 2012*b*; Rosén *et al.* 2014). These rotational states are illustrated in figure 2(c–e). It was also observed that these states always coexist with a stable tumbling solution, and that the final state depends on the initial orientation. Even though previous numerical works (Qi & Luo 2002, 2003; Yu *et al.* 2007; Huang *et al.* 2012*b*; Rosén *et al.* 2014) agree qualitatively, the quantitative critical values of Re_p characterizing the transitions differ between the studies. The reasons for these discrepancies are believed to be differences in spatial resolution and the fact that determination of these parameters is very sensitive to how well the boundary layer around the particle is resolved. This issue will be investigated thoroughly in the present work. What is also missing from most numerical studies is discussion of the fact that Re_p and St are dependent on one another through the relation $St = \alpha Re_p$, where α is the solid-to-fluid density ratio. This relationship means that even a neutrally buoyant particle will experience particle inertial effects for $Re_p > 0$, since $St = Re_p$ in that case.

Each rotational state mentioned here is associated with fluid and/or particle inertial effects being dominant, and can be seen as a combination of two dynamical states for the fluid and the particle. The dominant inertial effects for the different states are summarized in table 1. Generally, particle inertia is the dominant effect if the final rotational state is a time-periodic state for both particle and fluid, whereas fluid inertia

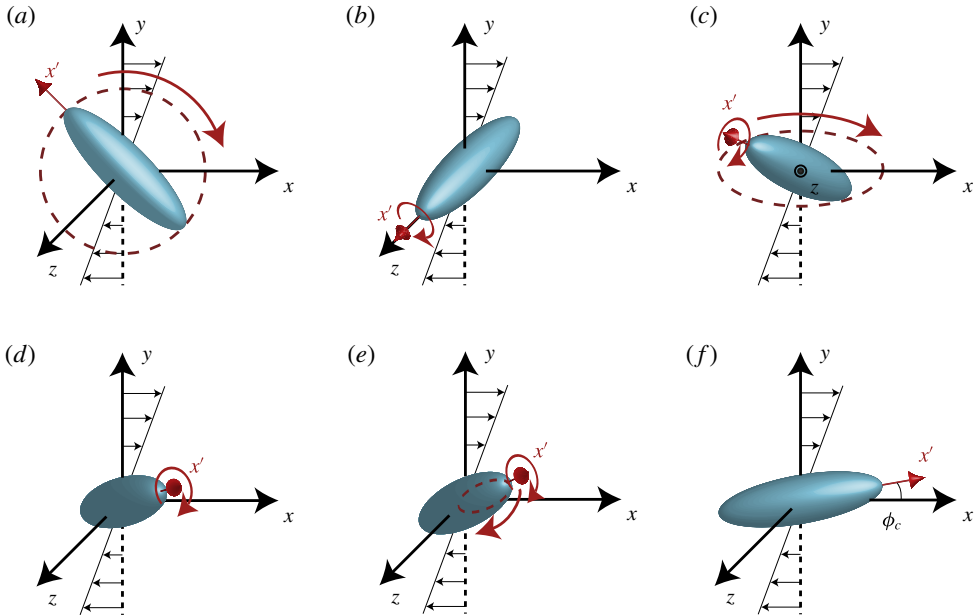


FIGURE 2. (Colour online) Rotational states of the prolate spheroid in a linear shear flow: (a) tumbling ($\dot{\phi} = f(\phi)$) or rotating ($\dot{\phi} = \text{const.}$); (b) log-rolling; (c) kayaking; (d) inclined rolling; (e) inclined kayaking; (f) steady state (where the particle symmetry axis is located in the flow-gradient plane, making an angle ϕ_c with the flow (i.e. x) direction); (a, f) are planar rotational states ($\theta = \pi/2$), while (b–e) are non-planar ($0 \leq \theta < \pi/2$); the rotational states can also be viewed in a movie clip provided in the supplementary data.

dominates if the final rotational state is a steady-state solution for the fluid. All the rotational states discussed in this work are illustrated in a movie clip, available as supplementary data at <http://dx.doi.org/10.1017/jfm.2015.127>.

Just as in previous work, a distinction will be made between planar rotational states and non-planar rotational states. In planar states, including the tumbling, rotating and steady states, the particle symmetry axis is located in the flow-gradient plane ($\theta = \pi/2$); in the non-planar states, i.e. log-rolling, inclined rolling, inclined kayaking and kayaking, the particle symmetry axis is located at $0 \leq \theta < \pi/2$.

The previous results indicate that the final rotational state is dependent not only on Re_p but also on St , since these parameters control the competition between fluid inertia and particle inertia. This competition determines most of the transitions between the rotational states in table 1, and examining the state space in the Re_p – St plane is necessary to understanding the physical mechanisms. At this point, it is appropriate to comment on the effect of gravity, since $St \neq Re_p$ means that the solid-to-fluid density ratio is not equal to 1. Obviously, sedimentation in itself affects particle rotation at non-zero Reynolds numbers. When combined with shear, a huge parameter space opens up (both the direction and the strength of gravity must be considered), but this is beyond the scope of the present study. For the case of small Re_p and St and nearly spherical particles, a perturbation analysis including sedimentation was done by Subramanian & Koch (2006), but for wider ranges of Re_p , St and r_p complete parameter sweeps are unfeasible. Nevertheless, the identification of critical transitions in the Re_p – St plane will help to reduce the complexity of future studies.

Rotational state (descriptive)	Rot. state (abbrev.)	Dynamical state (fluid)	Dynamical state (particle)	Dominant inertial effects P = particle inertia F = fluid inertia
Tumbling	T	Time-periodic	Time-periodic	P
Rotating	R	Time-periodic	Time-periodic	P
Log-rolling	LR	Steady state	Time-periodic	F
Kayaking	K	Time-periodic	Quasi-periodic	P and F
Inclined rolling	IR	Steady state	Time-periodic	F
Inclined kayaking	IK	Time-periodic	Quasi-periodic	P and F
Steady state	S	Steady state	Steady state	F

TABLE 1. Description of the rotational states of the particle, together with their abbreviations, and the corresponding states in terms of fluid and particle dynamics.

Similar arguments can be made when it comes to other aspects that affect rotation, such as Brownian diffusion.

Still, previous studies have almost solely been concerned with neutrally buoyant ($\alpha = 1$) spheroidal particles of fixed aspect ratio. Here, we extend the work of Rosén *et al.* (2014) to cover all the rotational states of a prolate spheroidal particle in a Re_p - St parameter space and also look closely at the transitions as functions of the particle aspect ratio r_p . The aim of this work is to predict the rotational behaviour of a prolate spheroidal particle with any density and aspect ratio in the absence of gravity. We will also discuss the implications this has for the rheological behaviour of dilute suspensions of such particles.

The outline of the paper is as follows. Section 2 will cover the flow problem and the range of parameters used in this numerical study. Section 3 will explain in detail the important knowledge gained from the works of Lundell & Carlsson (2010) (only particle inertial effects) and Rosén *et al.* (2014) (only neutrally buoyant particles of $r_p = 4$) and show that a general description of the flow problem has not yet been obtained. In § 4, we first discuss the difficulties in obtaining quantitative results due to the sensitivity to grid resolution; secondly, the qualitative rotational dynamics in the entire Re_p - St parameter space will be described for a particle of aspect ratio $r_p = 4$ (the same aspect ratio as in Rosén *et al.* (2014)). Finally, in § 5 we discuss how the dynamics changes with the aspect ratio r_p .

2. Method

The method used in the present work is the lattice Boltzmann method with external boundary forcing (LB-EBF). A description of the computational method can be found in appendix A, and further details are provided in the work by Wu & Aidun (2010). For the case of a spheroidal particle in a shear flow, the method was also validated and verified by Rosén *et al.* (2014) and so will not be repeated here. Only the case of $\alpha \geq 1$ (i.e. the particle has density equal to or higher than the surrounding fluid) will be simulated in this study, but the explanation of the physical processes will allow us to predict the behaviour of light particles (with $\alpha < 1$) as well.

2.1. The flow problem

The problem is set up by simulating the flow between two parallel moving walls located at $y = 0$ and $y = N$ moving with velocities U and $-U$, respectively, in the

x direction. The boundaries at $x=0$, $x=N$, $z=0$ and $z=N$ are treated with periodic boundary conditions. The linear shear flow has a shear rate of $G=2U/N$, which is also the relevant inverse time scale for the flow problem. The prolate particle, placed in the middle of the domain, is described by the equation

$$\frac{x'^2}{a^2} + \frac{y'^2}{b^2} + \frac{z'^2}{b^2} = 1 \quad (2.1)$$

where (x', y', z') denotes a body-fixed coordinate system. The length of the prolate particle symmetry axis a is greater than the equatorial radius b , and we define an aspect ratio $r_p = a/b$. The orientation and rotation of the particle are defined according to figure 1 with the Euler angles (ϕ, θ, ψ) , and a corresponding rotational velocity vector is defined as $\boldsymbol{\omega} = (\dot{\phi}, \dot{\theta}, \dot{\psi})$. The important dimensionless numbers in this problem are the aspect ratio r_p , the particle Reynolds number Re_p (associated with fluid inertia) and the Stokes number St (associated with particle inertia). The latter two quantities are defined as

$$Re_p = \frac{4Ga^2}{\nu}, \quad (2.2)$$

$$St = \alpha Re_p, \quad (2.3)$$

where ν is the kinematic viscosity of the fluid and α is the solid-to-fluid density ratio.

The aim of the present work is to determine the rotational motion of a spheroidal particle solely due to shear, while all forces giving a translational motion (e.g. gravity) are neglected.

The domain is chosen to be cubic with confinement $\kappa = 2a/N = 0.2$ to ensure results independent of domain size, as demonstrated by Rosén *et al.* (2014). A too-high confinement can influence the dynamics since streamlines close to the walls are forced to be in the horizontal direction. To get good resolution of the surface forces on the particle, the minor semi-axis was limited to $b \geq 3$ and the major semi-axis restricted to $a \geq 12$, where the length unit is the side of the cubic lattice. The lower limits were set by considering the smallest particle that still undergoes the same sequence of transitions as a larger particle. In particular, a neutrally buoyant particle of $a = 24$ and $b = 6$ was simulated with $N = 240$ and $G = 1/1200$ and was observed to undergo the same transitions with increasing Re_p (not presented in detail here) as a particle with $a = 12$ and $b = 3$. We also mention here that the correct Reynolds number for describing the flow around the minor axis should be equal to Re_p/r_p^2 . This means that the flow is much less complex around the minor dimension compared with that around the major dimension, and therefore the fluid–solid interaction is better captured when particle is in a non-planar motion (Do-Quang *et al.* 2014). Consequently, mesh convergence is ensured for the dynamics and any influence of a low mesh number can be ruled out. To simulate a spheroidal particle of aspect ratio $r_p > 4$ (i.e. $a > 12$), the domain size was increased to ensure that $\kappa = 0.2$, but with the shape remaining cubic. To simulate such low confinement, the range of Re_p is constrained since the flow becomes unstable at a too-high channel Reynolds number $Re_H = \kappa^{-2} Re_p$. In the range of $Re_p = 10\text{--}300$, the particle could reach its final rotational state before the flow becomes unsteady and non-periodic; hence this is considered a valid range for the problem. The lower limit on Re_p was set to ensure a low value of the lattice Boltzmann viscosity ν , which is connected to the Knudsen number through $Kn = \sqrt{3}\nu/(2a)$, and thus to ensure the absence of slipping effects at constant shear rate G (Rosén *et al.* 2014).

In general, throughout this study, all transitional Reynolds numbers are determined with an accuracy of $\Delta Re_p = 1$, meaning that if a transition is found at Re_p and not at $Re_p + \Delta Re_p$, a transition is said to have occurred at $Re_p + \Delta Re_p/2$. The same determination is used for critical Stokes numbers, with $\Delta St = 10$.

3. Summary of previous knowledge

3.1. Heavy particle in creeping shear flow

Jeffery (1922) derived analytical expressions for the torque on an ellipsoidal particle in a linear Stokes flow ($Re_p = 0$) for any angular velocity vector. By coupling this torque to the equation of motion for the particle, a differential equation could be obtained and integrated, which was done numerically by Lundell & Carlsson (2010). This differential equation in non-dimensional form has a dependency on the Stokes number $St = \alpha Re_p$, which determines the effect of particle inertia relative to viscous forces. Even though the torque was derived for $Re_p = 0$, a solution could be found for finite St . The Stokes flow solution can, however, be considered valid even in a low- Re_p regime. As mentioned in the introduction, Lundell & Carlsson (2010) and Nilsen & Andersson (2013) reported a transition for elongated heavy spheroids in linear shear flow from tumbling to rotating, due to particle inertial forces dominating viscous damping forces. The period is found to be a decreasing function of the Stokes number, with the period $GT_J = 2\pi(r_p^{-1} + r_p)$ obtained at $St = 0$ and the period $GT_H = 4\pi$ as $St \rightarrow \infty$. The transition is characterized by $St_{0.5}$, which is defined as the Stokes number at which the spheroid rotates with a period of $(GT_J + GT_H)/2$. For a particle of $r_p = 4$, this number is $St_{0.5} = 307$, and was observed by Lundell & Carlsson (2010) to be an increasing function of r_p . It should be mentioned that for a particle with this quite low value of r_p , the transition from tumbling to rotating will be rather smooth since $GT_J = 8.5\pi$, which is not much larger than $GT_H = 4\pi$. For a more slender particle the transition is sharper, as was also observed by Lundell & Carlsson (2010) and Nilsen & Andersson (2013).

The transition is illustrated in figure 3, where the angular velocity in the final rotational state is plotted against the angle at different St values. In the figure, a comparison is also made with the present LB-EBF model, where a prolate spheroid with $r_p = 4$ ($a = 12$, $b = 3$) is simulated for $Re_p = 0.5$ (with shear rate lowered to $G = 1/34560$ to ensure low Kn) in a computational domain of $N = 120$. The density ratio was chosen to be $\alpha = 2 \times 10^0, 2 \times 10^1, \dots, 2 \times 10^5$, leading to the same range of Stokes numbers as in the analytical case. The simulated results are compared with the analytical results at the same Stokes number. The simulated results agree well with the analytical ones, with some small discrepancies which can be explained by the slight non-zero Re_p in the simulations.

3.2. Neutrally buoyant particle ($\alpha = 1$, $r_p = 4$) at $Re_p > 0$

Rosén *et al.* (2014) investigated a prolate spheroid with $r_p = 4$ and reported that the particle inertia and fluid inertia are always in competition, even for a neutrally buoyant particle. As fluid inertia, associated with Re_p , increases, particle inertia also increases since it is associated with the Stokes number $St = 1 \cdot Re_p$. Therefore, if the particle is released at an orientation where the symmetry axis gains enough angular momentum, centrifugal forces will take it to a planar tumbling motion. Otherwise, fluid inertia effects will dominate, leading to non-planar motion (i.e. log-rolling, inclined rolling, inclined kayaking or kayaking). At $Re_p > Re_c$, fluid inertia can be strong enough

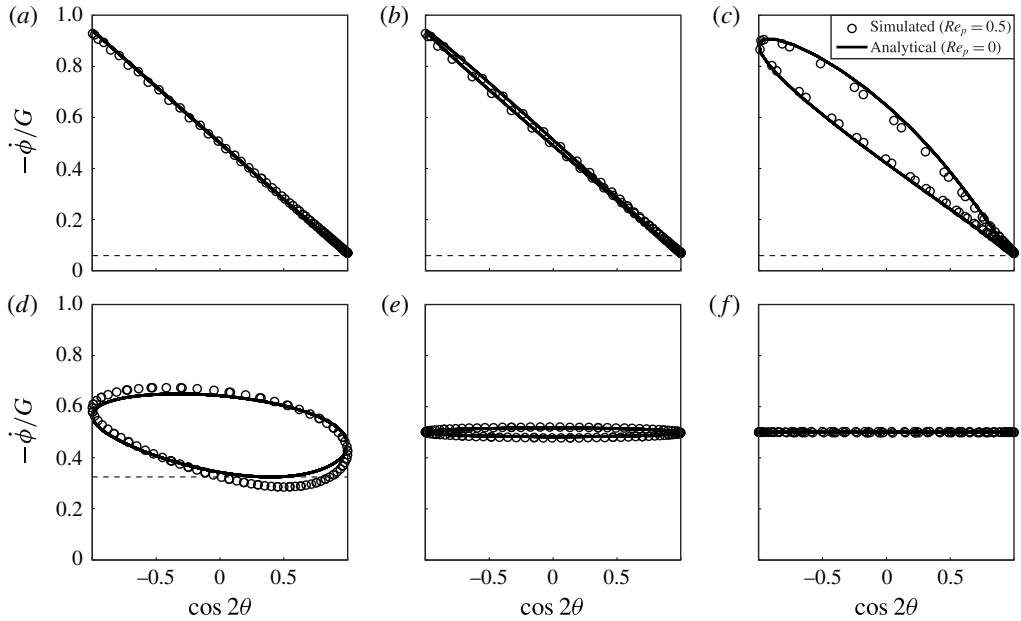


FIGURE 3. Phase diagrams showing the motion of a spheroid with $r_p = 4$ in a flow with negligible fluid inertia: our numerical model at $Re_p = 0.5$ (circles) is compared with analytical results from Lundell & Carlsson (2010) (solid lines) for: (a) $St = 1$; (b) $St = 10$; (c) $St = 100$; (d) $St = 1000$; (e) $St = 10\,000$; (f) $St = 100\,000$. Panels (a)–(f) show the transitions from tumbling to rotating; in each panel the dashed line indicates the location of $\dot{\phi}_{min}$ during a period.

to completely stop the motion, and the particle will end up in a steady state. The transitions for a neutrally buoyant particle with $r_p = 4$, as found by Rosén *et al.* (2014), are summarized in table 2, and the dynamical behaviour at $Re_p = 60$ (T/LR), 70 (T/IR), 74 (T/IK) and 80 (T) is shown in figure 4.

In the subcritical Hopf bifurcation at Re_{LR} , the log-rolling solution is stabilized, and there are two coexisting stable rotational states, one of which is associated with fluid inertial effects (log-rolling) and the other with particle inertial effects (tumbling). An unstable limit cycle (sketched as a dashed curve in figure 4*b,c*) is created in this transition. This limit cycle is located between the two solutions and distinguishes the initial orientations that lead to each rotational state. To illustrate this, a particle is simulated at $Re_p = 60, 70, 74$ and 80 with five different initial orientations according to

$$\left. \begin{aligned} (\phi_0, \theta_0)_1 &= (0, 0), \\ (\phi_0, \theta_0)_{2-5} &= \left(\frac{\pi n}{2}, \frac{\pi m}{8}\right), \quad n = 0, 1 \text{ and } m = 1, 2. \end{aligned} \right\} \quad (3.1)$$

In figure 4*(a)* the projection $(p_x, p_y) = (\sin \theta \cos \phi, \sin \theta \sin \phi)$ of a simulated particle trajectory is displayed, and figure 4*(b)* shows the projection of the five initial orientations from (3.1). Figure 4*(c)* shows that at $Re_p = 60$, the initial orientations 1, 2 and 4 are inside the unstable limit cycle and lead to log-rolling, whereas the initial orientations 3 and 5 are outside the limit cycle and end up in the tumbling state. The size of the limit cycle thus determines how probable it is that a random orientational configuration of particles, initialized at rest, will end up in the log-rolling state. After

Transition	Critical Reynolds number	Re_p	Dynamical transition/bifurcation	Dominant inertial effects
T → T/LR	Re_{LR}	14.5	Subcritical Hopf bifurcation	F and P
T/LR → T/IR	Re_{PF}	62.5	Supercritical pitchfork bifurcation	F
T/IR → T/IK	Re_{Hopf}	71.5	Supercritical Hopf bifurcation	F and P
T/IK → T/K	—	74.5	Breaking of homoclinic connection	F and P
T/K → T	Re_T	75.5	Saddle-node bifurcation of limit cycles	F and P
T → S	Re_c	89	Infinite-period saddle-node bifurcation of fixed points	F

TABLE 2. Summary of all dynamical transitions of a neutrally buoyant spheroidal particle with $r_p = 4$ in a linear shear flow, as found by Rosén *et al.* (2014).

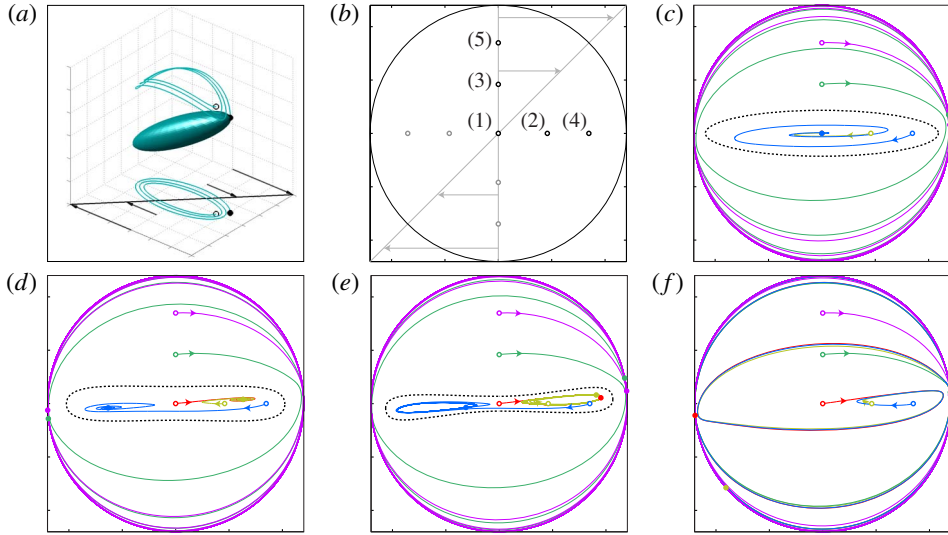


FIGURE 4. (Colour online) Trajectories of a neutrally buoyant particle with $r_p = 4$ initialized at rest from five different initial orientations: (a) particle motion projected onto the xy -plane by $(p_x, p_y) = (\sin \theta \cos \phi, \sin \theta \sin \phi)$; (b) the five initial orientations according to (3.1) projected onto the xy -plane, where the grey circles indicate the initial conditions, which due to symmetry in the flow problem are equivalent to the initial orientations 2–5, and the large circle represents orientations in the flow-gradient plane (when the particle is in planar rotation at $\theta = \pi/2$); projected trajectories from simulations at (c) $Re_p = 60$, (d) $Re_p = 70$, (e) $Re_p = 74$ and (f) $Re_p = 80$, where the empty circles represent the initial orientations and the filled circles the orientations at $G \cdot t = 1000$.

the supercritical pitchfork bifurcation at Re_{PF} , the unstable limit cycle shrinks and reshapes, as can be seen in figure 4(d,e); finally, the limit cycle is annihilated in a saddle-node bifurcation of limit cycles at Re_T , as seen in figure 4(f).

The transition at $Re_p = Re_c$, for a particle of $r_p = 4$, is through an infinite-period saddle-node bifurcation with a diverging tumbling period close to $Re_p = Re_c$ (Ding & Aidun 2000; Rosén *et al.* 2014). The bifurcation gives rise to two equilibrium orientations, where there is zero torque on the particle: one stable orientation ϕ_c and one unstable orientation ϕ_{us} ; see figure 5. This means that for a small deviation $\phi_c \pm \delta\phi$ or $\phi_{us} + \delta\phi$, the particle will go to ϕ_c , but a small negative deviation $\phi_{us} - \delta\phi$ will make the particle rotate around the vorticity direction until it is almost aligned with the flow again at $\phi_c - \pi$. The transition at Re_c is based on the balance between the torque on the particle from the primary flow and the opposing torque from the secondary flow, as explained in Ding & Aidun (2000). The magnitude of the torque from the secondary flow increases with the Reynolds number, and at the critical value it balances the torque from the primary flow. The quantitative value of Re_c predicted numerically depends on the exact point of balance reached. This makes the value of Re_c strongly dependent on precise prediction of the flow around the particle. This is true also for determination of the exact values of Re_{PF} , Re_{Hopf} and Re_T .

3.3. Bifurcation diagram

In practical applications, the solid-to-fluid density ratio of the particle and fluid rarely varies; instead, it is of greater significance what the rotational states of the particles

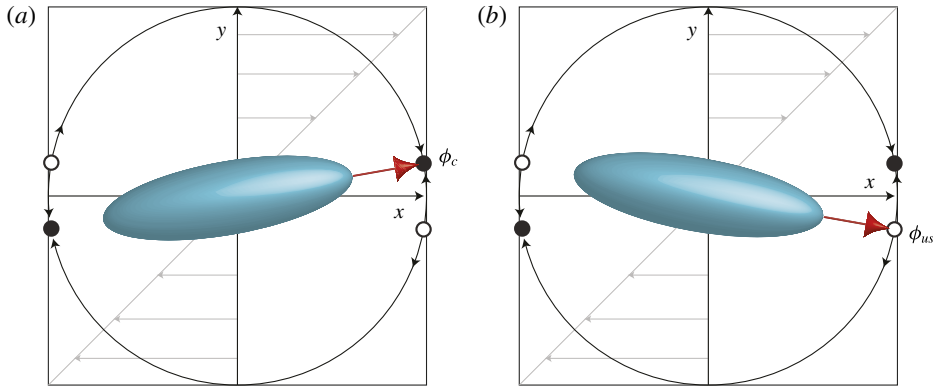


FIGURE 5. (Colour online) Illustration of equilibrium angles at $Re_p > Re_c$: (a) stable equilibrium angle at ϕ_c ; (b) unstable equilibrium angle at ϕ_{us} .

are with constant α and varying Re_p . This is best illustrated with a bifurcation diagram, shown in figure 6, following Rosén *et al.* (2014).

To draw bifurcation diagrams, a norm is needed to distinguish between the different rotational states. Rosén *et al.* (2014) used the measure $N_C = C_{norm}(1 + |\boldsymbol{\omega}/G|)$, where C_{norm} is an orbit parameter defined as

$$\left. \begin{aligned} C_{norm} &= \frac{C}{C+1}, \\ C &= r_p^{-1} \tan \theta (r_p^2 \sin^2 \phi + \cos^2 \phi)^{1/2}. \end{aligned} \right\} \quad (3.2)$$

This yields:

- (i) $N_C = \text{const.}$
 - (a) $N_C = 0$, log-rolling;
 - (b) $0 < N_C < 1$, inclined rolling;
 - (c) $N_C = 1$, steady state;
- (ii) $N_C = f(t)$
 - (a) $N_{C,mean} \geq 0.5$, tumbling/rotating;
 - (b) $N_{C,mean} < 0.5$, inclined kayaking, kayaking.

The definition above also makes it easy to distinguish planar rotational states ($N_C \geq 1$) from non-planar rotational states ($N_C < 1$). The oscillating states for which $N_C = f(t)$ are shown by plotting $N_{C,max}$ and $N_{C,min}$ during a period, represented by the grey area in figure 6. The dashed lines illustrate the approximate location of unstable fixed points, while the open circles illustrate the approximate location of $N_{C,max}$ of the unstable cycles. The bifurcation diagram reveals the following hysteresis behaviour: if one starts on the non-planar branch (where $N_C < 1$) and increases or decreases Re_p , the particle will eventually end up in a tumbling motion for $Re_p > Re_T$ or $Re_p < Re_{LR}$; however, since the tumbling solution is also available at $Re_{LR} < Re_p < Re_T$, if Re_p is now varied the particle will always stay in this planar motion.

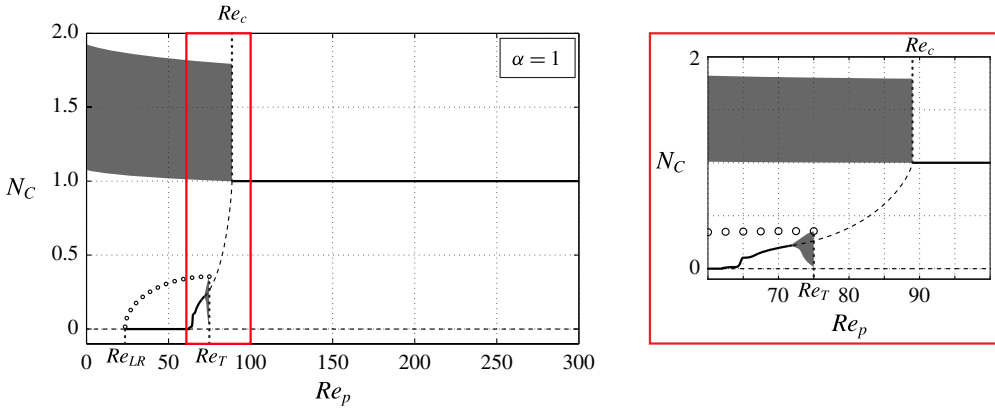


FIGURE 6. (Colour online) Bifurcation diagram for a spheroid with $r_p = 4$; the zoomed-in region with Re_p between 60 and 100 is displayed on the right. The stable rotational states have either a fixed orientation (solid lines) or an oscillating orientation (grey area), where in the latter case the norm is plotted using $N_{C,max}$ and $N_{C,min}$ during an oscillation period; the unstable fixed points are schematically drawn with dashed lines, and the approximate location of $N_{C,max}$ for the unstable limit cycle is indicated by the open circles.

3.4. Motivation for the present work

The knowledge from previous work can be summarized in Re_p - St parameter space, as is done in figure 7 for a prolate spheroid with $r_p = 4$. The entire regime of $St > 0$ and $Re_p = 0$ is well understood from the work of Lundell & Carlsson (2010), and the line where $St = Re_p$, i.e. $\alpha = 1$, is well understood from the work of Rosén *et al.* (2014) and others.

The region where $St > Re_p$ is not well documented. The critical Reynolds number Re_c is defined for a stationary particle and hence is dependent only on the geometry of the problem; this number therefore cannot depend on St . The dependence of the other critical Reynolds numbers (Re_{LR} , Re_{PF} , Re_{Hopf} and Re_T) on the Stokes number was not known prior to this work. Furthermore, the dependence of $St_{0.5}$ on Re_p and how all these transitions depend on the aspect ratio r_p are also not clear. All of these questions will be addressed in this work, where a complete picture of the dynamics of a single prolate spheroid will be provided. The results help to explain contradictions in the literature and provide a complete framework for analysing rotational motion of a particle in shear flow.

4. Results: dynamics of a spheroid with $r_p = 4$

4.1. Comments about the sensitivity of Re_c to the flow field near the particle

The transition at Re_c is very sensitive to small changes in the fluid field around the particle, as it is caused by a very delicate balance of torques on the particle. At $Re_p = Re_c$, the negative torque T^- from the primary flow is exactly equal to the positive torque T^+ from the secondary flow (illustrated in figure 8). Therefore, a small error of $\pm 1\%$ in determining T^+ or T^- can lead to a large error in the value of Re_c . This stringent requirement of accuracy in determining the torque results in difficulty pinpointing the critical Reynolds number using any numerical method without extreme grid refinement around the particle.

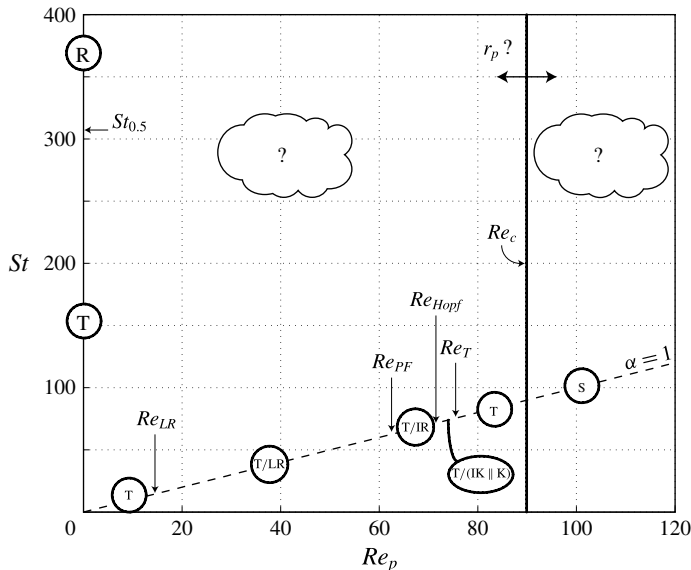


FIGURE 7. Dynamical transitions of a prolate spheroidal particle with $r_p = 4$ in the Re_p – St parameter space, as known from previous work by Lundell & Carlsson (2010) and Rosén *et al.* (2014). As in table 1, T = tumbling, R = rotating, LR = log-rolling, IR = inclined rolling, IK = inclined kayaking, K = kayaking and S = steady state.

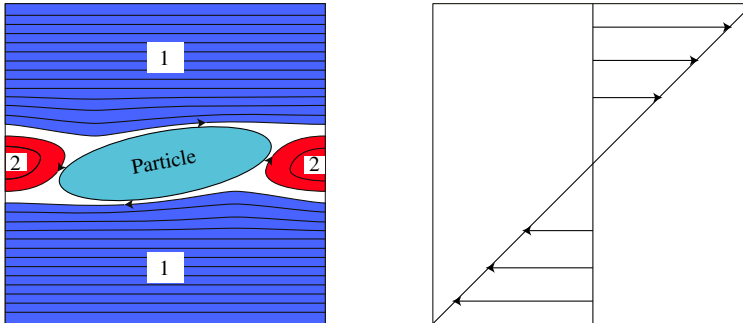


FIGURE 8. (Colour online) Qualitative flow field close to steady state at $Re_p \approx Re_c$; the primary flow (1) gives a negative torque (clockwise direction in left panel) on the particle, and the secondary flow (2) gives a positive torque (anticlockwise direction in left panel) on the particle (Ding & Aidun 2000).

The aim of the present work is to describe dynamical states of the particle in shear flow, and a moving grid is therefore crucial. The qualitative changes in the fluid field as Re_p changes are not sensitive to resolution, so the order in which the dynamical states occur and the particle behaviour in each dynamical region are resolution-independent. This is probably why the sensitivity described above has not been reported before. For example, the present model matches the transitional Reynolds numbers of Huang *et al.* (2012b) when the same resolution is used (Rosén *et al.* 2014). The critical Reynolds number Re_c was estimated by Huang *et al.* (2012b) to be $Re_c = 445 \pm 5$ for the case of $r_p = 2$ and $\kappa = 0.5$.

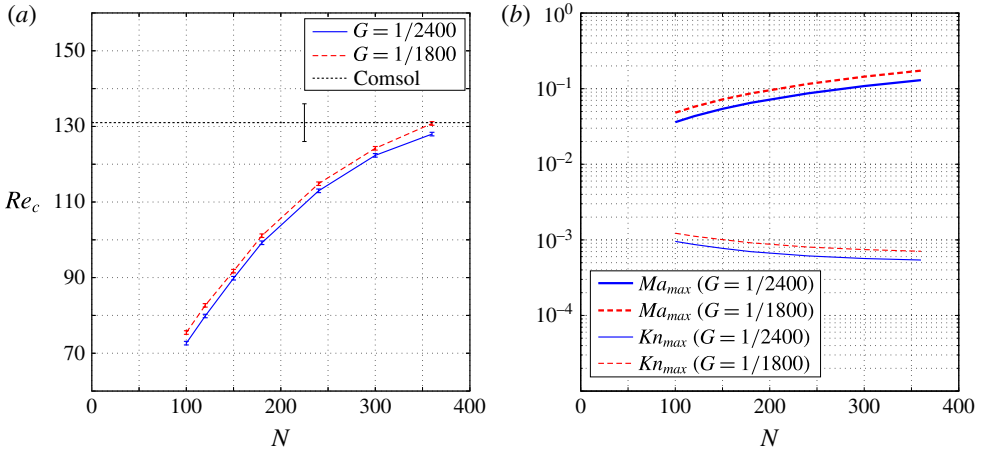


FIGURE 9. (Colour online) Resolution dependence of the critical Reynolds number Re_c for a prolate spheroid with $r_p = 4$ ($a = N/10$) and $\kappa = 0.2$: (a) Re_c versus the spatial resolution N with constant shear rate $G = 1/2400$ or $G = 1/1800$; (b) Ma_{max} and Kn_{max} versus the spatial resolution N .

Special consideration must be given to two parameters in lattice Boltzmann methods, due to the kinetic basis of the approach. One consideration is that the maximum Knudsen number Kn_{max} must be small enough in order to correctly describe a continuum fluid. This number is defined close to Re_c as $Kn_{max} = 2\sqrt{3}aGr_p/Re_c$, and can be seen in figure 9(b) to be small (approximately $O(10^{-3})$) and decreasing with N . These effects are therefore considered negligible. The second consideration is that the maximum lattice Boltzmann Mach number in the simulation, Ma_{max} , should be small in order to eliminate any compressible effects in the fluid. This number is linearly increasing with N via the relation $Ma_{max} = \sqrt{3}GN/2$. Therefore, to keep Ma_{max} small, as N increases G must become smaller to keep GN small. The smaller the value of G , the larger the number of time steps required to reach the equilibrium state. To pinpoint the quantitative value of Re_c requires very large N , very small G and very long computational time. However, in order to capture the dynamics of the particle motion and the equilibrium states, it is not necessary to pinpoint the value of Re_c .

To investigate the desired range of Re_p , the shear rate was set to $G = 1/600$, which was seen by Rosén *et al.* (2014) to give correct behaviour of particle motion. However, the physical transitional Re_p values, for example Re_c , could differ from the values presented here, since G is not small enough (and N not large enough) for the solution to have converged to the precise value of Re_c (Rosén *et al.* 2014). Figure 9 shows that the value of Re_p converges to the accurate Re_c as the resolution N increases (with particle size determined by $a = N/10$). It is found that, due to the slow convergence with respect to the spatial resolution, getting a converged value of Re_c will be extremely computationally expensive; the values obtained are in the range of 69–132.

Consequently, to obtain a converged solution of Re_c not only requires high spatial resolution through a high value of N but also a low value of the shear rate G in order to be able to neglect compressible effects. Because the lattice Boltzmann equation is explicit in time integration, the shear rate G is simultaneously connected to the time

resolution of the particle rotation, and more time steps are required for lower values of G . Furthermore, as Re_p approaches the critical point, i.e. Re_c , the period of oscillation increases to infinity asymptotically. In essence, the computational cost of achieving a simulation that is fully converged in space with a low Ma_{max} is huge.

However, a ‘steady state’ approach can be employed to accurately pinpoint Re_c , as shown by Ding & Aidun (2000). Determining the value of Re_c does not require performing a transient simulation, since it is defined as the minimum Re_p at which a stationary particle with a certain orientation ϕ ($\theta = \pi/2$) is experiencing zero torque. Therefore, one can fix the orientation of the particle, compute steady-state flow around the particle, and then change the orientation until the torque is balanced at the minimum value of Re_p , which would be Re_c . This approach was demonstrated by Ding & Aidun (2000) with the lattice Boltzmann method. However, because in this case the flow is steady state, one can solve the steady-state Navier–Stokes equation to pinpoint the value of Re_c without any need for time integration. This was done by setting up the problem in Comsol Multiphysics 4.3a. The same flow problem was considered, using a fixed particle and evaluating the torque for different particle orientations ϕ and Re_p . The minimum Re_p at which a zero torque exists is found to be $Re_c = 131 \pm 5$, and the corresponding orientation is $\phi = 6.6^\circ \pm 0.5^\circ$. This value seems reasonable, considering the LB-EBF data in figure 9. Using the same Comsol Multiphysics model, an estimate of $Re_c = 476 \pm 10$ is obtained for the case of $r_p = 2$ and $\kappa = 0.5$, as opposed to the lattice Boltzmann simulations which gave $Re_c = 445 \pm 5$.

To conclude this section, we observe that the same sequence of transitions between dynamical states as Re_p and St are changed is found regardless of the resolution (as long as the particle is large enough, as discussed in § 2.1). By fixing the spatial resolution and shear rate to constant values, as is done here, the resolution-dependence can be neglected and the transitional Re_p can be compared between the simulations. However, in order to pinpoint the physical transitional Reynolds numbers, other methods must be employed. At present this is feasible in cases where the particle is steady (such as Re_c), but not for transitions from one moving state to another.

4.2. Prediction of St_c

In the $Re_p = 0$ case, the spheroid in the tumbling motion has a minimum angular velocity $\dot{\phi}_{min}$, which increases with St as can be deduced from figure 3. On the other hand, for a neutrally buoyant particle, $\dot{\phi}_{min}$ decreases as Re_p increases from 10 to 50 (figure 10) and is the source of the infinite-period saddle-node bifurcation, which occurs at $Re_p = Re_c \approx 89$, as $\dot{\phi}_{min}$ becomes zero somewhere on the orbit. When St is increased to 1000 at $Re_p = 100$, an orbit with $\dot{\phi}_{min} > 0$ can again be found. Since the steady state must still exist, as it is dependent only on geometry, two solutions (tumbling and steady state) will exist above some critical Stokes number St_c . Hence, a heavy particle can maintain a tumbling motion due to particle inertia given sufficient initial angular momentum. If the particle is not initialized with enough angular momentum, the fluid inertia will dominate and take the particle to a steady state.

4.3. Rotational states

The results from the present numerical study for a prolate spheroid of $r_p = 4$ can be summarized in a single state plot as shown in figure 11. The figure shows the different rotational states that can be present for a certain pair of Re_p and St values. At every

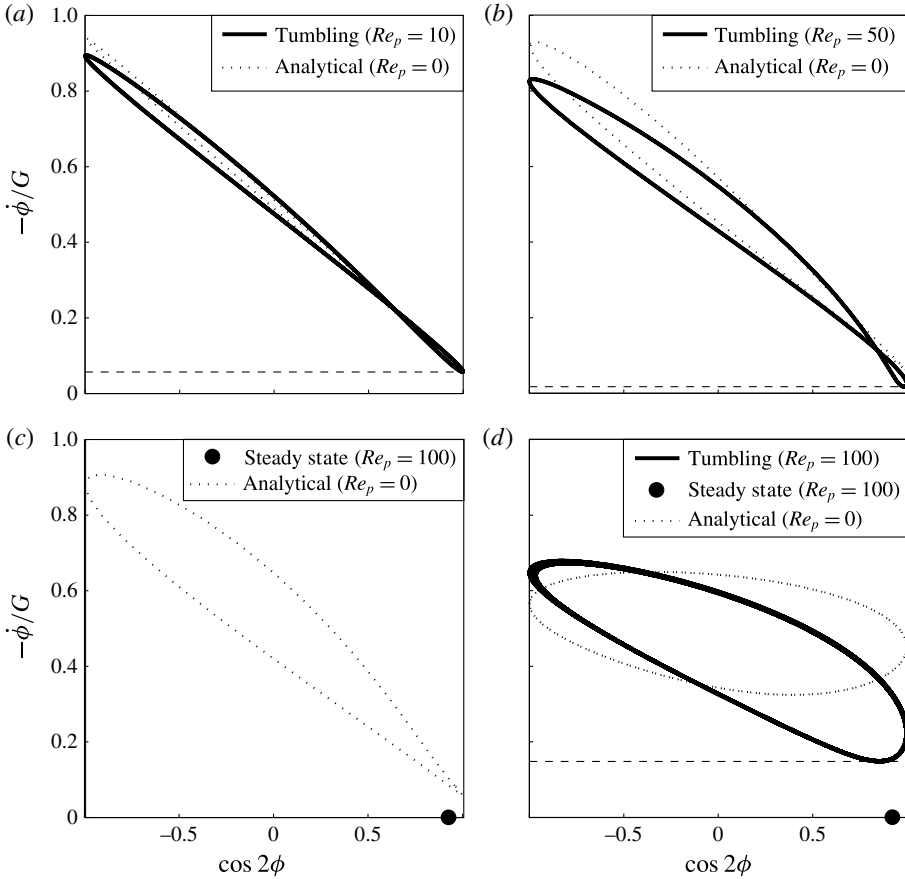


FIGURE 10. Phase diagrams of a tumbling particle with $r_p = 4$, for: (a) $St = 10$; (b) $St = 50$; (c) $St = 100$; (d) $St = 1000$. In each panel the dashed line indicates the location of $\dot{\phi}_{min}$, which does not exist in panel (c) (where $Re_p = St = 100$), since there is no stable tumbling solution; the steady state and the tumbling solution can coexist at $Re_p = 100$ if the Stokes number is increased to $St = 1000$, as in panel (d).

Re_p , there is a periodic solution at sufficiently high St where the particle rotates with axis of symmetry in the flow-gradient plane. This solution is tumbling in the moderate St regime, but at high St there is a transition to rotating motion. Since this solution is associated with high particle inertia, it will most likely be the preferred rotational state if the particle is initialized with sufficient angular momentum. Figure 11 shows thirteen distinct regimes depending on Re_p , which are listed below (an impressive number indeed for such a seemingly simple case). The details on how the transitions are determined will be given in the next subsection.

Regime Ia ($Re_p < Re_{LR}$ and $St < St_{0.5}$)

All initial conditions lead to tumbling.

Regime Ib ($Re_p < Re_{LR}$ and $St > St_{0.5}$)

All initial conditions lead to rotating.

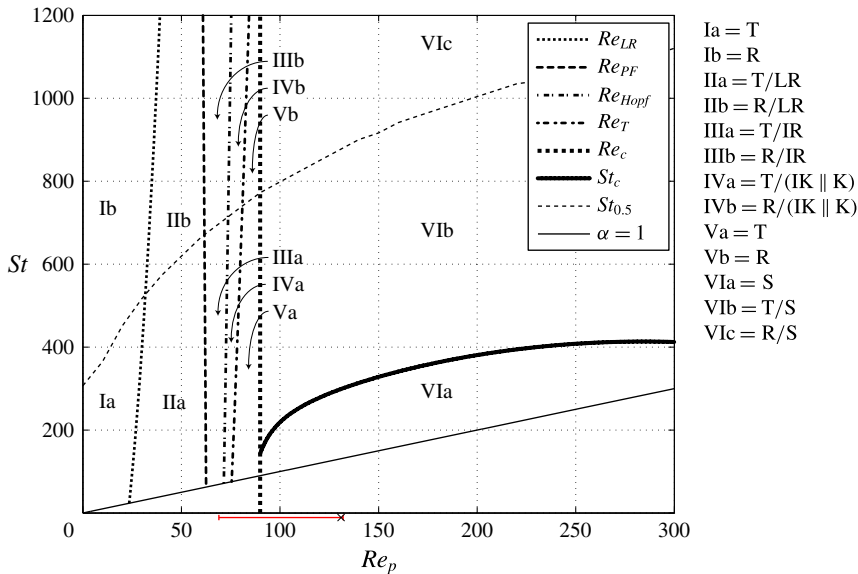


FIGURE 11. (Colour online) State plot showing the different rotational states for a spheroid with $r_p = 4$, depending on the values of Re_p and St (T = tumbling, R = rotating, LR = log-rolling, IR = inclined rolling, IK = inclined kayaking, K = kayaking, S = steady state); the error bar under the Re_p -axis indicates the range in which Re_c has been found due to a dependence on resolution, and the black cross indicates the value of Re_c obtained from Comsol (see § 4.1).

Regime IIa ($Re_{LR} < Re_p < Re_{PF}$ and $St < St_{0.5}$)

Tumbling and log-rolling are stable solutions; the final state depends on the initial conditions.

Regime IIb ($Re_{LR} < Re_p < Re_{PF}$ and $St > St_{0.5}$)

Rotating and log-rolling are stable solutions; the final state depends on the initial conditions.

Regime IIIa ($Re_{PF} < Re_p < Re_{Hopf}$ and $St < St_{0.5}$)

Tumbling and inclined rolling are stable solutions; the final state depends on the initial conditions.

Regime IIIb ($Re_{PF} < Re_p < Re_{Hopf}$ and $St > St_{0.5}$)

Rotating and inclined rolling are stable solutions; the final state depends on the initial conditions.

Regime IVa ($Re_{Hopf} < Re_p < Re_T$ and $St < St_{0.5}$)

Tumbling and inclined kayaking (and kayaking when the inclined kayaking solutions merge) are stable solutions; the final state depends on the initial conditions.

Regime IVb ($Re_{Hopf} < Re_p < Re_T$ and $St > St_{0.5}$)

Rotating and inclined kayaking (and kayaking when the inclined kayaking solutions merge) are stable solutions; the final state depends on the initial conditions.

Regime Va ($Re_T < Re_p < Re_c$ and $St < St_{0.5}$)

All initial conditions lead to tumbling.

Regime Vb ($Re_T < Re_p < Re_c$ and $St > St_{0.5}$)

All initial conditions lead to rotating.

Regime VIa ($Re_p > Re_c$ and $St < St_c$)

All initial conditions lead to steady state.

Regime VIb ($Re_p > Re_c$ and $St_c < St < St_{0.5}$)

Tumbling and steady state are stable solutions; the final state depends on the initial conditions.

Regime VIc ($Re_p > Re_c$ and $St_c < St < St_{0.5}$)

Rotating and steady state are stable solutions; the final state depends on the initial conditions.

The methods used to determine the curves in figure 11 will now be described in detail.

4.4. Determining St_c as function of Re_p

4.4.1. Definition of St_c

As already shown, the saddle-node bifurcation at $Re_p = Re_c$ gives rise to two equilibrium orientations, at which there is zero torque on the particle: one stable orientation, ϕ_c , and one unstable orientation, ϕ_{us} (see figure 5). This means that for a small deviation $\phi_c \pm \delta\phi$ or $\phi_{us} + \delta\phi$, the particle will go to ϕ_c , but a small negative deviation $\phi_{us} - \delta\phi$ will make the particle rotate around the vorticity direction until it is almost aligned with the flow again at $\phi_c - \pi$. If the particle has enough particle inertia, it will gain enough angular momentum during the rotation that the region of positive torque will be insufficient to stop the rotation. The particle will thus reach the symmetric (and dynamically identical) configuration $\phi_{us} - \pi$ with a non-zero angular velocity and consequently enter a periodic tumbling motion. This leads to a natural definition of a critical Stokes number St_c : when initializing the particle at rest at an angle $\phi_{us} - \delta\phi$, a particle with $St < St_c$ will end up in steady state whereas a particle with $St > St_c$ will end up in a tumbling motion.

Below St_c , all initial conditions will lead to the steady state, since the particle can never gain more angular momentum during a rotational period. So even if the particle is initialized with high angular velocity, it will eventually slow down until it reaches a steady state. Above St_c , the final solution can be either tumbling motion or steady state, depending on the initial condition. As will be shown in §5, the transition from tumbling to steady state at $St = St_c$ is through a homoclinic bifurcation. The critical Stokes number is determined by finding the critical density ratio $\alpha_c = St_c/Re_p$ as a function of Re_p .

4.4.2. Equilibrium angles

In order to find the critical density ratio α_c , the equilibrium angle ϕ_{us} must be found. Both ϕ_{us} and ϕ_c are found by using the fact that there should be zero torque on the particle, which means that $\dot{\phi} \approx 0$ should be achieved shortly after the particle is initialized in an equilibrium angle. This fact is used to systematically find the

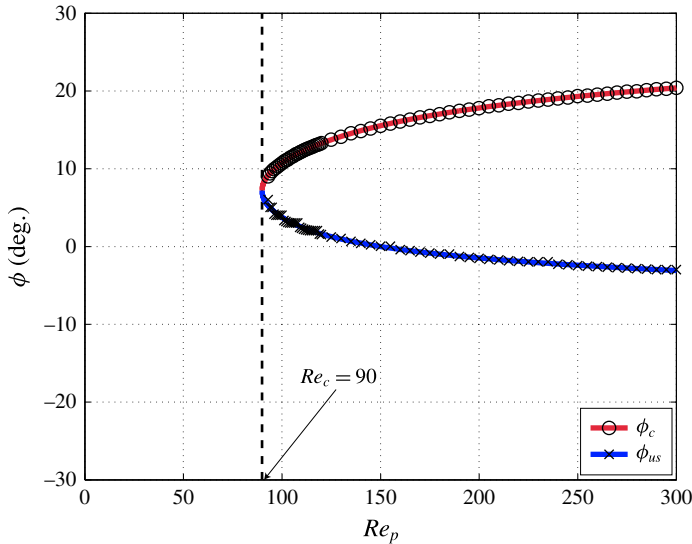


FIGURE 12. (Colour online) Equilibrium angles ϕ_c (circles) and ϕ_{us} (crosses) plotted as functions of Re_p for a spheroid with $r_p = 4$; the data are fitted with a polynomial function (solid line), for which the minimum is found at $Re_p = Re_c = 90$.

equilibrium angles for a given Re_p with a half-interval search (the exact algorithm is described in appendix B).

In figure 12, ϕ_c and ϕ_{us} are plotted as functions of Re_p . It can be seen that ϕ_{us} and ϕ_c are decreasing and increasing functions of Re_p , respectively. A seventh-order polynomial is fitted to the computed data points, and using the polynomial function the bifurcation is found to occur at $Re_c \approx 90$, which is slightly higher than the previously obtained value of $Re_c \approx 89$.

4.4.3. Critical density ratio as a function of Re_p

The critical density ratio, α_c , is obtained as described in § 4.4.1, with $\delta\phi$ chosen sufficiently large so that the particle is definitely moving with $\dot{\phi} < 0$. In this study, $\delta\phi$ was chosen to be 1° . The critical density ratio α_c is found by using the fact that the particle with $\alpha = \alpha_c$ initialized at rest from $\phi_0 = \phi_{us} - \delta\phi$ should reach $\phi = \phi_0 - \pi$ with $\dot{\phi} = 0$. The critical density ratio is found by using half-interval search (the exact algorithm is provided in appendix C).

Figure 13 shows α_c as a function of Re_p , where the data points are fitted with a rational function of the form $\alpha_c = P(Re_p)/Q(Re_p)$, with P and Q being fifth- and fourth-degree polynomials, respectively. The error bars indicate the minimum intervals $\Delta St = 10$ used in the half-interval algorithm. The maximum critical density ratio of $\alpha_{c,max} \approx 2.3$ is found close to $Re_p \approx 125$, while at higher Re_p particle inertia plays a greater role and α does not need to be much larger than unity in order to find a periodic state. It can also be seen from the figure that $\alpha_c > 1$ as $Re_p = Re_c$. This codimension-two point $\alpha_{cd2} = \alpha_c(Re_c) \approx 1.6$, where two bifurcations coexist (the saddle-node bifurcation at Re_c and the homoclinic bifurcation at α_c), is found by evaluating the fitted rational function at Re_c . A more detailed discussion of this bifurcation will be given in § 5. The line of St_c in figure 11 is drawn by the relation $St_c = \alpha_c Re_p$, which connects to the codimension-two point at $St_{cd2} = \alpha_{cd2} Re_c$.

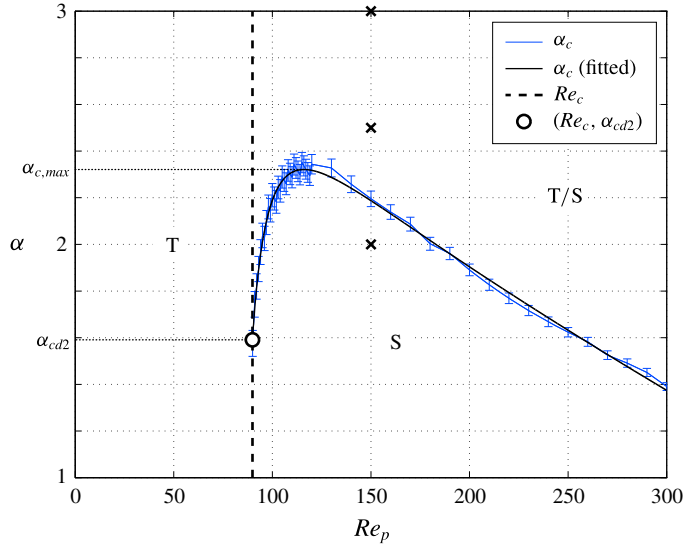


FIGURE 13. (Colour online) The critical density ratio α_c plotted as a function of Re_p for a spheroid with $r_p = 4$ (solid blue line), fitted with a rational function (solid black line). The critical Reynolds number $Re_c = 90$ (dashed line) is found through polynomial fitting of equilibrium angles, and the codimension-two point, where α_c connects to the Re_c line, is found at $\alpha = \alpha_{cd2} \approx 1.6$. Black crosses indicate $\alpha = 2, 2.5$ and 3 at $Re_p = 150$, for which the phase diagrams in figure 14 are drawn. Only planar rotational states, namely tumbling (T) and steady state (S), are shown in the figure.

4.5. Determining $St_{0.5}$ as function of Re_p

As already discussed, it was observed by Lundell & Carlsson (2010) and Nilsen & Andersson (2013) that the period of the tumbling motion is a steadily decreasing function of St in the absence of fluid inertia ($Re_p = 0$). At $St = 0$, the particle is tumbling with period $GT_J = 2\pi(r_p^{-1} + r_p)$; as $St \rightarrow \infty$, the particle rotates at a constant angular velocity with period $GT_H = 4\pi$. The transition between the two regimes is characterized by $St_{0.5}$, which is defined as the Stokes number at which the spheroid rotates with a period of $(GT_J + GT_H)/2$. However, $St_{0.5}$ was defined only for $Re_p = 0$, since it refers to the period without any fluid or particle inertia. For finite $Re_p > 0$, the period is also seen to decrease as St increases; therefore in this work we use the same definition of $St_{0.5}$, i.e. the Stokes number at which the period is $GT = (GT_J + GT_H)/2 = \pi(r_p^{-1} + r_p + 2)$. The value of $St_{0.5}$ is found by using a half-interval search (see appendix D for the exact algorithm). Since the value at $Re_p = 0$ can be analytically found to be $St_{0.5} = 307$, the line with the simulated values has been connected to this analytical value in figure 11.

4.6. Determining Re_{LR} , Re_{PF} , Re_{Hopf} and Re_T as functions of St

4.6.1. Re_{LR}

The definition of Re_{LR} is as the value at which the log-rolling solution goes from being unstable, with outward-spiralling trajectories, to stable, with nearby trajectories spiralling inwards. In the work by Rosén *et al.* (2014), this value was determined by initializing a particle in the log-rolling solution and observing whether it was still there

after a certain length of time or if it had started to spiral outwards. In this work, for better comparison with the heavier particle, we make use of the fact that the log-rolling solution should become attracting at Re_{LR} . Therefore, a particle is initialized at rest from a nearby location at $(\phi, \theta) = (0^\circ, 1^\circ)$, and the Re_p value at which this initial condition leads to log-rolling instead of tumbling is taken to be Re_{LR} . This is done for both a neutrally buoyant particle and a particle with $\alpha = 20$.

The value of Re_{LR} increases to $Re_{LR} \approx 33.5$ from $Re_{LR} \approx 23.5$ as α is increased from 1 to 20 (note that, with this definition, the value of Re_{LR} for the neutrally buoyant case is slightly higher than the value reported by Rosén *et al.* (2014)). With high α , small disturbances of the particle orientation will lead to a more considerable gain in angular momentum and an increased influence of particle inertia effects. Therefore, the transition that stabilizes the log-rolling solution is delayed in Re_p as α is increased.

4.6.2. Re_{PF}

The transition at Re_{PF} is associated with the fact that the log-rolling solution destabilizes and the particle ends up in inclined rolling with $\theta > 0$. The value of Re_{PF} is determined by finding the Re_p value at which a particle initialized close to the log-rolling solution at $(\phi, \theta) = (0^\circ, 1^\circ)$ ends up with an angle $\theta > 1^\circ$ after $Gt = 400$. This is done both for a neutrally buoyant particle and for a particle with $\alpha = 13$.

For the neutrally buoyant particle, the transition was found to occur at $Re_{PF} = 62.5$, and the heavy particle had a transition at $Re_{PF} = 61.5$. The transition point appears not to change for higher α , and the reason for the small variation in Re_{PF} is that the heavier particle assumes the final rotational state more quickly than the lighter one. The alignment to the new inclined rolling state is very slow, and so particle inertia is concluded to be negligible for this transition. This can be understood by recalling that both the log-rolling and inclined rolling solutions correspond to steady states of the fluid field.

4.6.3. Re_{Hopf}

The transition at Re_{Hopf} is associated with a final rotational state where θ goes from being a constant to an oscillating function of time. To determine Re_{Hopf} , we use the fact that a particle with an initial orientation of $(\phi, \theta) = (0, \pi/8)$ will quickly enter the final rotational state close to Re_{Hopf} (this can be seen in figure 4e for initial orientation 2). The Re_p value at which the oscillations of this particle become larger than $\Delta\theta = 1^\circ$ after $Gt = 1000$ is taken to be Re_{Hopf} . This is simulated for both $\alpha = 1$ and $\alpha = 13$.

For the transition at Re_{Hopf} , particle inertia plays a role, but small disturbances do not affect the particle that much. Most likely because of this, Re_{Hopf} increases slightly from $Re_{Hopf} \approx 71.5$ to $Re_{Hopf} \approx 73.5$ as α is increased from 1 to 13.

4.6.4. Re_T

The transition at Re_T is associated with a final rotational state that is tumbling regardless of the initial orientation. Here Re_T is defined as the Re_p value at which a particle initialized at $(\phi, \theta) = (0, 0)$ spirals outward towards a tumbling motion after $Gt = 400$. This is investigated both for a light particle ($\alpha = 1$) and a heavy particle ($\alpha = 13$).

A bigger difference is seen for this transition than for the previous ones. The oscillations of the heavier particle in the inclined kayaking state have smaller amplitude due to lower angular acceleration. This leads to a delayed transition to the pure tumbling region from $Re_T \approx 75.5$ for $\alpha = 1$ to $Re_T \approx 83.5$ for $\alpha = 13$.

In figure 11, the dependence of Re_{LR} , Re_{Hopf} and Re_T on the solid-to-fluid density ratio α is estimated by linearly interpolating the two results at low α and at high α given above. The curves therefore take on a slightly parabolic shape, as they are drawn according to the relation $St = \alpha Re_p$ for Re_{LR} , Re_{PF} , Re_{Hopf} and Re_T .

The Stokes number is seen to have a small or no effect on the transitions at Re_{LR} , Re_{PF} , Re_{Hopf} and Re_T , and we conclude that particle inertia has little effect due to the slow non-planar motions. These transitions are thus mainly controlled by fluid inertia.

5. Results: bifurcation analysis ($r_p = 4$)

In this section we discuss the results on the spheroid in a shear flow using general theory about nonlinear dynamical systems and bifurcations. The concepts discussed can be found in the books by Hale & Koçak (1991) and Strogatz (1994).

5.1. Bifurcation at $\alpha = \alpha_c$ ($St = St_c$)

From figure 13 it can be seen that for a particle of $\alpha > \alpha_{c,max}$, fluid inertia will never be enough to fully stop a tumbling motion, and there will be no diverging period when Re_p increases or decreases (a steady state will still always coexist at $Re_p > Re_c$). However, by fixing $Re_p > Re_c$ and decreasing α , the period diverges at $\alpha = \alpha_c$. This transition is not the infinite-period saddle-node bifurcation seen at $Re_p = Re_c$, since no new fixed points are created. To understand this process, phase diagrams of the motion are drawn in figure 14(b–d), where a particle is initialized at rest from eight different initial orientations, $(\phi, \theta)_{1-8} = (\pi n/8, \pi/2)$ for $n = 0, 1, \dots, 7$ (see figure 14a), at $Re_p = 150$ and three different density ratios $\alpha = 2, 2.5$ and 3 (crosses in figure 13). As shown in figure 12, the equilibrium angles at $Re_p = 150$ are $\phi_{us} \approx 0^\circ$ and $\phi_c \approx 15.5^\circ$.

Starting with the case where $\alpha = 3$ in figure 14(b), the point at $(\dot{\phi}, \phi) = (0, \phi_{us})$ is a saddle (open circle, with stable dynamics in one eigendirection and unstable dynamics in another eigendirection) while $(\dot{\phi}, \phi) = (0, \phi_c)$ is a sink (filled circle, with stable dynamics, i.e. nearby trajectories go to this point). The trajectory of the initial condition ϕ_1 almost coincides with the saddle (i.e. $\phi_1 = \phi_{us}$) and can be considered a branch of the unstable manifold of the saddle. This manifold asymptotically approaches the stable limit cycle, i.e. a tumbling motion. At some trajectory between the trajectories of initial conditions ϕ_6 and ϕ_7 , there is a branch of the stable manifold of the saddle; this means that initial conditions above it (e.g. initial conditions 2–6) lead to the sink, i.e. steady state. As α is decreased close to α_c , the stable limit cycle goes very near the saddle, as seen in figure 14(c). At $\alpha = \alpha_c \approx 2.2$, the stable limit cycle is destroyed as the limit cycle merges with the unstable manifold of the saddle. The manifold becomes a homoclinic orbit which connects with itself at the saddle. At $\alpha < \alpha_c$, even the unstable manifold of the saddle leads to the sink and therefore the steady state is the only solution (figure 14d).

The behaviour is very similar to that of a damped pendulum driven by a constant torque (Strogatz 1994), which undergoes a homoclinic bifurcation when the driving torque is varied under low damping conditions. The period of the limit cycle is divergent, but unlike the case of an infinite-period saddle-node bifurcation, where the period scales as $GT \propto |\beta - \beta_c|^{-0.5}$, the period close to the homoclinic bifurcation scales as $GT \propto \ln |\beta - \beta_c|$, where β is any parameter leading to the transition at β_c . In figure 15(a,b), the period is plotted as a function of $\alpha - \alpha_c$ and is compared with the two types of scaling at $Re_p = 150$. It is clear that the logarithmic scaling gives a much better fit, and the existence of a homoclinic bifurcation at $\alpha = \alpha_c$ is confirmed. The reason that the power law fails to describe the diverging period in this case is

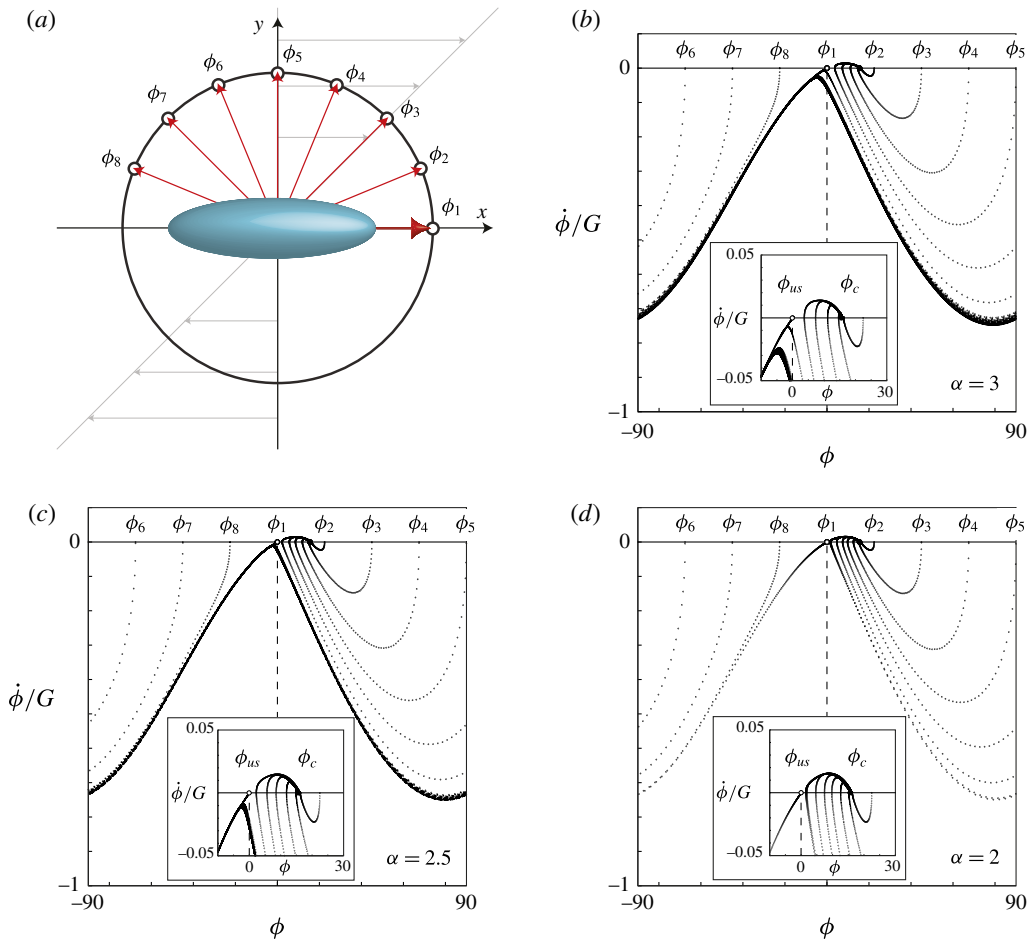


FIGURE 14. (Colour online) Phase diagrams showing homoclinic bifurcation for a spheroid with $r_p = 4$ at $Re_p = 150$, initialized at rest in the flow-gradient plane ($\theta = 90^\circ$) from eight different initial orientations, shown in (a), and with three different density ratios: (b) $\alpha = 3$; (c) $\alpha = 2.5$; (d) $\alpha = 2$. A limit cycle (tumbling) is seen to collapse with the saddle ϕ_{us} (open circle) at $\alpha = \alpha_c = 2.2$; at $\alpha < \alpha_c$, all initial conditions lead to the sink ϕ_c (solid circle) and thus stay in steady state.

that the angular velocity becomes a non-differentiable function of the angle close to the saddle (seen also in figure 13b), which is an important criterion for this scaling (Strogatz 1994; Rosén *et al.* 2014). The shape of $\dot{\phi}(\phi)$ close to ϕ_{us} is determined by the two eigendirections of the saddle when $\alpha \gtrsim \alpha_c$ and therefore has a non-parabolic shape.

Combining this knowledge with what was found regarding the coexistence of the tumbling solution and a steady state (figure 13), the following conclusions can be drawn for a prolate spheroid of aspect ratio $r_p = 4$ with increasing Re_p .

- (i) If $\alpha < \alpha_{cd2} \approx 1.6$, the transition from tumbling to steady state will be through an infinite-period saddle-node bifurcation at $Re_p = Re_c$, with a diverging period scaling as $GT \propto |Re_p - Re_c|^{-0.5}$.

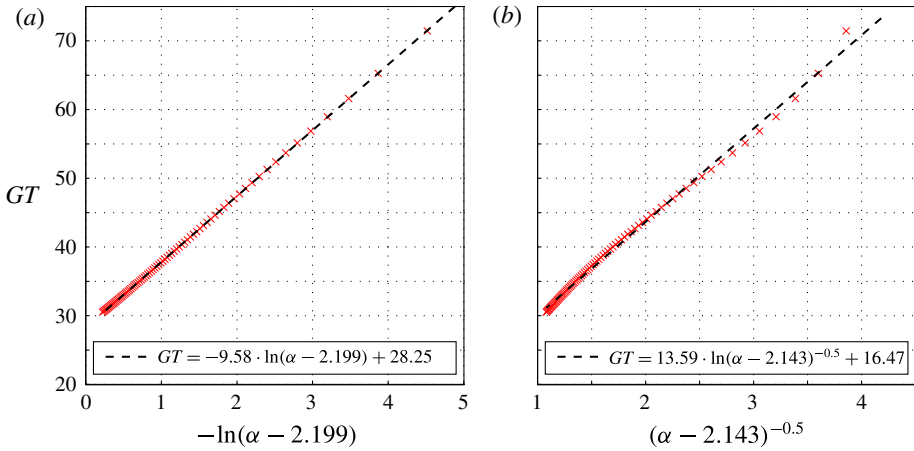


FIGURE 15. (Colour online) Scaling of the diverging period at $Re_p = 150$ for a spheroid with $r_p = 4$: (a) data fitted using the assumption of a homoclinic bifurcation with $GT \propto \ln |\beta - \beta_c|$; (b) data fitted using the assumption of an infinite-period saddle-node bifurcation with $GT \propto |\beta - \beta_c|^{-0.5}$.

- (ii) If $\alpha_{cd2} < \alpha < \alpha_{c,max}$ (i.e. $1.6 \lesssim \alpha \lesssim 2.3$), the transition will be through a homoclinic bifurcation at $Re_p = Re_{c1}$, where Re_{c1} is defined as the Re_p value at which $\alpha = \alpha_c$. The tumbling period will diverge according to $GT \propto \ln |Re_p - Re_{c1}|$.
- (iii) If $\alpha = \alpha_{cd2} \approx 1.6$, a transition from tumbling to steady state will occur at $Re_p = Re_c$, but no distinction can be made between the types of bifurcations and both types of scalings will apply.
- (iv) If $\alpha > \alpha_{c,max} \approx 2.3$, there is no diverging period at any Re_p , and a tumbling solution will always exist.

5.2. Bifurcation diagrams

The bifurcation diagrams for $\alpha = 2$ and 3 are shown in figure 16(a,b) in the same way as in §3.3. Looking at figure 16(a), for $\alpha = 2$, which lies in the interval $\alpha_{cd2} < \alpha < \alpha_{c,max}$, there are two critical particle Reynolds numbers which correspond to intersections of the α_c line (see figure 13). Due to higher particle inertia, the fluid inertia is not sufficient to stop the tumbling motion at $Re_p = Re_c$ (note that Re_c is dependent only on geometry and fluid properties and is not changed due to high particle inertia). The tumbling period instead diverges to infinity at $Re_p = Re_{c1} > Re_c$ through a homoclinic bifurcation. Similar behaviour is seen when Re_p is decreased from high values, with the tumbling period diverging to infinity at Re_{c2} . In the region $Re_{c1} < Re_p < Re_{c2}$, the steady state is thus the only solution, while a tumbling solution coexists in the regions $Re_c < Re_p < Re_{c1}$ and $Re_p > Re_{c2}$. For the particle with $\alpha = 3$ (figure 16b), the density ratio is higher than $\alpha_{c,max}$, and hence there is no diverging period and a tumbling solution will always coexist with the steady state at $Re_p > Re_c$. The bifurcation diagrams indicate the following hysteresis behaviour as Re_p varies.

- (i) For a particle with $\alpha_{cd2} < \alpha < \alpha_{c,max}$ (figure 16a) in a tumbling state at $Re_p < Re_c$, increasing Re_p will lead to a diverging period and a transition to steady state at $Re_p = Re_{c1}$. However, to reach a tumbling state again, Re_p must be decreased to $Re_p = Re_c$, since the steady-state solution (bold line at $N_c = 1$) exists also for $Re_c < Re_p < Re_{c1}$.

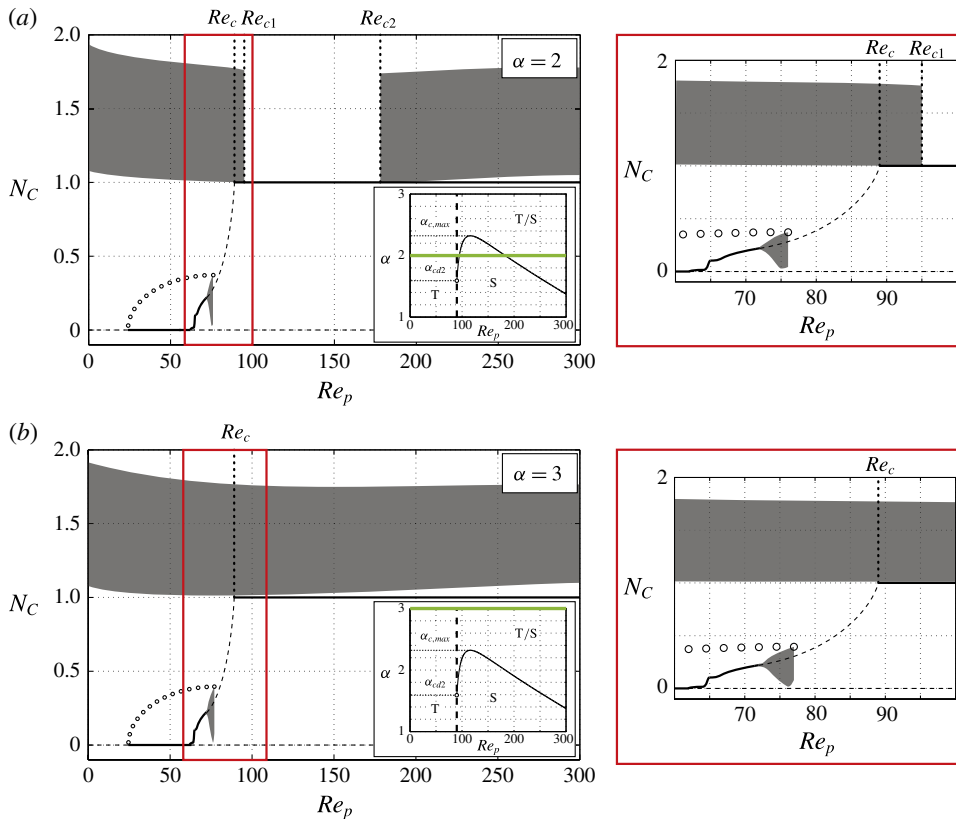


FIGURE 16. (Colour online) Bifurcation diagrams for a spheroid of $r_p = 4$ with: (a) $\alpha = 2$; (b) $\alpha = 3$. In each panel, the zoomed-in region for Re_p between 60 and 100 is displayed on the right; the stable rotational states have either a fixed orientation (solid lines) or an oscillating orientation (grey areas), where in the latter case the norm is plotted using $N_{C,max}$ and $N_{C,min}$ during an oscillation period. The unstable fixed points are schematically drawn with dashed lines, and the approximate location of N_{max} for the unstable limit cycle is indicated by open circles.

- (ii) For a particle with $\alpha < \alpha_{c,max}$ (figure 16a) in a tumbling state at $Re_p > Re_{c2}$, decreasing Re_p will lead to a diverging period and a transition to steady state at $Re_p = Re_{c2}$. Upon increasing $Re_p > Re_{c2}$, the particle will stay in steady state.
- (iii) For a particle with $\alpha > \alpha_{c,max}$ (figure 16b) in a steady state, decreasing Re_p past Re_c will make the particle enter a tumbling motion. If Re_p is increased again, the particle will stay in a tumbling rotational state.

6. Results: transient behaviour and orbit drift ($r_p = 4$)

Although we saw in § 4.6 that the transitions at Re_{LR} , Re_{PF} , Re_{Hopf} and Re_T are not affected much by α , a significant difference emerges when one studies the rate at which the particle assumes an equilibrium solution. Figure 17(a,b) show the projected trajectories of a particle with $\alpha = 13$ at $Re_p = 60$ and $Re_p = 70$, respectively, for the initial orientations given by (3.1). Compared with the neutrally buoyant particle in figure 4(b,c), the unstable orbit appears unchanged, leading to a similar probability

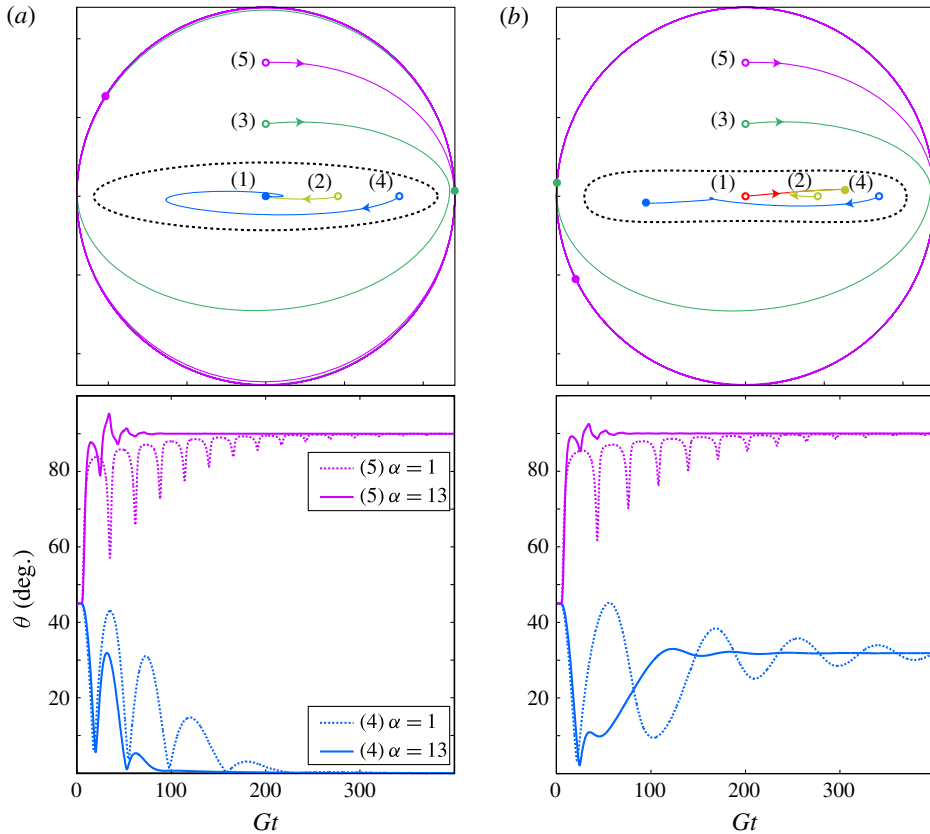


FIGURE 17. (Colour online) Projected trajectories of a heavy particle with $r_p = 4$ and $\alpha = 13$ at: (a) $Re_p = 60$ (T/LR); (b) $Re_p = 70$ (T/IR). The bottom panels show the evolution of the particle orientation (with just the θ angle plotted) for the heavy particle (solid line) compared with that of a neutrally buoyant particle (dashed line); the projected trajectories are as presented earlier in figure 4.

that a randomly oriented particle initialized at rest will end up in each rotational state. However, the oscillations of the particle orientation are almost completely damped out by particle inertia. The evolution of the θ angle for the heavy particle is shown in figures 17(c) and 4(d) for $Re_p = 60$ and 70, respectively. It is obvious that, although the final rotational state remains unchanged when the density ratio is altered, the rate of approach to the final state is strongly affected. In the range of St studied here, the heavier particle always reaches the final rotational state more rapidly than does a lighter particle. However, as observed by Lundell & Carlsson (2010) for $Re_p = 0$, at higher Re_p there is probably also an optimal Stokes number, above which the rate of reaching the final state increases instead, due to a slow initial transient.

7. Results: consequences for suspension rheology

One of the key consequences of the preferred rotational states due to inertia is the effect on the rheological properties of the particle suspension. Einstein (1906, 1911) already found that the shear viscosity of a dilute suspension ν_{susp} of spherical particles is modified through a relation $\nu_{susp} = \nu(1 + \eta\Phi)$, where ν is the kinematic viscosity of

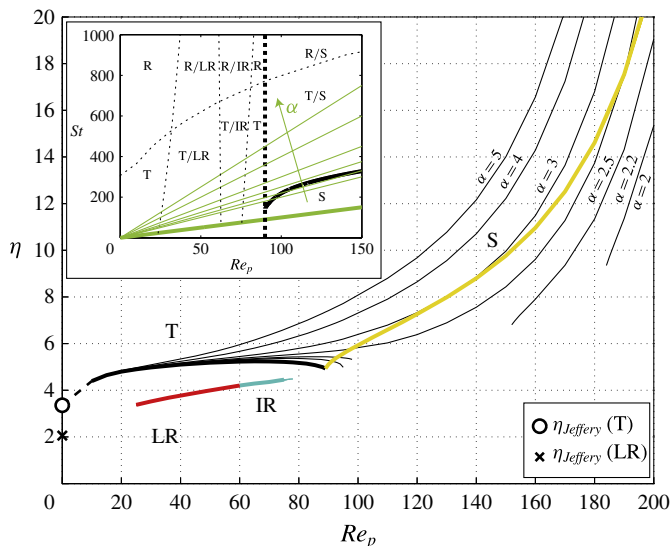


FIGURE 18. (Colour online) Intrinsic viscosity of one-particle suspension ($r_p = 4$) as a function of Re_p for $\alpha = 1$ (thick lines in main plot) and $\alpha = 2, 2.2, 2.5, 3, 4, 5$ (thin lines in main plot); the dashed line connects the analytical value of η of the tumbling motion derived at $Re_p = 0$ by Jeffery (1922) to the first simulated value at $Re_p = 10$, since it is known that this is the only stable rotational state in the low- Re_p regime. The inset shows the corresponding parameter sweep in the Re_p - St plane. Here T = tumbling (black lines in main plot), S = steady state (yellow), LR = log-rolling (red) and IR = inclined rolling (cyan); for simplicity, the rotational states of inclined kayaking and kayaking are included in the inclined rolling regime.

the suspending fluid, Φ is the particle volume fraction and η is the intrinsic viscosity. For a suspension of spherical particles, the intrinsic viscosity was found to be $\eta = 2.5$. Jeffery (1922) extended this work to account for spheroids, and found that η depends on the rotational orbit. For an inertia-less prolate spheroid with $r_p = 4$, the intrinsic viscosity averaged over a full period has a value ranging from $\eta_{Jeffery} \approx 2.05$ (in the log-rolling orbit) to $\eta_{Jeffery} \approx 3.36$ (in the tumbling orbit). In the work by Huang *et al.* (2012a,b) using a lattice Boltzmann method, η was found by evaluating the shear stress σ at the fluid nodes closest to the wall. The same method is employed here for each final rotational state at different values of Re_p and α (the method is described in detail in appendix E). The results are shown in figure 18. If we consider a constant fluid viscosity and constant particle size, the Re_p -axis can be interpreted as the shear rate. Both shear-thinning ($d\eta/dRe_p < 0$) and shear-thickening ($d\eta/dRe_p > 0$) can thus be directly deduced from the diagram. The thick lines correspond to the neutrally buoyant case and show a shear-thickening behaviour everywhere except in a small region just below $Re_p = Re_c$, where the diverging period, due to the infinite-period saddle-node bifurcation, causes shear-thinning. In the $Re_{LR} < Re_p < Re_T$ regime, two values of η are allowed, since we have coexisting rotational states corresponding to planar and non-planar motions. When $\alpha > \alpha_{c,max}$, the diverging period disappears and so does the small shear-thinning region, and the suspension has shear-thickening behaviour everywhere. We remark that it is only the tumbling branch that is strongly affected by the density ratio α , and this is due to the modified period as St increases. The reason the non-planar branch is not affected is that the rotation rate around the

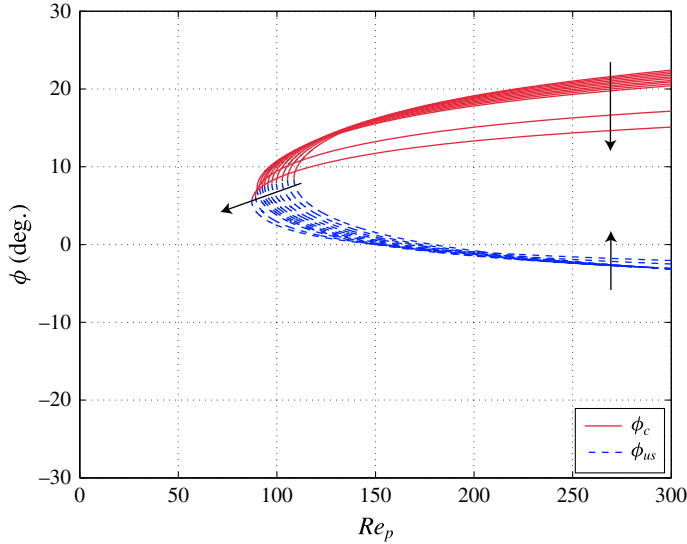


FIGURE 19. (Colour online) Fitted polynomials of the equilibrium angles ϕ_c (red solid line) and ϕ_{us} (blue dashed line) as functions of Re_p for spheroids with $r_p = 3.0, 3.1, \dots, 3.9, 4.0, 5.0$ and 6.0 ; the arrows indicate the effects of increasing r_p .

particle axis is constant, since it corresponds to a steady-state solution of the fluid at a given Re_p . The same is of course true for the steady-state branch, where the thick curve at $\alpha = 1$ coincides with the thinner lines for all α .

8. Results: extension to $r_p \neq 4$

To study the effect of aspect ratio on the behaviour described in the previous sections, the same simulations were done for particles with aspect ratios in the range of $r_p = 2$ – 6 . As mentioned in § 2.1, the higher-aspect-ratio particles were simulated in a larger domain to ensure that $\kappa = 0.2$ and $b \geq 3$. As an example, for the spheroid with $r_p = 6$, a particle of $a = 18$ and $b = 3$ was simulated in the domain $(180, 180, 180)$, while still keeping $G = 1/600$.

8.1. Dependence of aspect ratio on fluid inertial transitions

In § 4.6, it was seen that the transitions at Re_{PF} and Re_c are independent of the density ratio and are caused only by fluid inertia. Therefore the r_p -dependence is analysed for these transitions using a neutrally buoyant particle ($\alpha = 1$).

In order to find Re_{PF} , the same procedure as described in § 4.6.2 was performed for aspect ratios $r_p = 2, 2.1, 2.2, 2.3, 2.4, 2.6, 2.8, 3, 4, 5$ and 6 . The critical Reynolds number Re_c was found by first finding the equilibrium angles, using the procedure described in § 4.4.2, and then fitting a polynomial function to the data. The fitted polynomials are shown in figure 19. The value of Re_c is found by seeking the minimum of the polynomial function. Apart from the aspect ratios used for Re_{PF} , aspect ratios of $r_p = 3.1, 3.2, \dots, 3.9$ were also investigated. When r_p is increased, the difference between ϕ_{us} and ϕ_c decreases, and at the same time Re_c decreases.

Figure 20 illustrates how Re_{PF} and Re_c vary as functions of r_p . The transitional Reynolds numbers are found to be decreasing functions of r_p , and since the transitions

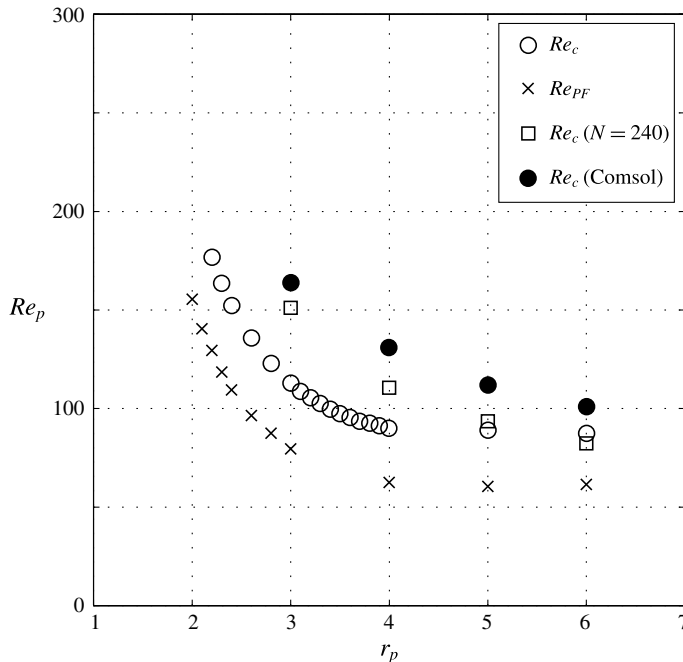


FIGURE 20. Plots of Re_c and Re_{PF} as functions of r_p .

at Re_{PF} and Re_c depend only on fluid inertia (i.e. the emerging solutions correspond to steady states of the fluid field), the two critical numbers decay in a similar manner as r_p is increased. However, due to the resolution issues discussed in § 4.1, it may not be easy to compare the values because the spatial resolution cannot be held exactly constant when the geometry of the particle is changed. Therefore, in figure 20 the trend of Re_c is analysed at higher resolution ($N=240$) and by using Comsol, following the procedure described in § 4.1. It is clear that the critical Reynolds numbers are indeed decreasing with increasing r_p . The sensitivity of these transitions indicates that the force balance which causes the transitions at Re_{PF} and Re_c becomes even more sensitive to the resolution as r_p gets larger.

8.2. Dependence of aspect ratio on particle inertial transitions

To study the effect of aspect ratio on α_c , the simulation procedure from § 4.4 was performed for $r_p = 3-6$. The critical density ratio α_c is plotted as a function of Re_p and r_p in figure 21. From this figure it is observed that both $\alpha_{c,max}$ and α_{cd2} increase rapidly with increasing r_p . These dependencies are better illustrated in figure 22, and it seems that $\alpha_{c,max}$ and α_{cd2} are linearly dependent on r_p . It can also be seen that below a certain aspect ratio, $r_p = r_{p,c1} \approx 3.2$, a critical density ratio of $\alpha_c > 1$ does not exist. This means that neutrally buoyant particles with low r_p will never experience a diverging period in the tumbling motion, a fact that could also be seen from the results of Huang *et al.* (2012a,b). In a range $r_{p,c1} < r_p < r_{p,c2}$ ($3.2 \lesssim r_p \lesssim 3.7$), it is found that $\alpha_{c,max} > 1$ but $\alpha_{cd2} < 1$. Thus, in this range of aspect ratios, the transition to steady state is through a homoclinic bifurcation at $Re_p = St_c$ for neutrally buoyant particles.

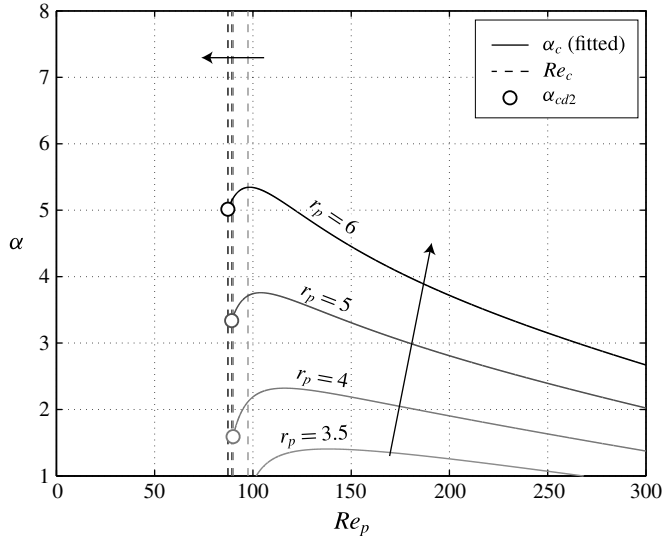


FIGURE 21. The critical density ratio α_c plotted as a function of Re_p (solid curve) and Re_c (dashed line) for a spheroid with $r_p = 3.5, 4, 5$ and 6 ; the arrows show the effect of increasing r_p .

At larger aspect ratios, $r_p > r_{p,c2}$, both $\alpha_{c,max}$ and α_{cd2} are greater than 1 and the transition to steady state will instead be through an infinite-period saddle-node bifurcation at $Re_p = Re_c$ for neutrally buoyant particles.

The fact that fluid inertia effects dominate over particle inertia effects when $r_p \rightarrow \infty$ is also supported by physical reasoning. Particle inertia torques scale with the magnitude of the inertial tensor, which behaves as $1/r_p^2$ for large r_p , whereas torques from fluid stresses on the particle probably scale with the surface area of the particle, behaving as $1/r_p$ for large r_p .

Furthermore, the Re_p value at which $\alpha_{c,max}$ occurs seems to get closer to Re_c , i.e. α_{cd2} approaches $\alpha_{c,max}$ as r_p increases. At $r_p = r_{p,c3} \approx 8.0$ they coincide, and this could mean that for more slender particles, no Re_{c1} will exist and a transition from tumbling to steady state for particles with $\alpha < \alpha_{cd2}$ will always be through an infinite-period saddle-node bifurcation with increasing Re_p . However, the increased sensitivity of determining Re_c for more slender particles will result in increased sensitivity for the determination of α_{cd2} as well. This will probably decrease the slope of α_{cd2} in figure 22, and thus it cannot be concluded that $r_{p,c3}$ exists.

Lundell & Carlsson (2010) and Nilsen & Andersson (2013) already studied the aspect ratio dependence on $St_{0.5}$ with negligible fluid inertia and found it to be an increasing function of r_p . The same can be said of the problem with fluid inertia in figure 23, where $St_{0.5}$ is plotted as function of Re_p and r_p . The graphs are connected to the analytical value at $Re_p = 0$. From the dent in the graph at $Re_p = 10$, it can be seen that the error is larger at higher values of r_p . This is because small numerical errors in determining $\dot{\phi}_{min}$ during the slow part of the tumbling motion for slender particles have a large influence on the error in the period, which is used for the determination of $St_{0.5}$.

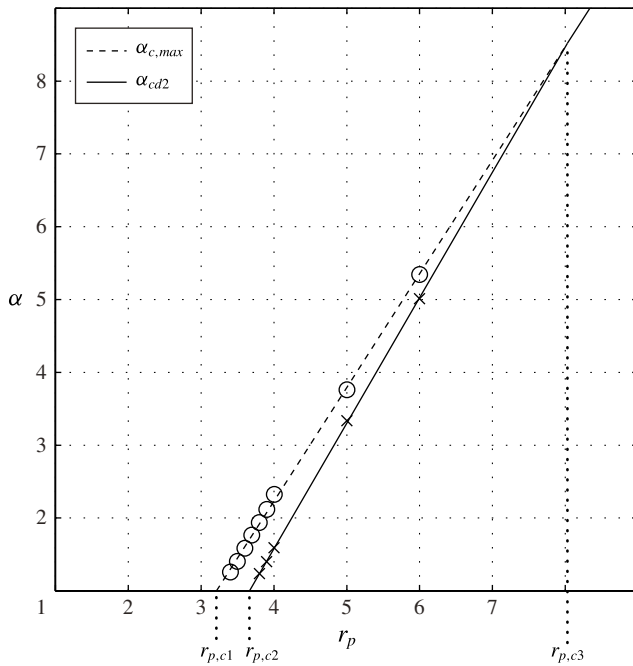


FIGURE 22. Plots of the codimension-two point α_{cd2} (crosses with fitted solid line) and the maximum critical density ratio $\alpha_{c,max}$ (circles with fitted dashed line) as functions of the particle aspect ratio r_p . At a certain $r_p = r_{p,c3}$, the fitted lines intersect and $\alpha_{c,max} = \alpha_{cd2}$ for $r_p > r_{p,c3}$.

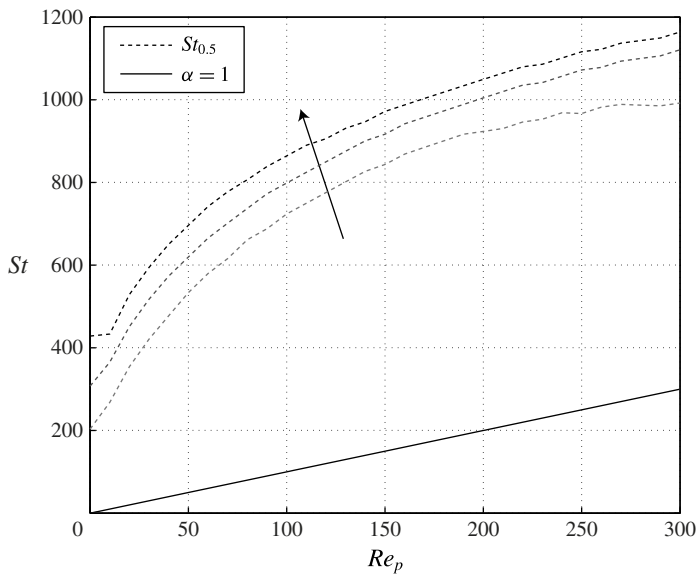


FIGURE 23. Plots of $St_{0.5}$ as a function of Re_p for a spheroid with $r_p = 3, 4$ and 5 ; the black arrow shows the effect of increasing r_p .

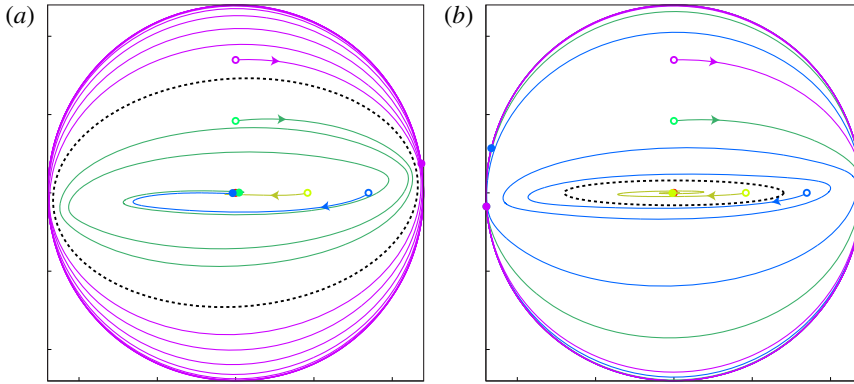


FIGURE 24. (Colour online) Projected trajectories close to Re_{PF} , for: (a) $r_p = 3$ and $Re_p = 79$; (b) $r_p = 6$ and $Re_p = 60$. In each panel, the open circles indicate initial orientations and the filled circles indicate final orientations at $Gt = 400$; the dashed line indicates the approximate location of an unstable limit cycle, such that initial orientations inside lead to log-rolling and initial orientations outside lead to tumbling.

8.3. Transient behaviour and orbit drift

In the region $Re_{LR} < Re_p < Re_T$, the particle can end up in one of two final rotational states depending on the initial conditions. This splitting of solutions is described by a subcritical Hopf bifurcation and the emergence of an unstable limit cycle, outside of which initial orientations go to tumbling (associated with particle inertia), while initial orientations inside go to the coexisting solution (associated with fluid inertia). The size of the unstable limit cycle is therefore a measure of the probability of a randomly oriented particle initialized at rest ending up in the respective states. This was described in § 3.2. To determine how the unstable limit cycle changes shape due to the aspect ratio, a comparison is made of the initial orientation dependence for $r_p = 3$ and 6 around Re_{PF} . The simulation was initialized with the particle at rest in five different initial orientations given by (3.1), and was run up to $Gt = 400$.

Figure 24(a,b) show the projected trajectories for $r_p = 3$ ($Re_p = 79$) and $r_p = 6$ ($Re_p = 60$), respectively. It is clear that the unstable limit cycle shrinks with increasing aspect ratio, which leads to the conclusion that as r_p increases, more initial orientations will lead to a tumbling motion. This is also supported by physical reasoning, as the inertial tensor element for minor axis rotation, $I_{minor} \propto r_p^{-2}(1 + r_p^{-2})$, is orders of magnitude greater than the element for major axis rotation, $I_{major} \propto 2r_p^{-4}$, when $r_p \rightarrow \infty$. It therefore seems reasonable that the tumbling state would be favoured over log-rolling as the particle becomes more slender.

8.4. Consequences for suspension rheology

In § 7, it was discussed how the intrinsic viscosity η is modified by the rotational states at different Re_p and St . To investigate the r_p -dependence of η , three neutrally buoyant particles of aspect ratios $r_p = 4, 5$ and 6 are simulated and compared in figure 25. The analytical value of $\eta_{Jeffery}(LR)$ at $Re_p = 0$ is known to go as $\eta_{Jeffery}(LR) \rightarrow 2$ when $r_p \rightarrow \infty$, whereas $\eta_{Jeffery}(T) \rightarrow \infty$ as $r_p \rightarrow \infty$ (Jeffery 1922). At the same time $\eta(Re_c)$ seems to decrease. This observation is also supported by the fact that $\phi_c(Re_c)$ gets closer to zero as r_p is increased (figure 19), which would make

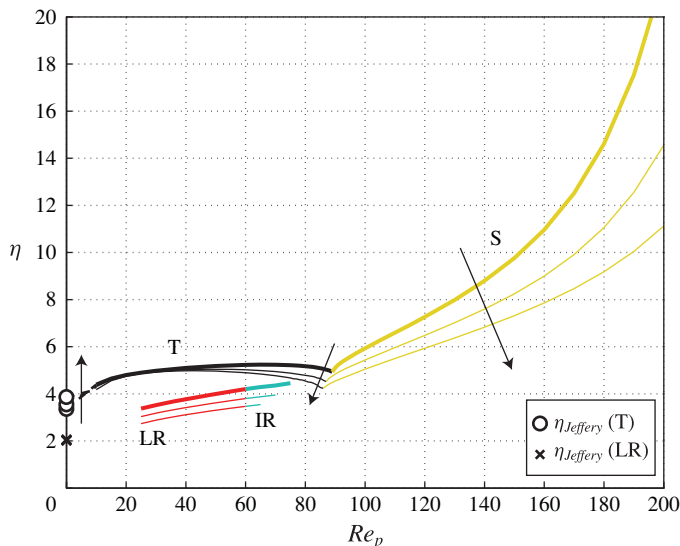


FIGURE 25. (Colour online) Intrinsic viscosity of a one-particle suspension plotted as a function of Re_p with $r_p = 4, 5$ and 6 for a neutrally buoyant particle ($\alpha = 1$); the arrows show the effect of increasing r_p .

a smaller contribution to the suspension viscosity (Jeffery 1922; Mueller, Llewellyn & Mader 2009). The consequence is that the suspension goes from shear-thickening to shear-thinning behaviour on the tumbling branch as r_p increases. This is in line with the results of Huang *et al.* (2012a). It should be noted that, even though the shear-thickening behaviour of the non-planar branch still holds for high-aspect-ratio particles, the results from the previous section show that slender particles have a higher probability of ending up in a tumbling motion. Therefore the rheological properties of dilute fibre suspensions will rather be determined by the shear-thinning of the tumbling branch. The largest contribution to the suspension viscosity without inertia occurs when the particle is aligned with $\phi = \pm 45^\circ$ (Jeffery 1922; Mueller *et al.* 2009). Knowing that the stable equilibrium angle ϕ_c is an increasing function of Re_p (figure 12) and that the viscosity increases as this angle comes closer to $\phi = 45^\circ$, the shear-thickening behaviour on the steady-state branch when $Re_p > Re_c$ is believed to remain as the particles become more slender. However, the trend is that $d\eta/dRe_p$ decreases as r_p increases for $Re_p > Re_c$.

8.5. Low-aspect-ratio particles

In this section, we report our findings on the rotational behaviour of neutrally buoyant, low-aspect-ratio particles, which deviates from the behaviour described by Rosén *et al.* (2014). A spheroid with $r_p = 2$ is typically considered in this section.

Up to $Re_p = Re_{pF}$, the behaviour is similar to that of the higher-aspect-ratio particles. The big difference occurs at $Re_p > Re_{pF}$ and is illustrated in figure 26 for $Re_p = 210$ and three different initial orientations: $(\phi, \theta)_1 = (0, \pi/8)$, $(\phi, \theta)_2 = (0, 0)$ and $(\phi, \theta)_3 = (\pi/2, \pi/8)$.

Here a third rotational state is seen to coexist with the tumbling and inclined rolling solutions. The unstable manifold of the log-rolling saddle is attracted by the other unstable manifold of the same saddle. The result is a rotational state which

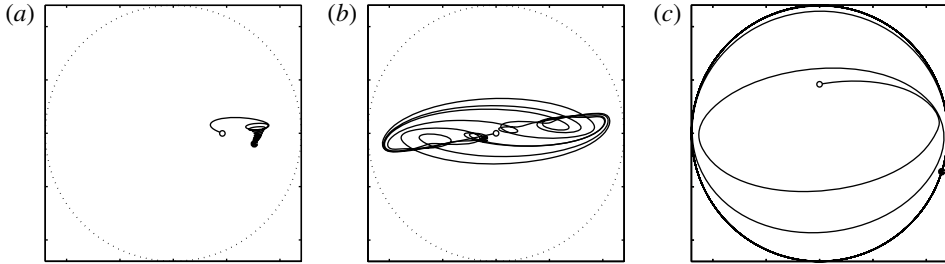


FIGURE 26. Projected trajectories of the $r_p = 2$ particle's symmetry axis $(p_x, p_y) = (\sin \theta \cos \phi, \sin \theta \sin \phi)$ at $Re_p = 210$, for different initial orientations that lead to: (a) inclined rolling; (b) 'chaotic' kayaking; (c) tumbling. The open circles indicate the initial orientations and the filled circles indicate the final orientations at $Gt = 1000$.

is seemingly chaotic. While all the other rotational states discussed previously could be attributed to either fluid or particle inertial effects, this 'chaotic' kayaking state seems to involve both effects. Fluid inertia dominates close to the log-rolling saddle, but the angular acceleration of the particle leads to increased particle inertial effects, bringing it close to the saddle again. The simulations were not run for long enough (simulation time $Gt = 1000$) to exclude the possibility that the particle actually does not settle to a stable limit cycle, and thus it cannot be concluded that this state is truly chaotic.

The reason for both these effects being present can be understood by again looking at the terms in the inertial tensor of the spheroid. Using the same reasoning as in § 8.3, we find that the inertial tensor element for minor axis rotation, $I_{minor} \propto r_p^{-2}(1 + r_p^{-2})$, is of the same order of magnitude as the inertial tensor element for major axis rotation, $I_{major} \propto 2r_p^{-4}$, when $r_p \gtrsim 1$. Particle inertia therefore cannot be neglected when the particle is in non-planar motion and will play a role in any type of rotation.

Above $Re_p = 215$, no stable fixed point was found, and due to the existence of the 'chaotic' kayaking state, it is impossible to see by simple observation whether a supercritical Hopf bifurcation has occurred or if the fixed point becomes unstable because it has moved outside the unstable limit cycle. A value of Re_{Hopf} thus becomes very hard to determine.

As Re_p is increased further, the oscillations become so large that the particle eventually spirals out to the stable limit cycle corresponding to tumbling.

Another difference for the $r_p = 2$ spheroid is observed when analysing the tumbling motion. Above $Re_p \approx 210$, the particle in tumbling motion does not necessarily rotate with axis of rotation parallel to the vorticity axis. Depending on the initial conditions, it may end up in a tumbling motion for which the angle between the vorticity axis and the axis of rotation is somewhere between 0 and $\pi/6$. For one value of Re_p , the particle can thus end up in one of an infinite number of these inclined tumbling states. The explanation might be that the particle is interacting with its own wake, which creates forces that counteract the centrifugal forces that want to take the particle to a rotation in the flow-gradient plane.

It is clear that the rotational behaviour of low-aspect-ratio particles, i.e. ones with $r_p \leq 2$, is very complex, and further investigation is needed to make a full characterization. This is beyond the scope of the present study.

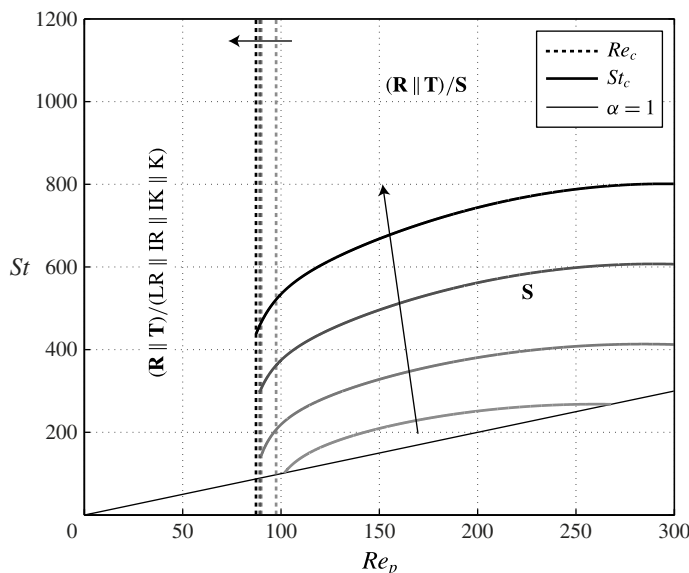


FIGURE 27. State plot showing dependence of the different rotational states on the parameters Re_p and St for a spheroid with $r_p = 3.5, 4, 5$ and 6 , excluding the transitional Reynolds numbers Re_{LR} , Re_{PF} , Re_{Hopf} and Re_T and excluding $St_{0.5}$. The states shown are T = tumbling, R = rotating, LR = log-rolling, IR = inclined rolling, IK = inclined kayaking, K = kayaking and S = steady state; the planar rotational states appear in bold while the non-planar states associated with fluid inertia are in regular font; the arrows show the effect of increasing r_p .

9. Conclusions

The work in this paper was performed numerically using the lattice Boltzmann method with external boundary forcing (LB-EBF) in order to fully describe the inertial effects on a prolate spheroidal particle with higher density than the fluid in shear flow. The parameters that influence the effects of particle and fluid inertia are Re_p , St and r_p . For a given r_p , the results can be summarized in a state-space diagram of the Re_p – St plane; see figure 11. The influence of r_p is shown in figure 27 (for the reader's convenience, only Re_c and St_c are included in this figure).

9.1. Quantitative versus qualitative conclusions

One of the main conclusions drawn from this work is that finding the exact physical values of the transitional Re_p (e.g. Re_c) is very difficult, because the spatial resolution requirement makes it computationally unfeasible to obtain grid-independent values. This is believed to be the reason that the values of the critical Reynolds numbers differ between different studies (e.g. Qi & Luo 2003, Yu *et al.* 2007 and Huang *et al.* 2012b). However, to the authors' knowledge, currently there is no method available that can provide exact results without extreme computational effort.

Nevertheless, the qualitative features and, in particular, the bifurcation sequences of the particle motion are not grid-dependent in the present LB-EBF method, as long as the minor dimension is large enough ($b \geq 3$). Therefore the LB-EBF method provides an excellent exploratory tool for elucidating the relevant physical phenomena; other methods should, however, be used to pinpoint the exact physical values of the

governing parameters. We can draw several conclusions based on the results of this numerical study combined with physical and mathematical reasoning.

9.2. Rotational states and bifurcations

The conclusions regarding rotational states and bifurcations are summarised as follows.

- (i) The final rotational state of the particle is determined by the parameters Re_p , St and r_p . In some parameter regions, two solutions can coexist, one that is due to fluid inertia effects and one that is due to particle inertia effects. The solution created by particle inertia effects is a planar rotation (tumbling or rotating) in the flow-gradient plane. Increasing St will thus increase the probability that the particle will end up in these states.
- (ii) The tumbling period is dependent on St , and the particle undergoes a transition from having an orientation-dependent angular velocity (tumbling) to a constant angular velocity (rotating). The transition is characterized by $St_{0.5}$, which is an increasing function of both Re_p and r_p .
- (iii) The transitions between rotational states due to fluid inertia are characterized by the critical Reynolds numbers Re_{LR} , Re_{PF} , Re_{Hopf} , Re_T and Re_c , which are fairly unaffected by St but are decreasing functions of r_p .
- (iv) At $Re_p > Re_c$, there exists a St_c below which the steady state is the only long-term solution (below the solid line in figure 27); St_c is an increasing function of Re_p and r_p . The transition from tumbling to steady state at $St = St_c$ is through a homoclinic bifurcation with a diverging period that scales as $GT \propto \ln |\beta - \beta_c|$, where β is any parameter leading to the transition at $\beta = \beta_c$, such as St or r_p . This is in contrast to the infinite-period saddle-node bifurcation at $Re_p = Re_c$ discussed by Rosén *et al.* (2014), where the period scales as $GT \propto |\beta - \beta_c|^{-0.5}$. Changing r_p can lead to both types of bifurcations, since it affects both Re_c and St_c .
- (v) For a given r_p , the two bifurcations connect at a codimension-two point (marked with circles in figure 27) where $(Re_p, St) = (Re_c, St_c(Re_c)) = (Re_c, St_{cd2})$. The density ratio $\alpha_{cd2} = St_{cd2}/Re_c$ increases linearly with r_p .
- (vi) The sequence of bifurcations between the non-planar rotational states (Re_{LR} , Re_{PF} , Re_{Hopf} , Re_T) is typical for a nonlinear dynamical system with odd symmetry around a double zero eigenvalue and further supports the existence of such an eigenvalue proposed by Rosén *et al.* (2014).
- (vii) The bifurcations and transitions between the planar rotational states (Re_c , St_c , $St_{0.5}$) are typical for a damped rotating system with an orientation-dependent torque, and are exactly the same as those seen for a driven damped pendulum (Strogatz 1994).
- (viii) For a neutrally buoyant particle, the effect of increasing Re_p depends on r_p in the following way.
 - (a) If $r_p < r_{p,c1}$ ($r_{p,c1} \approx 3.2$), there will be no diverging tumbling period since $\alpha_{c,max} < 1$. This means that the St_c for $r_p = 3.2$ (not drawn) would be below the $Re = St$ line in figure 27.
 - (b) If $r_{p,c1} < r_p < r_{p,c2}$ ($r_{p,c2} \approx 3.7$), the tumbling period will diverge through a homoclinic bifurcation at $Re_p = St_c$ since $\alpha_{cd2} < 1$. This is seen for $r_p = 3.5$ in figure 27 from the fact that the $Re = St$ line crosses Re_c before crossing St_c .
 - (c) If $r_p > r_{p,c2}$, the period will diverge through an infinite-period saddle-node bifurcation at $Re_p = Re_c$, as seen for $r_p = 4.5$ and 6 in figure 27.

9.3. Rheological consequences of the particle behaviour

In addition to direct observation of the particle motion, we obtain from the simulations the stress distribution on the walls. The total stress on the wall makes it possible to determine the intrinsic viscosity of the (very dilute) suspension system. In terms of rheology, the conclusions are as follows.

- (i) At $Re_p < Re_c$:
 - (a) the rotation of high-aspect-ratio particles is mainly dependent on particle inertia effects, and most initial orientations lead to planar rotation, i.e. tumbling; a dilute suspension of such particles will be shear-thinning;
 - (b) the rotation of low-aspect-ratio particles is dependent on both fluid and particle inertia effects, and it is more likely for the particles to end up in a non-planar motion, i.e. log-rolling, inclined rolling, inclined kayaking or kayaking; a dilute suspension of such particles will be shear-thickening.
- (ii) At $Re_p > Re_c$:
 - (a) the rotation of high-aspect-ratio particles is mainly dependent on fluid inertia effects, and the particle is likely to end up in a steady state; a dilute suspension of such particles will be weakly shear-thickening;
 - (b) the rotation of low-aspect-ratio particles is dependent on both fluid and particle inertia effects, and it is more likely for the particle to end up in a tumbling motion; a dilute suspension of such particles will be significantly shear-thickening.
- (iii) With increasing α of the particles, the dilute suspension containing these particles will probably become more shear-thickening at any Re_p . If $Re_p > Re_c$ and $\alpha < \alpha_c$, a change in α will not have an influence on the rheology since the particles are in steady state.

9.4. Final remarks

This work, combined with the work of Rosén *et al.* (2014), provides fundamental information on the behaviour of solid prolate spheroids in a linear shear flow. The results can also be considered fundamental to understanding the behaviour of any elongated particles in any flow, as the rotational dynamics is mainly governed by the local velocity gradient around the particle. The description of the behaviour of single particles in shear flow provides the basic knowledge necessary to understanding the behaviour of dispersed particle flows. Properties characterizing the dispersion, such as rheology, mixing or particle–particle interactions, are highly dependent on the particle orientation distribution, which in turn is highly affected by inertial effects as observed in this work.

Of course, the results themselves should only be considered as fundamental research and of limited direct practical use. In principle, the results about neutrally buoyant particles can be used practically in a wall-bounded flow with constant flow rate and straight streamlines. The particle will then be in an inertial frame of reference and the translational equation of motion for the particle can be neglected upon applying a Galilean transformation of the particle coordinate system. Local velocity gradients around the particle, and hence the local Re_p and aspect ratio r_p , will determine the rotational states and therefore also the orientation distribution and rheological features of the flow. However, due to the difficulty in determining the exact transitional Reynolds numbers, only a relative evaluation can be done; that is, given a flow and a

distribution of particles and orientations, what is likely to happen if the flow rate is increased/decreased? The results in this work enable such experimental observations to be analysed based on physical arguments.

In many cases, the influence of gravity will cause non-neutrally buoyant particles to sediment, and particle inertia will cause the particles to not follow curved streamlines of the flow. Both of these examples will probably result in a relative velocity between particles and fluid, which would give rise to additional forces and hence additional complexity of the problem (Crowe *et al.* 2012).

The results of this work concern only axisymmetric particles. However, as soon as the axisymmetry is broken (e.g. triaxial particles), the particles will exhibit more complex dynamical behaviour, since the tumbling solution may become unstable at low Re_p (Hinch & Leal 1979; Yarin, Gottlieb & Roisman 1997; Lundell 2011).

Additionally, if the particles become smaller, i.e. comparable to the molecular scale, Brownian diffusion will disturb the rotational motion. In such cases, the results of this work can provide the fundamental knowledge that could be referred to when performing other parameter variations; for example, what happens to the Re_p - St - r_p rotational state space when a relative velocity is varied? These are just some of many examples of additional studies that can be performed in the future based on the present results.

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Supplementary data

Supplementary data is available at <http://dx.doi.org/10.1017/jfm.2015.127>.

Appendix A. Description of the LB-EBF method

This appendix provides a summary of the simulation steps performed in the LB-EBF method, described in detail by Wu & Aidun (2010).

A.1. Fluid grid

The fluid is defined on a cubic Eulerian grid denoted by

$$\mathbf{x}^e \in \Pi_f. \quad (\text{A } 1)$$

The nodes are separated by distance Δx in each direction. The domain is bounded by walls at $y=0$ and $y=N$ with velocities U and $-U$ in the x direction, which are treated with non-slip conditions; periodic boundary conditions are applied at $x=0$, $x=N$, $z=0$ and $z=N$.

A.2. Particle grid

The particle is defined using a Lagrangian grid consisting of boundary nodes denoted by

$$\mathbf{x}_j^l \in \Gamma_s \tag{A 2}$$

for the j th node. Each node is associated with an area element of size ΔA_j and is assigned a characteristic volume $\Delta v_j = \Delta A_j^{3/2}$. The best results are obtained by keeping $\Delta A_j \approx (\Delta x)^2$.

A.3. The discrete Dirac function

Since the fluid and particle nodes are not at the same position, the mapping of forces and velocities between fluid and particle grids is done through a discrete Dirac function, defined by Peskin (2002) as

$$D(\mathbf{x}) = \begin{cases} \frac{1}{64\Delta x^3} \left(\left(1 + \cos\left(\frac{\pi x}{2\Delta x}\right) \right) \left(1 + \cos\left(\frac{\pi y}{2\Delta x}\right) \right) \left(1 + \cos\left(\frac{\pi z}{2\Delta x}\right) \right) \right), & |\mathbf{x}| \leq 2\Delta x, \\ 0, & |\mathbf{x}| > 2\Delta x. \end{cases} \tag{A 3}$$

A.4. Principle of the lattice Boltzmann method

The principle of lattice Boltzmann methods is based on the motion of gas particles. The streaming of and collisions between the particles are simulated using particle distribution functions $f_i(\mathbf{x}^e, t)$, with 19 components ($i = 0, \dots, 18$ for a D3Q19 lattice) and defined in each fluid grid node \mathbf{x}^e . Each component can be viewed as the probability of a fluid particle having an associated velocity $\mathbf{c}_i = (c_{i,x}, c_{i,y}, c_{i,z})$, where $\mathbf{x} + \mathbf{c}_i \delta t \in \Pi_f$ (δt is the discrete time step). In the streaming step, each distribution $f_i(\mathbf{x}^e, t)$ is translated to $f_i(\mathbf{x}^e + \mathbf{c}_i \delta t, t + \delta t)$. Macroscopic quantities can be obtained in each fluid node through the relations

$$\rho(\mathbf{x}, t) = \sum_{i=0}^{18} f_i(\mathbf{x}^e, t), \tag{A 4}$$

$$\rho \mathbf{u}_f(\mathbf{x}, t) = \sum_{i=0}^{18} \mathbf{c}_i f_i(\mathbf{x}^e, t). \tag{A 5}$$

In the collision step, the particle distributions $f_i(\mathbf{x}^e, t)$ are locally redistributed towards a local equilibrium distribution $f_i^{eq}(\mathbf{x}^e, t)$, still preserving local density and momentum. The equilibrium distribution is defined as

$$f_i^{eq}(\mathbf{x}^e, t) = w_i \rho \left(1 + 3\mathbf{c}_i \cdot \mathbf{u}_f + \frac{9}{2} (\mathbf{c}_i \cdot \mathbf{u}_f)^2 - \frac{3}{2} \mathbf{u}_f^2 \right), \tag{A 6}$$

where $w_0 = 1/3$, $w_{1-6} = 1/18$ and $w_{7-18} = 1/36$.

A.5. Simulation steps

A.5.1. Initialization

- (a) The initial particle node position \mathbf{x}_j^l and velocity $\mathbf{U}_p(\mathbf{x}_j^l, t_0)$ are calculated by using information about the particle's initial position, orientation, velocity and angular velocity.

- (b) The initial fluid velocity at \mathbf{x}^e is set to the linear velocity profile defined by the wall velocity U .
- (c) The density is set to $\rho(\mathbf{x}^e, t_0) = \rho_0$.
- (d) The populations are set to the equilibrium distributions, $f_i(\mathbf{x}^e, t_0) = f_i^{eq}(\mathbf{x}^e, t_0)$.

A.5.2. Iterative steps

- (i) Streaming: the new pre-collisional populations $f_i^{pre}(\mathbf{x}^e, t)$ are obtained from

$$f_i^{pre}(\mathbf{x}^e, t) = f_i(\mathbf{x}^e - \mathbf{c}_i \delta t, t - \delta t), \tag{A 7}$$

and the velocity $\mathbf{u}_f(\mathbf{x}^e, t)$ and density $\rho(\mathbf{x}^e, t)$ are calculated as above.

- (ii) The fluid velocity on the particle boundary is calculated from

$$\mathbf{u}_f(\mathbf{x}_j^l, t) = \sum_{\mathbf{x}^e \in \Pi_f} \mathbf{u}_f(\mathbf{x}^e, t) D(\mathbf{x}^e - \mathbf{x}_j^l) (\Delta x)^3. \tag{A 8}$$

- (iii) The force acting on a particle node is calculated by

$$\mathbf{F}^{fsi}(\mathbf{x}_j^l, t) = \mathbf{f}^{fsi}(\mathbf{x}_j^l, t) \cdot \Delta v_j = \rho_0 (\mathbf{u}_f(\mathbf{x}_j^l, t) - \mathbf{U}_p(\mathbf{x}_j^l, t - \delta t)) \cdot \Delta v_j / \delta t \tag{A 9}$$

with $\mathbf{f}^{fsi}(\mathbf{x}_j^l, t)$ defined as a force density.

- (iv) The particle force \mathbf{F} and torque \mathbf{T} are determined by summation of the contributions from the nodes:

$$\mathbf{F}(t) = \sum_{\mathbf{x}_j^l \in \Gamma_s} \mathbf{F}^{fsi}(\mathbf{x}_j^l, t), \tag{A 10}$$

$$\mathbf{T}(t) = \sum_{\mathbf{x}_j^l \in \Gamma_s} (\mathbf{x}_j^l - \mathbf{x}^{lc}) \times \mathbf{F}^{fsi}(\mathbf{x}_j^l, t), \tag{A 11}$$

where \mathbf{x}^{lc} is the centre of gravity of the particle.

- (v) The velocity and angular velocity of the particle at time t are determined by integration of

$$\frac{d\mathbf{U}}{dt} = \frac{\mathbf{F}}{M}, \tag{A 12}$$

$$\mathbf{I} \frac{d\boldsymbol{\Omega}}{dt} + \boldsymbol{\Omega} \times (\mathbf{I} \cdot \boldsymbol{\Omega}) = \mathbf{T}, \tag{A 13}$$

where M is the particle mass, \mathbf{I} the inertia tensor and $\boldsymbol{\Omega}$ the particle angular velocity. From this, the new position $\mathbf{x}_j^l(t)$ and velocity $\mathbf{U}_p(\mathbf{x}_j^l, t)$ of the particle nodes can be obtained through a fourth-order-accurate Runge–Kutta integration procedure.

- (vi) The force acting on the fluid is determined by

$$\mathbf{g}(\mathbf{x}^e, t) = - \sum_{\mathbf{x}_j^l \in \Gamma_s} \mathbf{f}^{fsi}(\mathbf{x}_j^l, t) D(\mathbf{x}^e - \mathbf{x}_j^l) \Delta v_j. \tag{A 14}$$

- (vii) Collision step: the post-collision distribution f_i is obtained by colliding the pre-collision distribution f_i^{pre} with the local equilibrium distribution f_i^{eq} , accounting for the external boundary force \mathbf{g} . All distribution functions relax towards equilibrium with the same time scale governed by the relaxation parameter τ ,

$$f_i(\mathbf{x}^e, t) = f_i^{pre}(\mathbf{x}^e, t) + \frac{1}{\tau} [f_i^{eq}(\mathbf{x}^e, t) - f_i^{pre}(\mathbf{x}^e, t)] + 3w_i \mathbf{g}(\mathbf{x}^e, t) \cdot \mathbf{c}_i. \quad (\text{A } 15)$$

- (viii) Repeat from step (i) for $t = t + \delta t$.

The speed of sound in this model is a constant equal to $c_s = \sqrt{1/3}$, to preserve symmetries with the choice of equilibrium functions in (A 6) (Latt 2007). The kinematic viscosity ν is related to the parameter τ in the last equation through $\nu = c_s^2(\tau - 0.5)$. By keeping $Ma = U/c_s < O(0.1)$ and $Kn = \nu/(c_s \cdot L) < O(0.1)$ (where U and L are the characteristic velocity and length scale, respectively), the lattice Boltzmann fluid solver is equivalent to solving the incompressible Navier–Stokes equations.

Appendix B. Determination of the equilibrium angles ϕ_c and ϕ_{us}

The angles are found using half-interval search in the following way for a given value of Re_p .

- (i) Initialize at $\phi_n = -30^\circ$ ($\theta = 90^\circ$).
- (ii) The particle is held steady for the first 3000 time steps, in order for the fluid to be resolved around the particle.
- (iii) The simulation runs for 500 time steps.
 - (a) If $\dot{\phi} < 0$ and $\phi_n < 30^\circ$:
 - (i) repeat from step (ii) with $\phi_{n+1} = \phi_n + 1^\circ$.
 - (b) If $\dot{\phi} > 0$:
 - (i) find ϕ_{us} through a half-interval search between ϕ_n and ϕ_{n-1} , with the tolerance set to $\Delta\phi = 0.1^\circ$;
 - (ii) find ϕ_c through a half-interval search between ϕ_n and $\phi = 30^\circ$, with the tolerance set to $\Delta\phi = 0.1^\circ$.
 - (c) If $\phi_n = 30^\circ$:
 - (i) there are no equilibrium angles at the given Re_p (i.e. $Re_p < Re_c$).

Appendix C. Determination of the critical density ratio α_c

The critical density ratio α_c is found in the following way for a given value of Re_p .

- (i) Initialize at $\phi_0 = \phi_{us} - \delta\phi$ ($\theta = 90^\circ$) with $\alpha_n = 1$.
- (ii) The particle is held steady for the first 3000 time steps, in order for the fluid to be resolved around the particle.
- (iii) The particle is released and starts rotating with $\dot{\phi} < 0$.
 - (a) If the particle reaches $\phi \leq \phi_0 - \pi$: there is no $\alpha_c > 1$ at the given Re_p .
 - (b) If the particle stops ($\dot{\phi} \geq 0$) at $\phi > \phi_0 - \pi$: find α_c by a half-interval search between $\alpha = 1$ and $\alpha = St_{max}/Re_p$ ($St_{max} = 600$ for the $r_p = 4$ spheroid), with the tolerance set to $\Delta St = 10$.

Appendix D. Determination of $St_{0.5}$

The critical Stokes number $St_{0.5}$ is found in the following way for a given value of Re_p .

- (i) Initialize at $\phi_0 = -45^\circ$ ($\theta = 90^\circ$).
- (ii) The particle is held steady for the first 500 time steps, in order for the fluid field to be resolved around the particle.
- (iii) Set the initial angular velocity of the particle to $\dot{\phi} = -G/2$, where G is the shear rate of the flow.
- (iv) $St_{0.5}$ is defined as the value of St at which $GT = Gt_{\phi=-5\pi} - Gt_{\phi=-3\pi} = \pi(r_p^{-1} + r_p + 2)$, and is found by a half-interval search between $St = Re_p$ (i.e. $\alpha = 1$) and $St = St_{max}$ ($St_{max}=2500$ for the $r_p=4$ spheroid), with the tolerance set to $\Delta St=10$.

Appendix E. Determination of the intrinsic viscosity η

In this section we describe how to obtain the intrinsic viscosity of the one-particle suspension from the simulations in this work. The procedure is identical to that used by Huang *et al.* (2012a,b). In the lattice Boltzmann scheme employed in this study, the local shear stress can be evaluated in each fluid node according to

$$\sigma(\mathbf{x}^e, t) = - \left(1 - \frac{1}{2\tau}\right) \sum_{i=0}^{18} (f_i(\mathbf{x}^e, t) - f_i^{eq}(\mathbf{x}^e, t)) c_{i,x} c_{i,y}. \quad (\text{E } 1)$$

The total shear stress on the moving walls, $\sigma_w(t)$, is evaluated by taking the mean value of all nodes closest to the walls (i.e. at $y=1$ and $y=N-1$). The time-dependent intrinsic viscosity $\eta^*(t)$ is calculated from

$$\eta^*(t) = \frac{1}{\Phi} \left(\frac{\sigma_w(t)}{\rho_0 \nu G} - 1 \right), \quad (\text{E } 2)$$

where ν and ρ_0 are the kinematic viscosity and density of the fluid, respectively, and Φ is the volume fraction calculated from

$$\Phi = \left(\frac{4\pi ab^2}{3} \right) / N^3, \quad (\text{E } 3)$$

where $4\pi ab^2/3$ is the volume of a prolate spheroid with major semi-axis a and minor semi-axis b and N^3 is the volume of the simulation domain. In order to evaluate the intrinsic viscosity for a certain rotational state at a certain Re_p , the simulation is initialized at rest in an orientation from which the particle assumes the stable rotational state quickly, i.e. $(\phi, \theta)_1 = (0, 0)$ for non-planar motion at $Re_{LR} < Re_p < Re_T$, $(\phi, \theta)_2 = (5^\circ, 90^\circ)$ for the steady state at $Re_p > Re_c$ and $(\phi, \theta)_3 = (-7^\circ, 90^\circ)$ for tumbling at any Re_p . The simulation is run up to $Gt = 1000$, which seemed to be enough for the value of $\eta^*(t)$ to converge at the walls for $0 < Re_p < 200$. The final (time-independent) value of η was evaluated through time-averaging $\eta^*(t)$ from $Gt = 667$ to $Gt = 1000$.

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